

Solid Mechanics (plate/shell, 3D)

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CONTENTS

- **Plate and Shell Model**
 - **Thick plate**
 - **Hood**
- **3D Model**

SHELLS AND PLATES

SHELLS AND PLATES

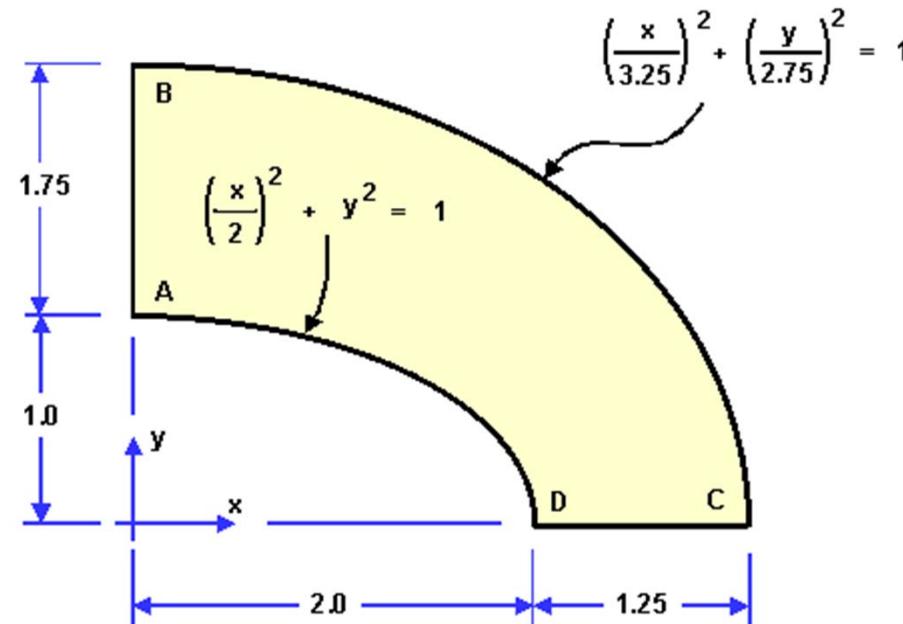
The Shell interface () is intended for the structural analysis of thin-walled structures. The formulation used in the Shell interface is a Mindlin-Reissner type, which means that transverse shear deformations are accounted for, and it can therefore be used for rather thick shells. It is also possible to prescribe an offset in the direction normal to a selected surface. The Shell interface also includes other features such as damping, thermal expansion, and initial stresses and strains. The preset studies available are the same as for the Solid Mechanics interface.

The Plate interface () is the 2D analogy to the 3D Shell interface. Plates are similar to shells but act in a single plane and usually only with out-of-plane loads.

NAFEMS BENCHMARK: LE10

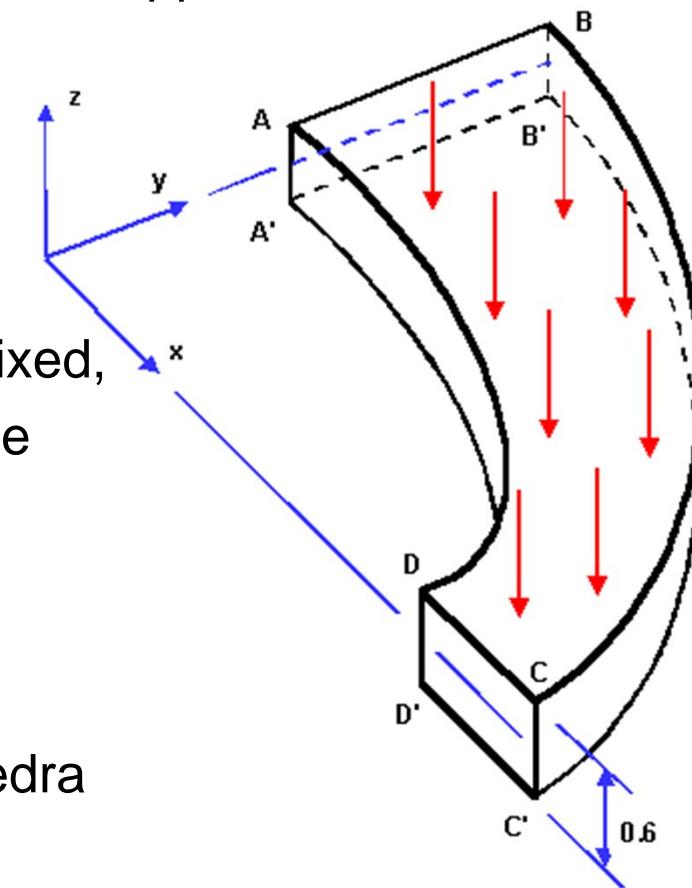
Thick Plate – Pressure

- Analysis Type: Linear elastic solid vs. plate
- Geometry: meters, thickness 0.6

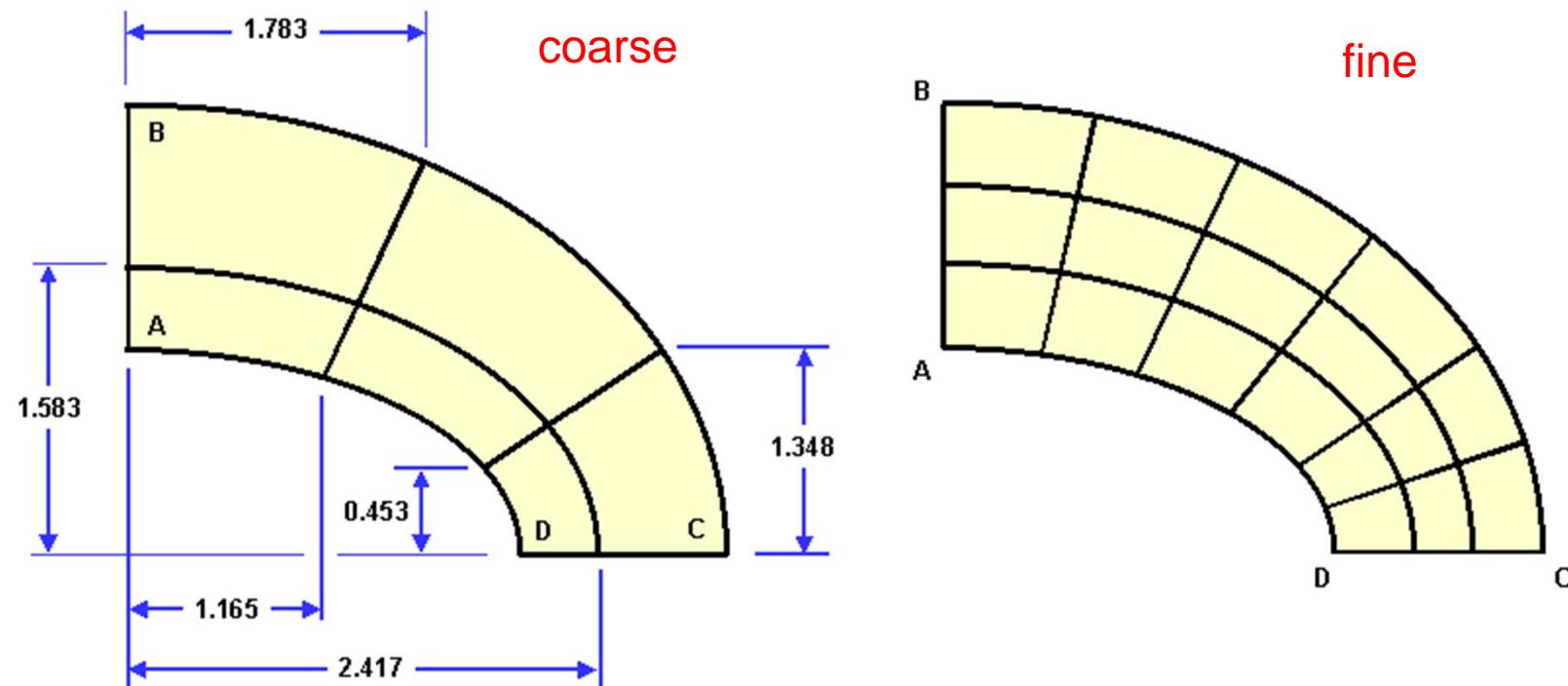


COMSOL Solid Mechanics Module Verification Models: Thick Plate Stress Analysis

- Loading
 - Uniform normal pressure of 1 MPa on the upper surface of the plate
- Boundary conditions
 - Face DCD'C': zero y-displacement
 - Face ABA'B': zero x-displacement
 - Face BCB'C': x and y-displacement fixed, z-displacements fixed along mid-plane
- Material properties
 - Isotropic: $E = 210 * 10^3$ MPa, $\nu = 0.3$
- Element types
 - Solid hexahedra, wedges and tetrahedra



- Meshes

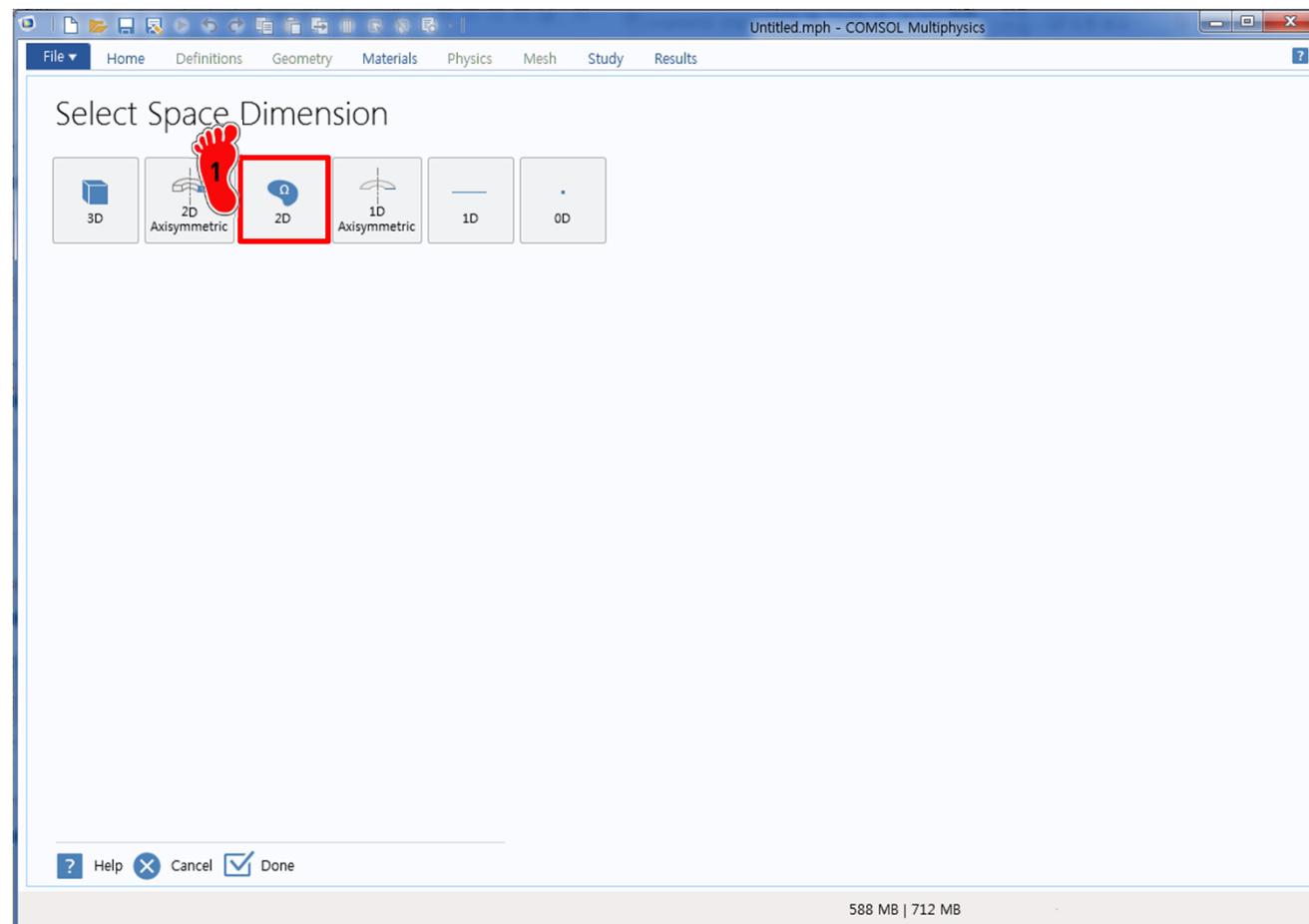


- Output
 - Direct stress (σ_{yy}) at point D: -5.38 MPa

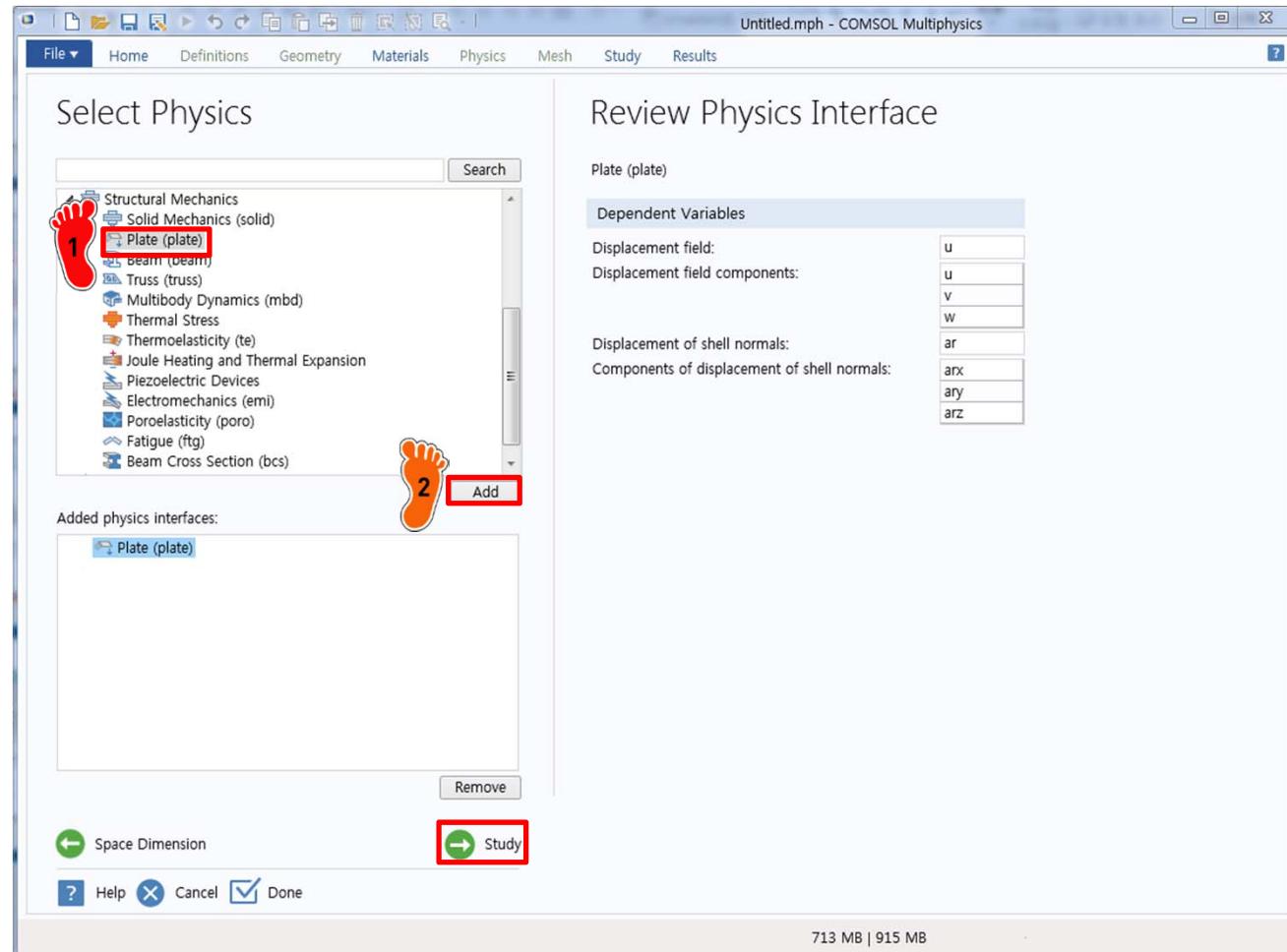
LE10 COMPARISON

| LE 10 | order | integration points | element | coarse mesh | | fine mesh | |
|----------|-------|--------------------|------------|-------------|--------|-----------|-------|
| | | | | s22 | error | s22 | error |
| Abaqus * | 2 | 8 | C3D20R | -7.93E+06 | 47.40% | -5.53E+06 | 2.79% |
| Ansys | 2 | 8 | SOLID95 | -5.36E+06 | -0.32% | -5.61E+06 | 4.26% |
| CalculiX | 2 | 8 | C3D20R | -5.36E+06 | -0.32% | -5.61E+06 | 4.26% |
| Tahoe | 2 | 8 | hexahedron | | | | |
| Abaqus * | 2 | 27 | C3D20 | -6.72E+06 | 24.91% | -5.64E+06 | 4.83% |
| Ansys | 2 | 14 | SOLID95 | -5.40E+06 | 0.46% | -5.61E+06 | 4.33% |
| CalculiX | 2 | 27 | C3D20 | -5.20E+06 | -3.32% | -5.50E+06 | 2.18% |
| Tahoe | 2 | 27 | hexahedron | | | | |

