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- Multifreedom constraints
- Master-slave method
- Penalty method
- Lagrange multiplier method

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## Multifreedom Constraints (1)

Single freedom constraint examples

$$u_{x4} = 0$$
 (linear, homogeneous)  
 $u_{y9} = 0.6$  (linear, non-homogeneous)

Multifreedom constraint examples

$$u_{x2} = \frac{1}{2}u_{y2} \text{ (linear, homogeneous)}$$

$$u_{x2} - 2u_{x4} + u_{x6} = 0.25 \text{ (linear, non-homogeneous)}$$

$$\left(x_5 + u_{x5} - x_3 + u_{x3}\right)^2 + \left(y_5 + u_{y5} - y_3 + u_{y3}\right)^2 = 0 \text{ (nonlinear, homogeneous)}$$

## Multifreedom Constraints (2)

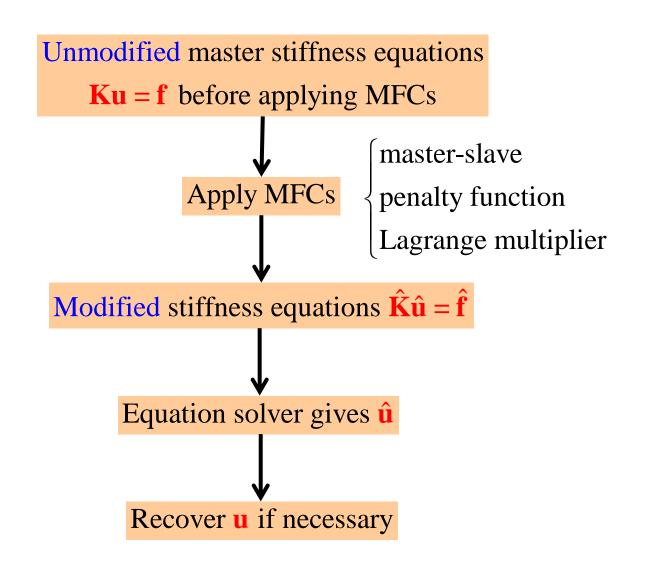
#### Sources

- "skew" displacement BCs
- Coupling nonmatched FEM meshes
- Global-local and multiscale analysis
- Incompressibility
- Model reduction

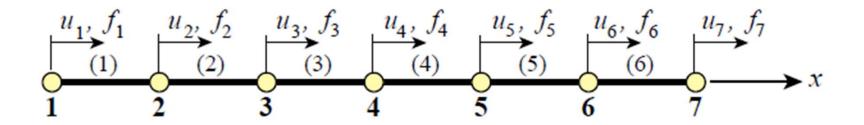
#### MFC application methods

- Master-slave elimination
- Penalty function augmentation
- Lagrange multiplier adjunction

## Procedure Summary in Static Analysis



#### Example: 1D structure



multifreedom constraint:  $u_2 = u_6$  or  $u_2 - u_6 = 0$  (rigid link) unconstrained master stiffness equations

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} & K_{23} & 0 & 0 & 0 & 0 \\ 0 & K_{23} & K_{33} & K_{34} & 0 & 0 & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 & 0 \\ 0 & 0 & 0 & K_{45} & K_{55} & K_{56} & 0 \\ 0 & 0 & 0 & 0 & K_{56} & K_{66} & K_{67} \\ 0 & 0 & 0 & 0 & 0 & K_{67} & K_{77} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} \Leftrightarrow \mathbf{Ku} = \mathbf{f}$$

#### Example: Master-Slave Method

taking  $u_2$  as master and  $u_6$  as slave

$$\begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_7 \end{bmatrix} \Leftrightarrow \mathbf{u} = \mathbf{T}\hat{\mathbf{u}}$$

unconstrained master stiffness equations:  $\mathbf{K}\mathbf{u} = \mathbf{f}$  master-slave transformation:  $\mathbf{u} = \mathbf{T}\hat{\mathbf{u}}$  replace  $\mathbf{u}$  and premultiply both sides by  $\mathbf{T}^T$ :  $\mathbf{T}^T\mathbf{K}\mathbf{T}\hat{\mathbf{u}} = \mathbf{T}^T\mathbf{f}$  modified stiffness equations:  $\hat{\mathbf{K}}\hat{\mathbf{u}} = \hat{\mathbf{f}}$ 