DIMENSION SELECTION

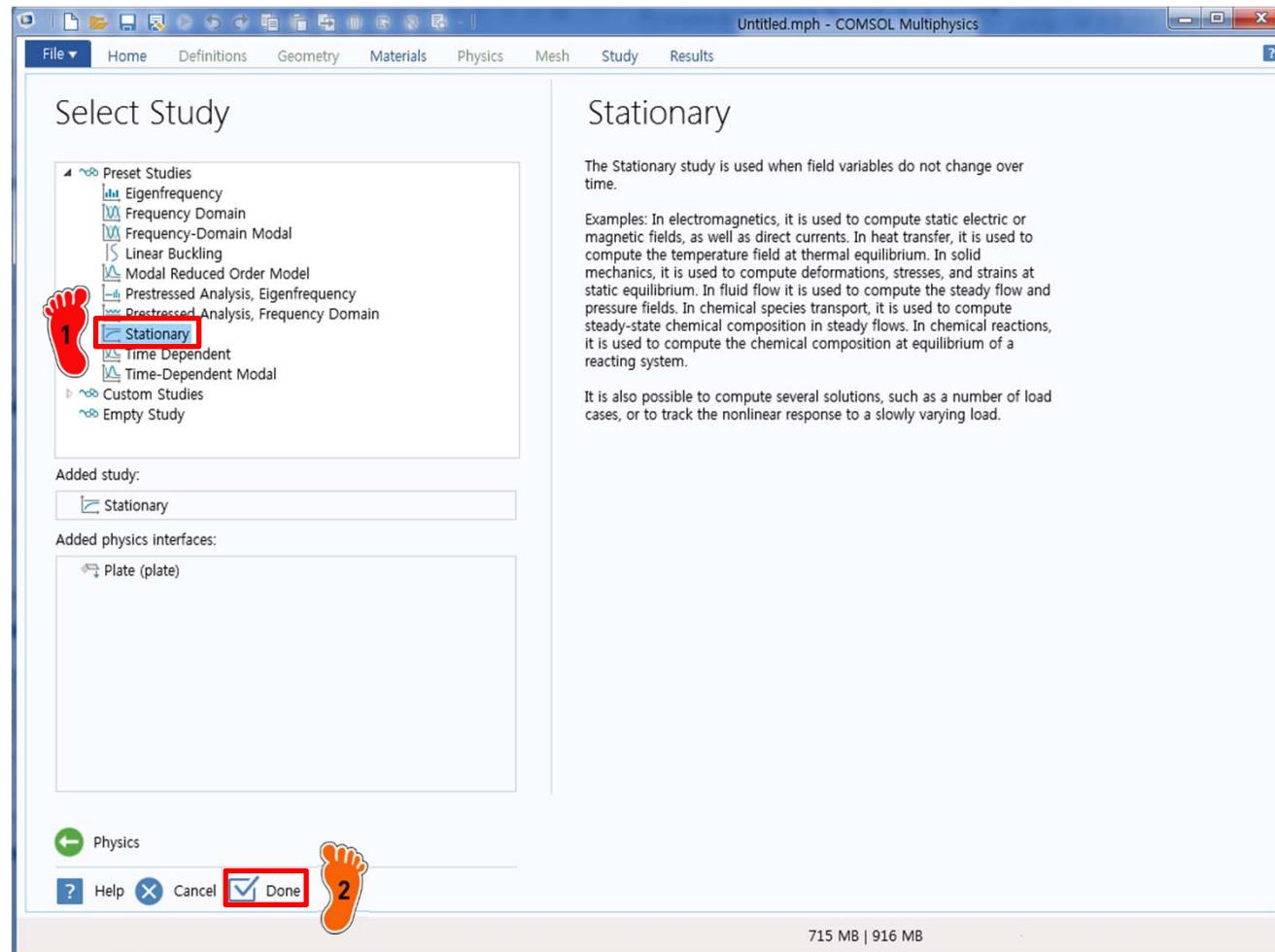


PHYSICS SELECTION

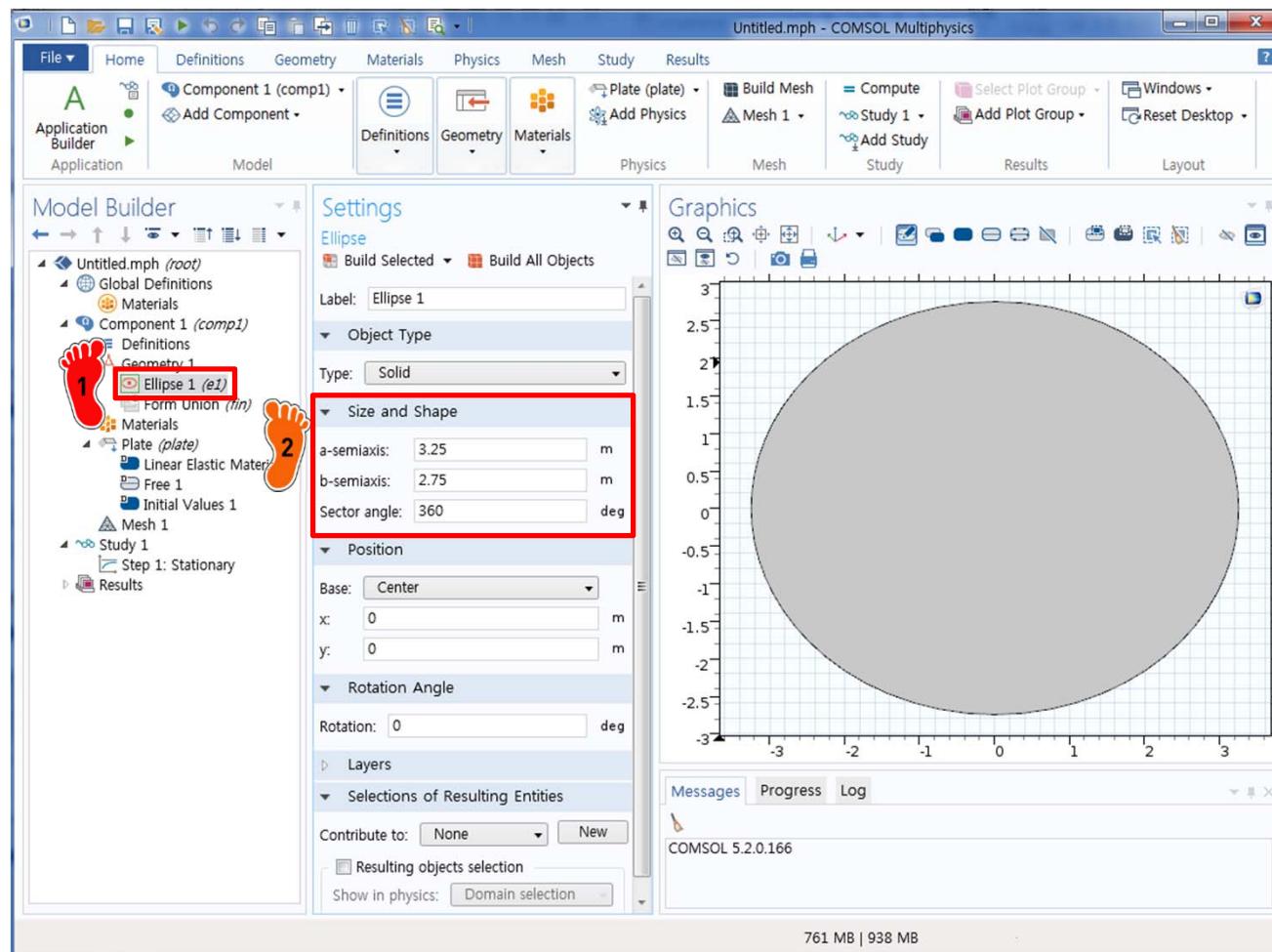


- 1 Structural Mechanics 의 Plate 선택
- 2 Add 클릭
- 3 Study 클릭

STUDY TYPE SELECTION



GEOMETRY CREATION

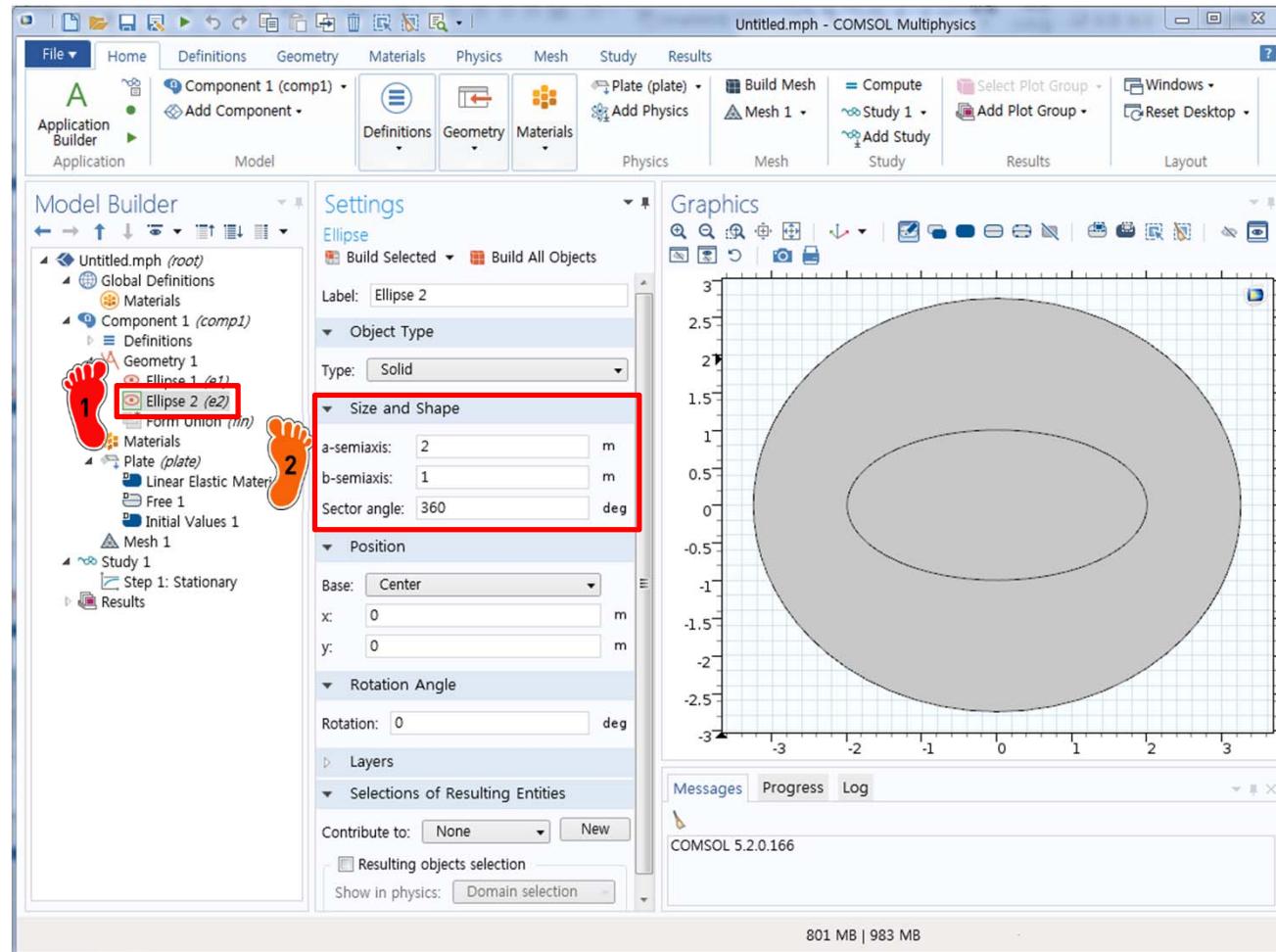


1 Ellipse 1 생성

2 a-semiaxis: 3.25
b-semiaxis: 2.75

입력 후 타원 생성

GEOMETRY CREATION



1 Ellipse 2 생성

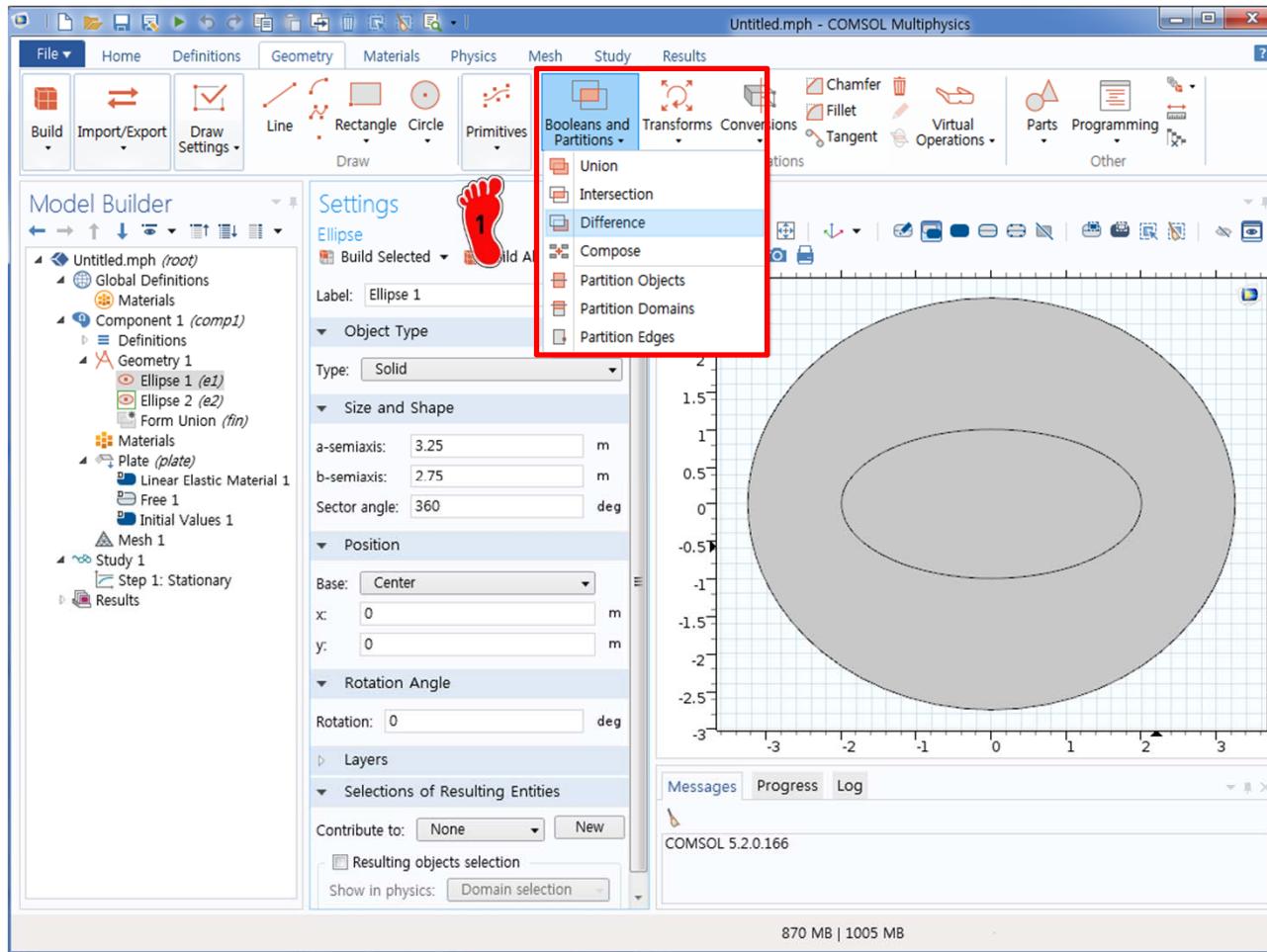
2 a-semiaxis: 2
b-semiaxis: 1

입력 후 타원 생성

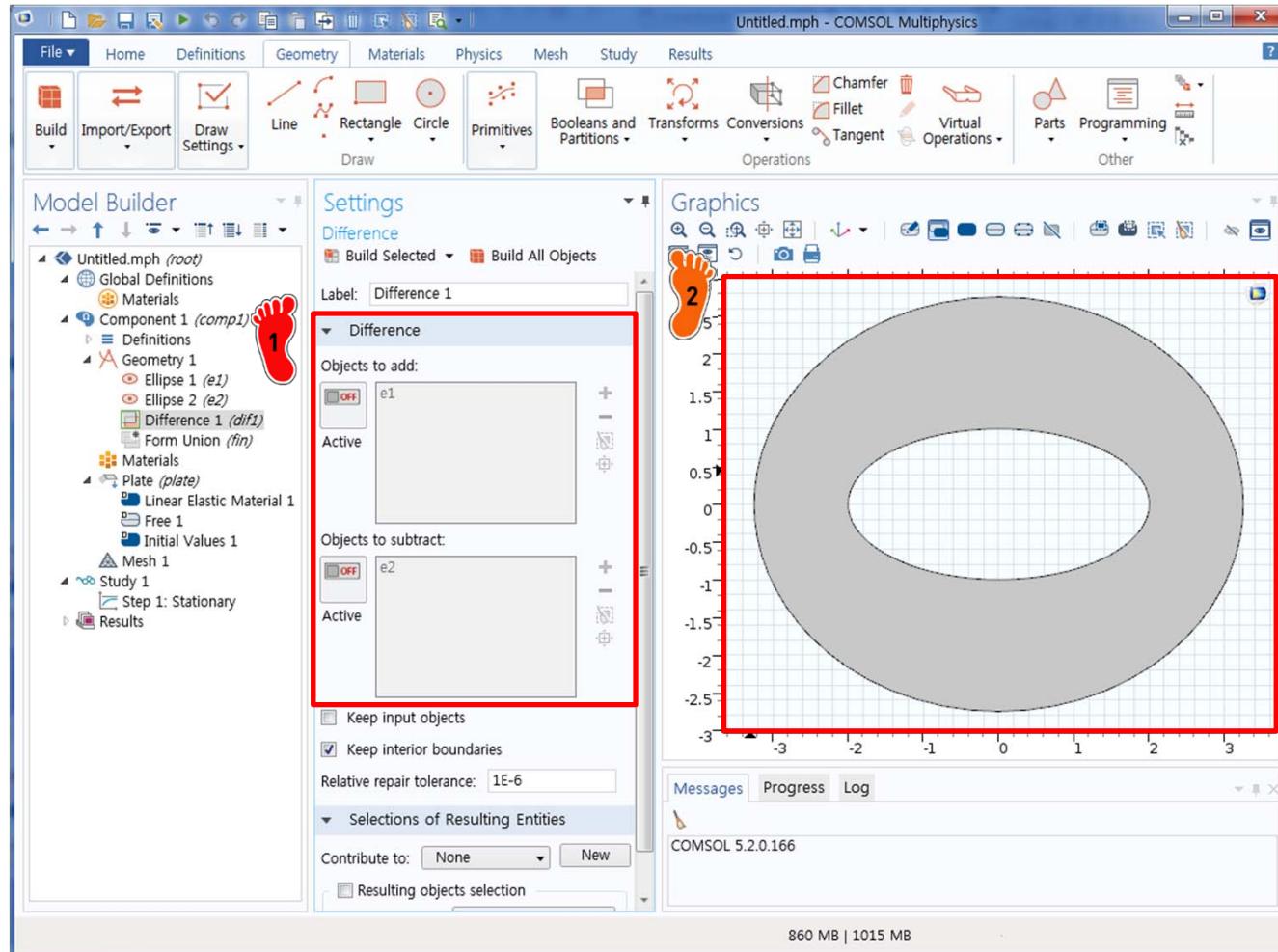
GEOMETRY CREATION



Booleans and Partitions에서 Difference를 선택



GEOMETRY CREATION



Objects to add에서

바깥 타원 선택

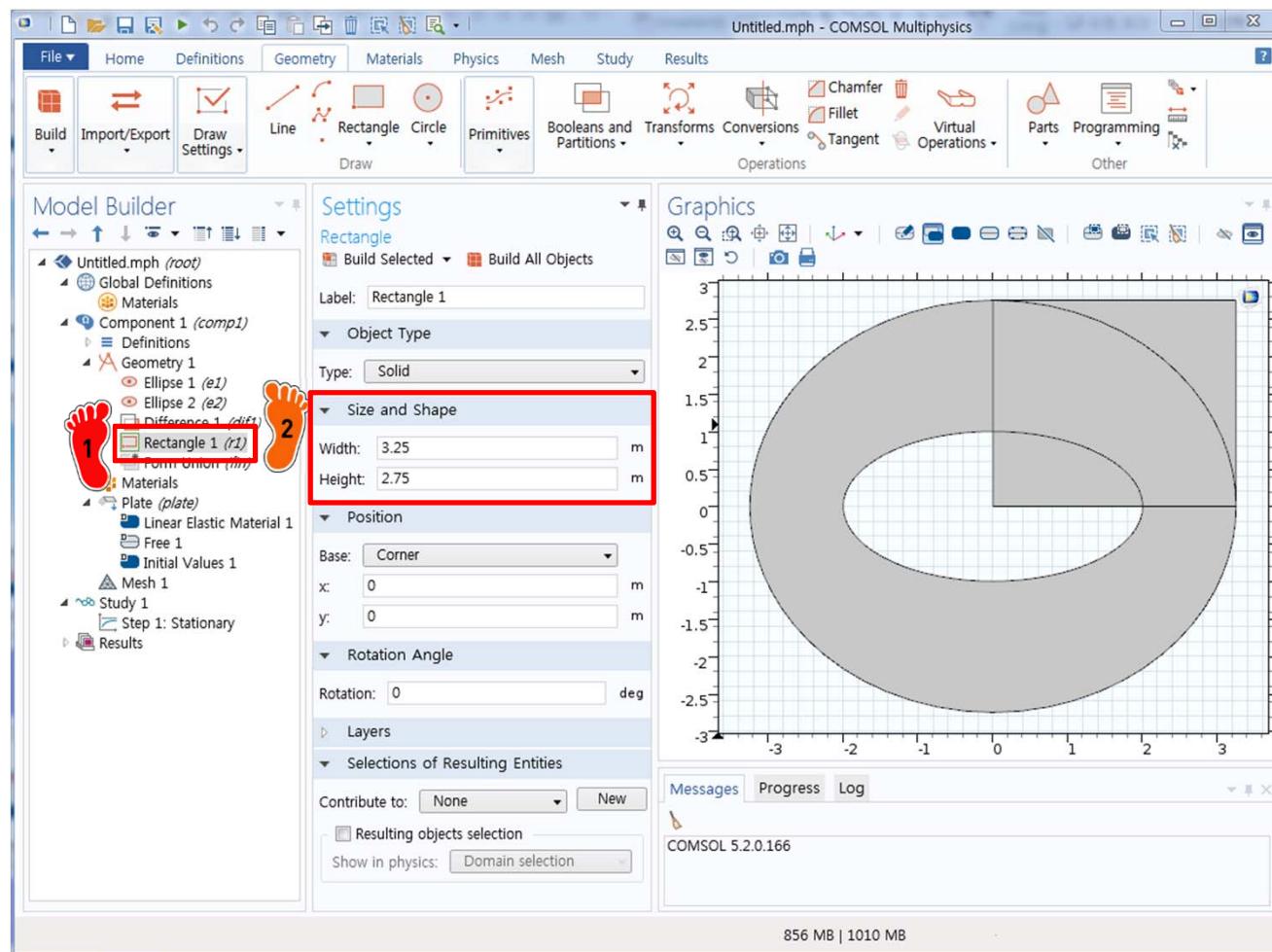
Objects to subtract에서

안쪽 타원 선택



타원 구멍 생성

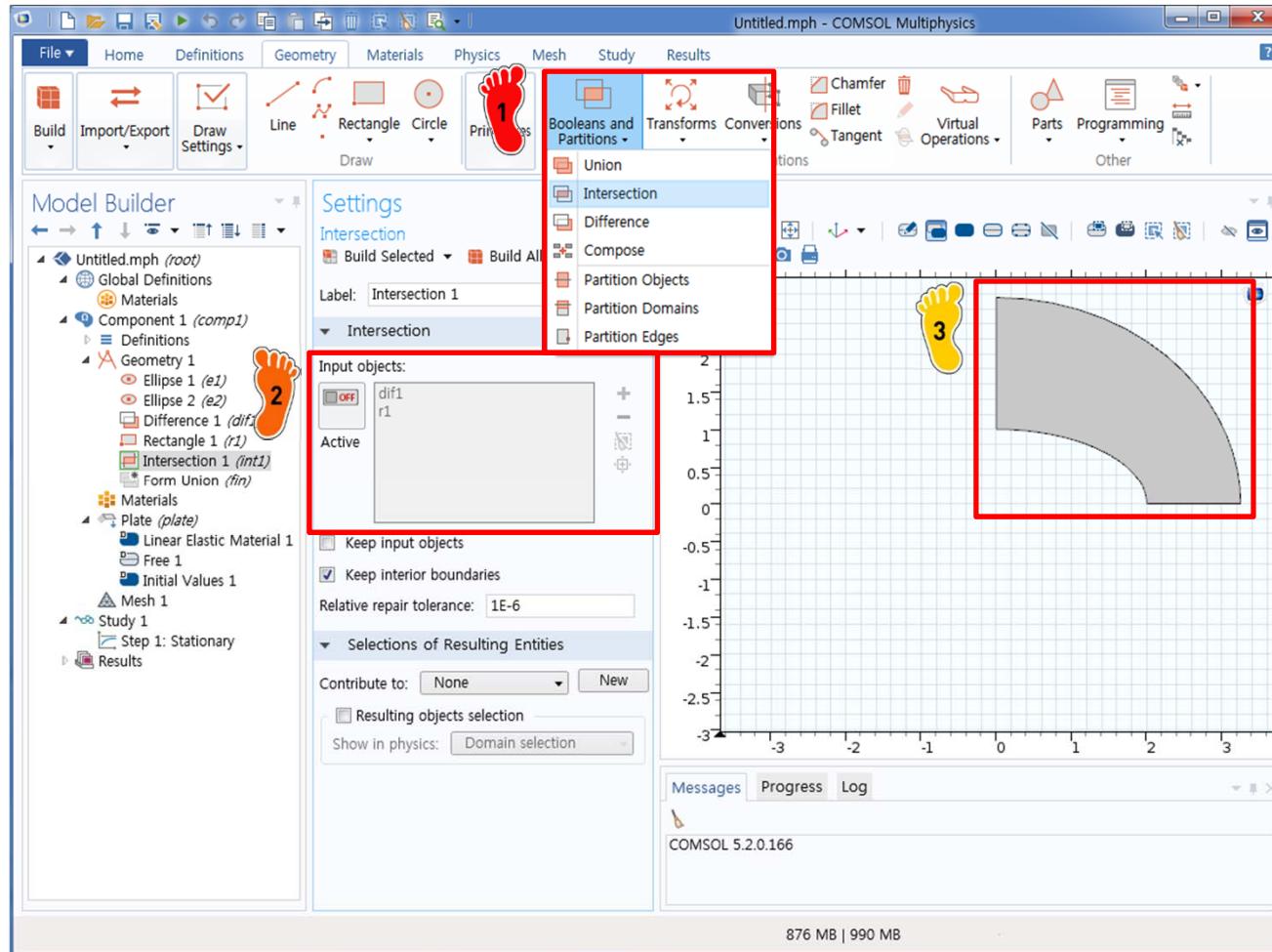
GEOMETRY CREATION



- 1 Rectangle 1 생성
- 2 Width: 3.25
Height: 2.75

입력 후 사각형 생성

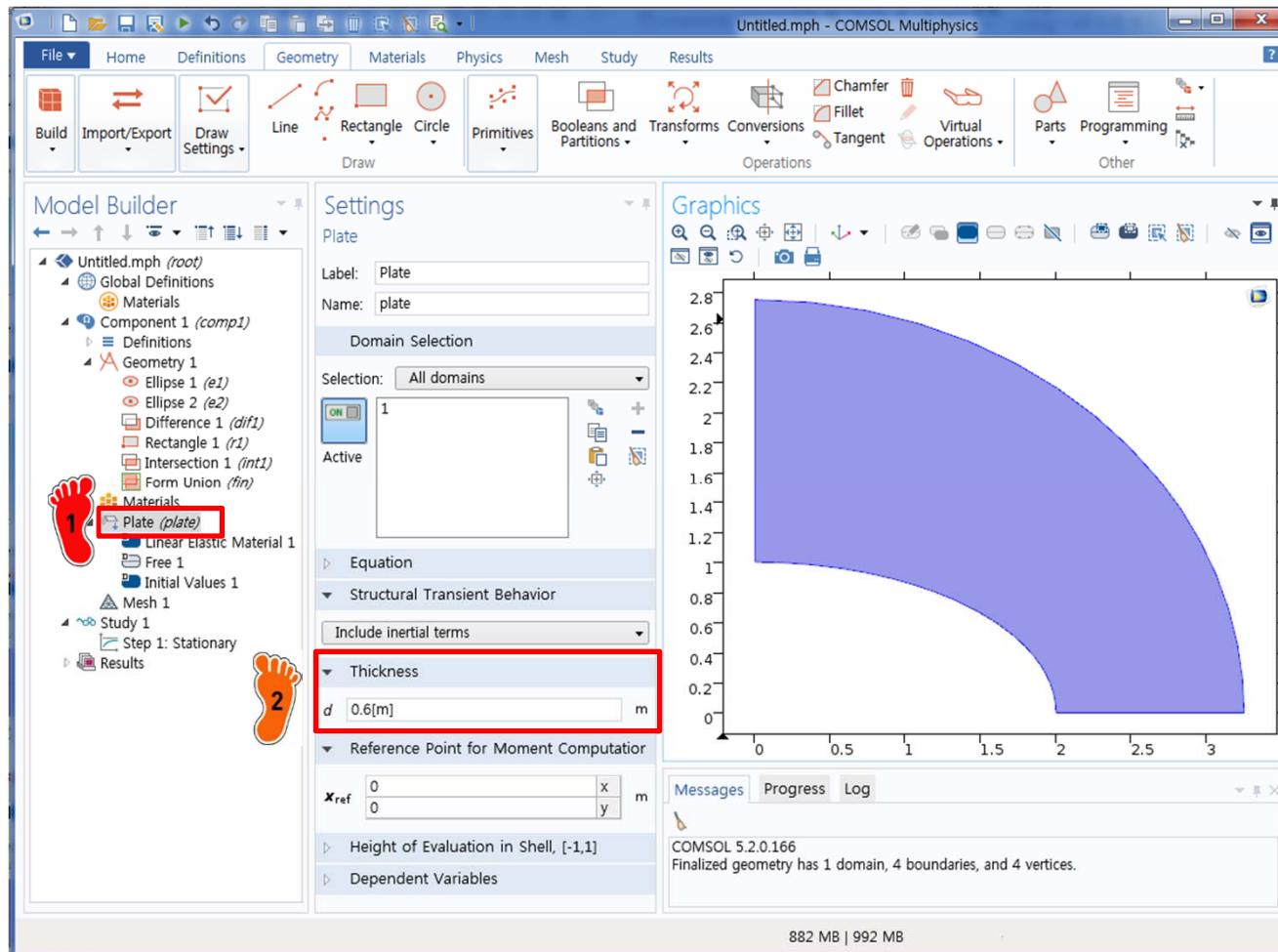
GEOMETRY CREATION



- 1 Booleans and Partitions에서 Intersection 선택
- 2 Input object에 두 형상 선택

3 다음과 같은 형상 생성

THICKNESS



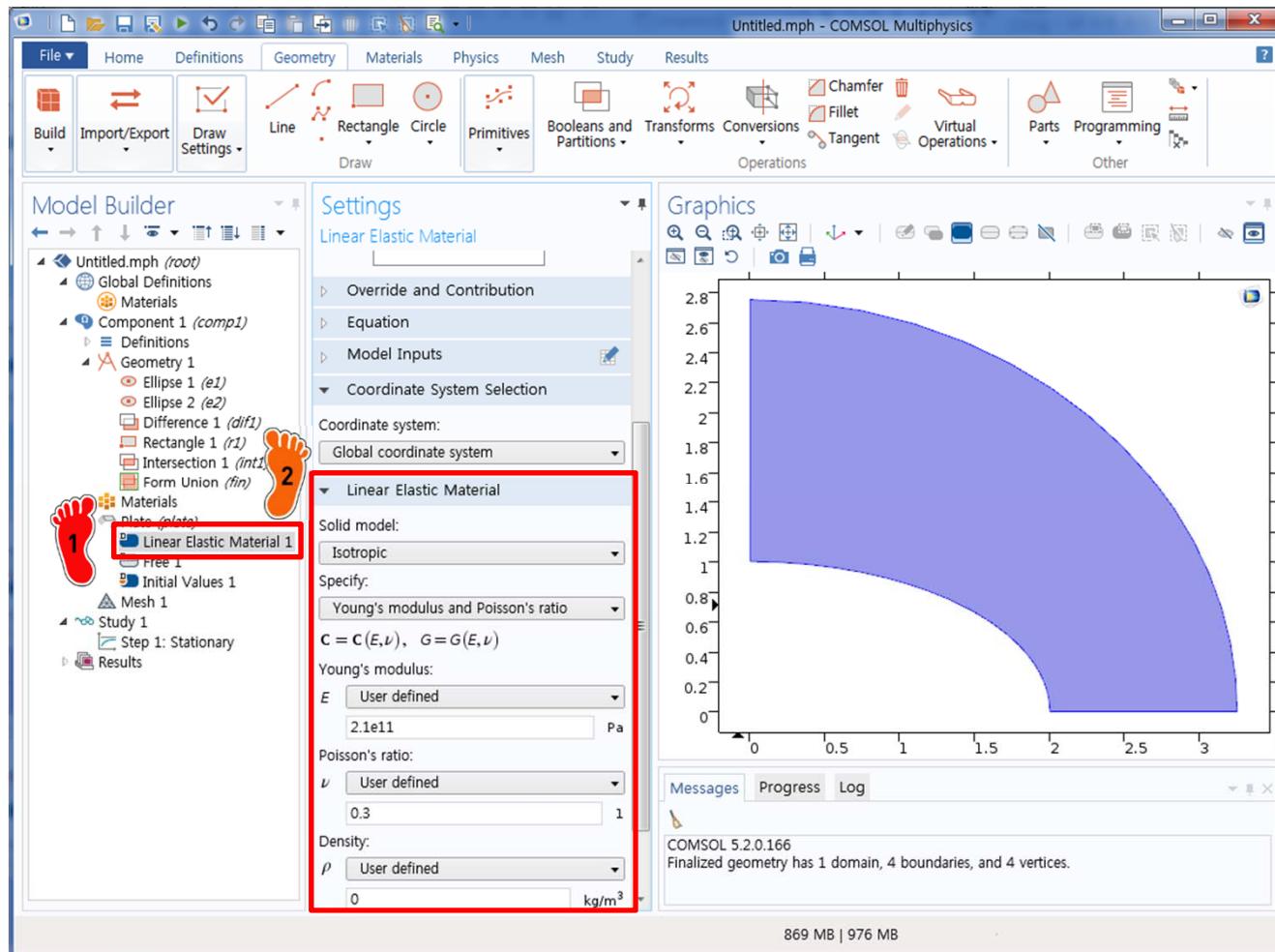
1 Plate 클릭

2 두께 d: 0.6

입력

요소 차수는
quadratic(default)

MATERIAL PROPERTY

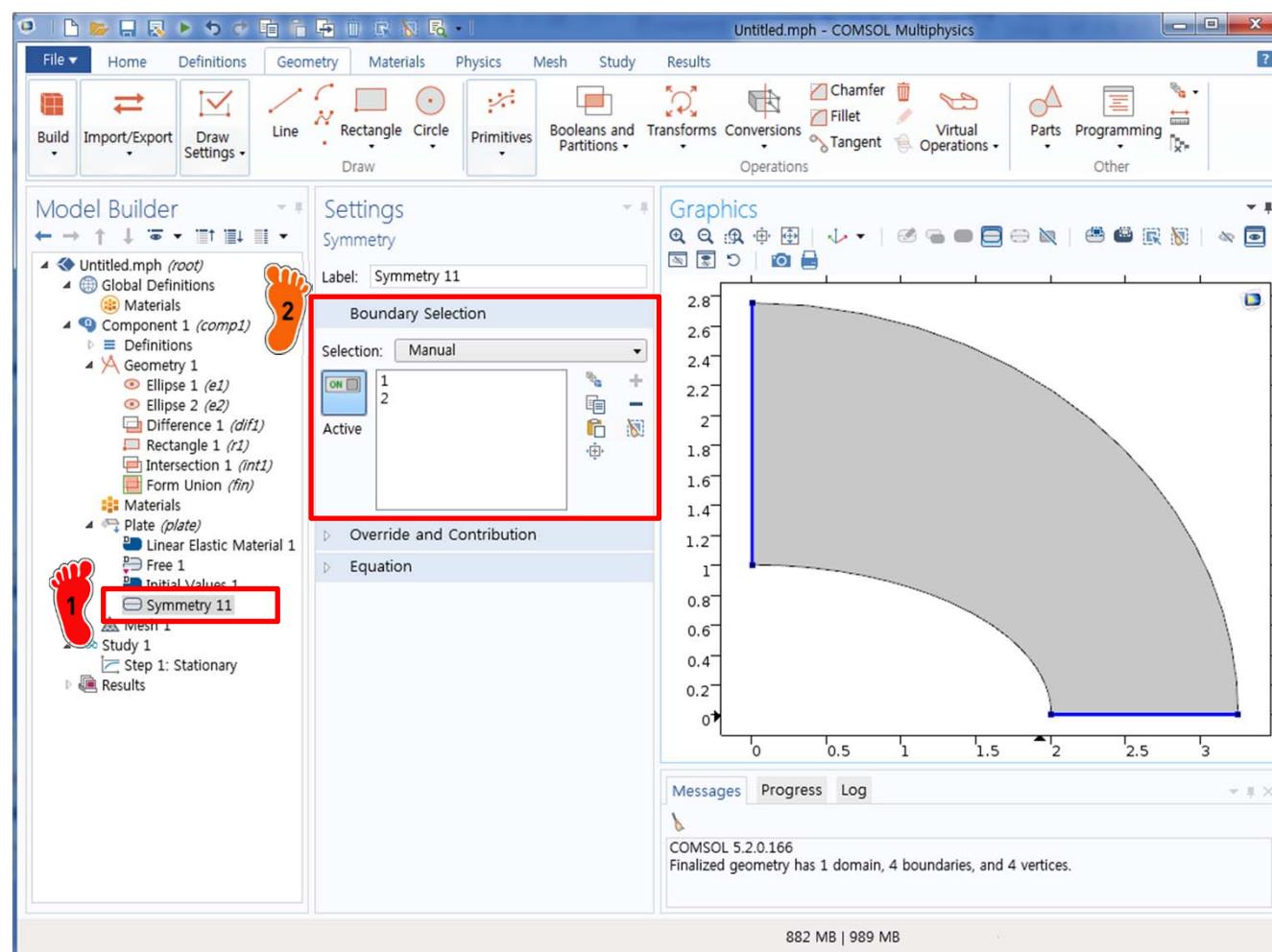


1 Linear Elastic Material 1
클릭

2 E: 2.1e11
mu: 0.3
rho : 0

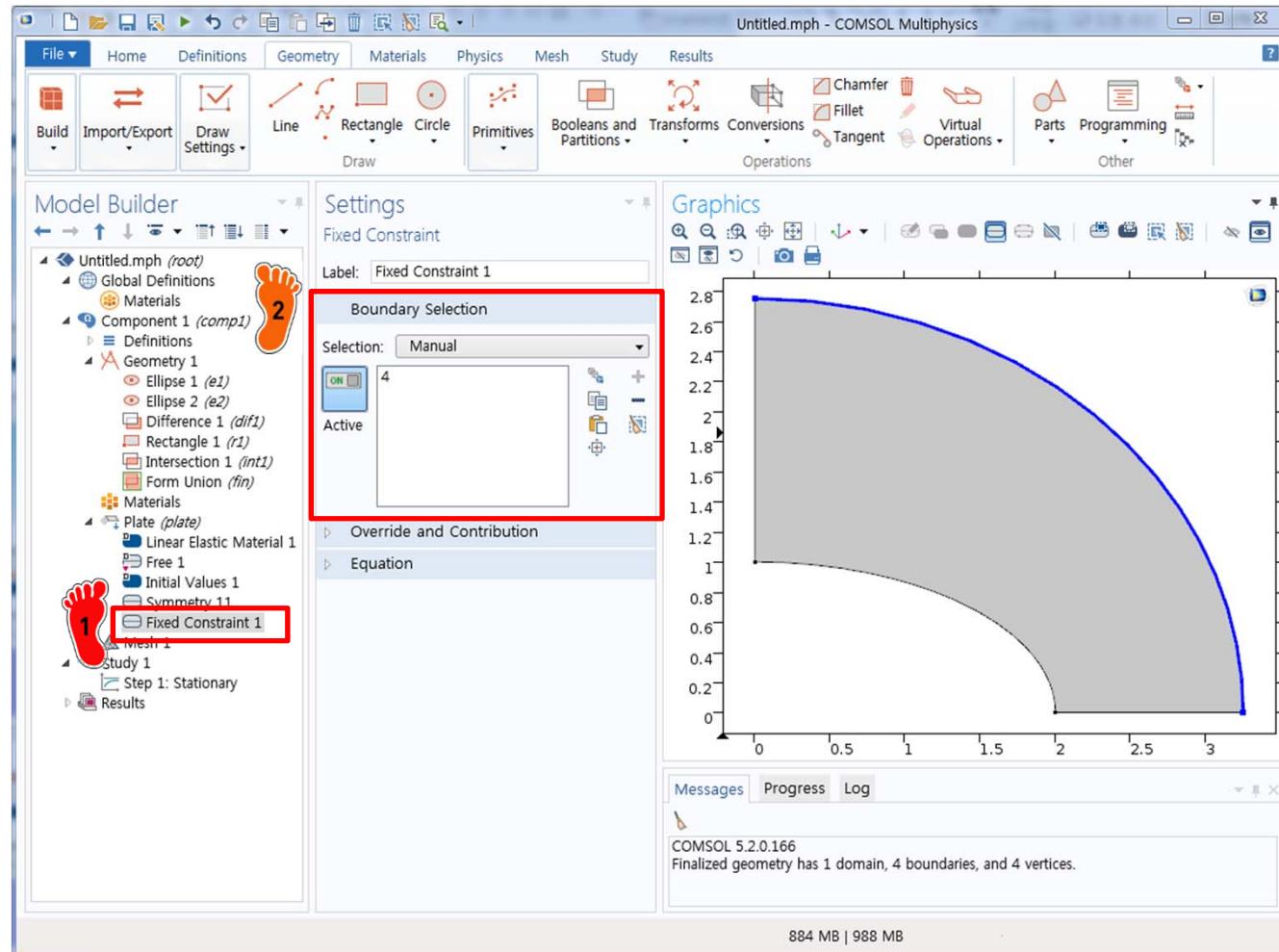
입력

BOUNDARY CONDITION



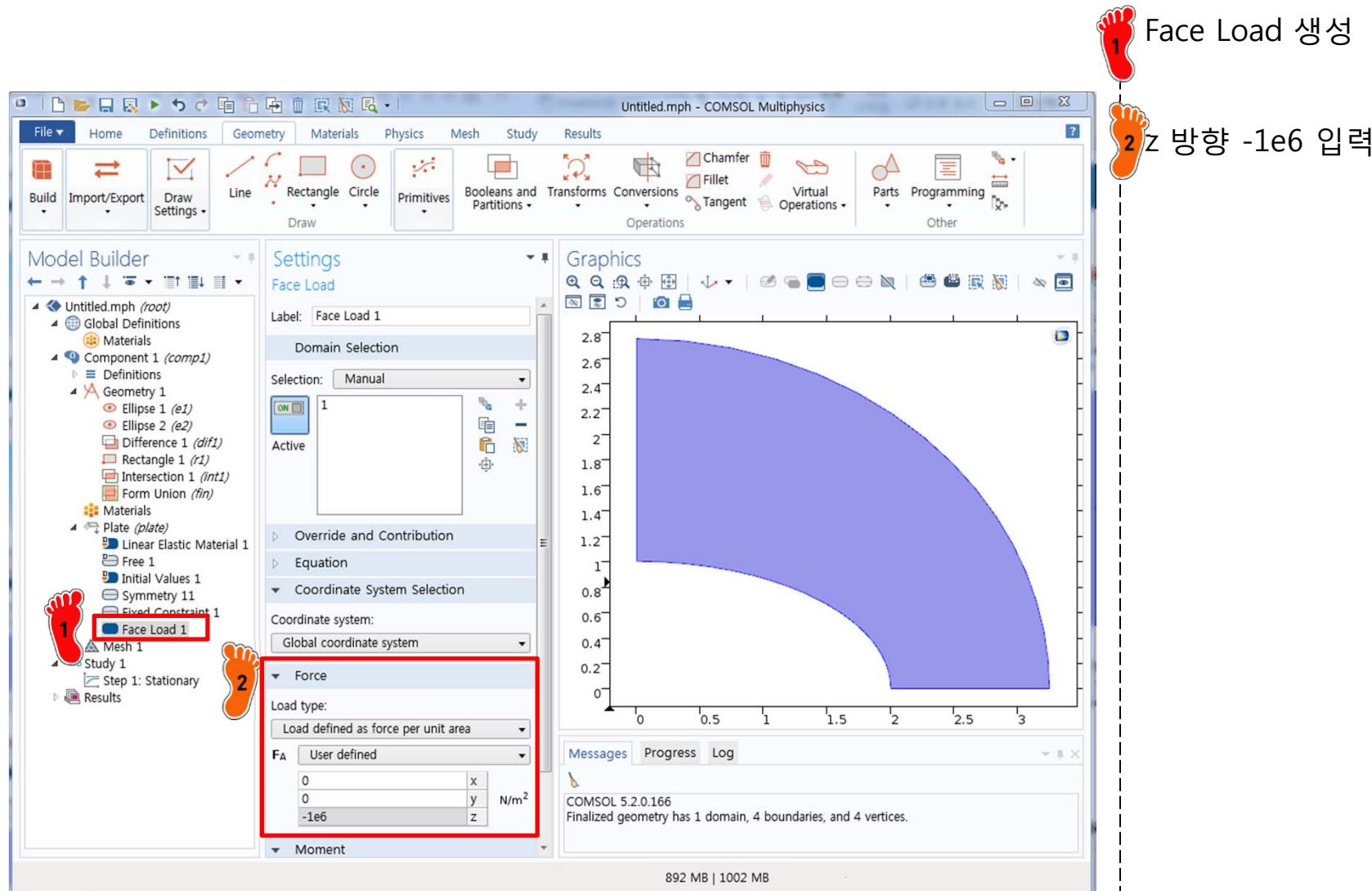
- 1 Symmetry 경계조건 생성
- 2 1, 2번 경계 선택

BOUNDARY CONDITION



- 1 Fixed 경계조건 생성
- 2 4번 경계 선택

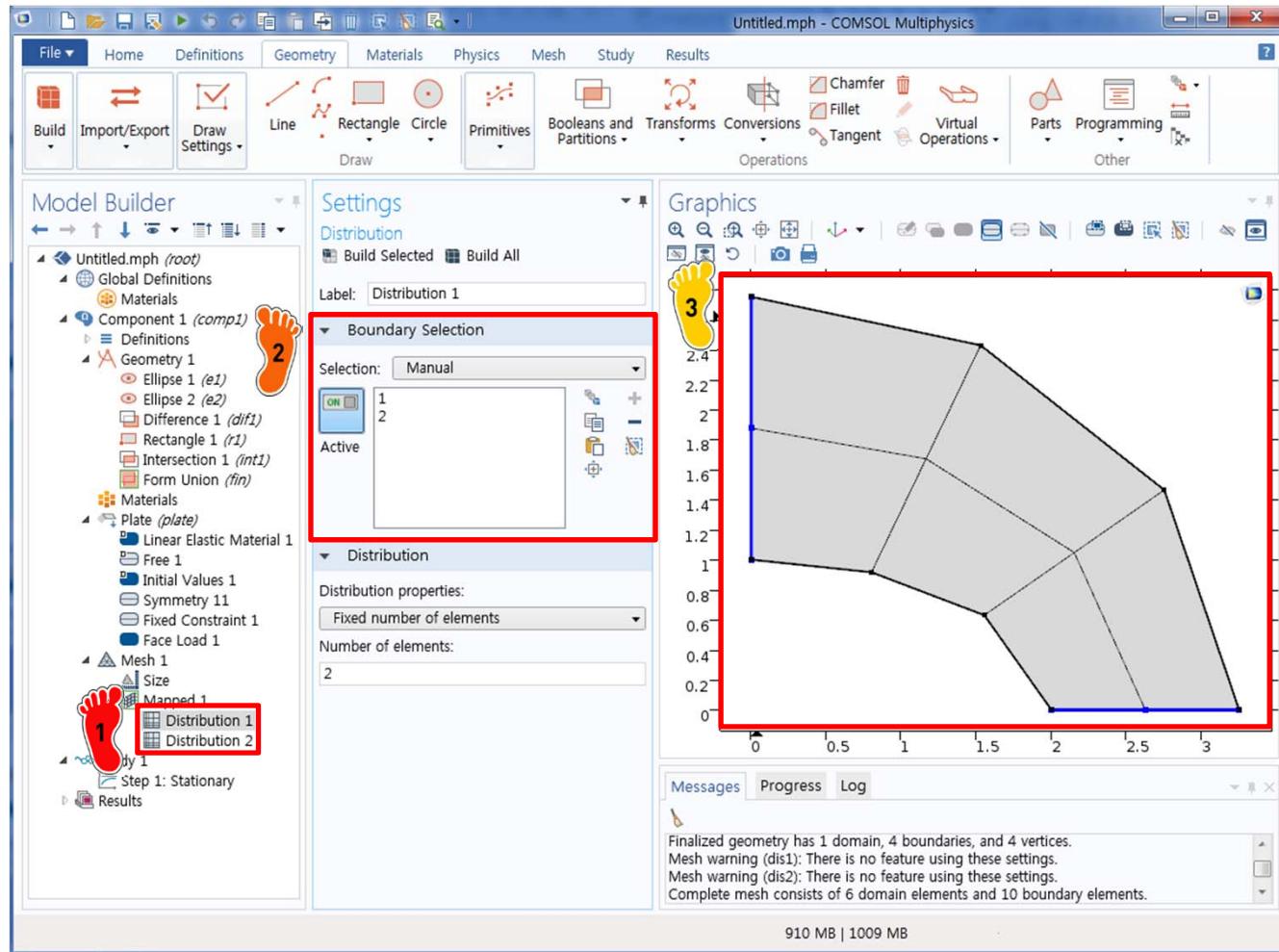
LOADING CONDITION



1 Face Load 생성

2 z 방향 $-1e6$ 입력

MESH



1 Mesh1에서 Mapped를 생성하고 Distribution1,2 생성

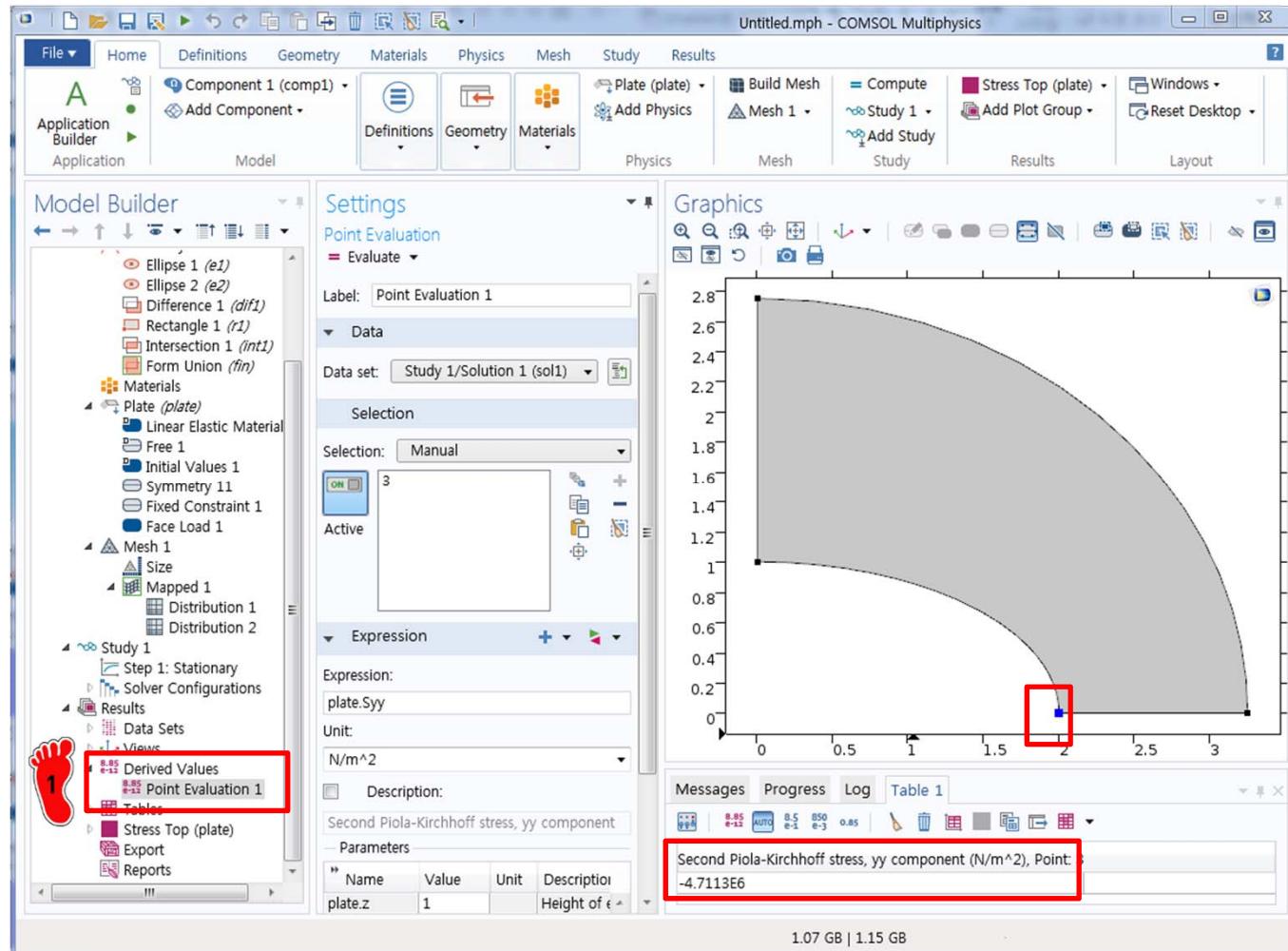
2 Distribution1에서 1,2번 경계 선택
Number of elements에 2 입력

Distribution2에서 3,4번 경계 선택
Number of elements에 3 입력

다음과 같은 격자 생성

그 후, 해석 실행

POST-PROCESSING

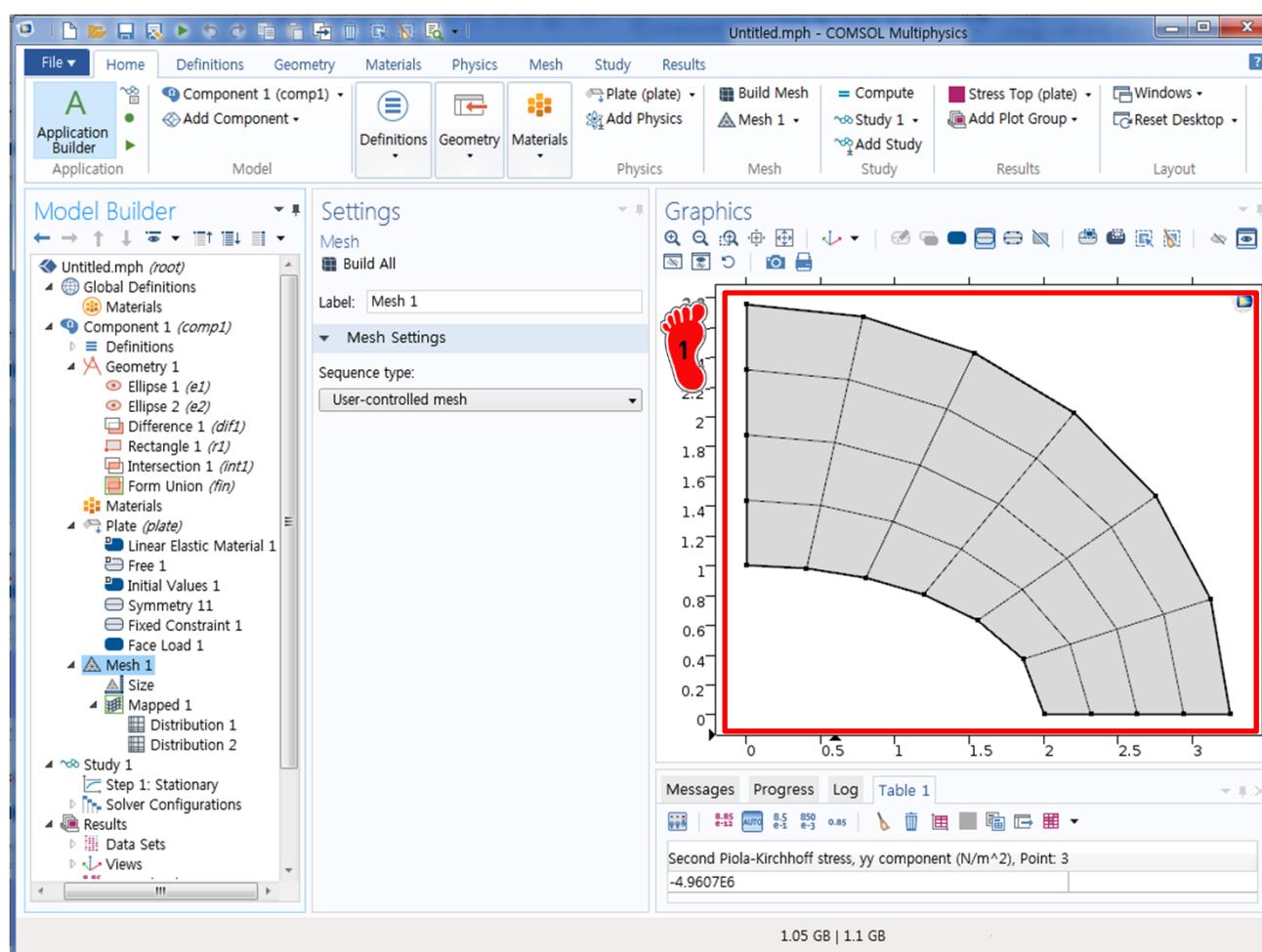


1 Derived Values 의 Point Evaluation 기능을 이용하여 3번 절점의 plate.Syy 응력을 계산

plate.Syy: Second Piola-Kirchhoff stress, yy component

결과값: -4.7113 MPa

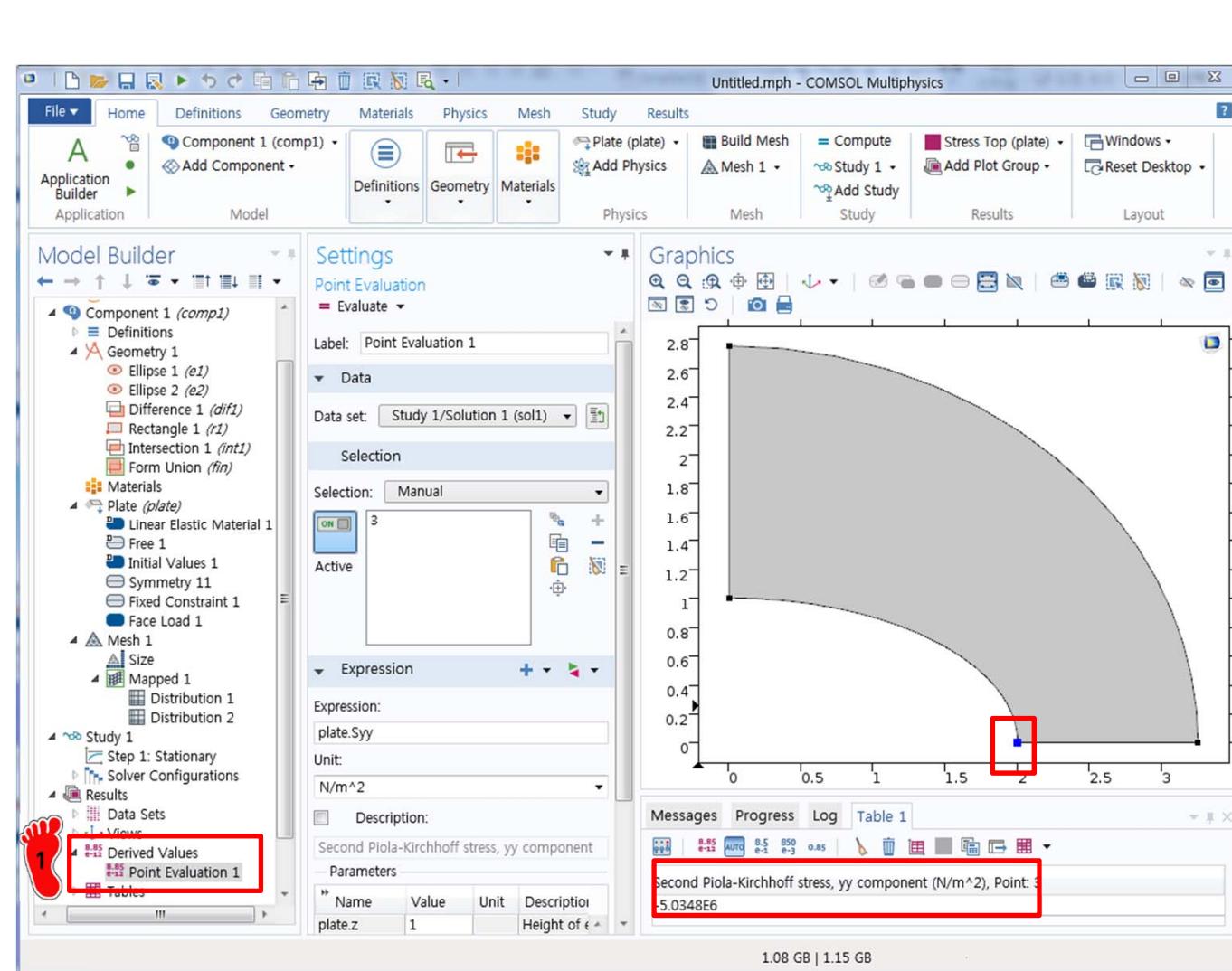
MESH



1 Fine 메시로 변경

그 후, 해석실행

POST-PROCESSING



1 Point Evaluation 1 계산

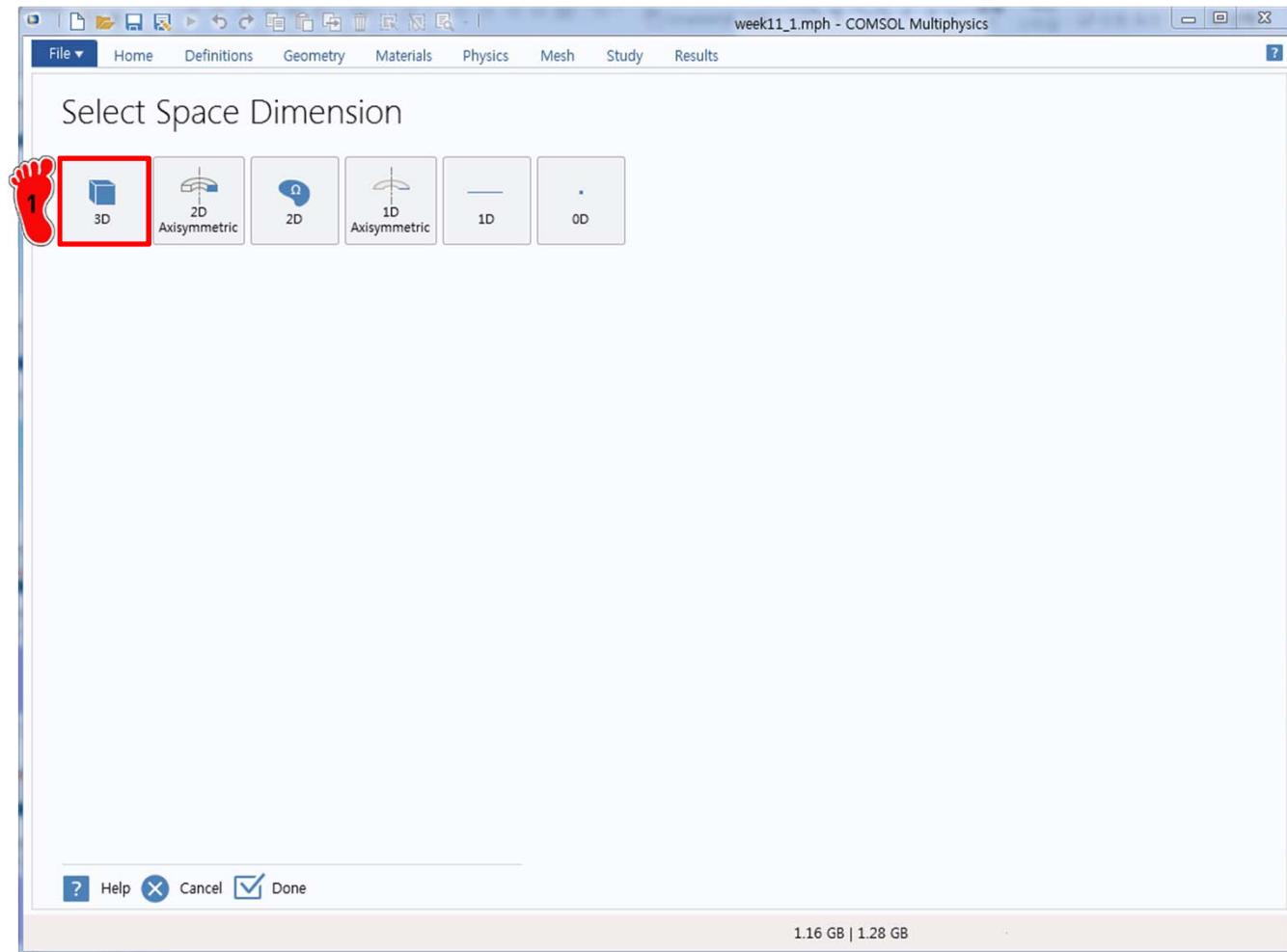
결과값: -5.0348 MPa

SUMMARY

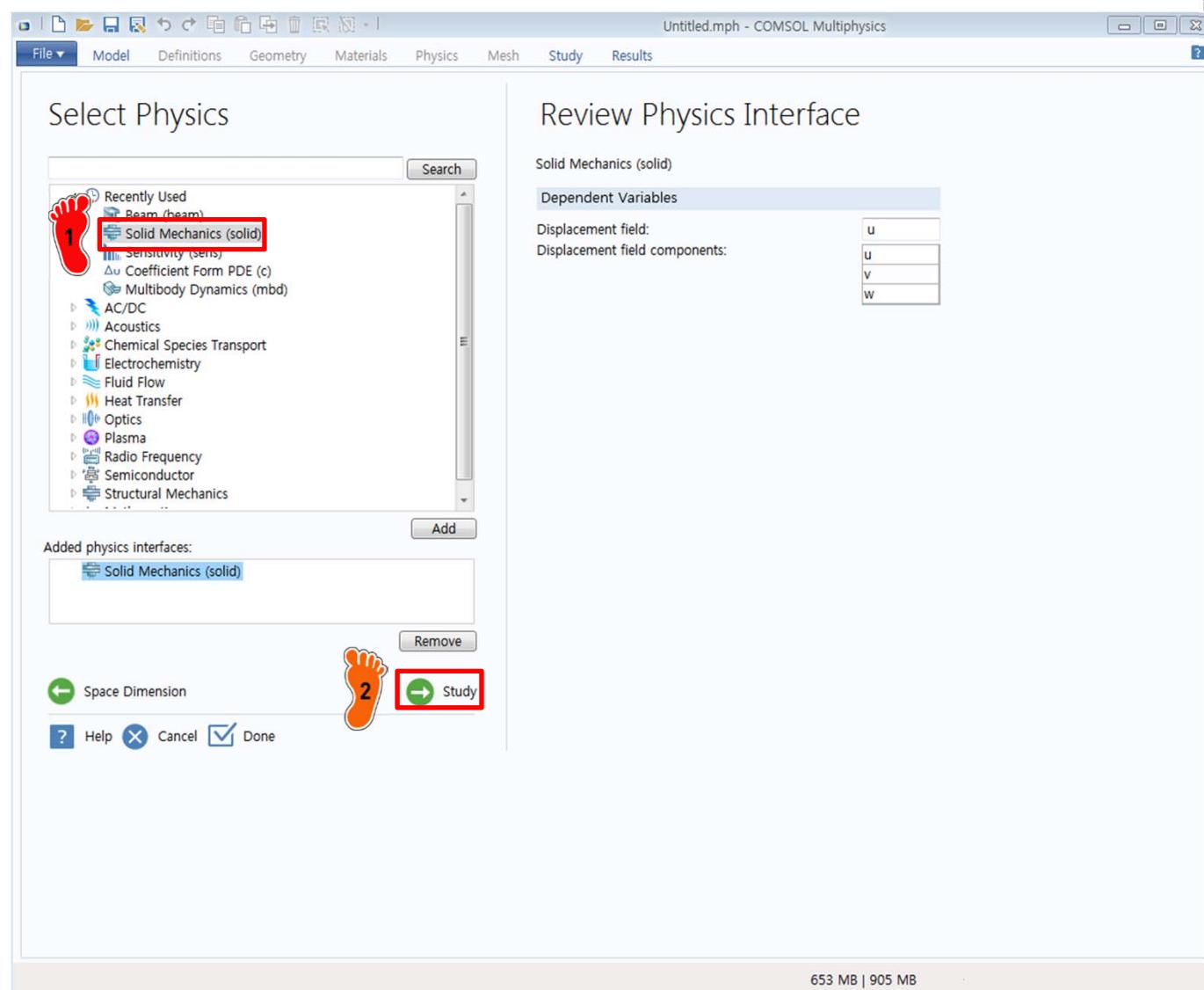
| LE 10 | order | integration points | element | coarse mesh | | fine mesh | |
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| Abaqus * | 2 | 8 | C3D20R | -7.93E+06 | 47.40% | -5.53E+06 | 2.79% |
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| Tahoe | 2 | 8 | hexahedron | | | | |
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| Ansys | 2 | 14 | SOLID95 | -5.40E+06 | 0.46% | -5.61E+06 | 4.33% |
| CalculiX | 2 | 27 | C3D20 | -5.20E+06 | -3.32% | -5.50E+06 | 2.18% |
| Tahoe | 2 | 27 | hexahedron | | | | |

| | | | | | | | |
|--------|---|--|------------|-----------|--------|-----------|-------|
| COMSOL | 2 | | plate | -4.71E+06 | 12.45% | -5.03E+06 | 6.51% |
| COMSOL | 2 | | hexahedron | | | | |

DIMENSION SELECTION



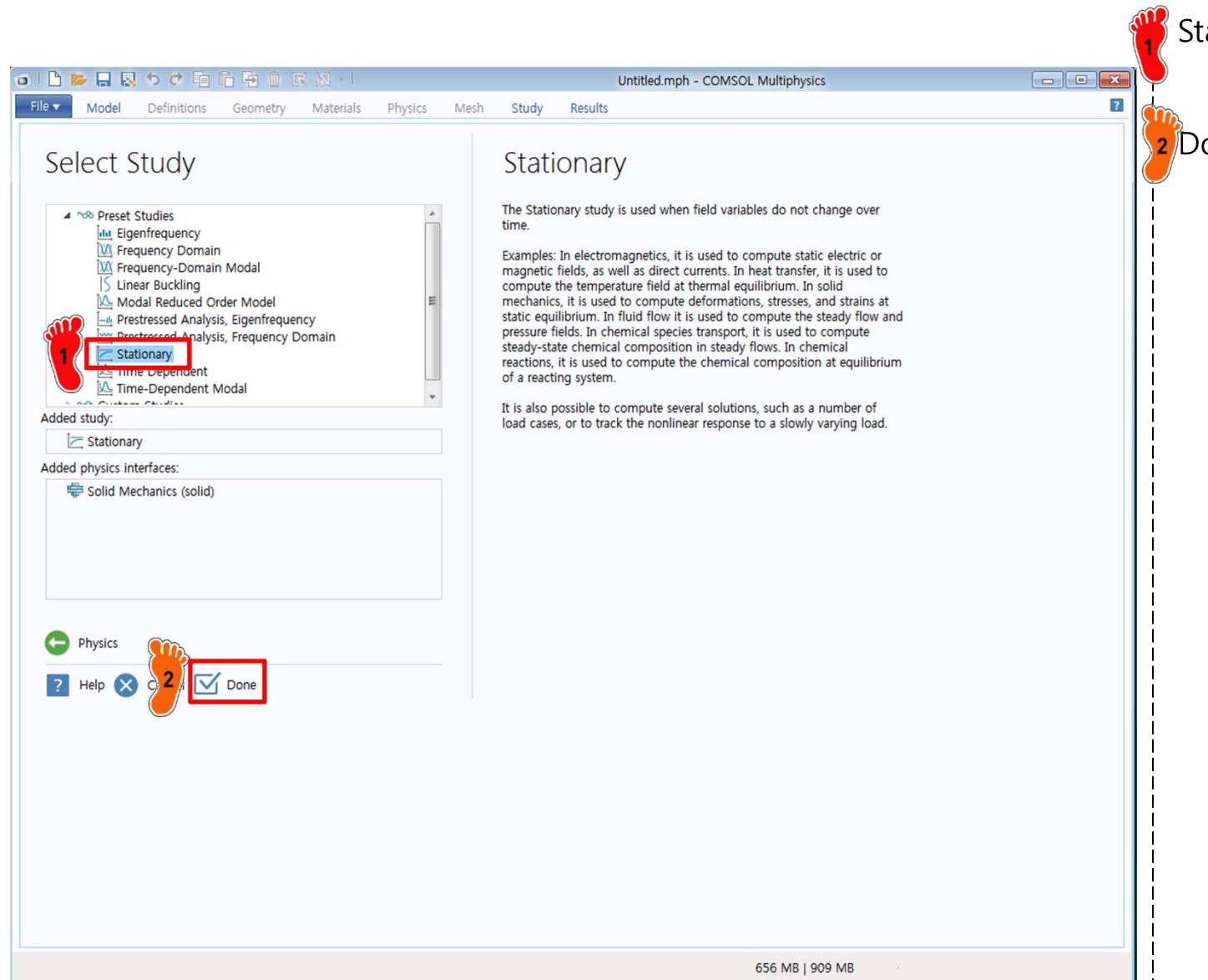
PHYSICS SELECTION



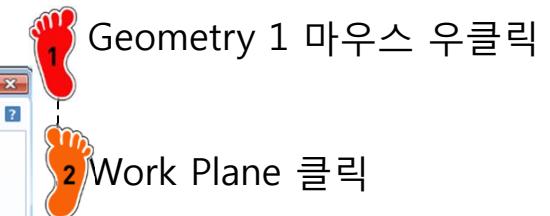
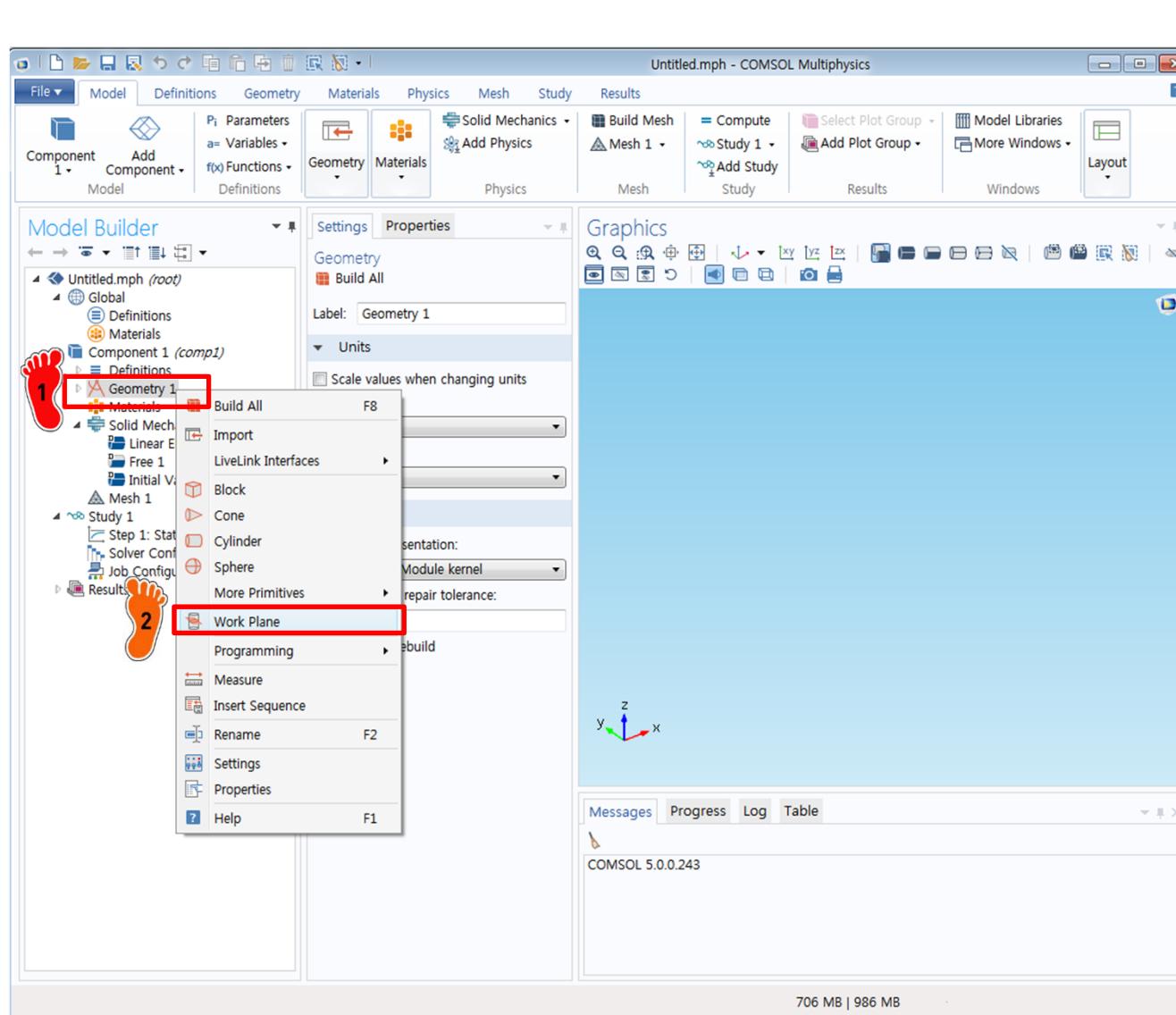
1 Structural Mechanics 의
Solid Mechanics 선택