#### Example: Master-Slave Method

modified stiffness equations

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} + K_{66} & K_{23} & 0 & K_{56} & K_{67} \\ 0 & K_{23} & K_{33} & K_{34} & 0 & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 \\ 0 & K_{56} & 0 & K_{45} & K_{55} & 0 \\ 0 & K_{67} & 0 & 0 & 0 & K_{77} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_7 \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 + f_6 \\ f_3 \\ f_4 \\ f_5 \\ f_7 \end{bmatrix}$$

$$\Leftrightarrow \hat{K}\hat{u} = \hat{f} \xrightarrow{\text{solve for } \hat{u}} u = T\hat{u}$$

## Example: Multiple MFCs (1)

Suppose 
$$u_{1} + 4u_{4} = 0$$
  
 $2u_{3} + u_{4} + u_{5} = 0$ 

$$\begin{cases}
u_{2} - u_{6} = 0 \\
u_{1} + 4u_{4} = 0 \\
2u_{3} + u_{4} + u_{5} = 0
\end{cases}$$

$$u_{3} = -\frac{1}{2}(u_{4} + u_{5}) = \frac{1}{8}u_{1} - \frac{1}{2}u_{5}$$

$$\begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 1/8 & 0 & -1/2 & 0 \\ -1/4 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_5 \\ u_7 \end{bmatrix} \Leftrightarrow \mathbf{u} = \mathbf{T}\hat{\mathbf{u}}$$

## Example: Multiple MFCs (2)

$$\begin{aligned}
u_{2} - u_{6} &= 0 \\
u_{1} + 4u_{4} &= 0 \\
2u_{3} + u_{4} + u_{5} &= 0
\end{aligned}
\xrightarrow{\text{master: 1,2,5,7}} \begin{cases}
u_{6} &= u_{2} \\
4u_{4} &= -u_{1} \\
2u_{3} + u_{4} &= -u_{5}
\end{cases}$$

$$\begin{bmatrix}
0 & 0 & 1 \\
0 & 4 & 0 \\
2 & 1 & 0
\end{bmatrix} \begin{bmatrix}
u_{3} \\
u_{4} \\
u_{6}
\end{bmatrix} = \begin{bmatrix}
0 & 1 & 0 \\
-1 & 0 & 0 \\
0 & 0 & -1
\end{bmatrix} \begin{bmatrix}
u_{1} \\
u_{2} \\
u_{5}
\end{bmatrix} \rightarrow \mathbf{A}_{s} \mathbf{u}_{s} + \mathbf{A}_{m} \mathbf{u}_{m} = 0 \rightarrow \mathbf{u}_{s} = -\mathbf{A}_{s}^{-1} \mathbf{A}_{m} \mathbf{u}_{m}$$

## Example: Non-homogeneous MFCs

$$u_2 - u_6 = 0.2$$

Pick again  $u_6$  as slave, put into matrix form:

$$\begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_7 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} \Leftrightarrow \mathbf{u} = \mathbf{T}\hat{\mathbf{u}} + \mathbf{g}$$

$$\overset{\text{gap}}{\overset{\text{gap}}{\text{vector}}}$$

premultiply both sides by  $\mathbf{T}^T \mathbf{K}$ , replace  $\mathbf{K} \mathbf{u} = \mathbf{f}$  and pass data to RHS

$$\mathbf{T}^{T}\mathbf{K}\mathbf{u} = \mathbf{T}^{T}\mathbf{K}\left(\mathbf{T}\hat{\mathbf{u}} + \mathbf{g}\right) \to \mathbf{T}^{T}\mathbf{K}\mathbf{T}\hat{\mathbf{u}} = \mathbf{T}^{T}\left(\mathbf{K}\mathbf{u} - \mathbf{K}\mathbf{g}\right)$$

$$\to \mathbf{T}^{T}\mathbf{K}\mathbf{T}\hat{\mathbf{u}} = \mathbf{T}^{T}\left(\mathbf{f} - \mathbf{K}\mathbf{g}\right) \xrightarrow{\hat{\mathbf{f}} = \mathbf{T}^{T}\left(\mathbf{K}\mathbf{u} - \mathbf{K}\mathbf{g}\right)} \hat{\mathbf{K}}\hat{\mathbf{u}} = \hat{\mathbf{f}}$$
modified force vector