2 Study 클릭

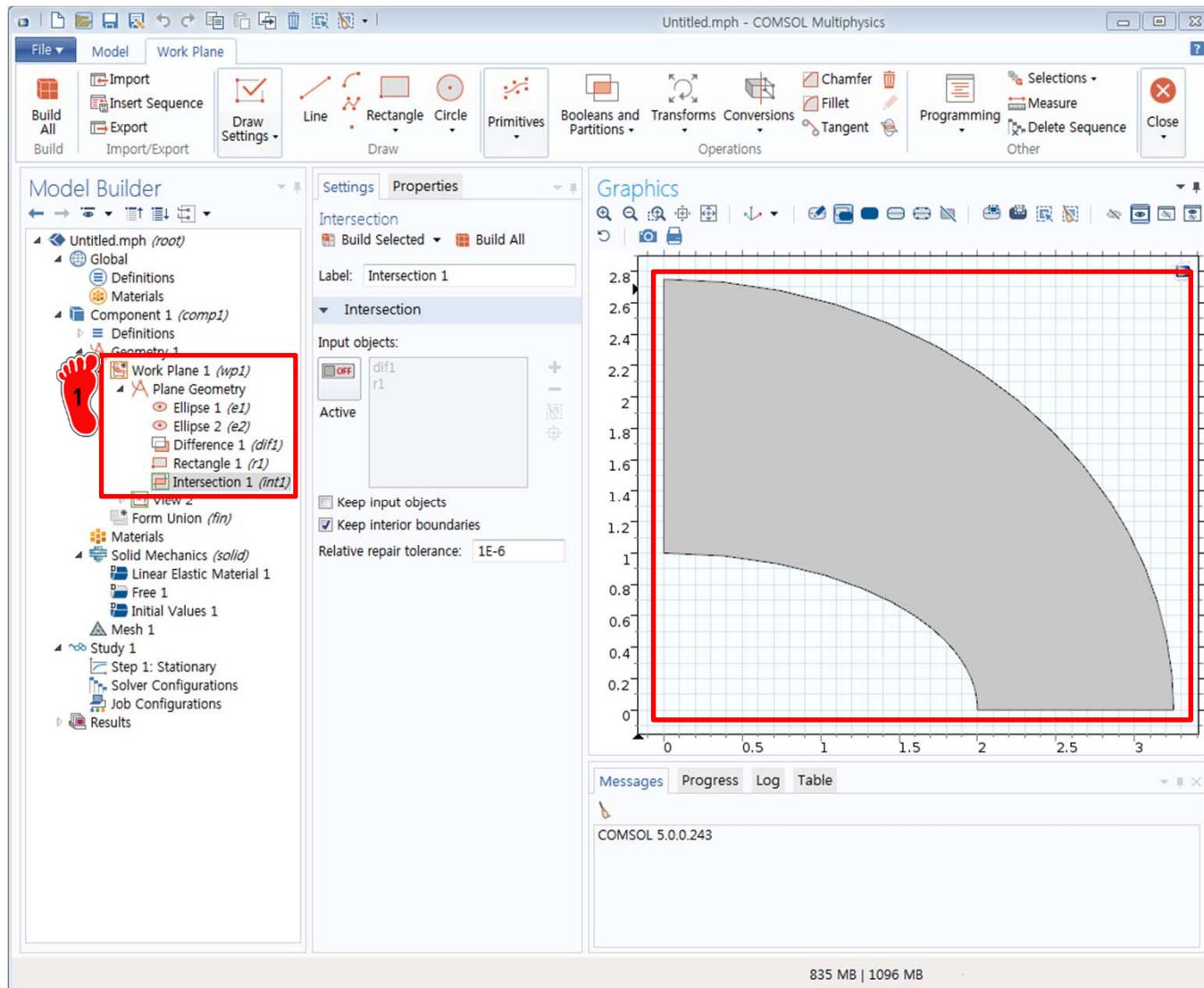
STUDY TYPE SELECTION



GEOMETRY CREATION

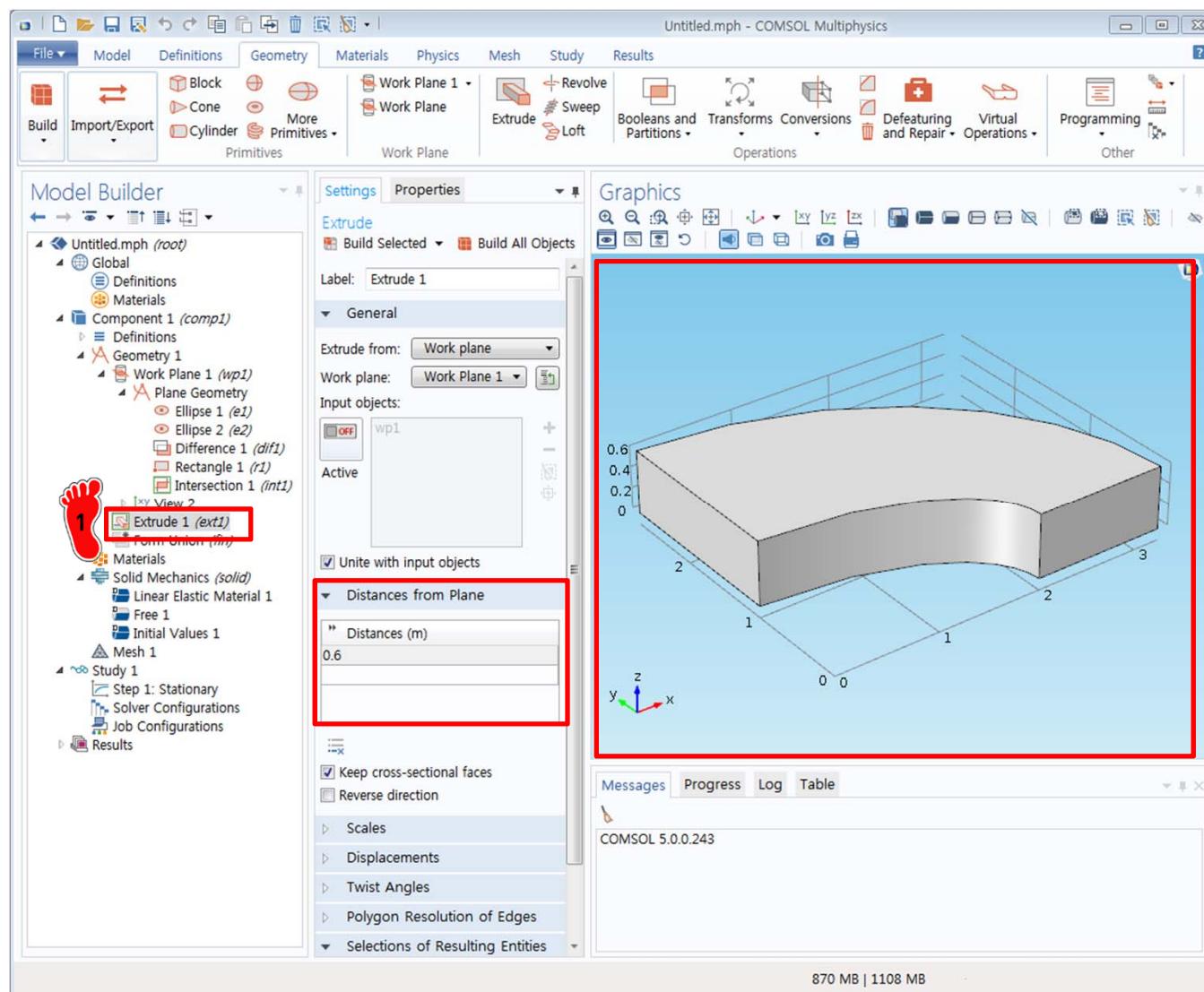


GEOMETRY CREATION



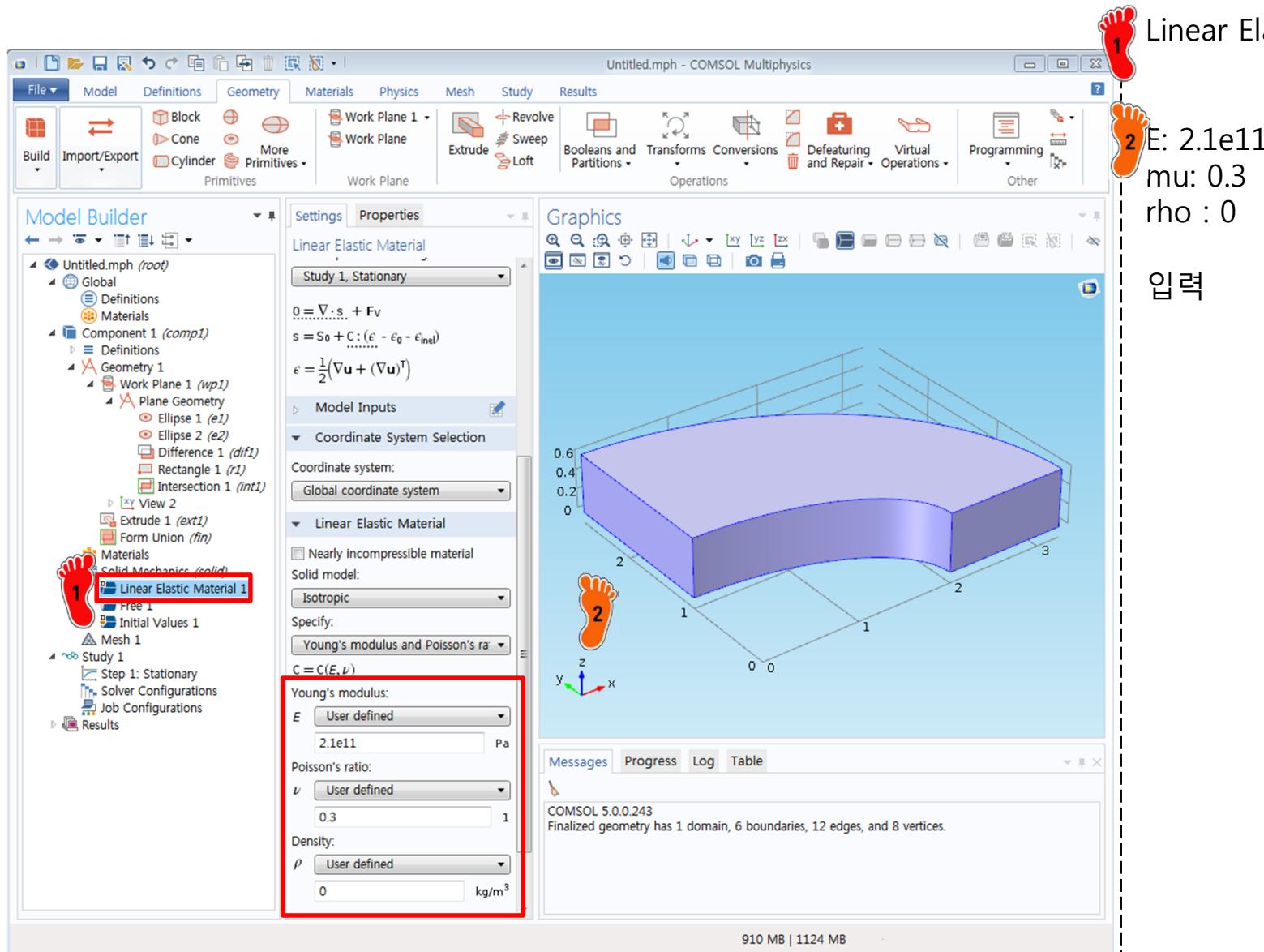
1 Plane Geometry 에 2차원
형상 생성

GEOMETRY CREATION



Extrude 메뉴를 이용하여 z 방향으로 0.6 m 두께를 가지는 형상 생성

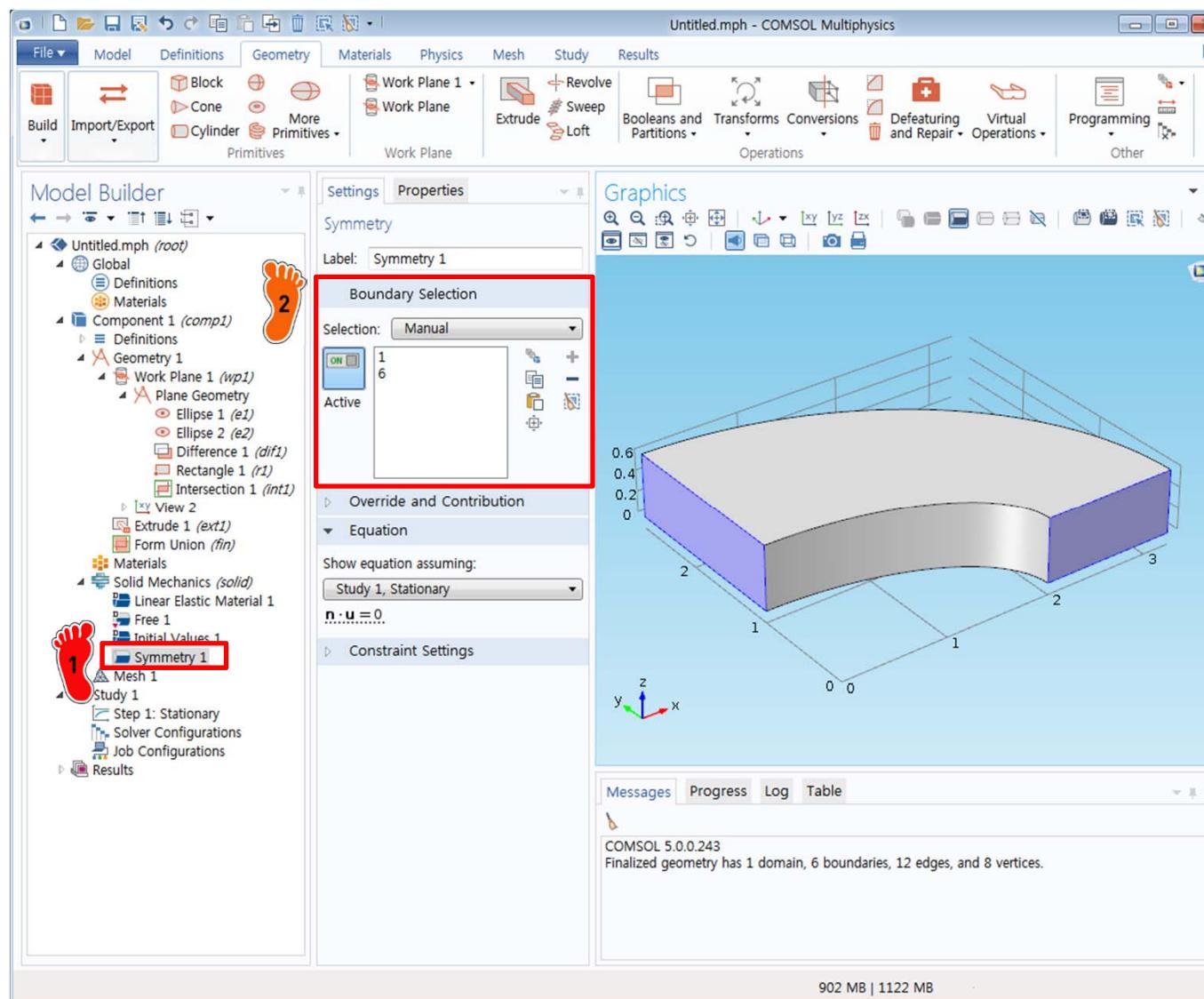
MATERIAL PROPERTY



1 Linear Elastic Material 클릭
 2 E: $2.1e11$
 mu: 0.3
 rho : 0

입력

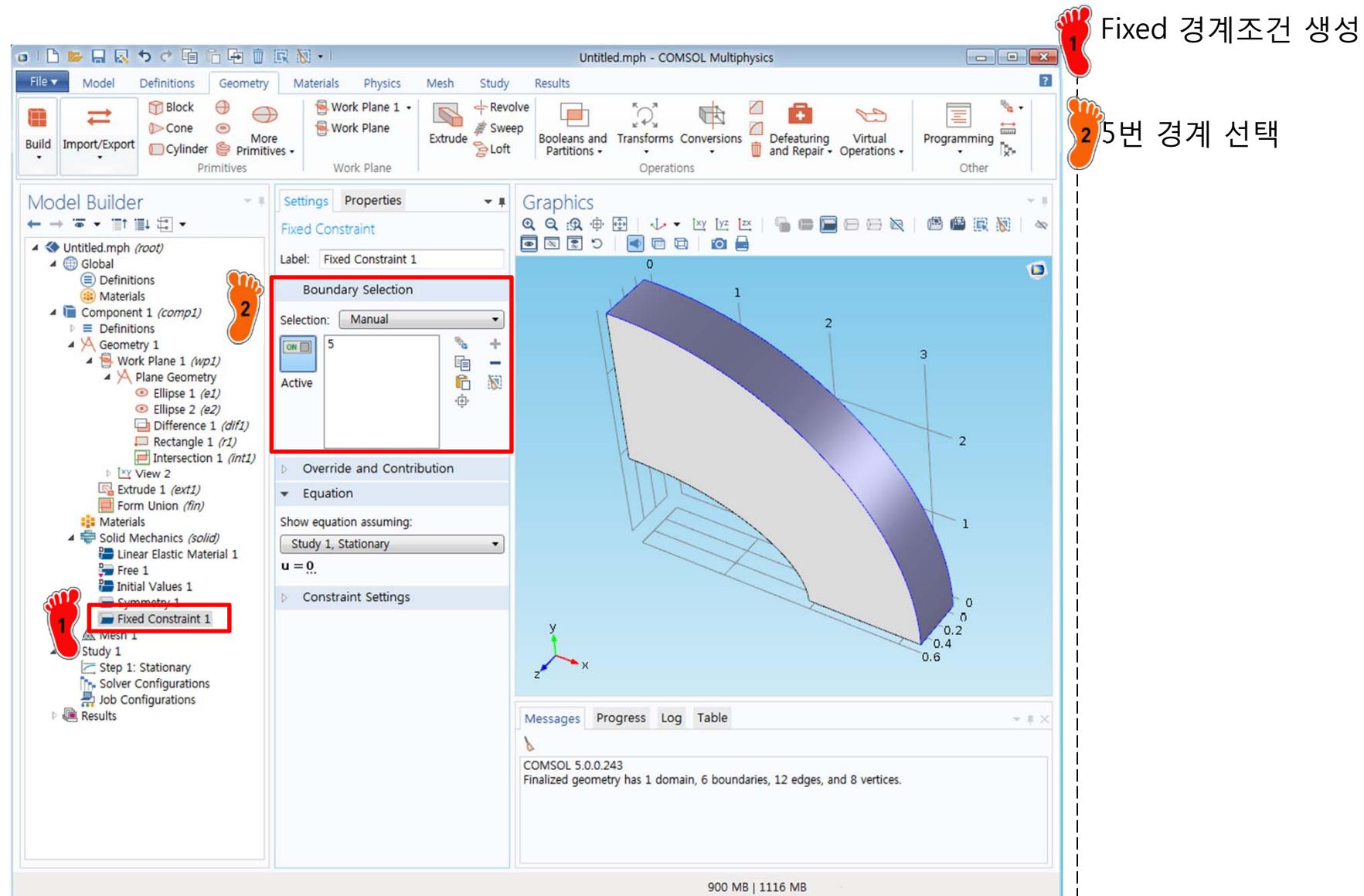
BOUNDARY CONDITION



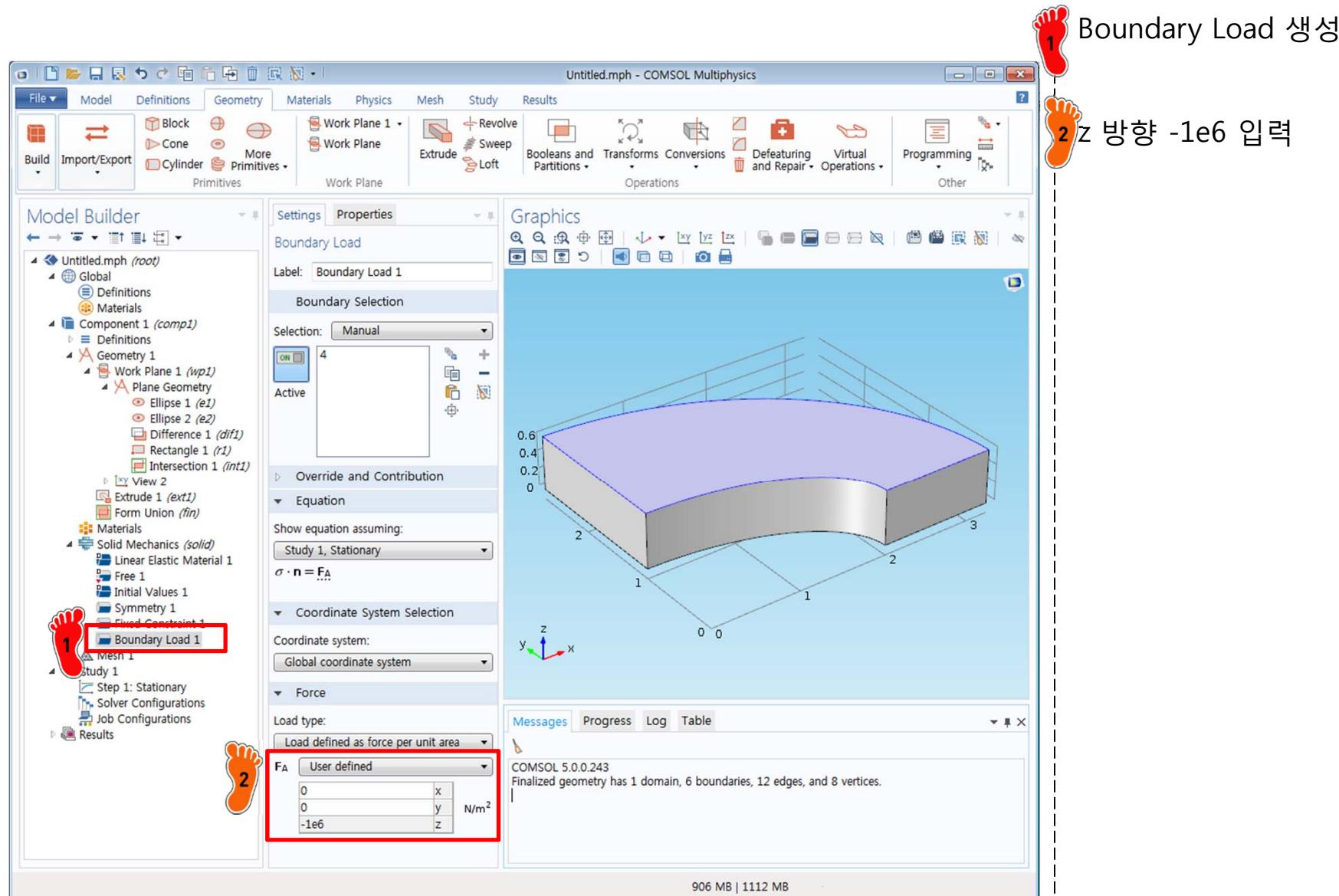
1 Symmetry 경계조건 생성

2 1, 6번 경계 선택

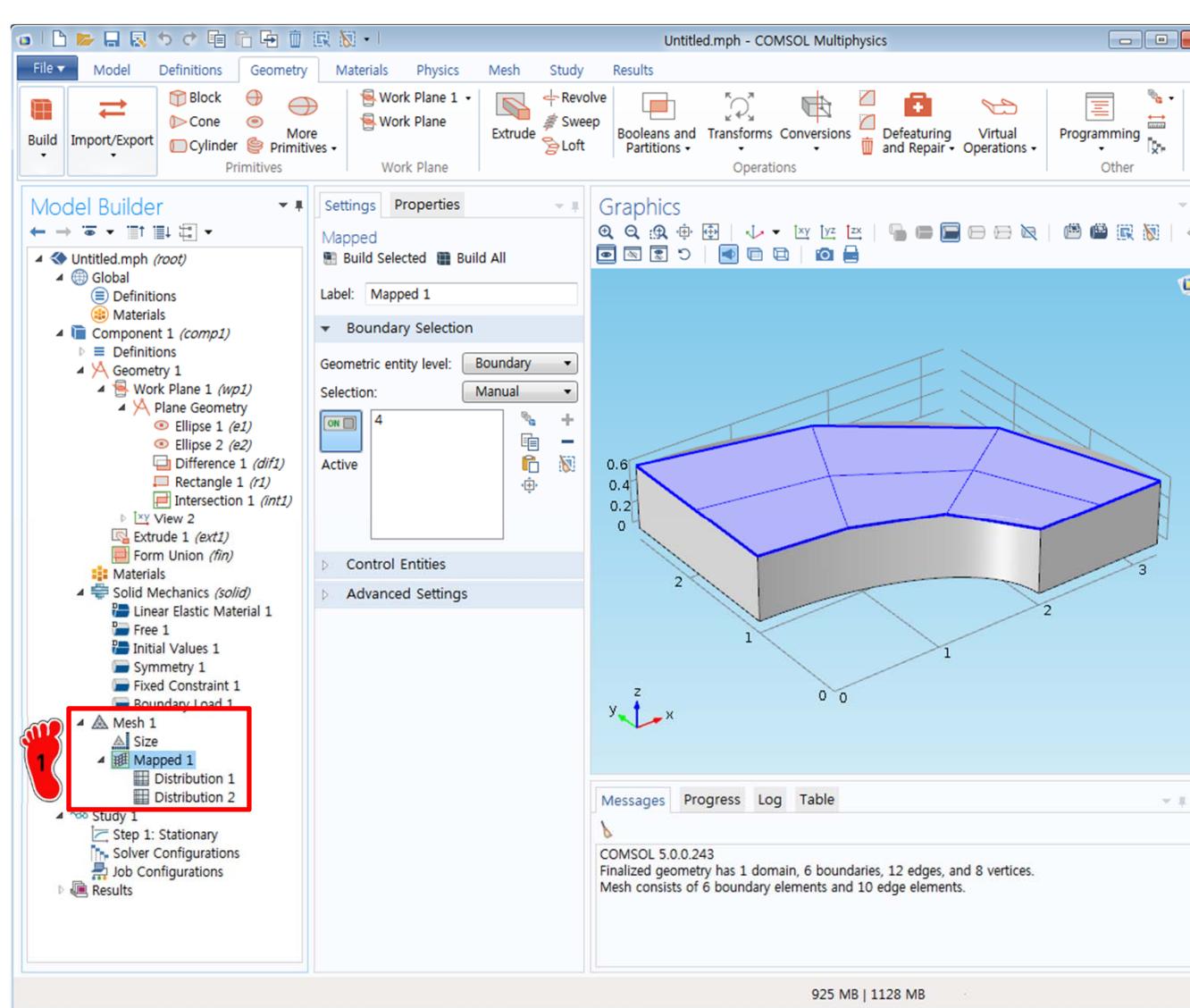
BOUNDARY CONDITION



LOADING CONDITION

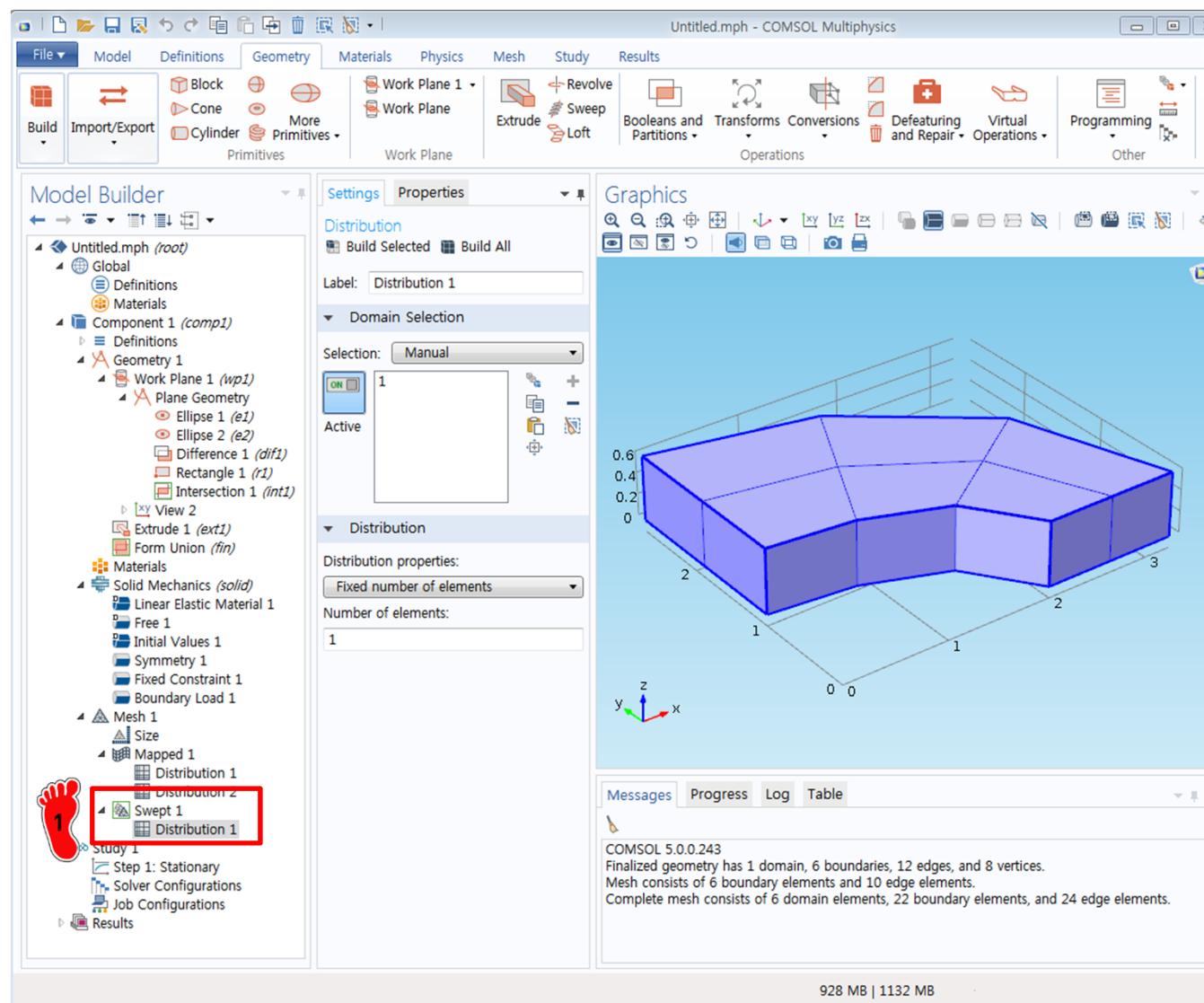


MESH



1 Mapped mesh의
Distribution 기능을 이용하
여 4번 평면에 요소 생성

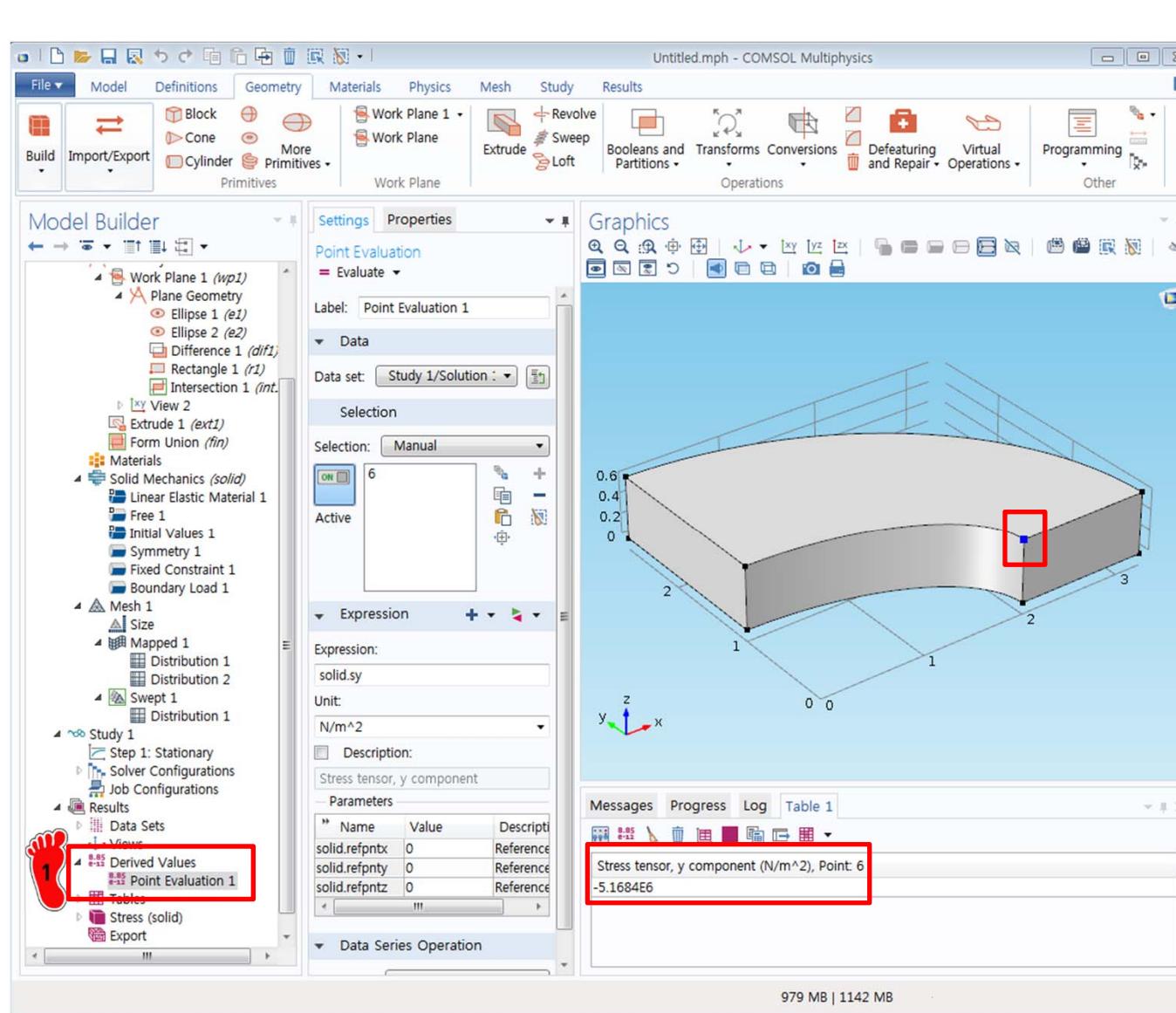
MESH



Swept 메뉴를 이용하여 전체 영역에 한 개의 layer 를 가지는 요소 생성

그 후 해석 실행

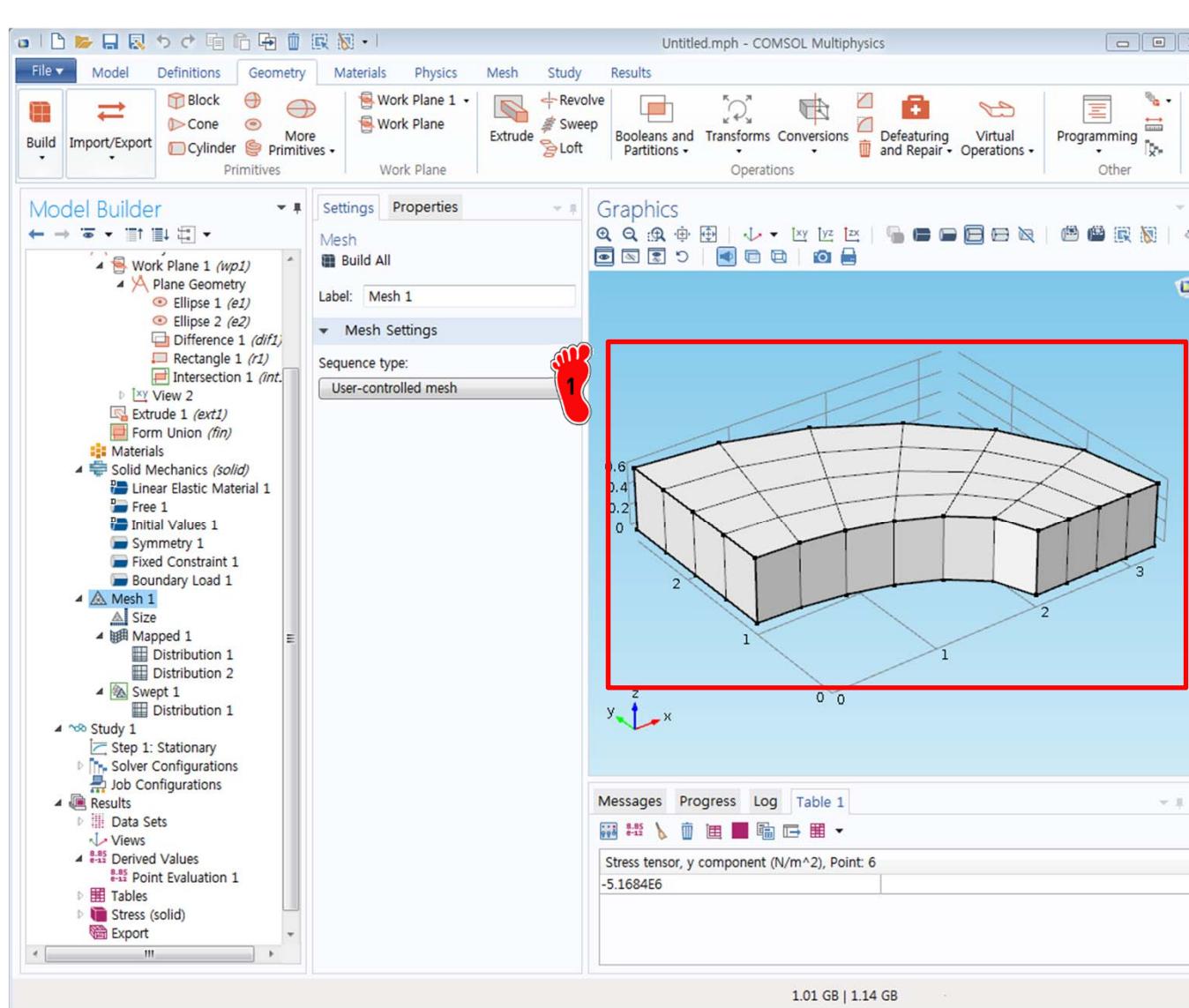
POST-PROCESSING



Derived Values 의 Point Evaluation 기능을 이용하여 6번 절점의 solid.sy 응력을 계산

결과값: -5.1684 MPa

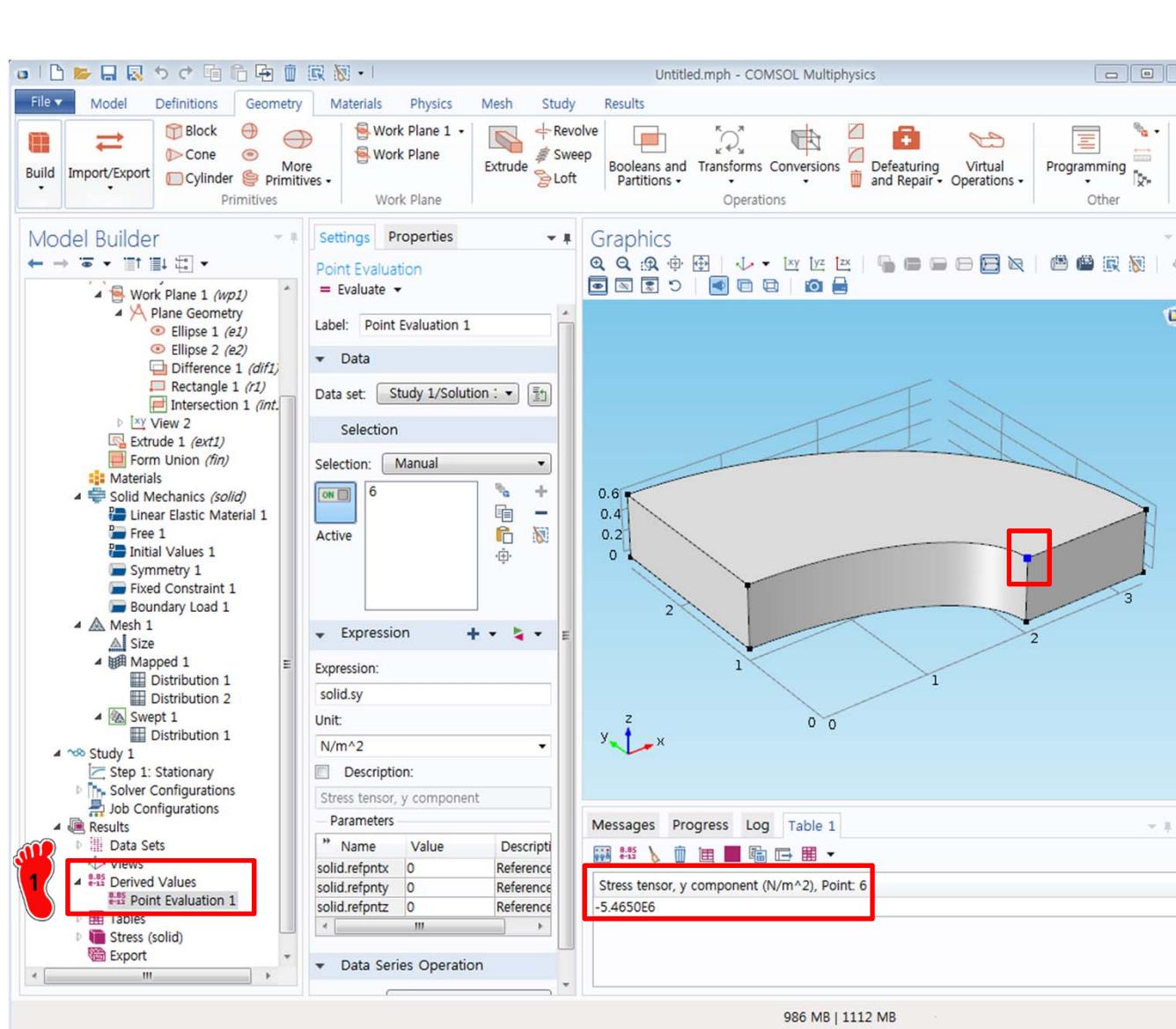
MESH



1 Fine 메시로 변경

그 후, 해석실행

POST-PROCESSING



Derived Values 의 Point Evaluation 기능을 이용하여 6번 절점의 solid.ssy 응력을 계산

결과값: -5.4650 MPa

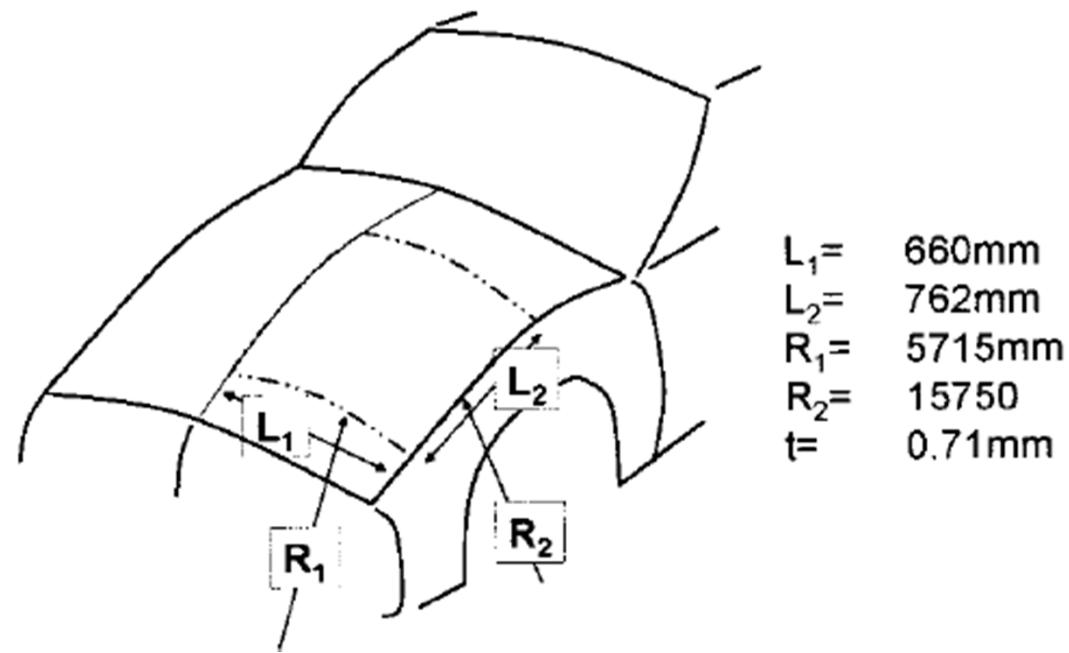
SUMMARY

| LE 10 | order | integration points | element | coarse mesh | | fine mesh | |
|----------|-------|--------------------|------------|-------------|--------|-----------|-------|
| | | | | s22 | error | s22 | error |
| Abaqus * | 2 | 8 | C3D20R | -7.93E+06 | 47.40% | -5.53E+06 | 2.79% |
| Ansys | 2 | 8 | SOLID95 | -5.36E+06 | -0.32% | -5.61E+06 | 4.26% |
| CalculiX | 2 | 8 | C3D20R | -5.36E+06 | -0.32% | -5.61E+06 | 4.26% |
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| Abaqus * | 2 | 27 | C3D20 | -6.72E+06 | 24.91% | -5.64E+06 | 4.83% |
| Ansys | 2 | 14 | SOLID95 | -5.40E+06 | 0.46% | -5.61E+06 | 4.33% |
| CalculiX | 2 | 27 | C3D20 | -5.20E+06 | -3.32% | -5.50E+06 | 2.18% |
| Tahoe | 2 | 27 | hexahedron | | | | |

| | | | | | | | |
|--------|---|--|------------|-----------|--------|-----------|-------|
| COMSOL | 2 | | plate | -4.71E+06 | 12.45% | -5.03E+06 | 6.51% |
| COMSOL | 2 | | hexahedron | -5.17E+06 | 3.90% | -5.47E+06 | 1.67% |