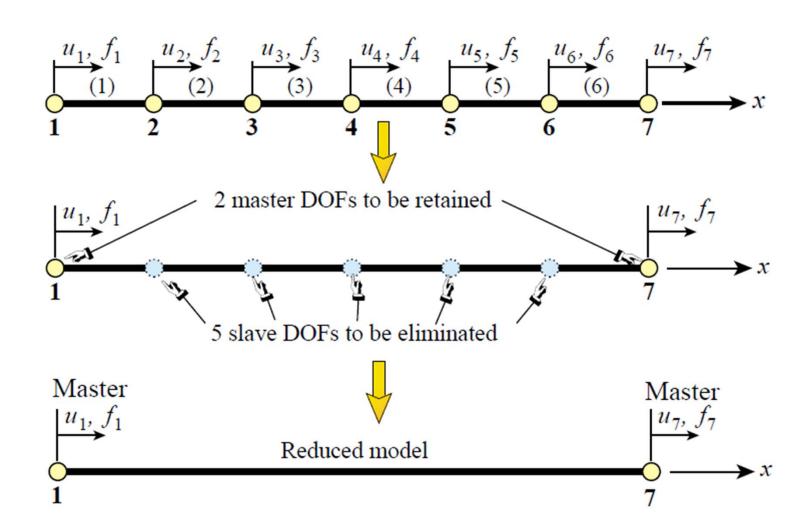
#### Example: Non-homogeneous MFCs

modified stiffness equations

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} + K_{66} & K_{23} & 0 & K_{56} & K_{67} \\ 0 & K_{23} & K_{33} & K_{34} & 0 & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 \\ 0 & K_{56} & 0 & K_{45} & K_{55} & 0 \\ 0 & K_{67} & 0 & 0 & 0 & K_{77} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_7 \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 + f_6 - 0.2K_{66} \\ f_2 + f_6 - 0.2K_{66} \\ f_3 \\ f_4 \\ f_5 - 0.2K_{56} \\ f_7 - 0.2K_{67} \end{bmatrix}$$

$$\Leftrightarrow \hat{\mathbf{K}}\hat{\mathbf{u}} = \hat{\mathbf{f}} \xrightarrow{\text{solve for } \hat{\mathbf{u}}} \mathbf{u} = \mathbf{T}\hat{\mathbf{u}} + \mathbf{g}$$

#### **Example: Model Reduction**



#### **Example: Model Reduction**

Lots of slaves, few masters. Only masters are left.

$$\begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 5/6 & 1/6 \\ 4/6 & 2/6 \\ 3/6 & 3/6 \\ 2/6 & 4/6 \\ 1/6 & 5/6 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_7 \\ 2 \text{ masters} \end{bmatrix}$$

apply the congruential transformation we get the reduced stiffness equations

$$\begin{bmatrix} \hat{K}_{11} & \hat{K}_{17} \\ \hat{K}_{17} & \hat{K}_{77} \end{bmatrix} \begin{bmatrix} u_1 \\ u_7 \end{bmatrix} = \begin{bmatrix} \hat{f}_1 \\ \hat{f}_7 \end{bmatrix}$$

5 slaves

#### Master-Slave Method

#### Advantages

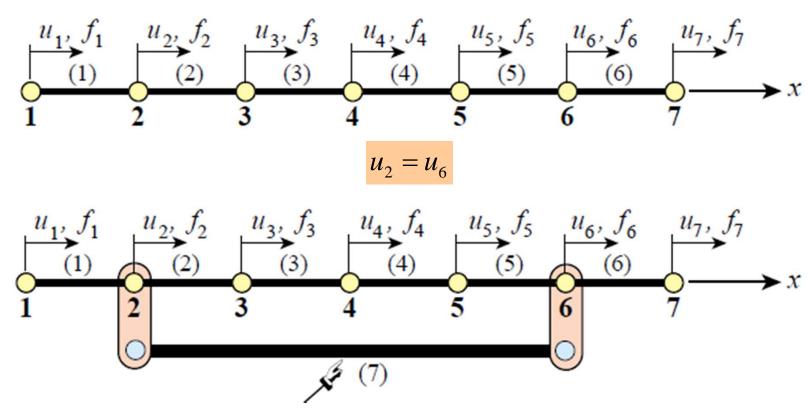
- Exact of precaution taken
- Easy to understand
- Retains positive definiteness
- Important applications to model reduction