HOOD PANEL ANALYSIS

- Simply supported boundary conditions
- Dynamic yield stress: $s_{yd} = 298 \text{ N/mm}^2$
- Panel stiffness: **load(1N)/deflection**
- Oil-can load: **buckling load**



$$K = \frac{B}{R} \frac{Et^2}{\sqrt{1 - \mu^2}} = 28.39 \text{ N/mm}$$

where

R Spherical radius

B Constant (2.309)

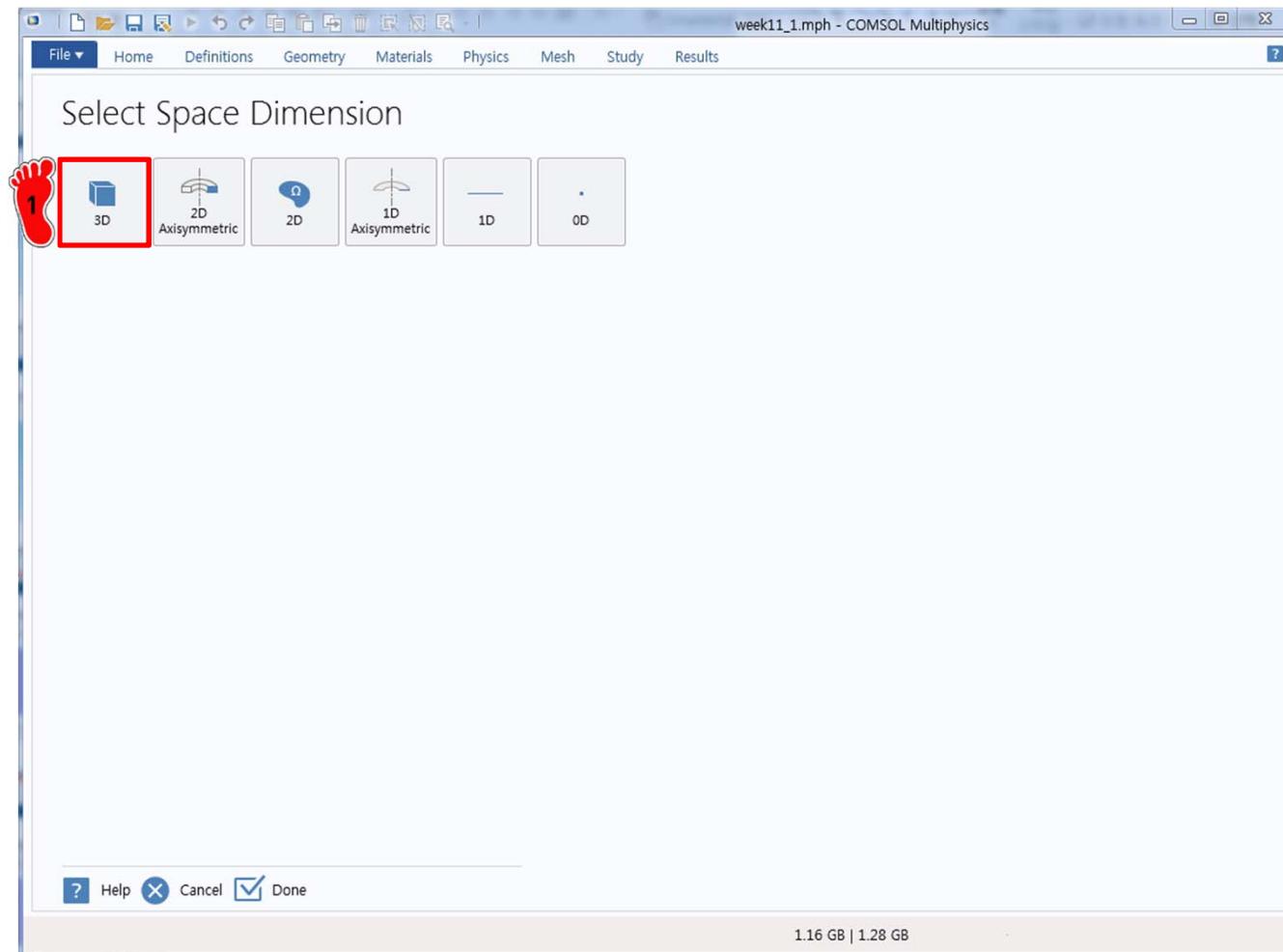
E Young's Modulus

μ Poisson's ratio

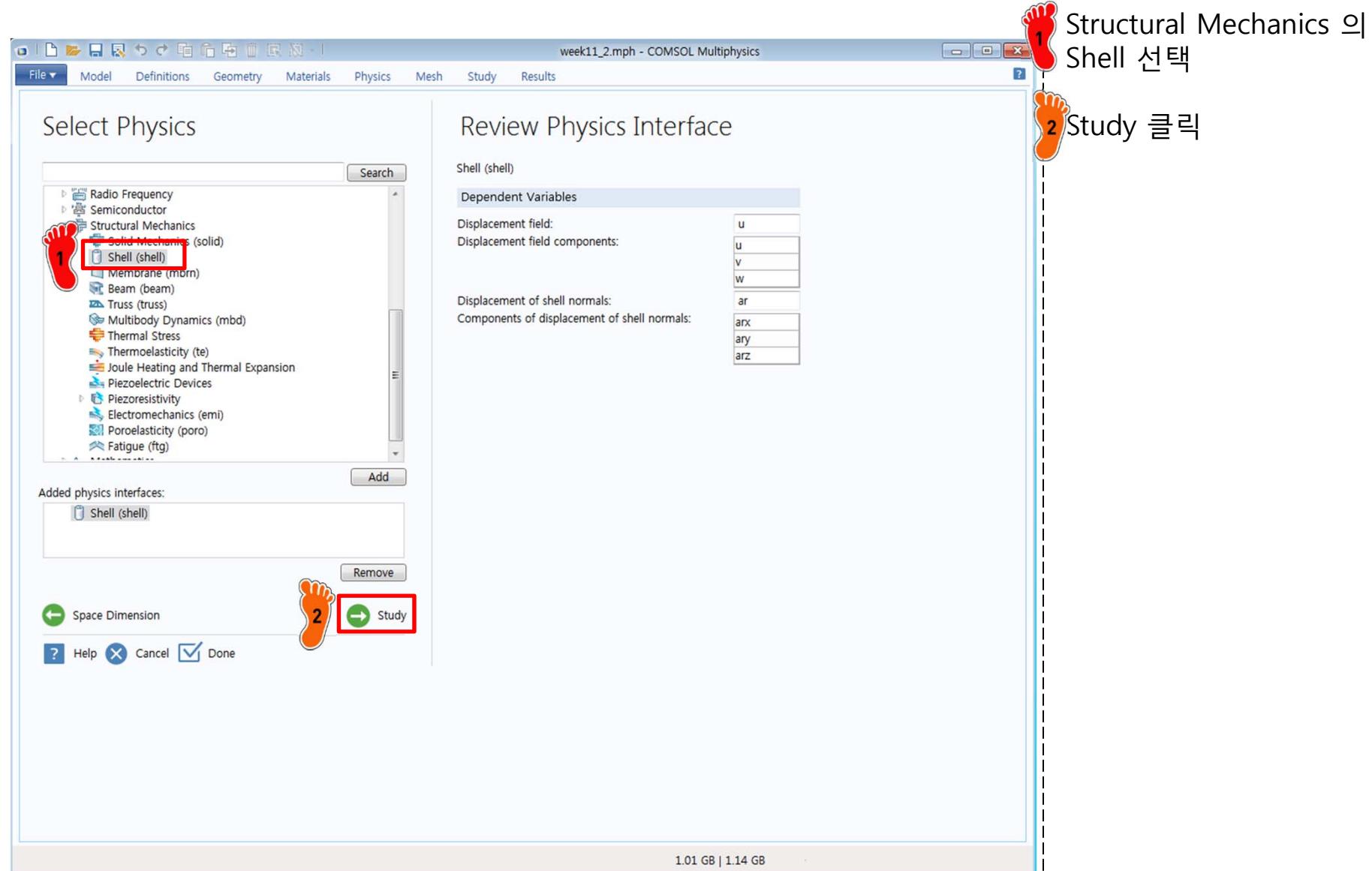
t Panel thickness

$$P_{CR} = \frac{CR_{CR}\pi^2 Et^4}{L_1 L_2 (1 - \mu^2)} = 50 \text{ N}$$

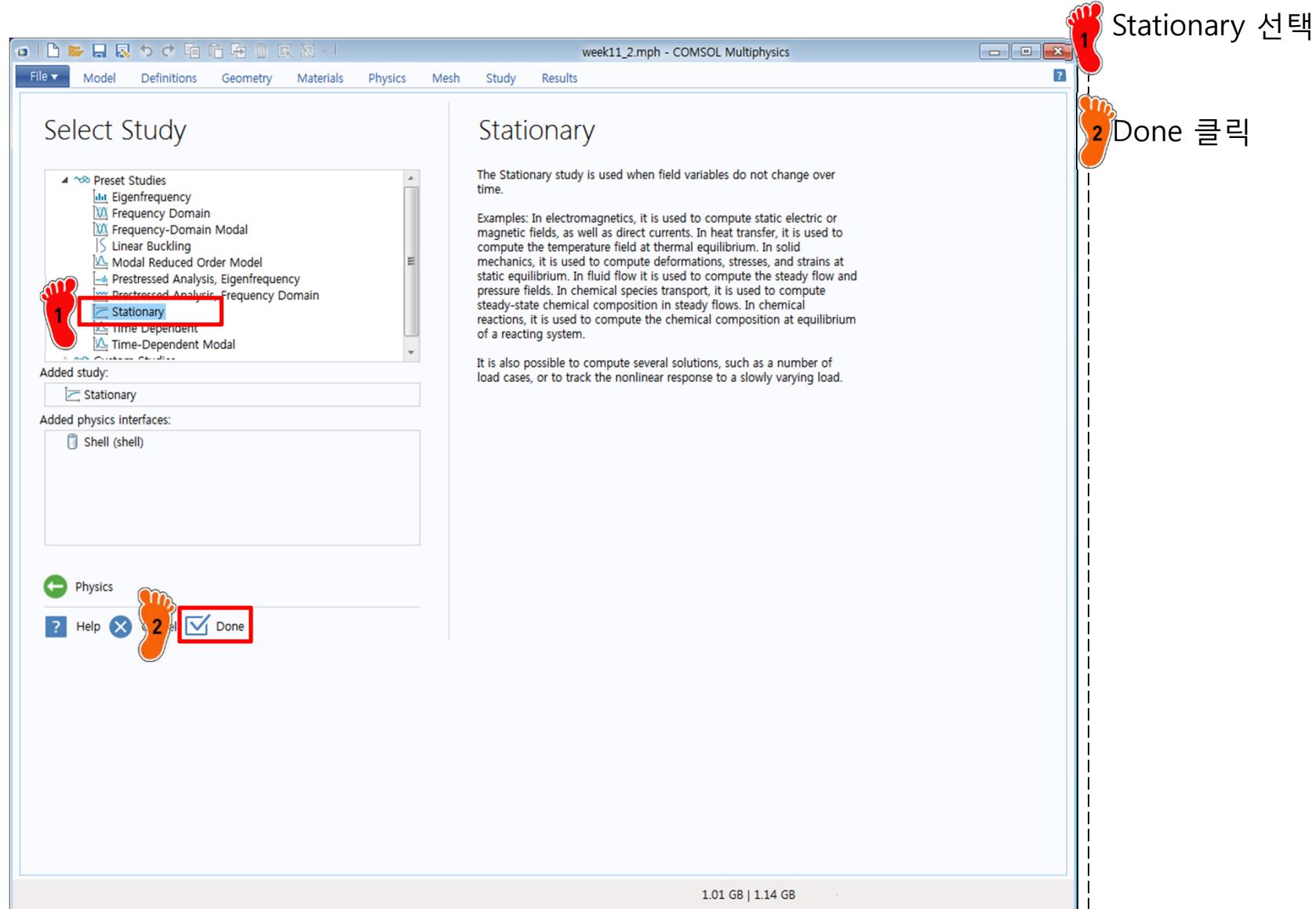
DIMENSION SELECTION



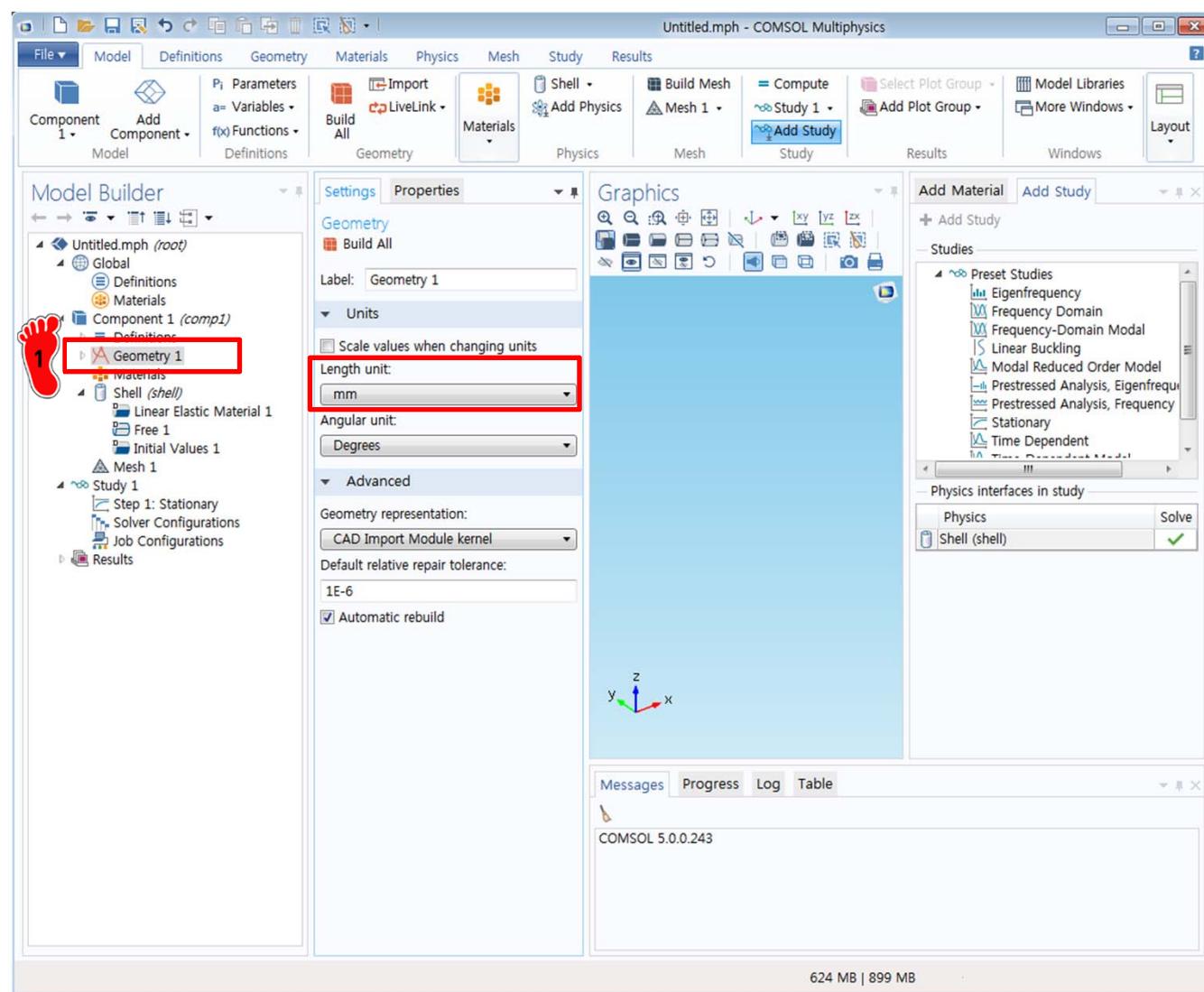
PHYSICS SELECTION



STUDY TYPE SELECTION

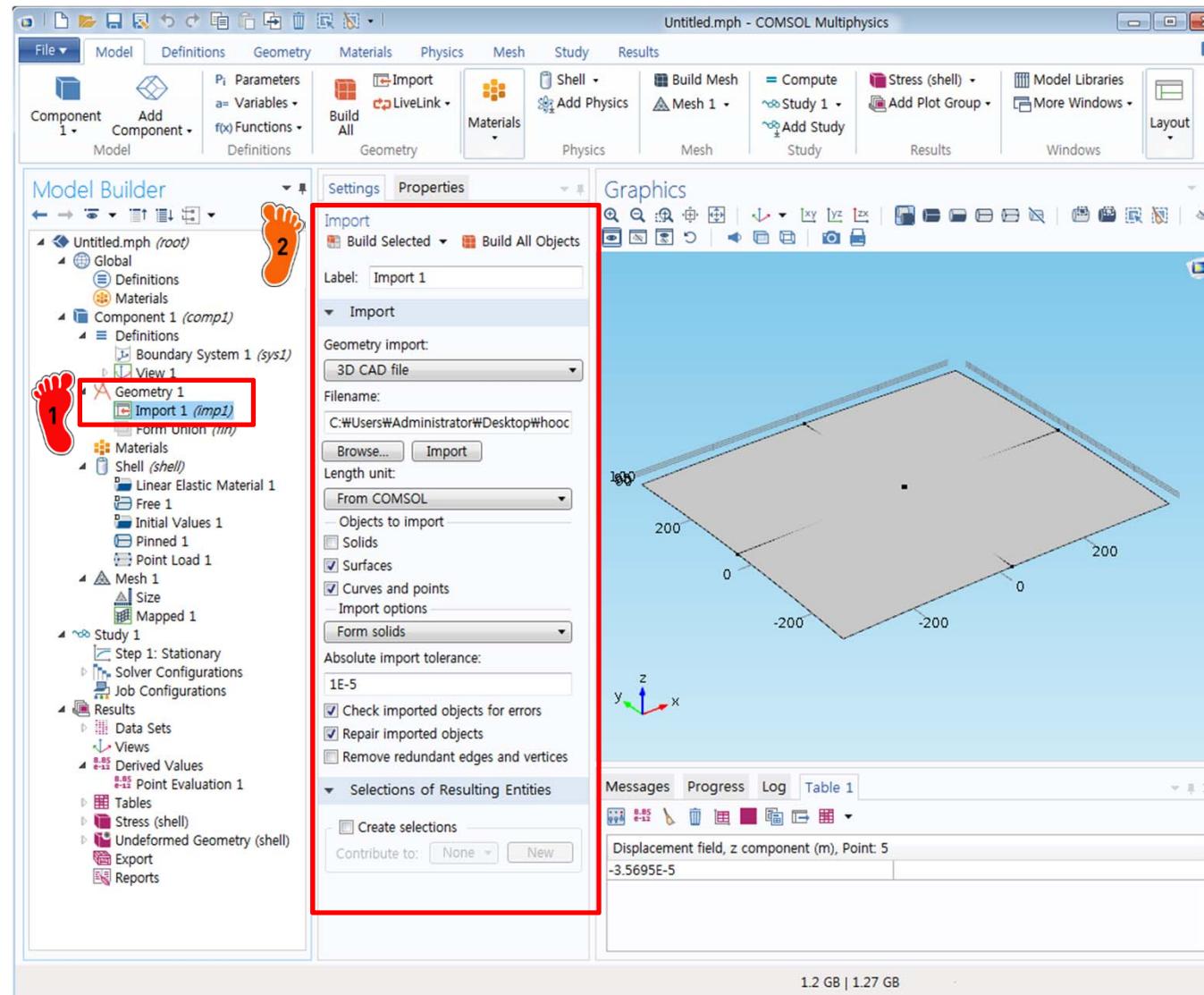


UNIT SETTING



1 Geometry 1에서 단위를
mm로 설정

GEOMETRY IMPORT

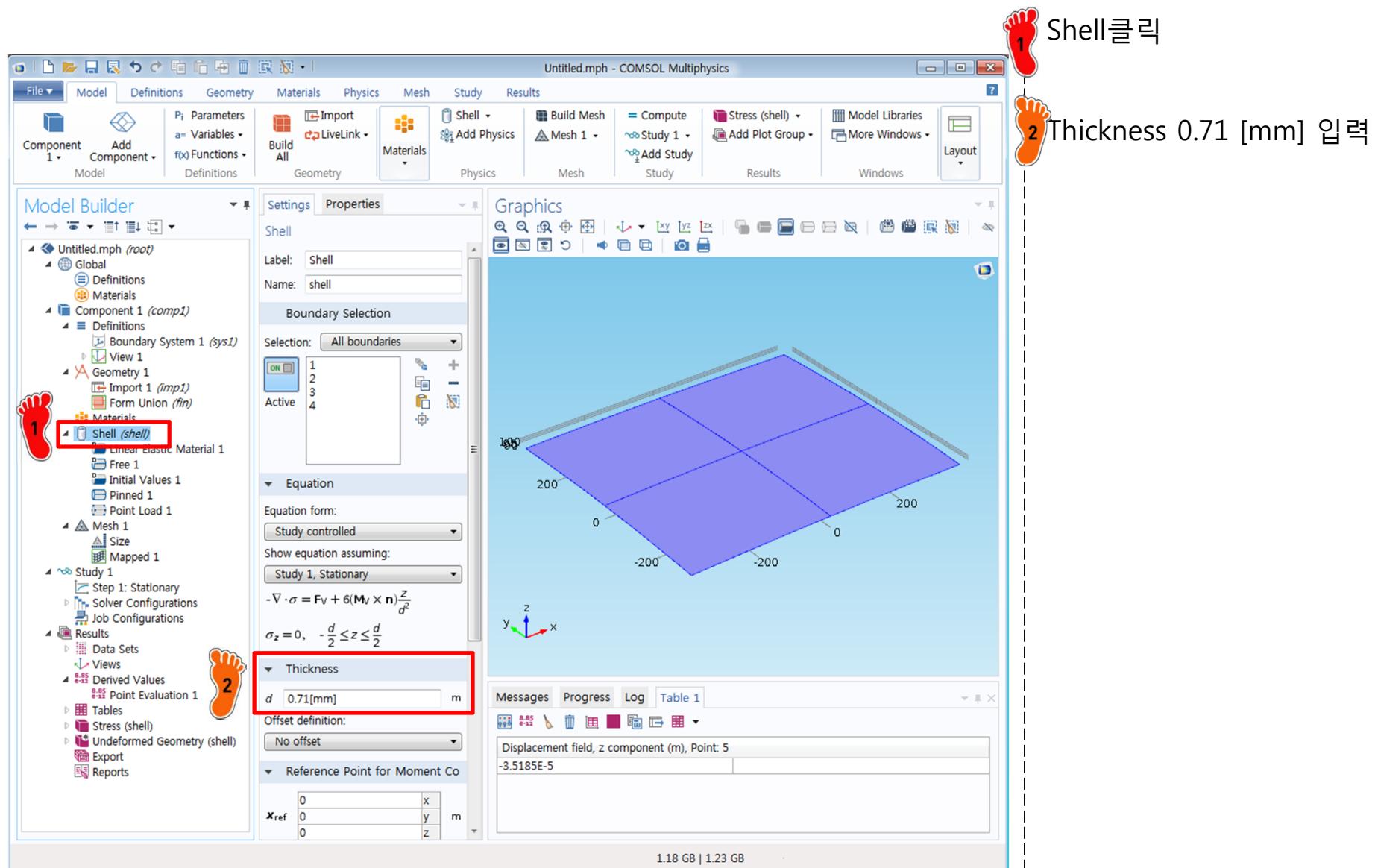


1 Import 메뉴 생성

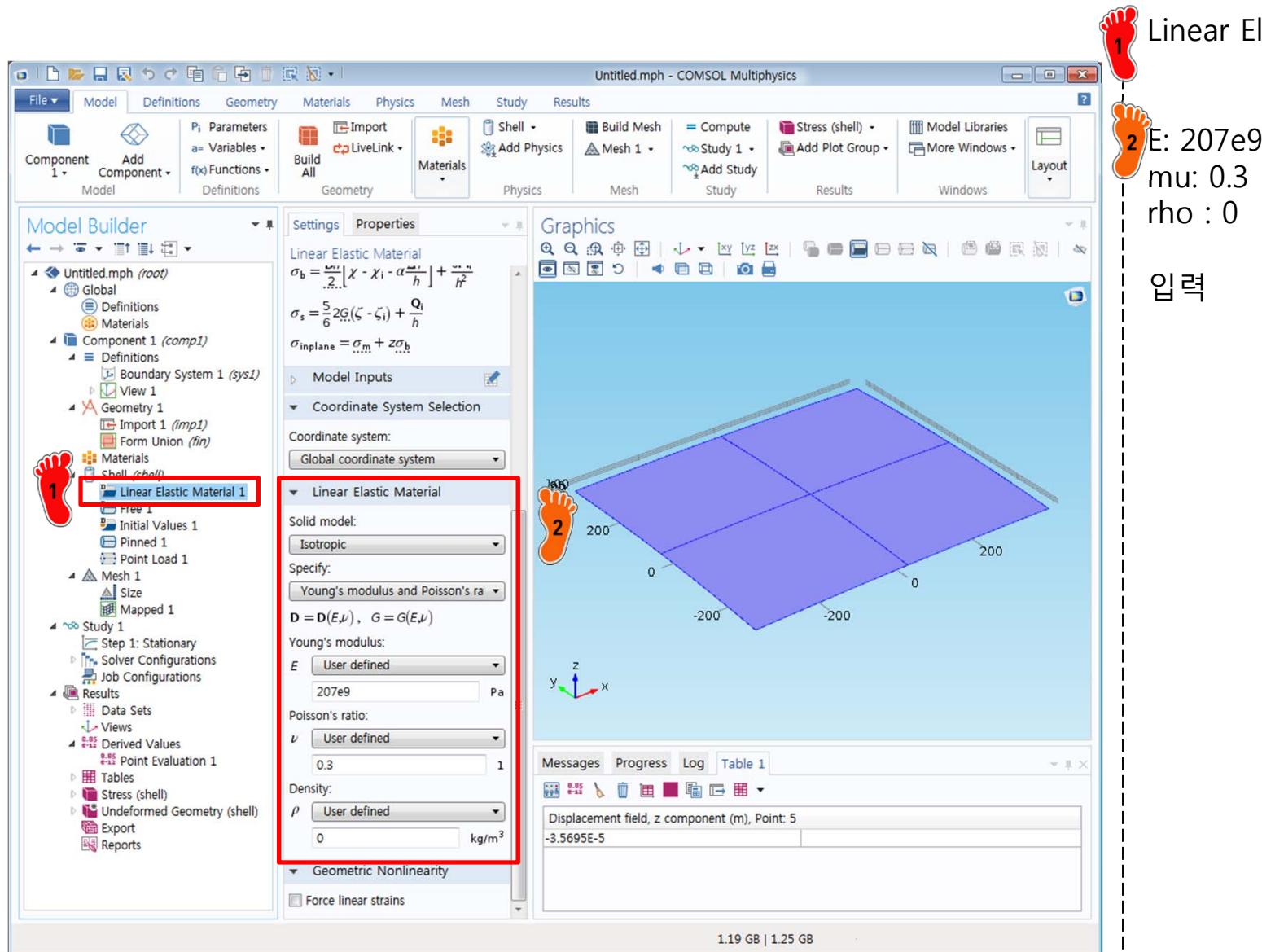
2 Solids: 체크 해제
Curves and points: 체크 활성화