#### Disadvantages

- Requires user decisions
- Messy implementation for general MFCs
- Hinders sparsity of master stiffness equations
- Sensitive to constraint dependence
- Restricted to linear constraints

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#### Penalty Function Method: Physical Interpretation



add "penalty element" of axial rigidity w

$$w\begin{bmatrix} 1 & -1 \end{bmatrix} \begin{bmatrix} u_2 \\ u_6 \end{bmatrix} = 0 \xrightarrow{\text{premultiply}} w \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{bmatrix} u_2 \\ u_6 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \rightarrow \mathbf{K}^{(7)} \mathbf{u}^{(7)} = \mathbf{f}^{(7)}$$

w: "penalty weight" assigned to the constraint

#### Penalty Function Method

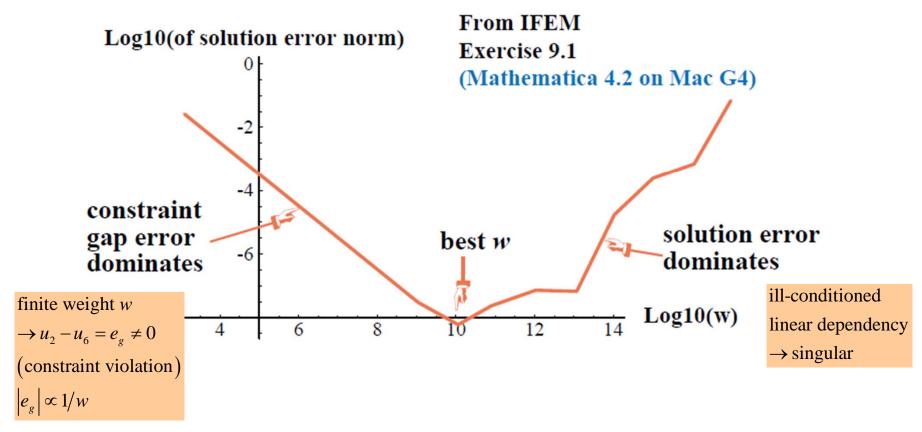
upon merging the penalty element, the modified stiffness equations are

$\int K_{11}$	$K_{12}$	0	0	0	0	0	$\lceil u_1 \rceil$		$\lceil f_1 \rceil$
$K_{12}$	$K_{22} + w$	$K_{23}$	0	0	-w	0	$u_2$		$f_2$
0	$K_{12}$ $K_{22} + w$ $K_{23}$ $0$ $0$ $-w$ $0$	$K_{33}$	$K_{34}$	0	0	0	$u_3$		$f_3$
0	0	$K_{34}$	$K_{_{44}}$	$K_{45}$	0	0	$u_4$	=	$f_4$
0	0	0	$K_{45}$	$K_{55}$	$K_{56}$	0	$u_5$		$f_5$
0	-w	0	0	$K_{56}$	$K_{66} + w$	$K_{67}$	$u_6$		$f_6$
$\bigcup_{i=1}^{n} 0_i$	0	0	0	0	$K_{67}$	$K_{77}$	$\lfloor u_7 \rfloor$		$\lfloor f_7 \rfloor$

This modified system is submitted to the equation solver.

Note that **u** remains the same arrangement of DOFs

## But which penalty weight to use?



Square Root Rule: 
$$w = 10^k \sqrt{10^p} = 10^{k+p/2}$$

k: order of the largest stiffness coefficient before adding penalty elements

p: digits of the working machine precison

#### Penalty Function Method: General MFCs

$$3u_3 + u_5 - 4u_6 = 1 \rightarrow \begin{bmatrix} 3 & 1 & -4 \end{bmatrix} \begin{bmatrix} u_3 \\ u_5 \\ u_6 \end{bmatrix} = 1$$

premultiply both sides by 
$$\begin{bmatrix} 3 & 1 & -4 \end{bmatrix}^T : \begin{bmatrix} 9 & 3 & -12 \\ 3 & 1 & -4 \\ -12 & -4 & 16 \end{bmatrix} \begin{bmatrix} u_3 \\ u_5 \\ u_6 \end{bmatrix} = \begin{bmatrix} 3 \\ 1 \\ -4 \end{bmatrix}$$

"penalty element" stiffness equations

scale by w and merge:

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} & K_{23} & 0 & 0 & 0 & 0 \\ 0 & K_{23} & K_{33} + 9w & K_{34} & 3w & -12w & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 & 0 \\ 0 & 0 & 3w & K_{45} & K_{55} + w & K_{56} - 4w & 0 \\ 0 & 0 & -12w & 0 & K_{56} - 4w & K_{66} + 16w & K_{67} \\ 0 & 0 & 0 & 0 & 0 & K_{67} & K_{77} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 \\ f_3 + 3w \\ f_4 \\ f_5 + w \\ f_6 - 4w \\ f_7 \end{bmatrix}$$

#### Penalty Function Method

#### Advantages

- General application including nonlinear MFCs
- Easy to implement using FE library and standard assembler
- No change in vector of unknowns
- Retains positive definiteness
- Insensitive to constraint dependence

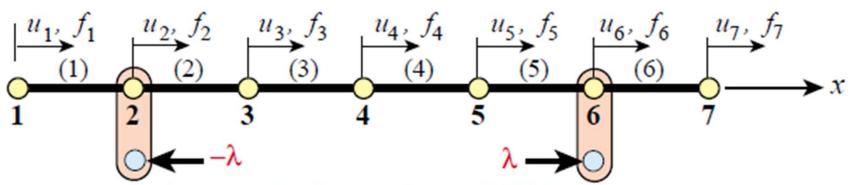
#### Disadvantages

- Selection of weights left to users: big burden
- Accuracy limited by ill-conditioning

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# Lagrange Multiplier Method: Physical Interpretation



force-pair that enforces MFC

$\int K_{11}$	$K_{12}$	0	0	0	0	0	$\lceil u_1 \rceil$		$\lceil f_1 \rceil$
$K_{12}$	$K_{22}$	$K_{23}$	0	0	0	0	$ u_2 $		$f_2 - \lambda$
0	$K_{23}$	$K_{33}$	$K_{34}$	0	0	0	$u_3$		$\begin{bmatrix} f_1 \\ f_2 - \lambda \\ f_3 \\ f_4 \\ f_5 \\ f_6 + \lambda \\ f_7 \end{bmatrix}$
0	0	$K_{34}$	$K_{_{44}}$	$K_{45}$	0	0	$u_4$	=	$f_4$
0	0	0	$K_{45}$	$K_{55}$	$K_{56}$	0	$u_5$		$f_5$
0	0	0	0	$K_{56}$	$K_{66}$	$K_{67}$	$u_6$		$f_6 + \lambda$
$\begin{bmatrix} 0 \end{bmatrix}$	0	0	0	0	$K_{67}$	$K_{77}$	$\lfloor u_7 \rfloor$		$oxed{ \int f_7 }$

## Lagrange Multiplier Method

Because  $\lambda$  is unknown, it is passed to the LHS and appended to the node-displacement vector:

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} & K_{23} & 0 & 0 & 0 & 0 & 1 \\ 0 & K_{23} & K_{33} & K_{34} & 0 & 0 & 0 & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 & 0 & 0 \\ 0 & 0 & 0 & K_{45} & K_{55} & K_{56} & 0 & 0 \\ 0 & 0 & 0 & 0 & K_{56} & K_{66} & K_{67} & -1 \\ 0 & 0 & 0 & 0 & K_{67} & K_{77} & 0 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \\ \lambda \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 \\ f_3 \\ f_4 \\ f_5 \\ f_6 \\ f_7 \end{bmatrix}$$

This is now a system of 7 equations and 8 unknowns.

Need an extra equation: MFC

#### Lagrange Multiplier Method

Appended MFC as an additional equation (adjunction):

$$\begin{bmatrix} K_{11} & K_{12} & 0 & 0 & 0 & 0 & 0 & 0 \\ K_{12} & K_{22} & K_{23} & 0 & 0 & 0 & 0 & 0 \\ 0 & K_{23} & K_{33} & K_{34} & 0 & 0 & 0 & 0 \\ 0 & 0 & K_{34} & K_{44} & K_{45} & 0 & 0 & 0 \\ 0 & 0 & 0 & K_{45} & K_{55} & K_{56} & 0 & 0 \\ 0 & 0 & 0 & 0 & K_{56} & K_{66} & K_{67} & -1 \\ 0 & 0 & 0 & 0 & 0 & K_{67} & K_{77} & 0 \\ 0 & 1 & 0 & 0 & 0 & -1 & 0 & 0 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ u_4 \\ u_5 \\ u_6 \\ u_7 \\ \lambda \end{bmatrix} = \begin{bmatrix} f_1 \\ f_2 \\ f_3 \\ f_4 \\ f_5 \\ f_6 \\ f_7 \\ 0 \end{bmatrix}$$

This is the *multiplier* - *augmented system*.