옵션 변경 후 캐드 파일 삽입

MATERIAL PROPERTY



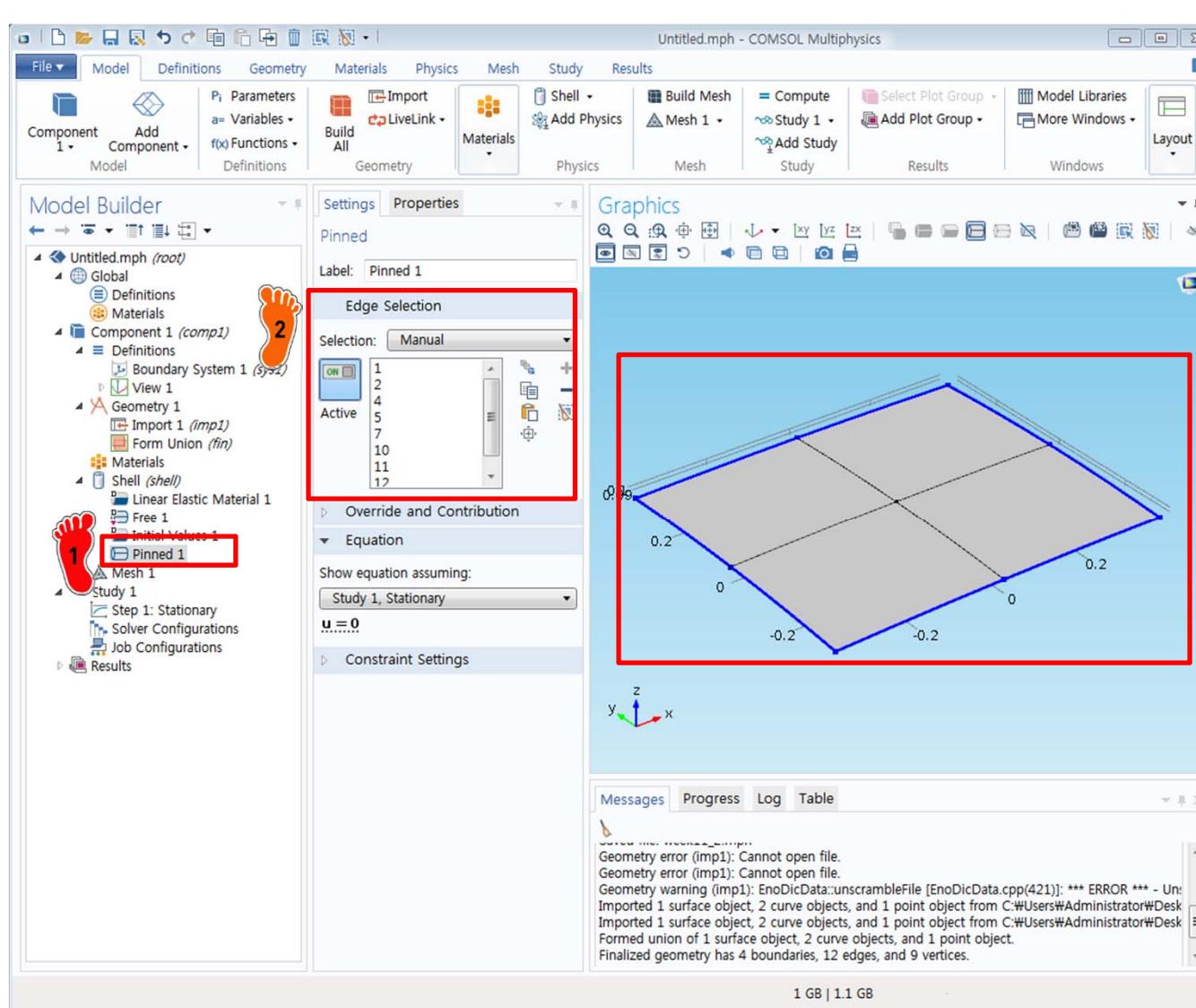
MATERIAL PROPERTY



- 1 Linear Elastic Material 클릭
- 2 E: 207e9
mu: 0.3
rho : 0

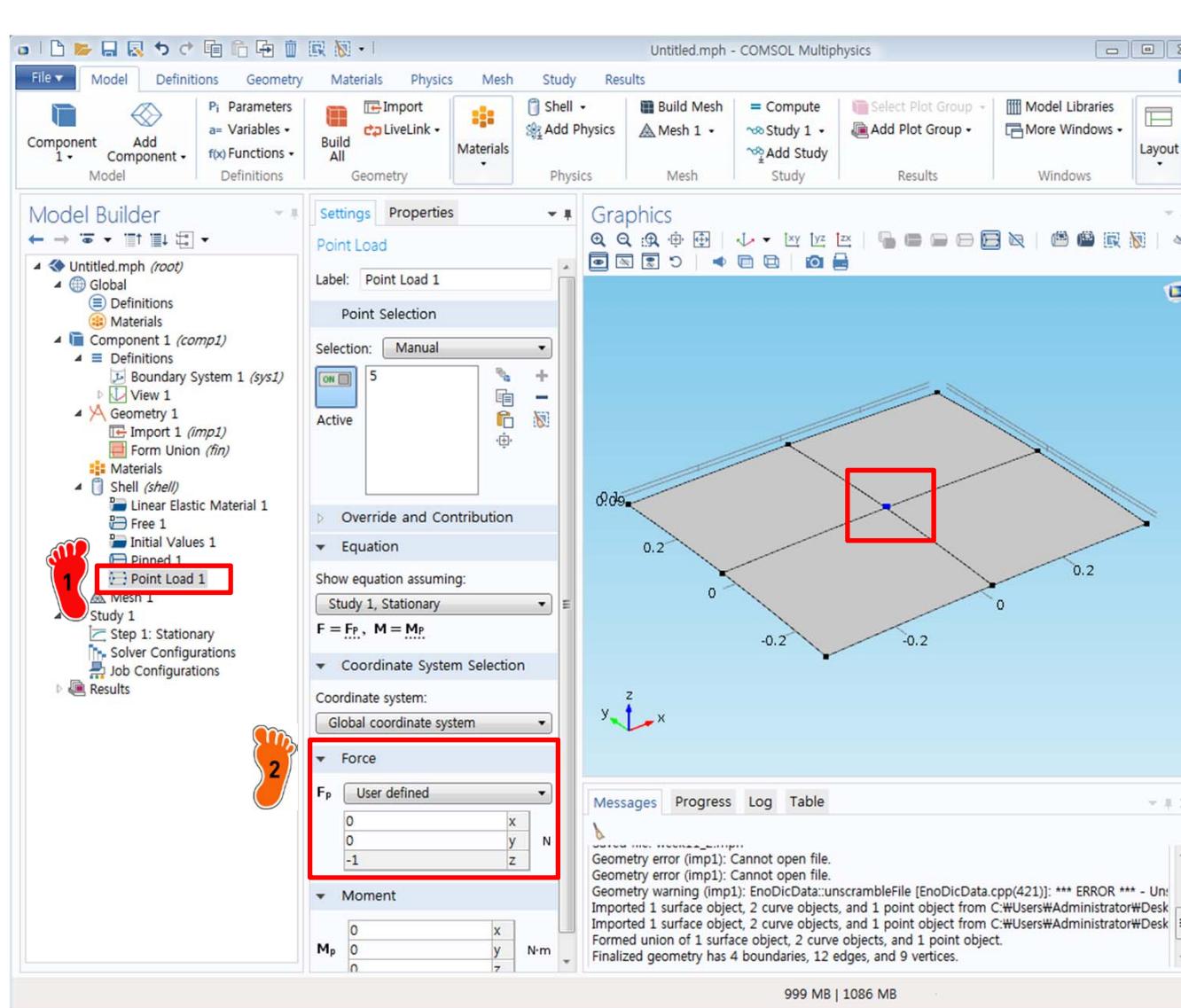
입력

BOUNDARY CONDITION



- 1 Pinned 경계조건 생성
2 최외각 경계 선택

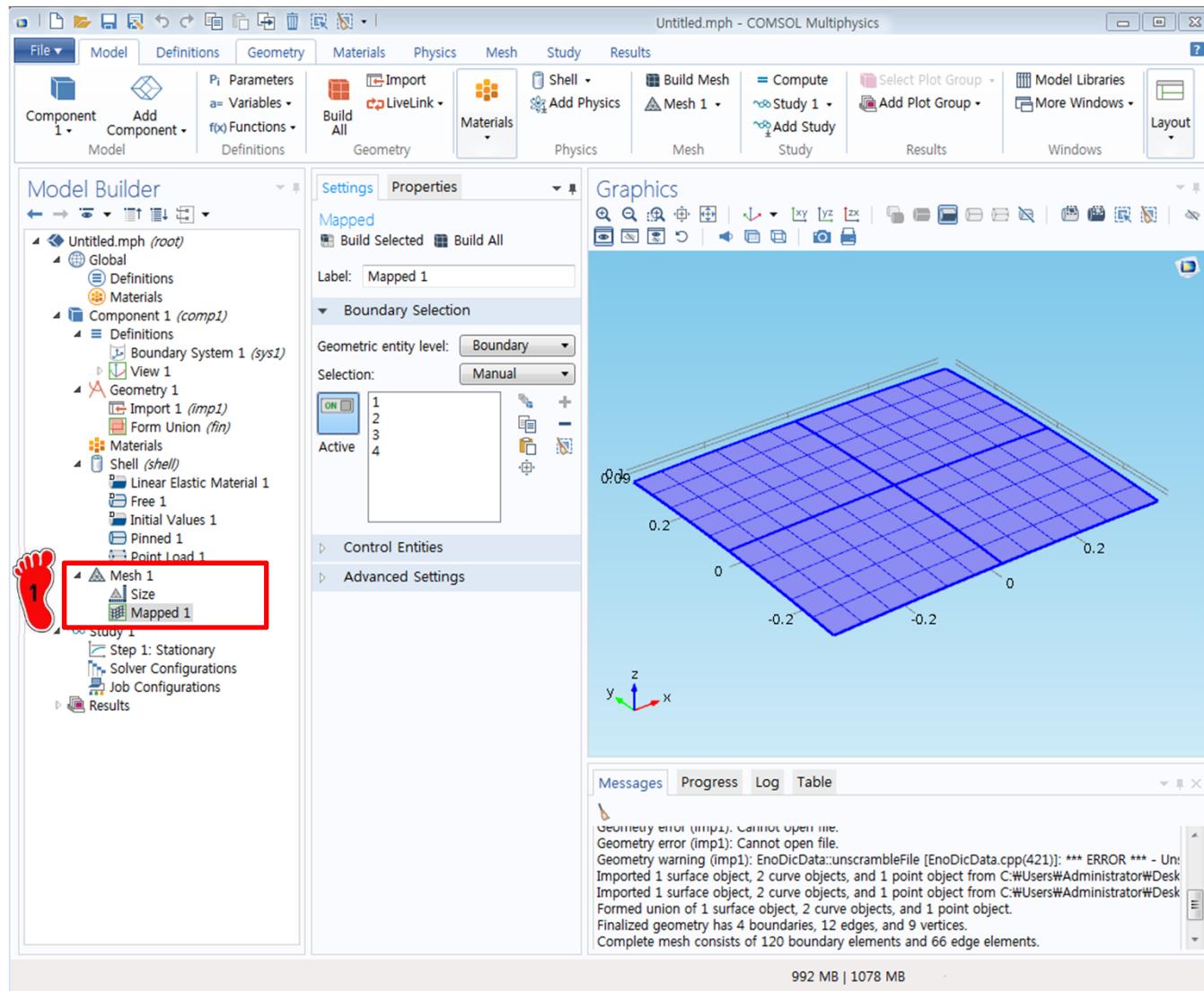
LOADING CONDITION



1 Point Load 메뉴 생성

2 5번 절점에 z 방향 -1 N 하 중 입력

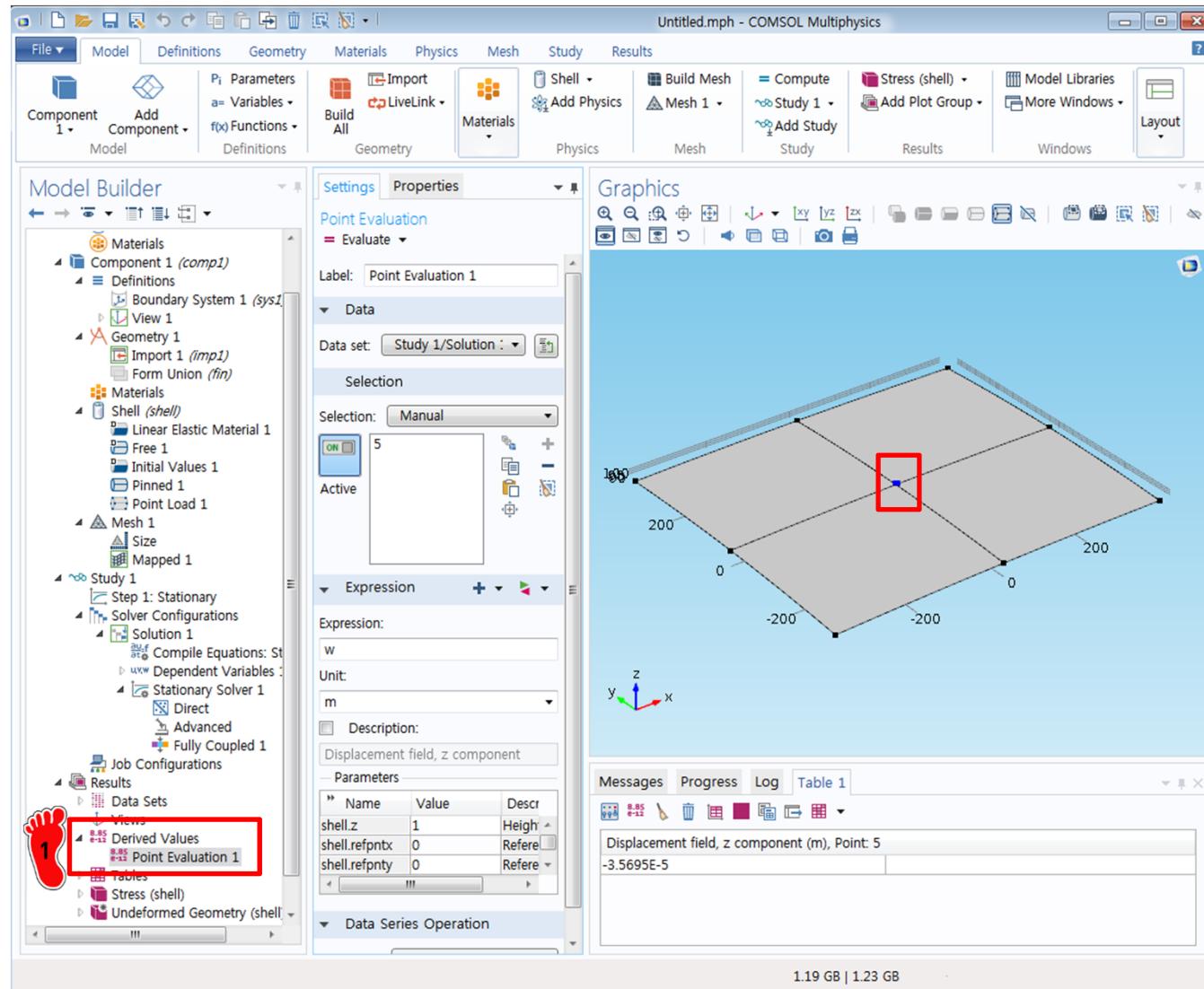
MESH



1 Mapped mesh 기능을 이용
하여 요소 생성

그 후, 해석실행

POST-PROCESSING



Derived Values 의 Point Evaluation 기능을 이용하여 5번 절점의 z 방향 변위를 계산

결과값: $-3.5695\text{e-}2 \text{ mm}$

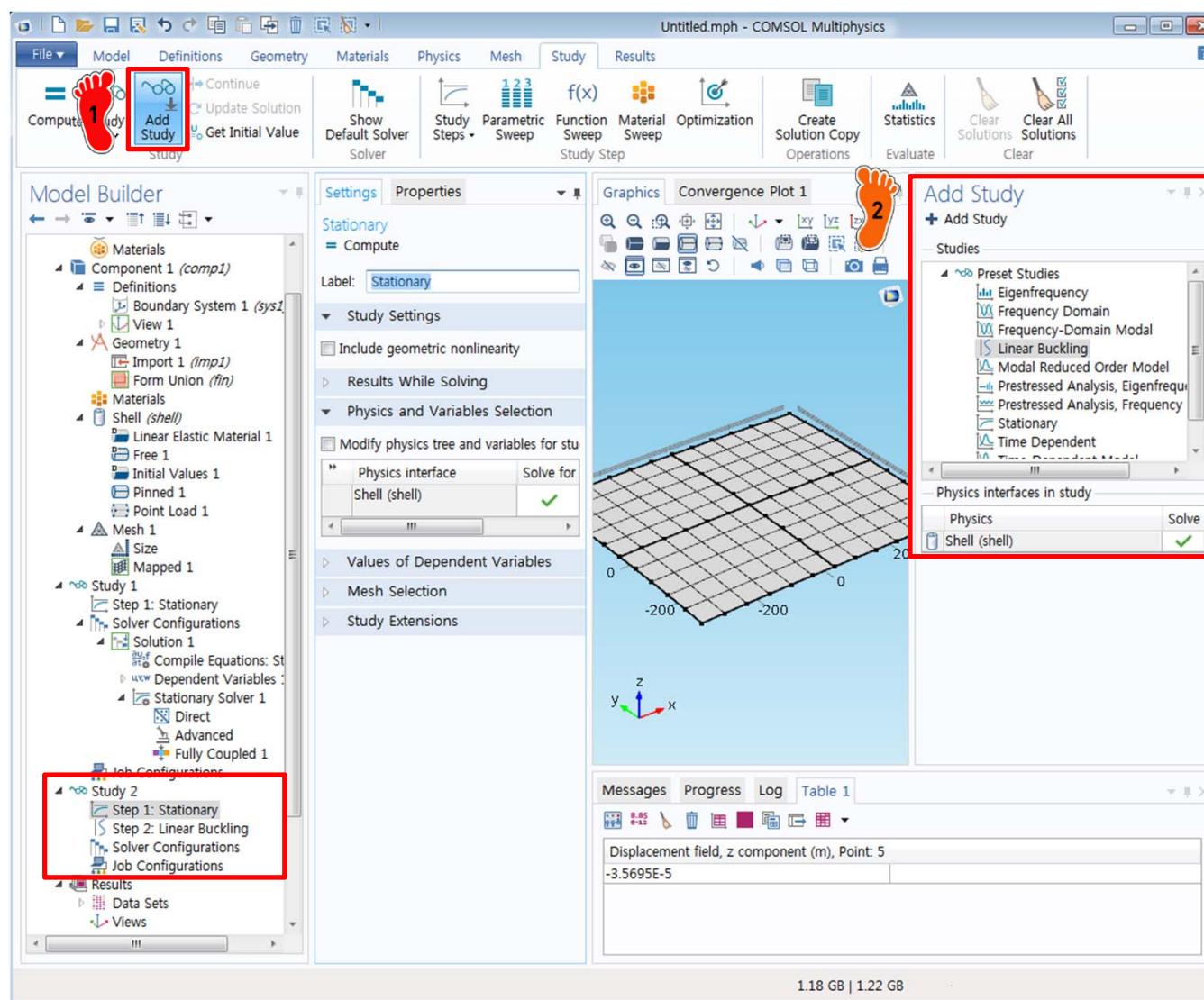
Analytic solution

$$K = \frac{B}{R} \frac{Et^2}{\sqrt{1-\mu^2}} = 28.39 \text{ N/mm}$$

COMSOL

$$K = \frac{P}{\delta} = \frac{1 \text{ N}}{3.5695\text{e-}2 \text{ mm}} \\ = 28.02 \text{ N/mm}$$

DIMENSION SELECTION

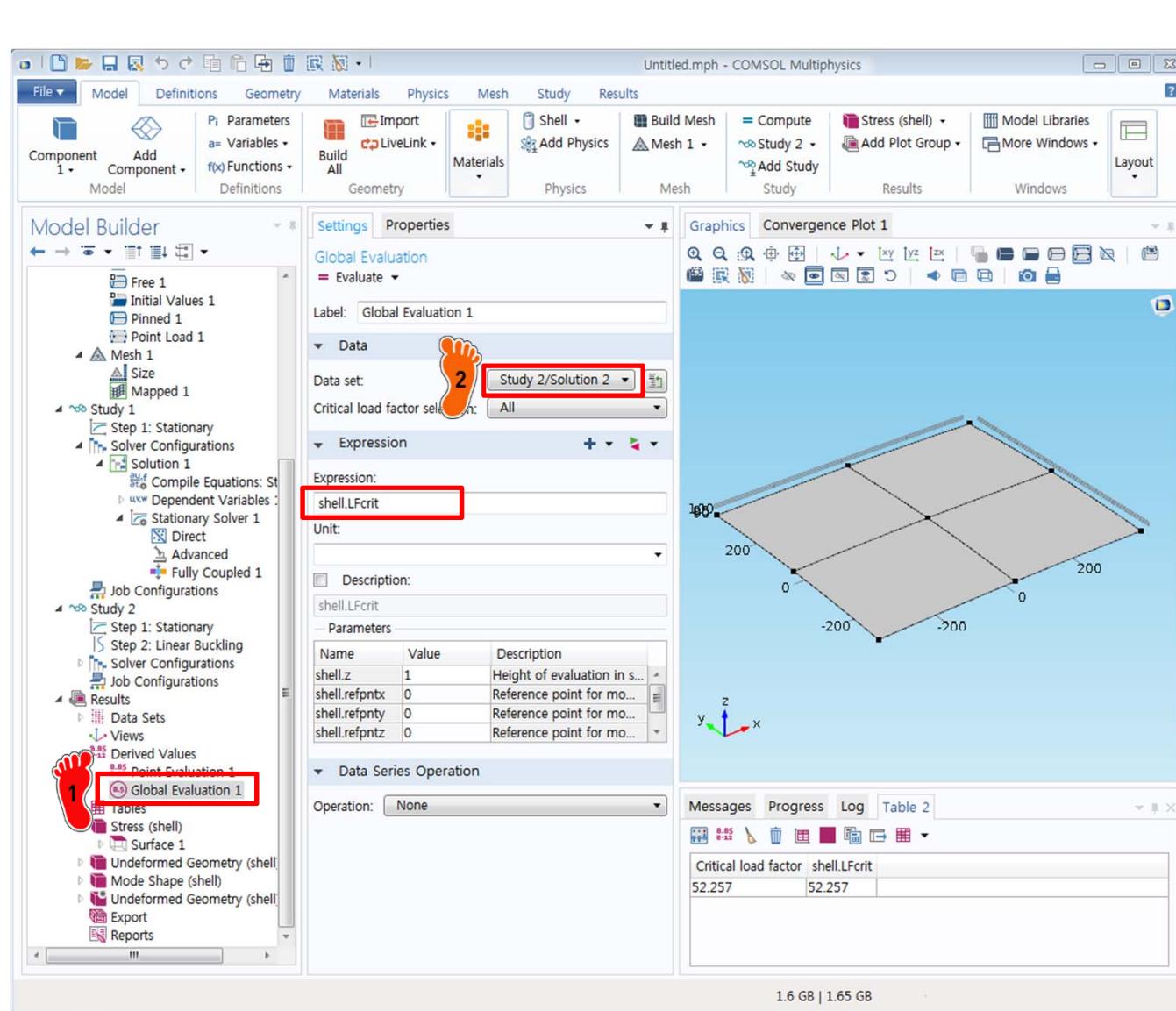


1 상단 메뉴에서 Add Study 선택

2 Linear Buckling 추가

Study 2 생성 확인 후
해석 실행

POST-PROCESSING



1 Global Evaluation생성

2 Data set을
Study 2/Solution 2로 변경,
shell.LFcrit값 확인

shell.LFcrit: Critical load
factor from linear buckling
analysis

결과값: 52.257 N

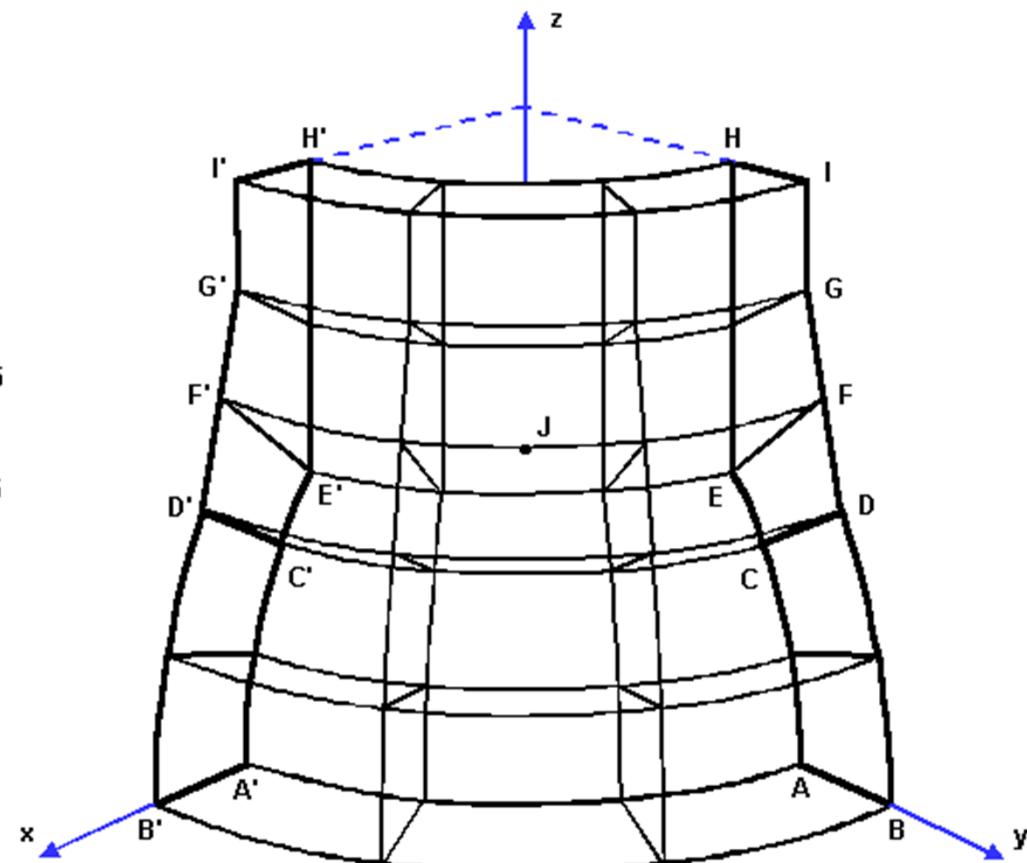
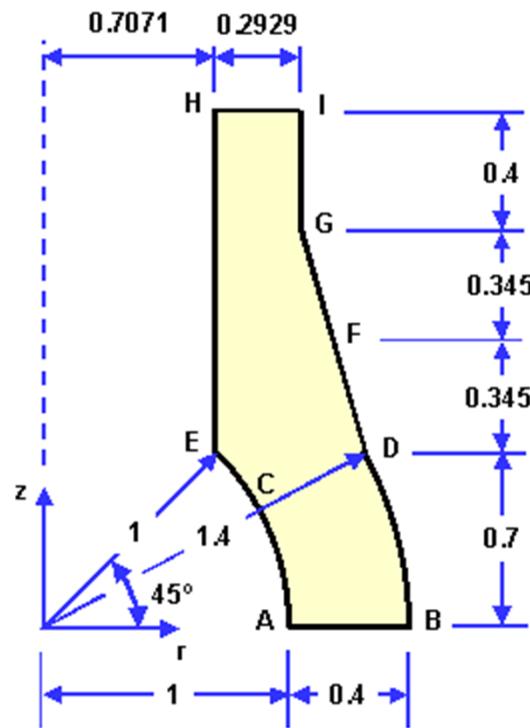
Analytic solution

$$P_{CR} = \frac{CR_{CR}\pi^2 Et^4}{L_1 L_2 (1 - \mu^2)} = 50 \text{ N}$$

NAFEMS BENCHMARK: LE11

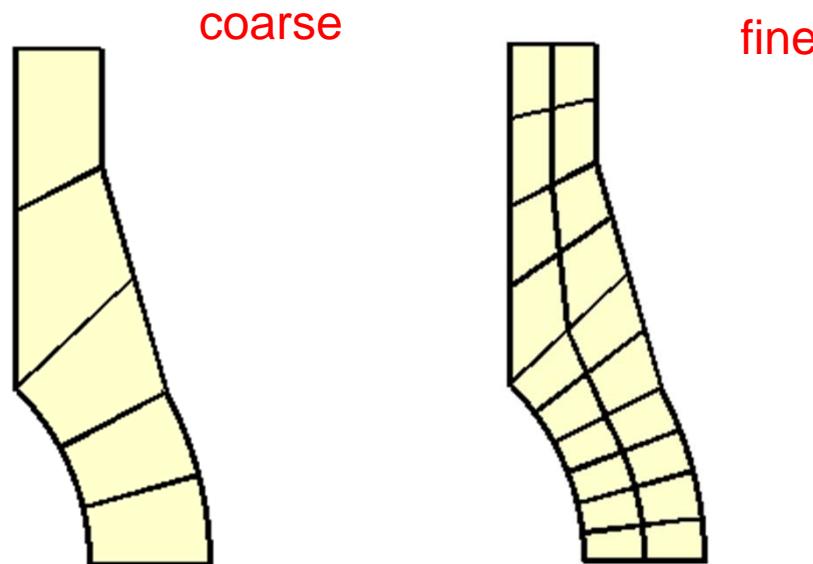
Solid Cylinder/Taper/Sphere – Temperature

- Analysis Type: Linear elastic solid
- Geometry: meters



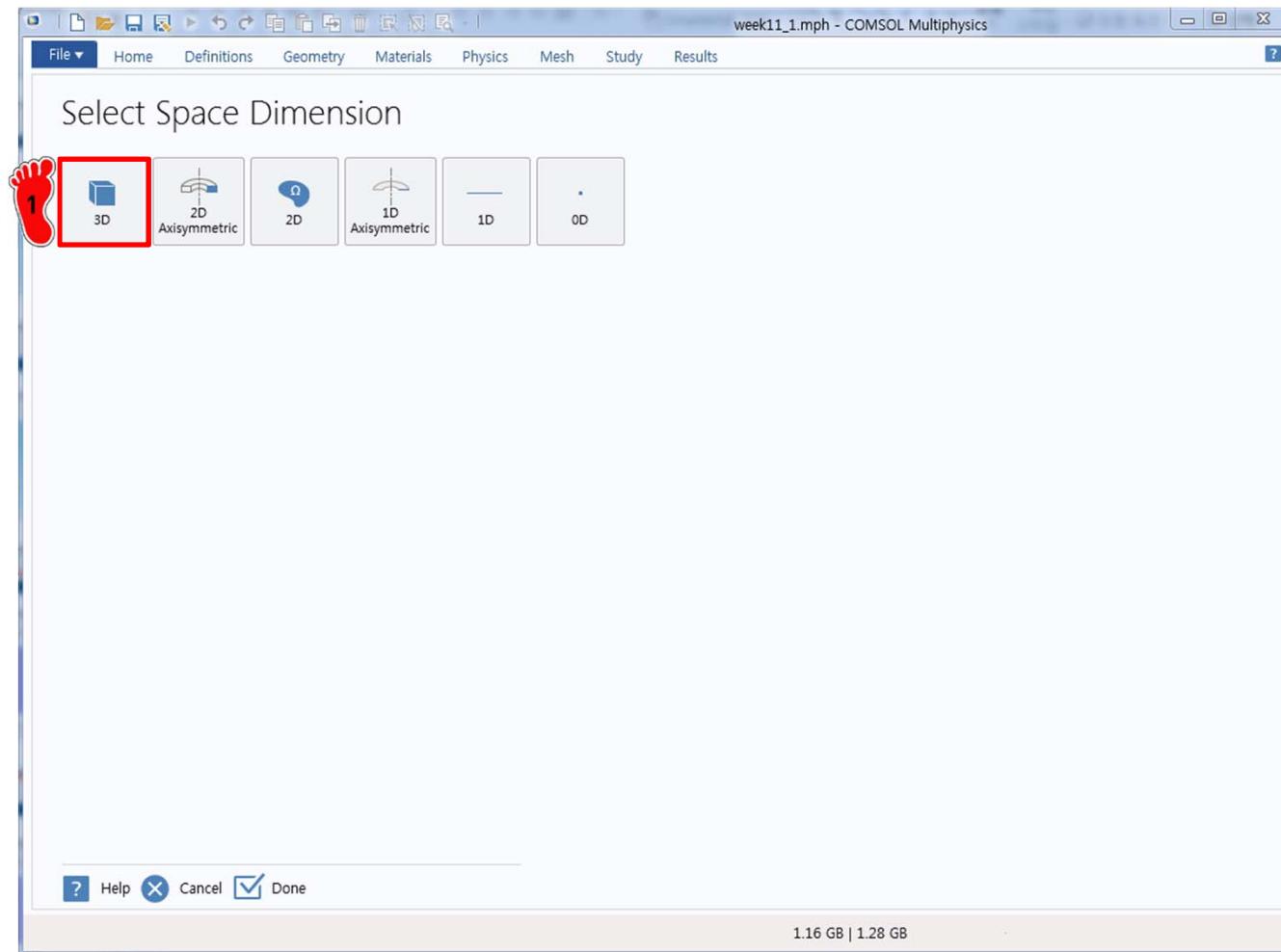
- Loading
 - Linear temperature gradient in the radial and axial direction
 - $T (\text{ }^{\circ}\text{C}) = (x^2 + y^2)^{1/2} + z$
- Boundary conditions
 - Symmetry on xz-plane
 - Symmetry on yz-plane
 - Face on xy-plane: zero z-displacement
 - Face HIH'I': zero z-displacement
- Material properties
 - Isotropic: $E = 210 * 10^3 \text{ MPa}$, $\nu = 0.3$
 - $\alpha = 2.3 * 10^{-4} / \text{ }^{\circ}\text{C}$
- Element types
 - Solid hexahedra, wedges and tetrahedra

- Meshes

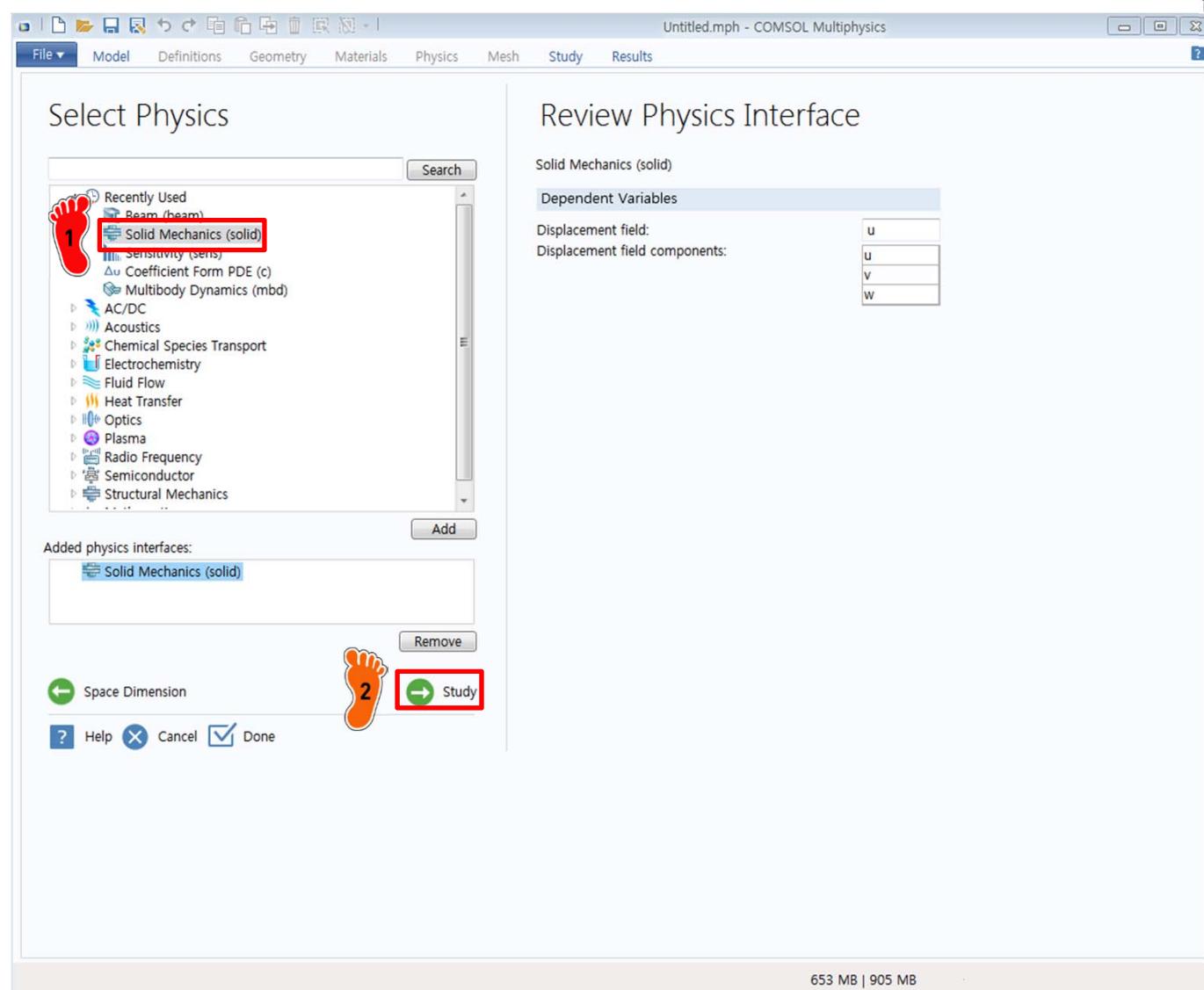


- Output
 - Direct stress (σ_{zz}) at point A: -105 MPa

DIMENSION SELECTION



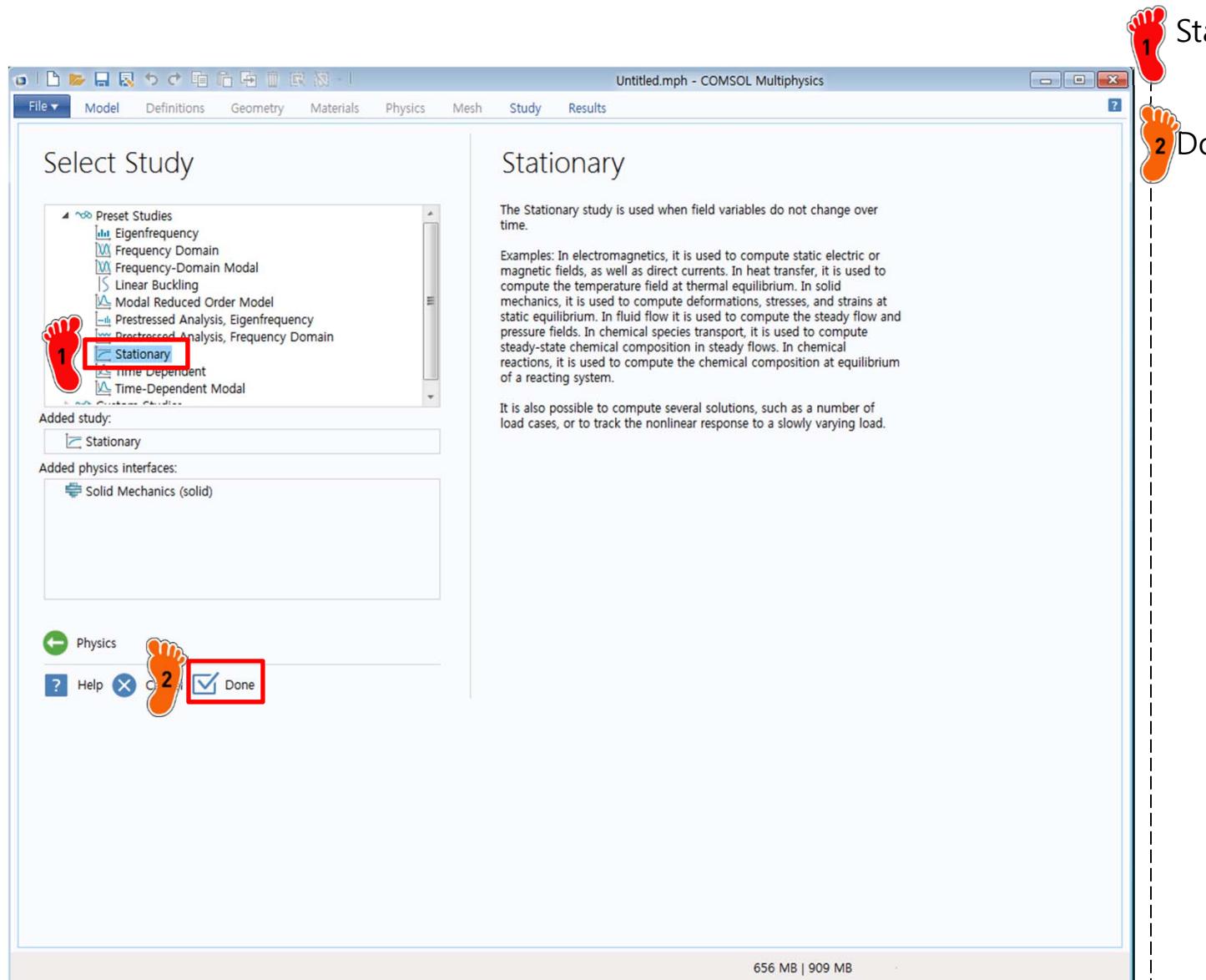
PHYSICS SELECTION



1 Structural Mechanics 의
Solid Mechanics 선택

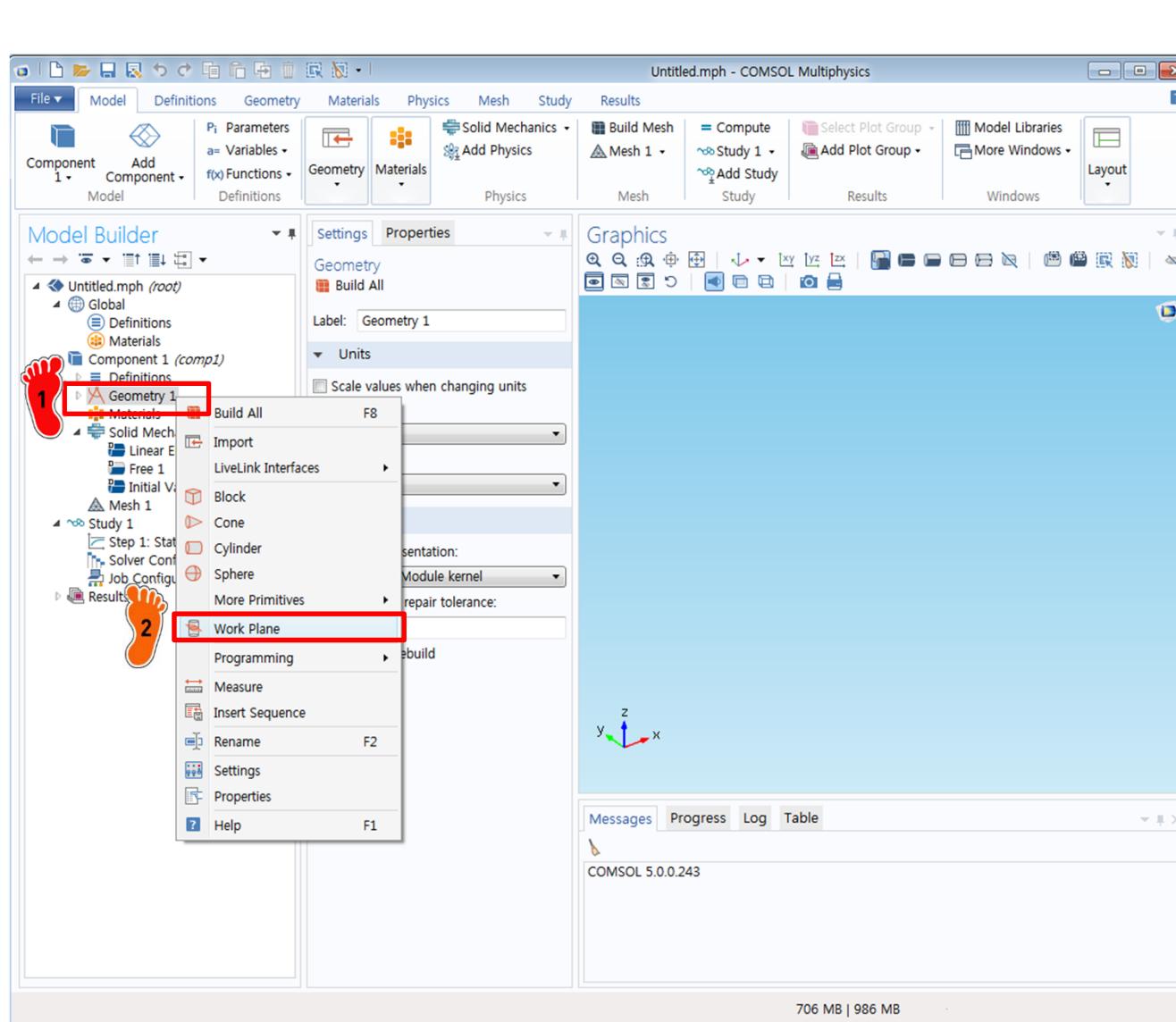
2 Study 클릭

STUDY TYPE SELECTION



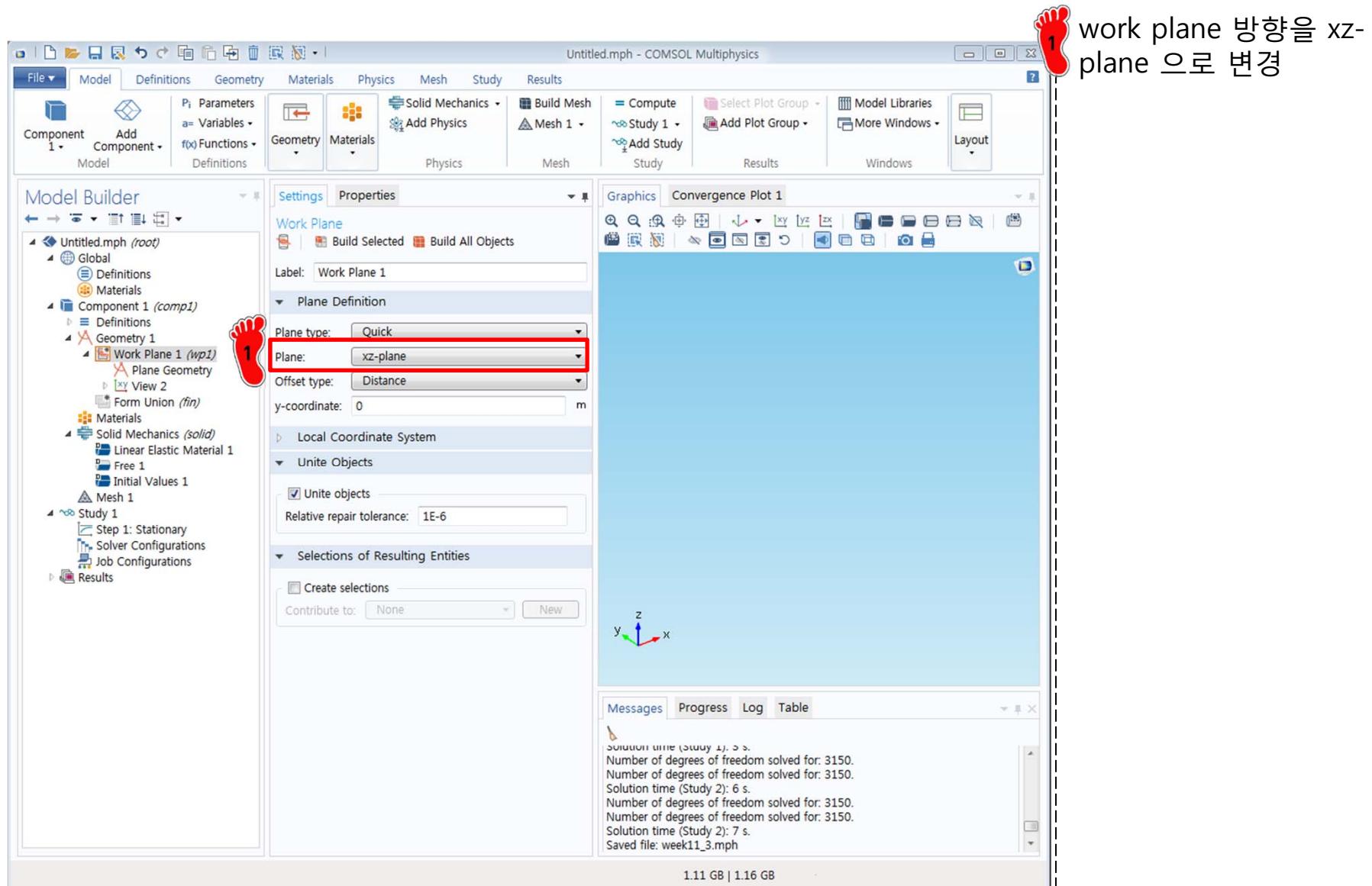
1 Stationary 선택
2 Done 클릭

GEOMETRY CREATION

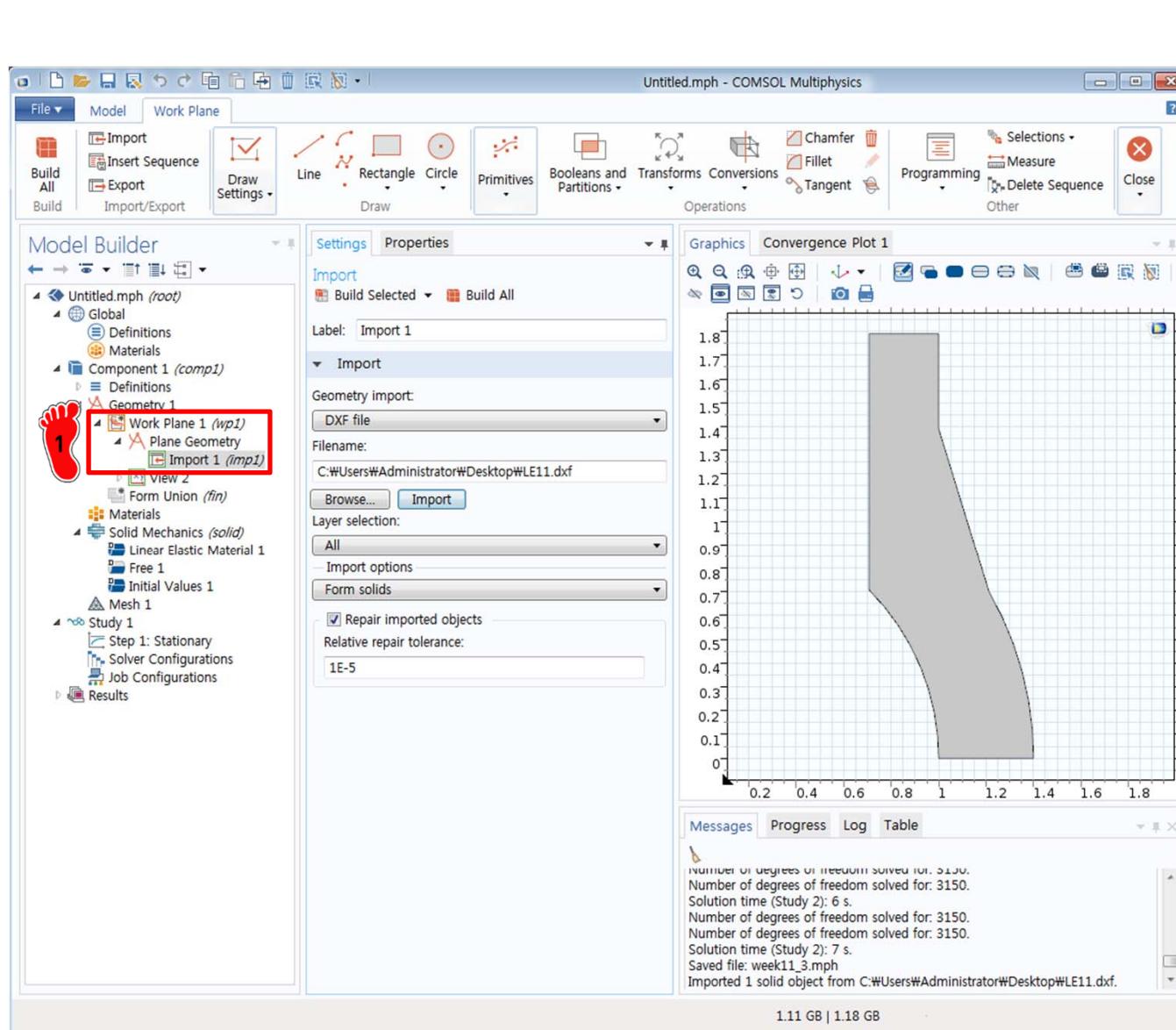


- 1 Geometry 1 마우스 우클릭
- 2 Work Plane 클릭

GEOMETRY CREATION

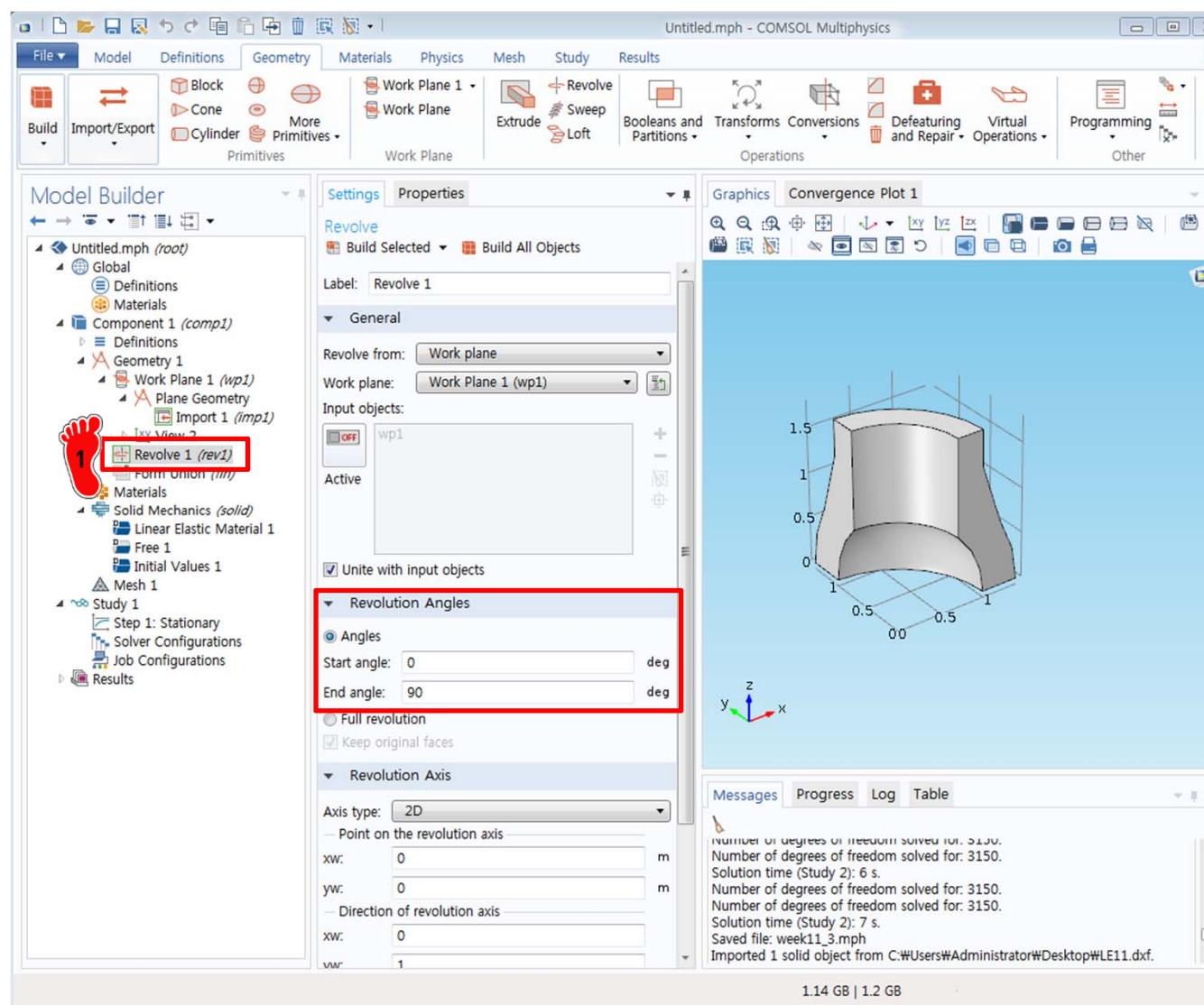


GEOMETRY CREATION



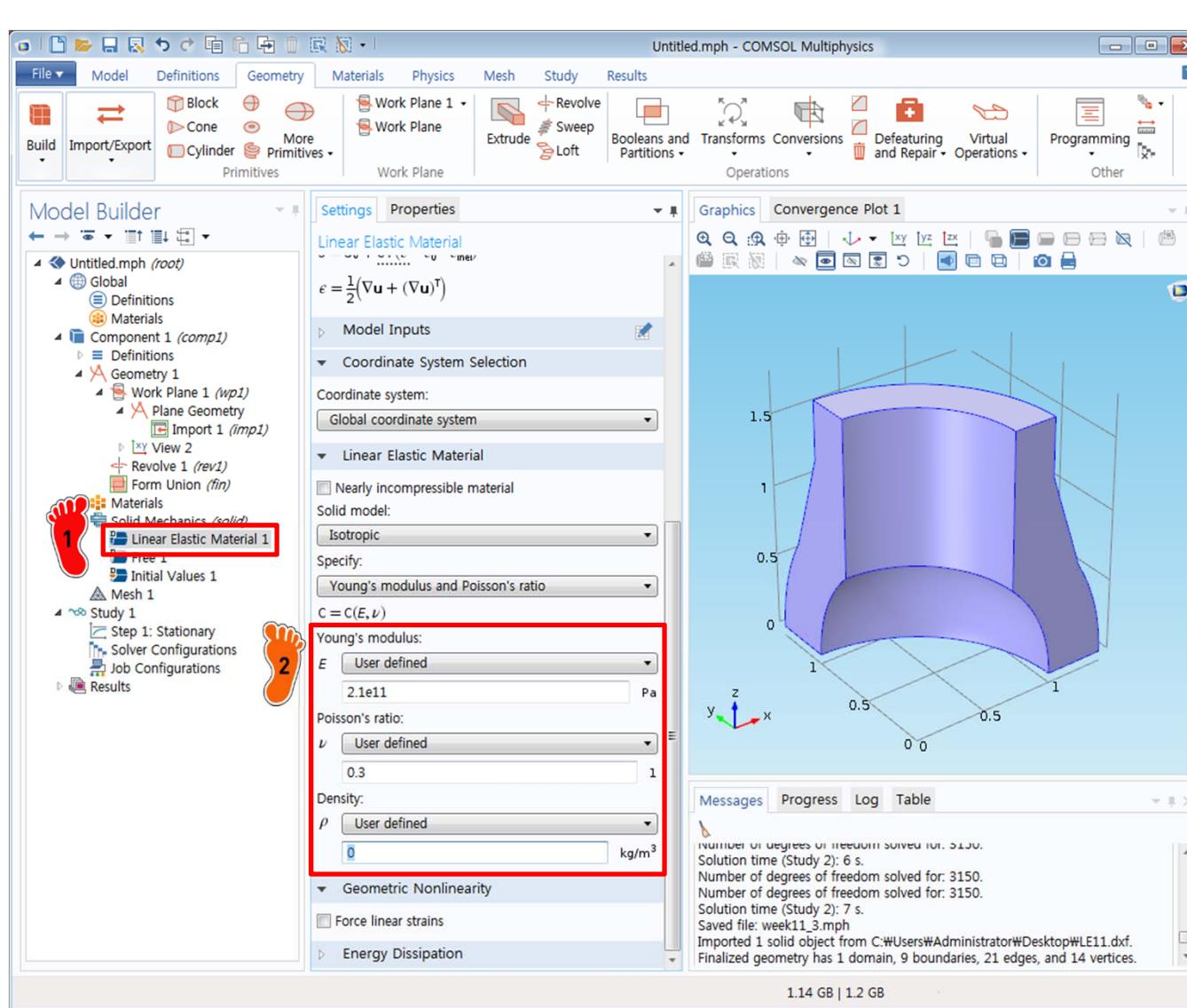
Plane Geometry에서
Import 메뉴를 이용하여 주
어진 캐드 기하형상을 삽입

GEOMETRY CREATION



1 Revolve 메뉴를 이용하여
90도 회전시킨 형상을 생성

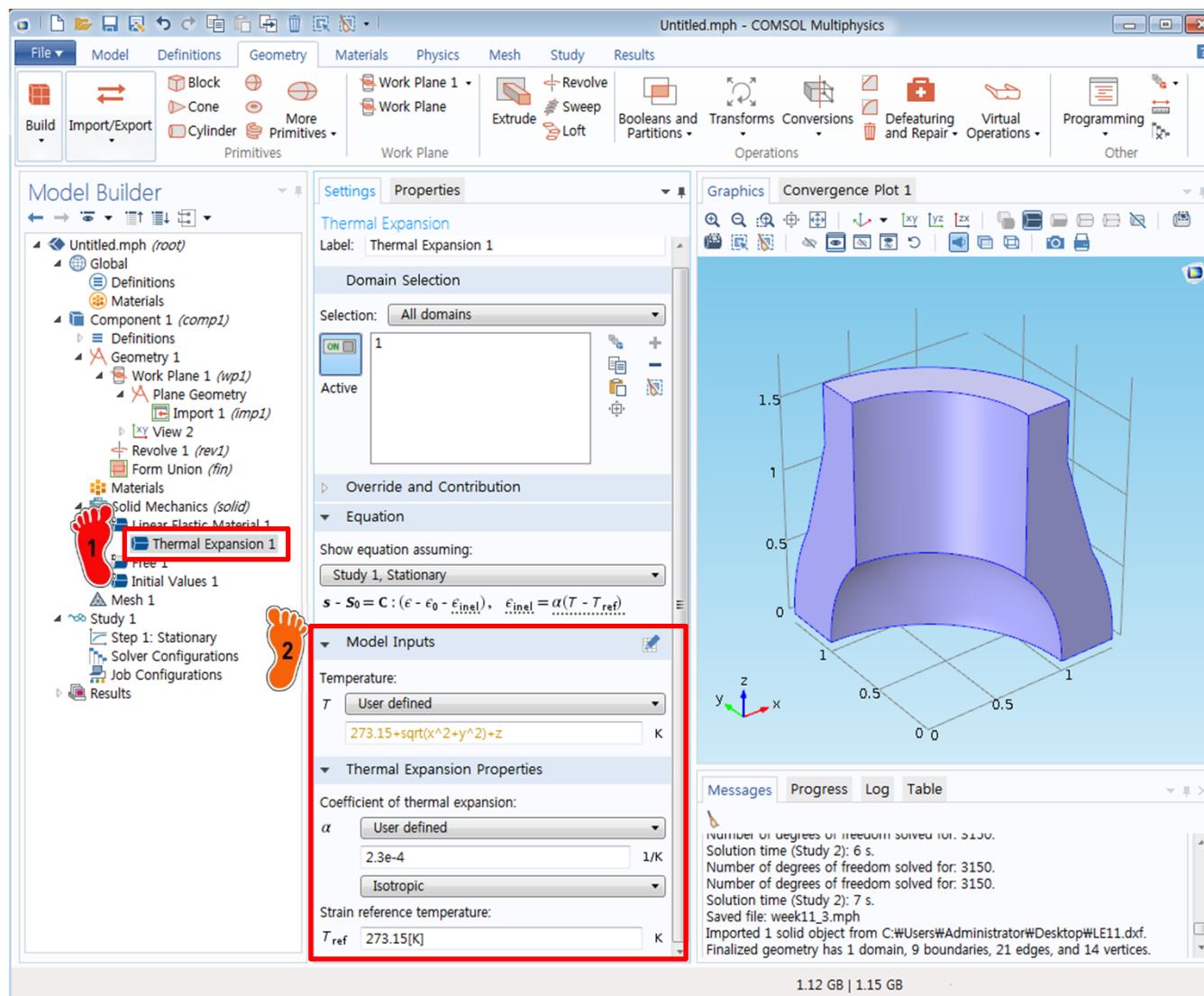
MATERIAL PROPERTY



1 Linear Elastic Material 클릭
 2 E: 2.1e11
 mu: 0.3
 rho : 0

입력

MATERIAL PROPERTY

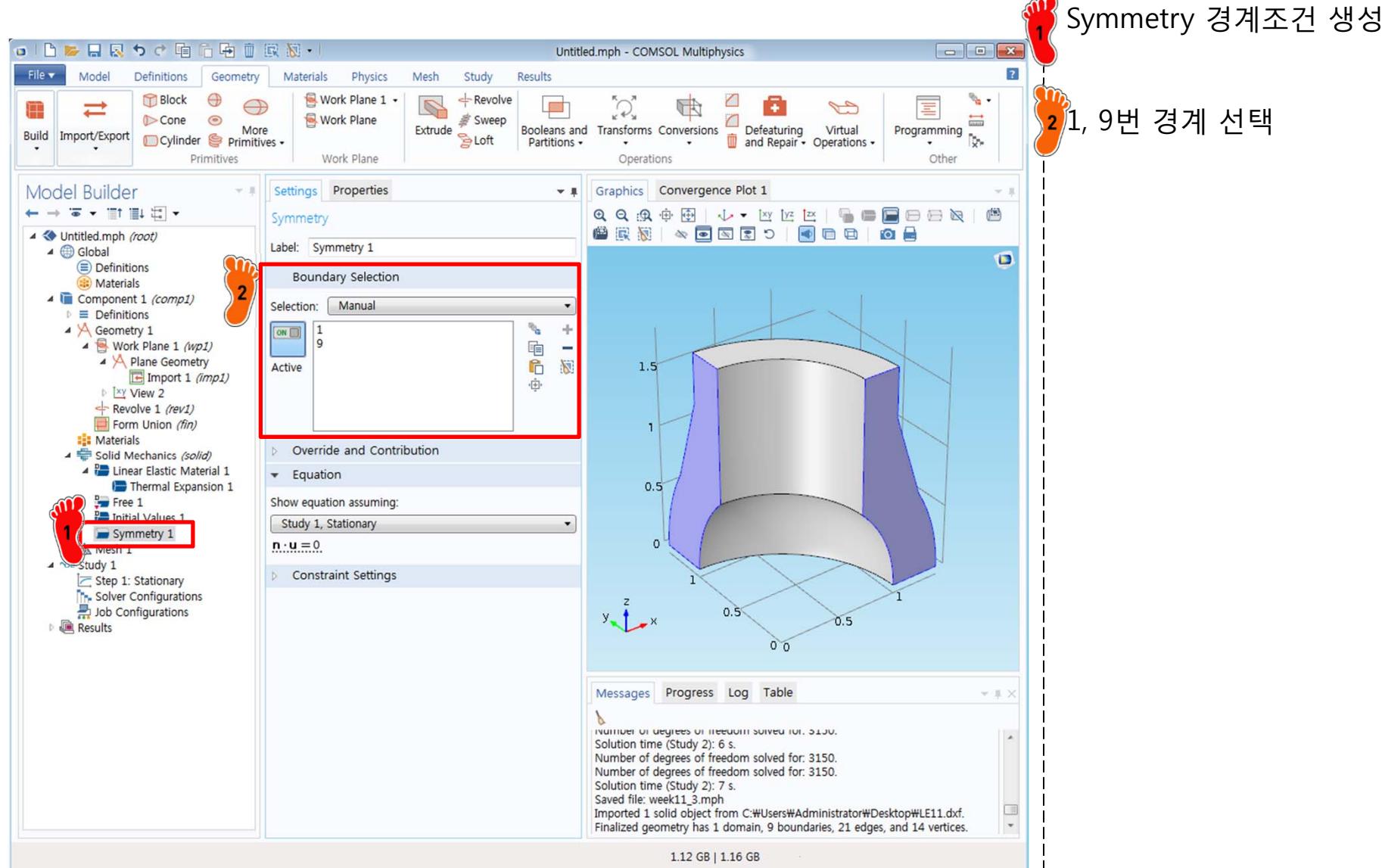


1 Linear Elastic Material 하위
메뉴로 Thermal Expansion
추가

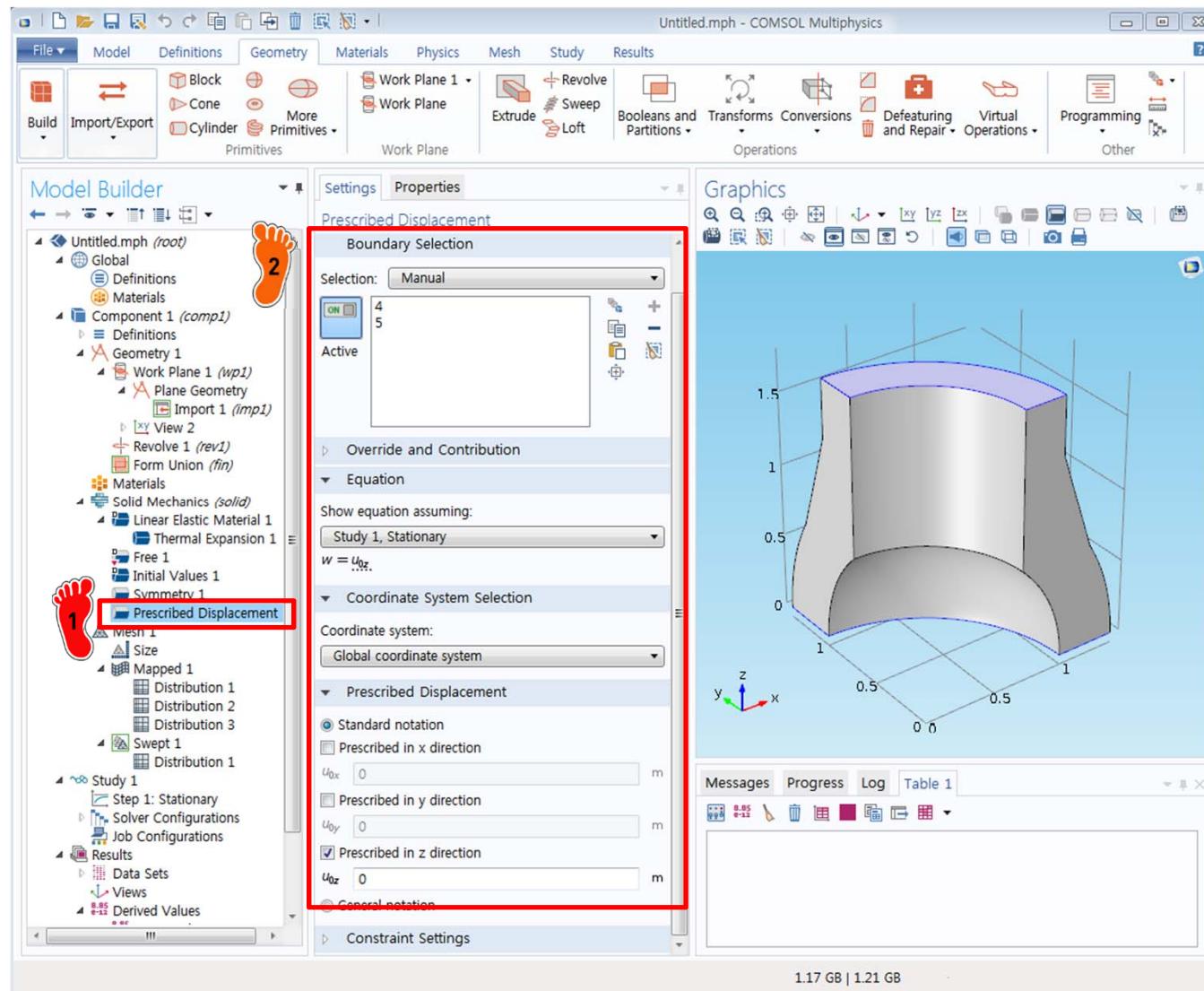
2 Temperature:
 $273.15 + \sqrt{x^2 + y^2} + z$
 $\alpha: 2.3e-4$
 Strain reference
 temperature: 273.15