The new coefficient matrix is called the *bordered stiffness*.

## Lagrange Multiplier Method: Multiple MFCs

Three MFCs:  $u_2 - u_6 = 0$ ,  $5u_2 - 8u_7 = 3$ ,  $3u_3 + u_5 - 4u_6 = 1$ Step#1: append the 3 constraints

$\int K_{11}$	$K_{12}$	0	0	0	0	0		$\lceil f_1 \rceil$
$K_{12}$	$K_{22}$	$K_{23}$	0	0	0	0	  Г., Т	$f_2$
0	$K_{23}$	$K_{33}$	$K_{34}$	0	0	0	$\begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$	$f_3$
0	0	$K_{34}$	$K_{44}$	$K_{45}$	0	0	$\begin{bmatrix} u_2 \end{bmatrix}$	$ f_4 $
0	0	0	$K_{45}$	$K_{55}$	$K_{56}$	0	$\begin{bmatrix} u_3 \\ u \end{bmatrix}$	$f_5$
0	0	0	0	$K_{56}$	$K_{66}$	$K_{67}$	$  u_4   =$	$f_6$
0	0	0	0	0	$K_{67}$	$K_{77}$	$\begin{bmatrix} u_5 \end{bmatrix}$	$ f_7 $
0	1	0	0	0	-1	0	$\begin{bmatrix} u_6 \end{bmatrix}$	0
0	5	0	0	0	0	-8	$\lfloor u_7 \rfloor$	3
0	0	3	0	1	<b>-4</b>	0		$\lfloor 1 \rfloor$

## Lagrange Multiplier Method: Multiple MFCs

Three MFCs: 
$$\underbrace{u_2 - u_6}_{\lambda_1} = 0$$
,  $\underbrace{5u_2 - 8u_7}_{\lambda_2} = 3$ ,  $\underbrace{3u_3 + u_5 - 4u_6}_{\lambda_3} = 0$ 

Step#2: append multipliers, symmetrize and fill

$K_{11}$	$K_{12}$	0	0	0	0	0	0	0	0	$\lceil u_1 \rceil$	$f_1$
$K_{12}$	$K_{22}$	$K_{23}$	0	0	0	0	1	5	0	$u_2$	$f_2$
0	$K_{23}$	$K_{33}$	$K_{34}$	0	0	0	0	0	3	$u_3$	$f_3$
0	0	$K_{34}$	$K_{44}$	$K_{45}$	0	0	0	0	0	$u_4$	$f_4$
0	0	0	$K_{45}$	$K_{55}$	$K_{56}$	0	0	0	1	$u_5$	$f_5$
0	0	0	0	$K_{56}$		$K_{67}$	-1	0	-4	$u_6$	$f_6$
0	_										
	0	0	0	0	$K_{67}$	$K_{77}$	0	-8	0	$u_7$	$f_7$
0	1	0	0	0	$K_{67}$ -1	$K_{77}$	0	-8 0	0	$u_7$ $\lambda_1$	$\begin{bmatrix} f_7 \\ 0 \end{bmatrix}$
	1 5	0 0 0	0 0 0	0 0 0		<i>K</i> <sub>77</sub> 0 −8	0 0 0			_ ′	_

#### Lagrange Multiplier Method

#### Advantages

- General application
- Exact
- No user decisions: black-box
- Disadvantages
  - Difficult implementation
  - Additional unknowns
  - Loses positive definiteness
  - Sensitive to constraint dependence

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## MFC Application Methods: Summary

	Master-slave Elimination	Penalty function	Lagrange multiplier
Physical interpretation	Model reduction	Penalty element (flexible link)	Rigid link (reaction force)
Generality	fair	Excellent	Excellent
Ease of implementation	Poor to fair	Good	Fair
Sensitivity to user decisions	High	High	Small to none
Accuracy	Variable	Mediocre	Excellent
Sensitivity as regards constraint dependence	High	None	High
Retains positive definiteness	Yes	Yes	No
Modifies unknown vector	Yes	No	Yes

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