입력

BOUNDARY CONDITION



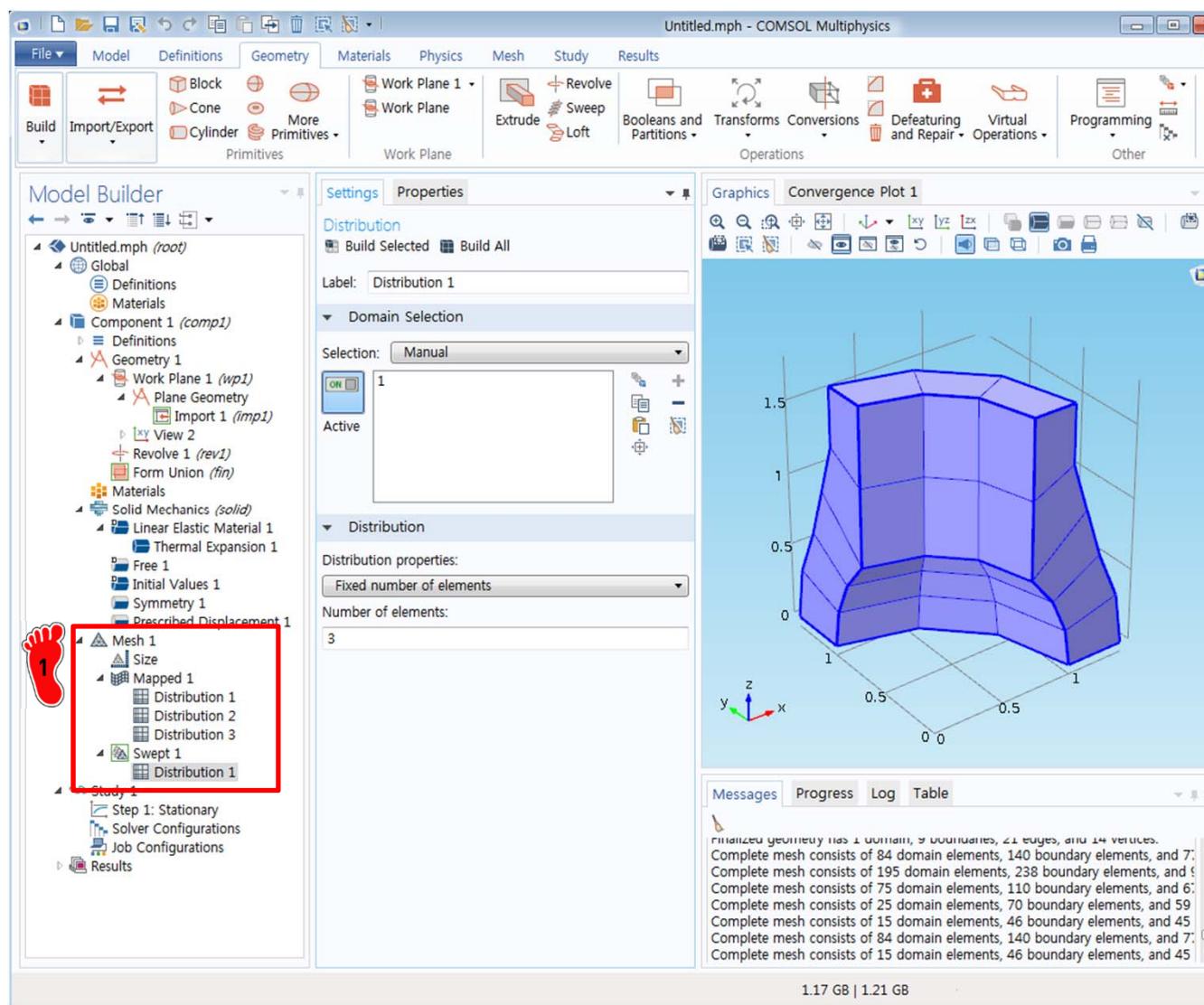
BOUNDARY CONDITION



1 Prescribed Displacement
경계조건 생성

2 4, 5번 경계 선택 후 z 방향
변위에 0 입력

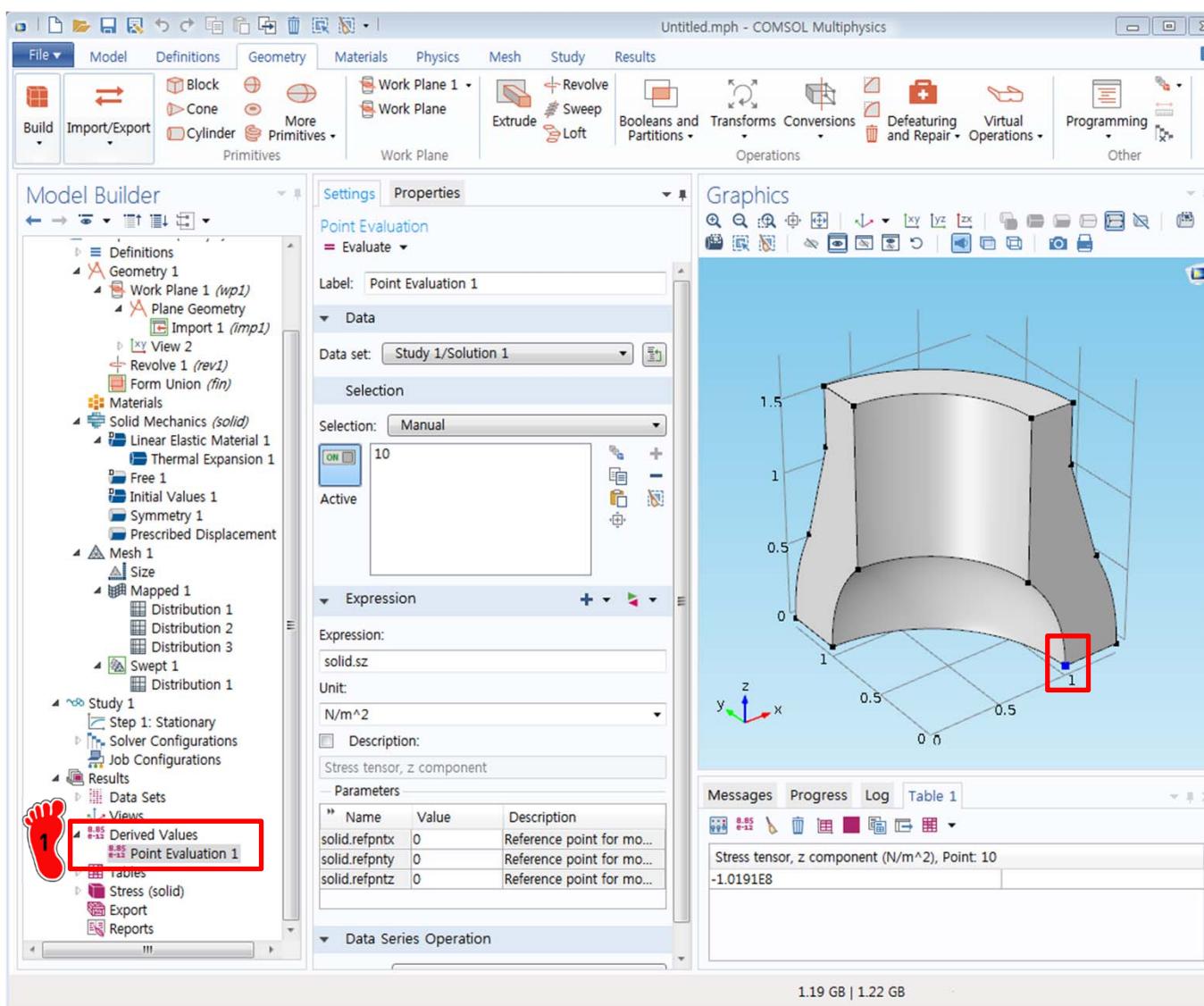
MESH



1 Mapped mesh 와 Swept 기
능을 이용하여 요소 생성

그 후, 해석실행

POST-PROCESSING

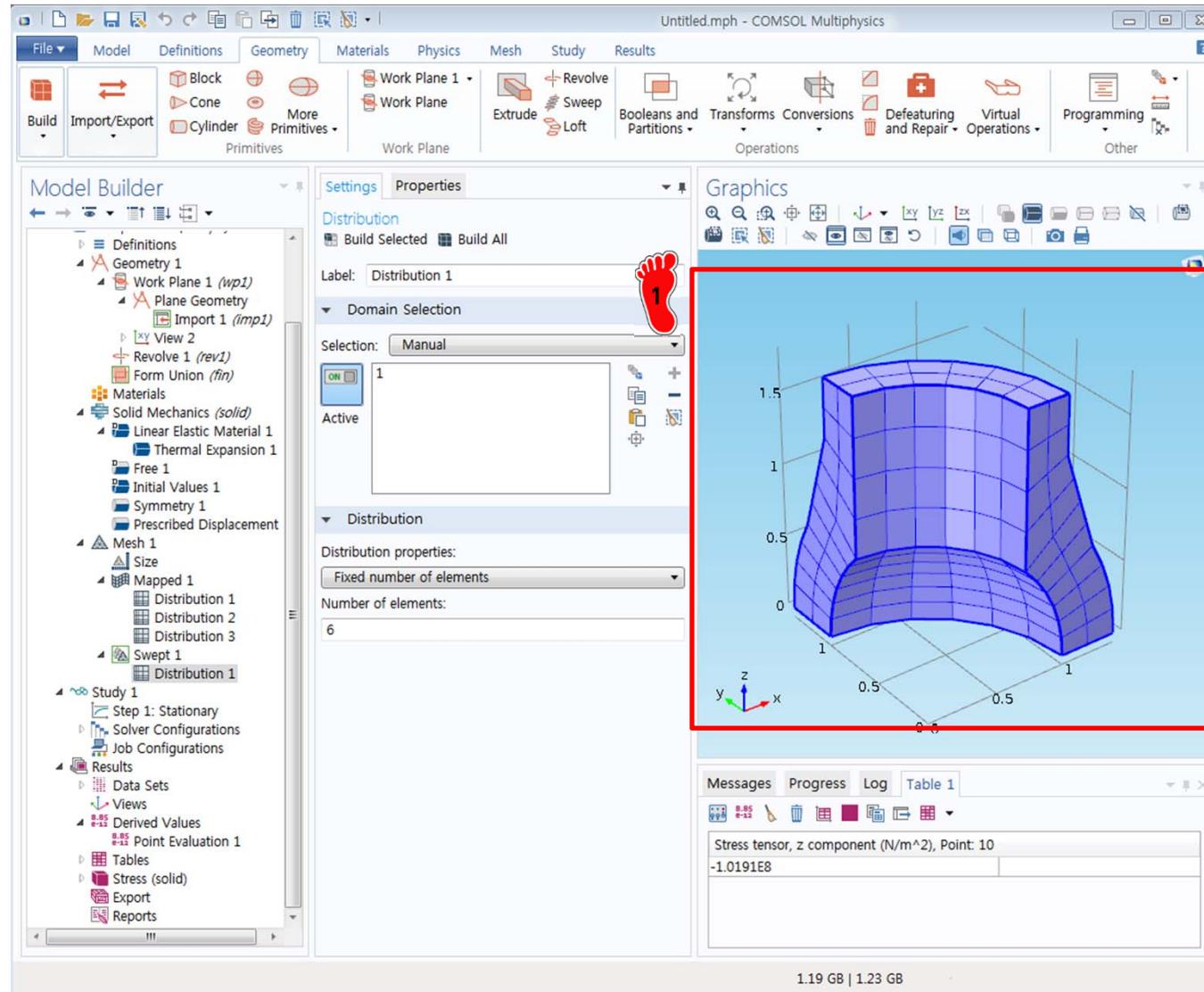


Derived Values 의 Point Evaluation 기능을 이용하여 10번 절점의 solid.sz 응력을 계산

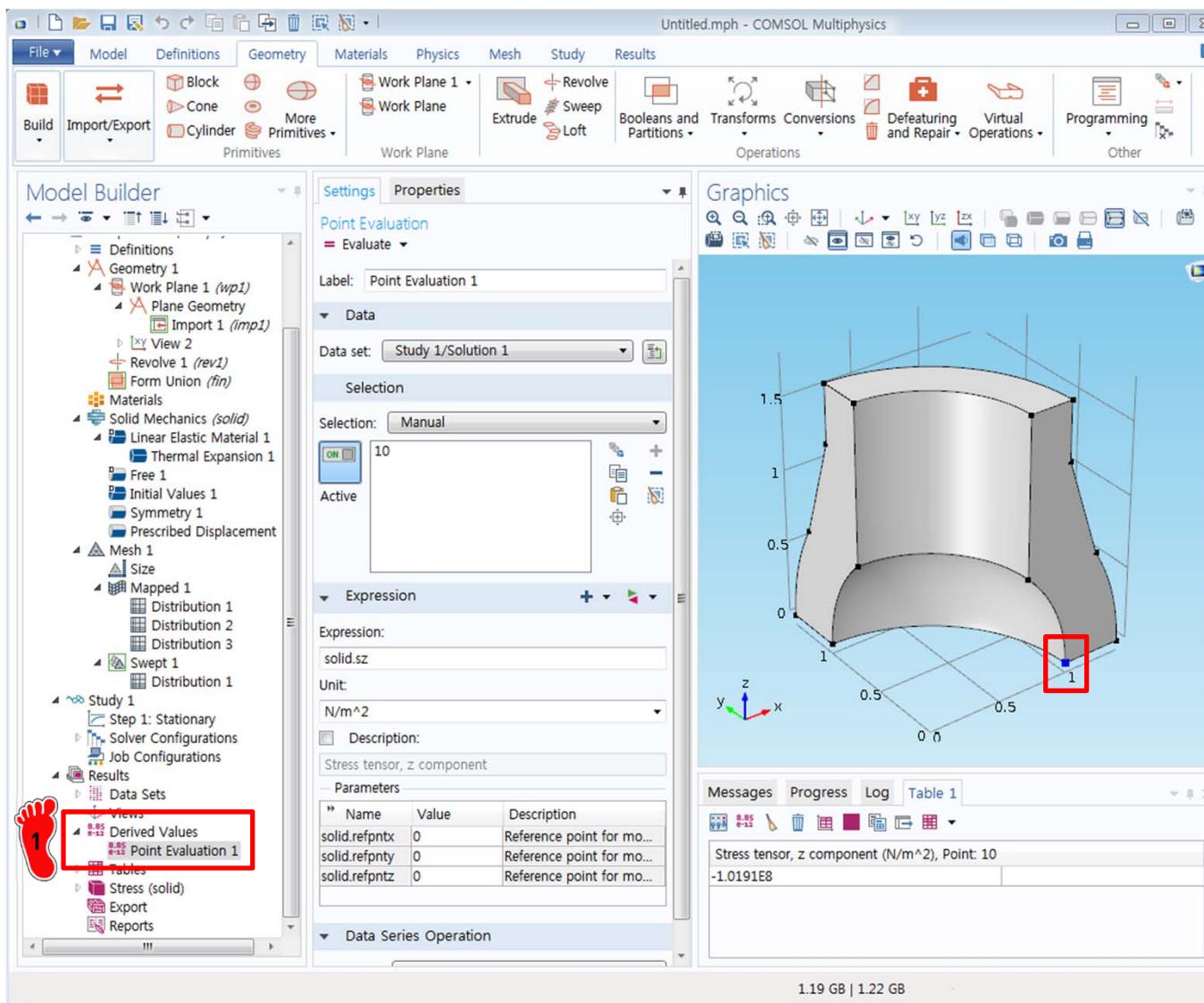
결과값: -101.91 MPa

MESH

1 Fine mesh로 변경 후 해석
실행



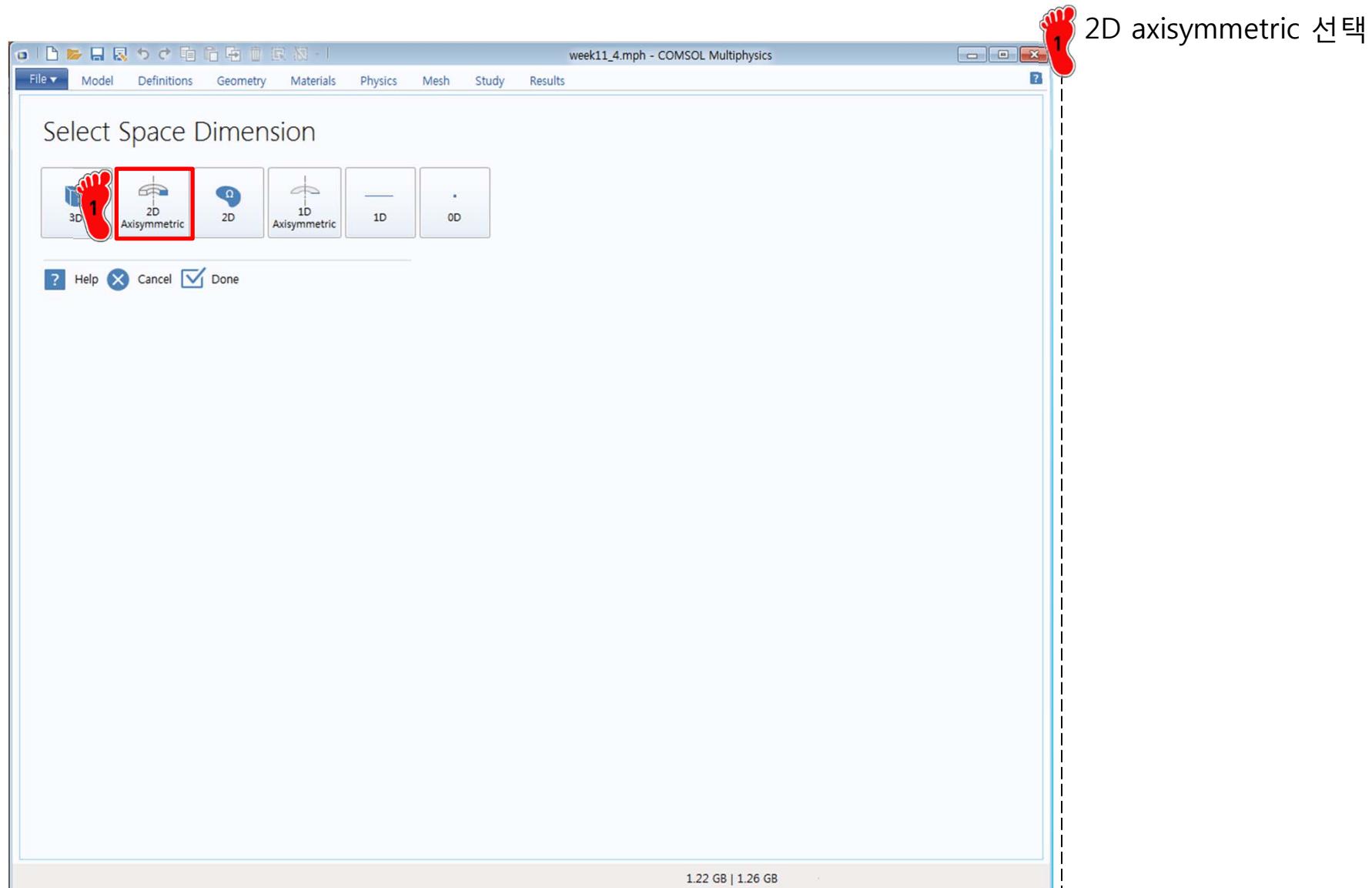
POST-PROCESSING



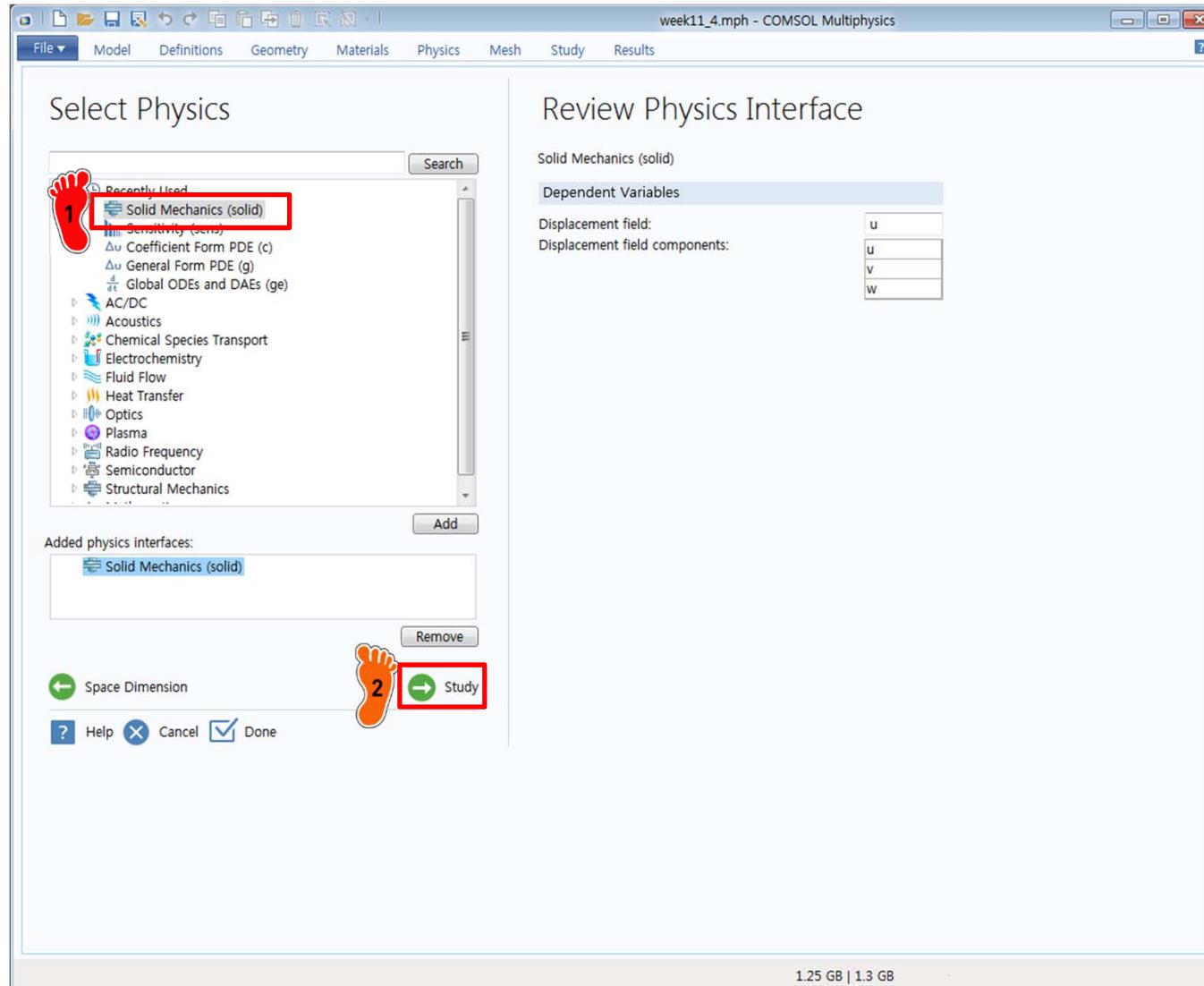
Derived Values 의 Point Evaluation 기능을 이용하여 10번 절점의 solid.sz 응력을 계산

결과값: -104.24 MPa

DIMENSION SELECTION

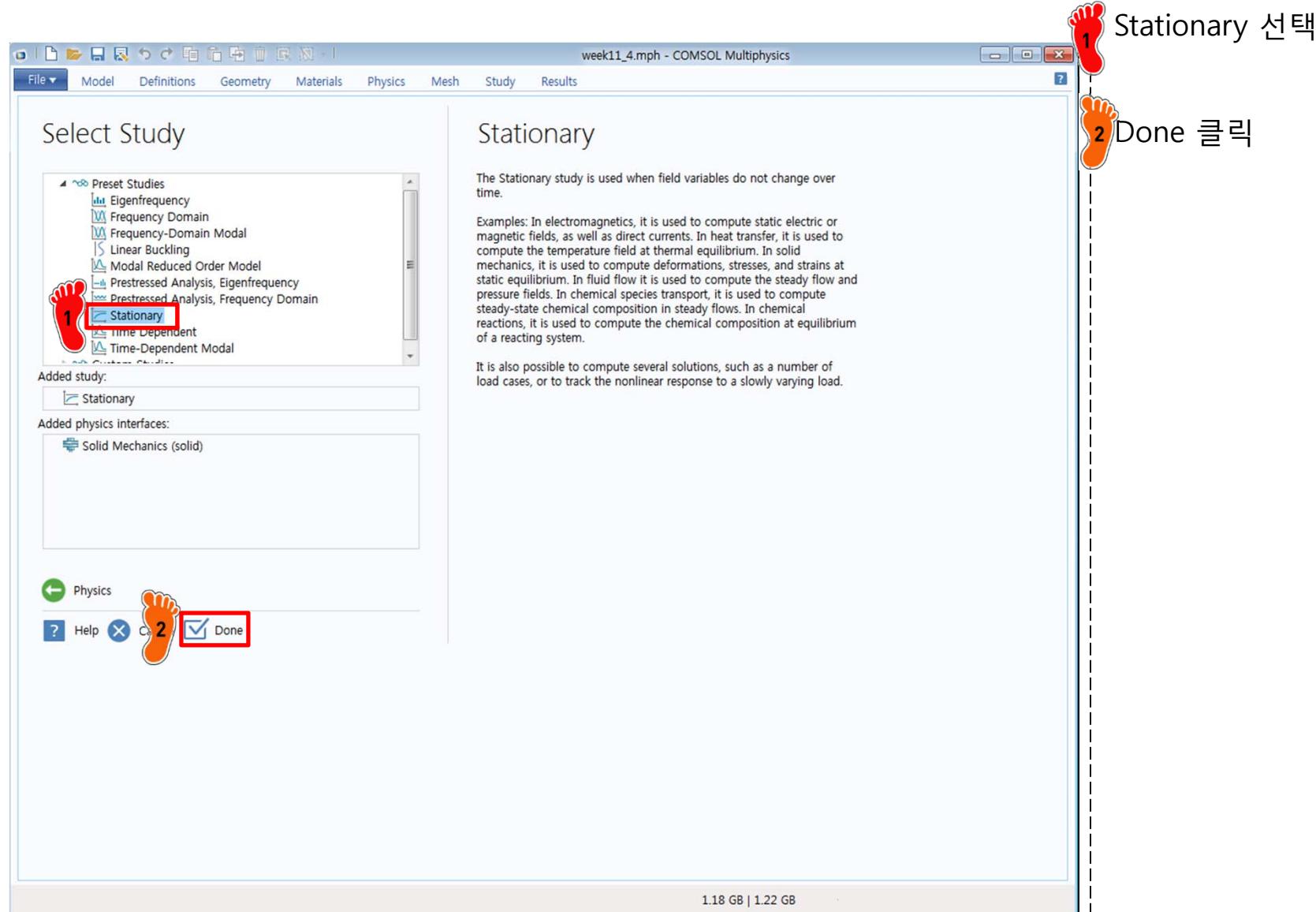


PHYSICS SELECTION



1 Structural Mechanics 의
Solid Mechanics선택
2 Study 클릭

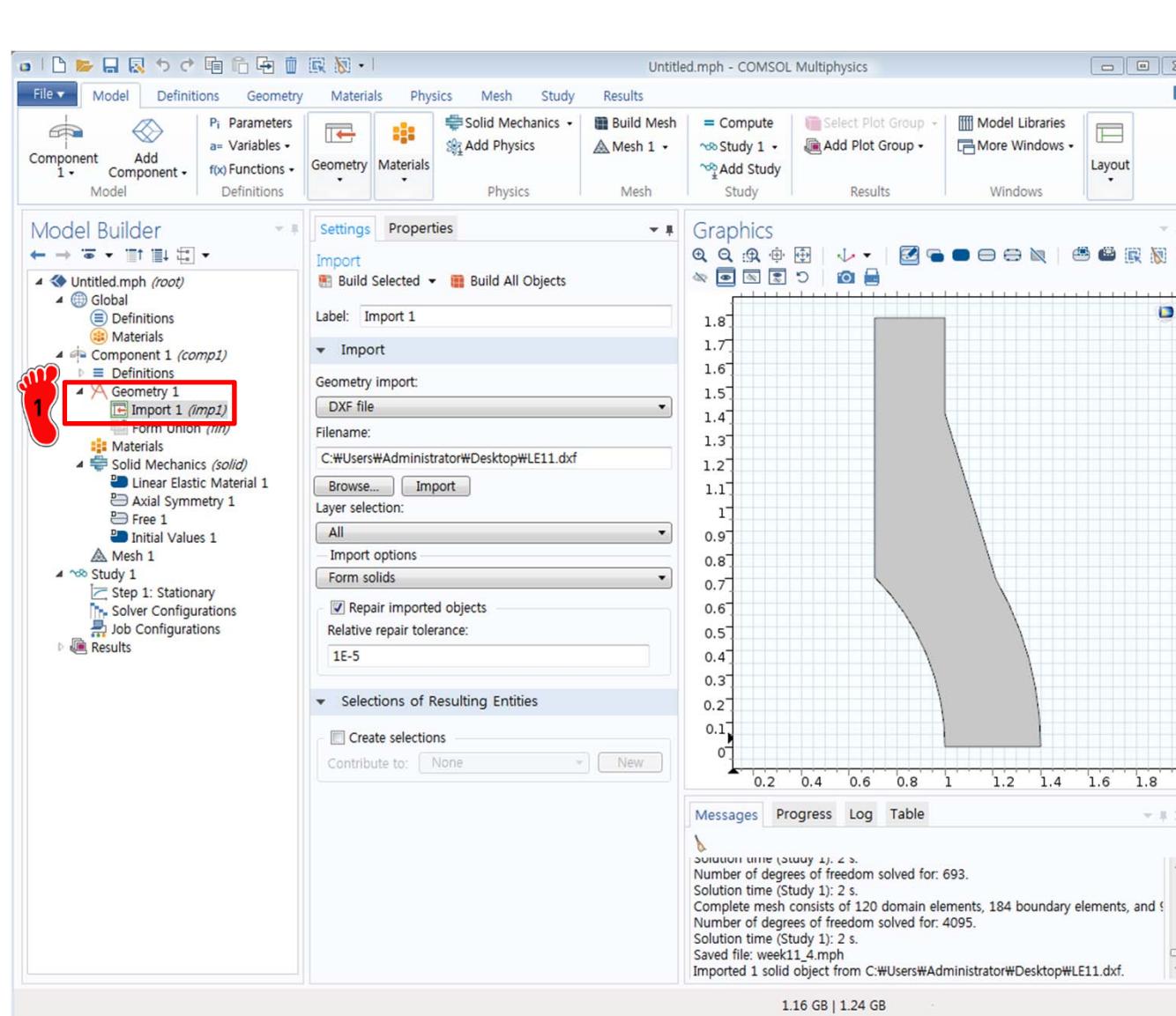
STUDY TYPE SELECTION



1 Stationary 선택

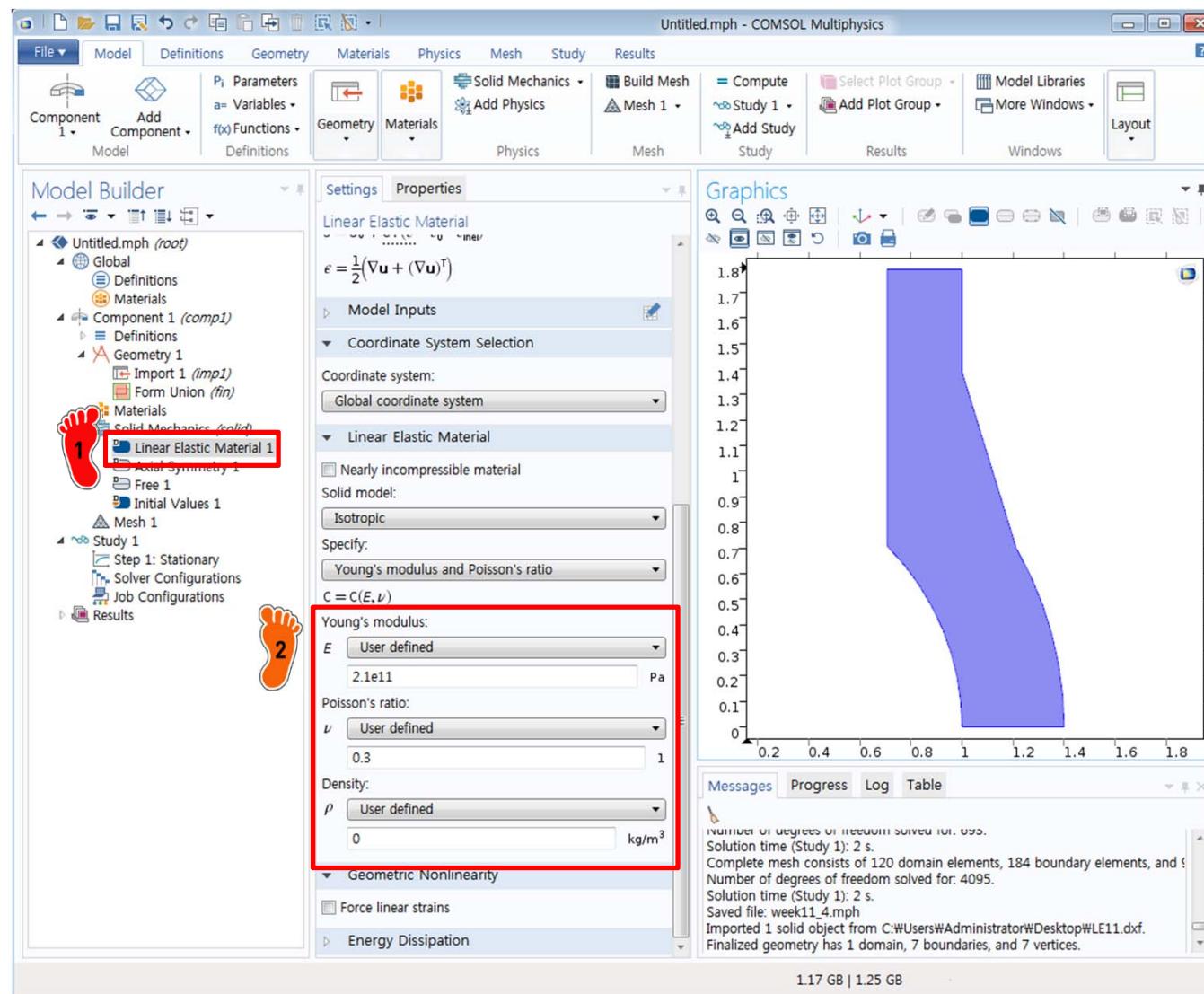
2 Done 클릭

GEOMETRY CREATION



1 Import 메뉴를 이용하여 주
어진 캐드 기하형상을 삽입

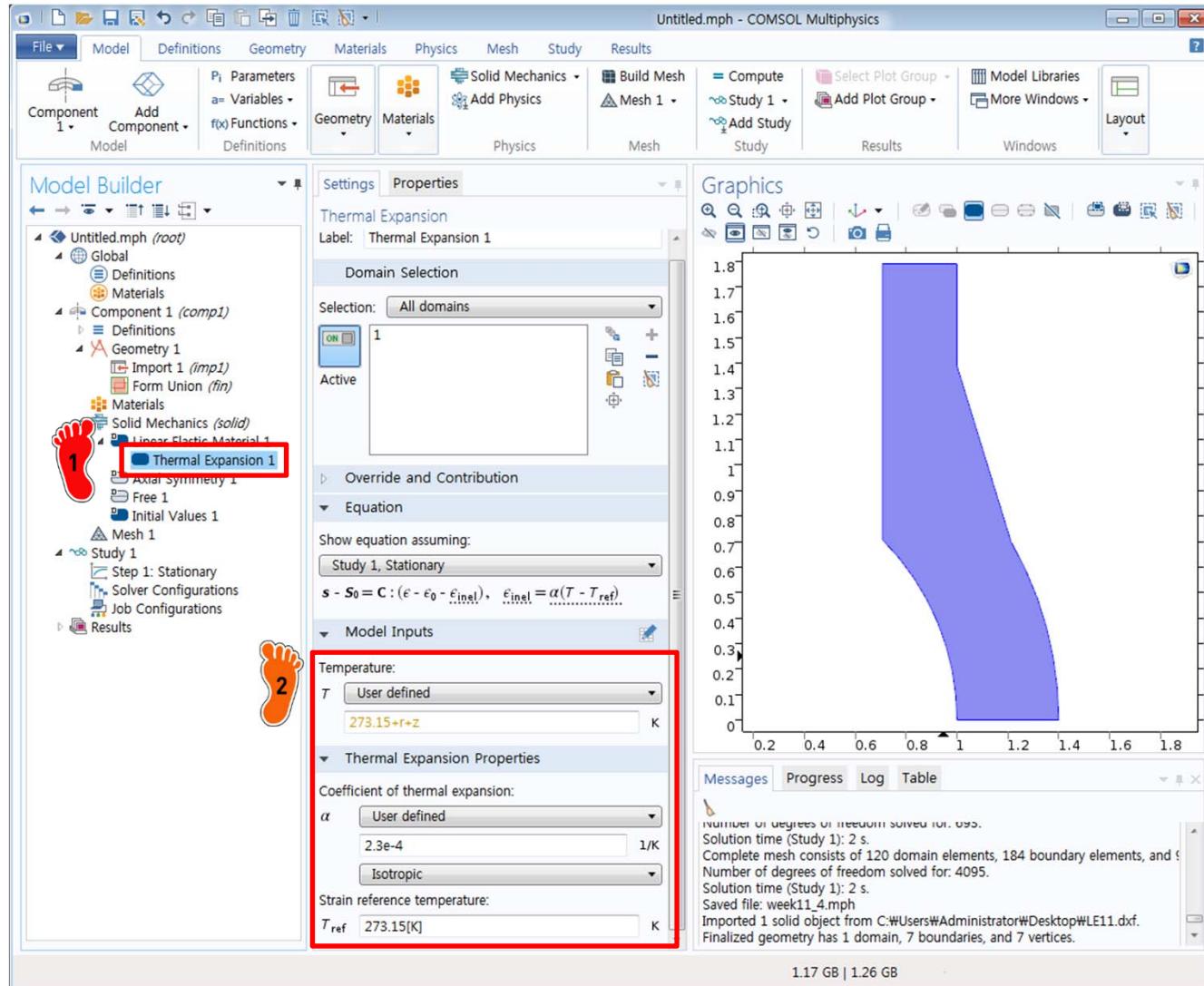
MATERIAL PROPERTY



- 1 Linear Elastic Material 클릭
- 2 E: 2.1e11
mu: 0.3
rho : 0

입력

MATERIAL PROPERTY

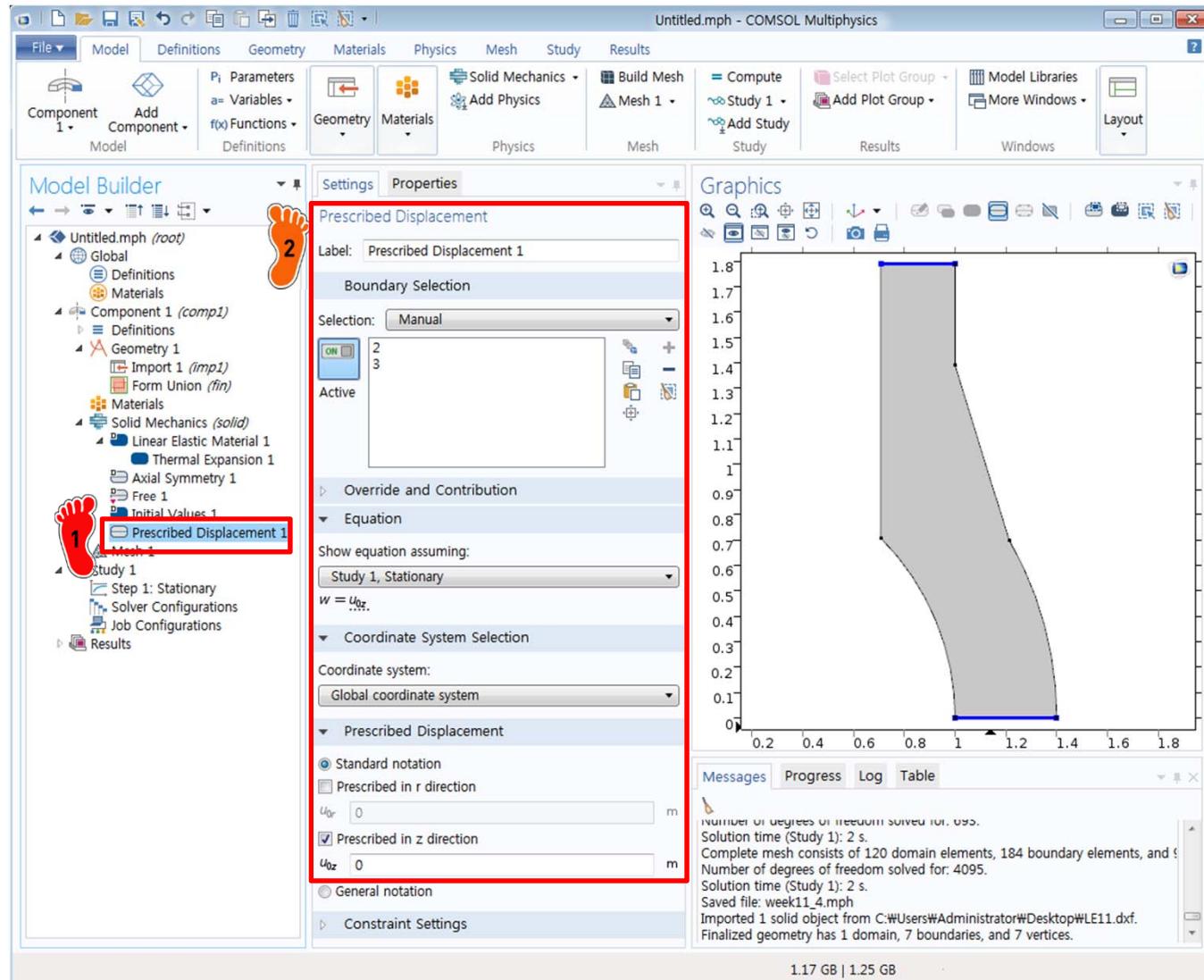


1 Linear Elastic Material 하위
메뉴로 Thermal Expansion
추가

2 Temperature:
 $273.15 + \sqrt{x^2 + y^2} + z$
 $\alpha: 2.3e-4$
 Strain reference
 temperature: 273.15

입력

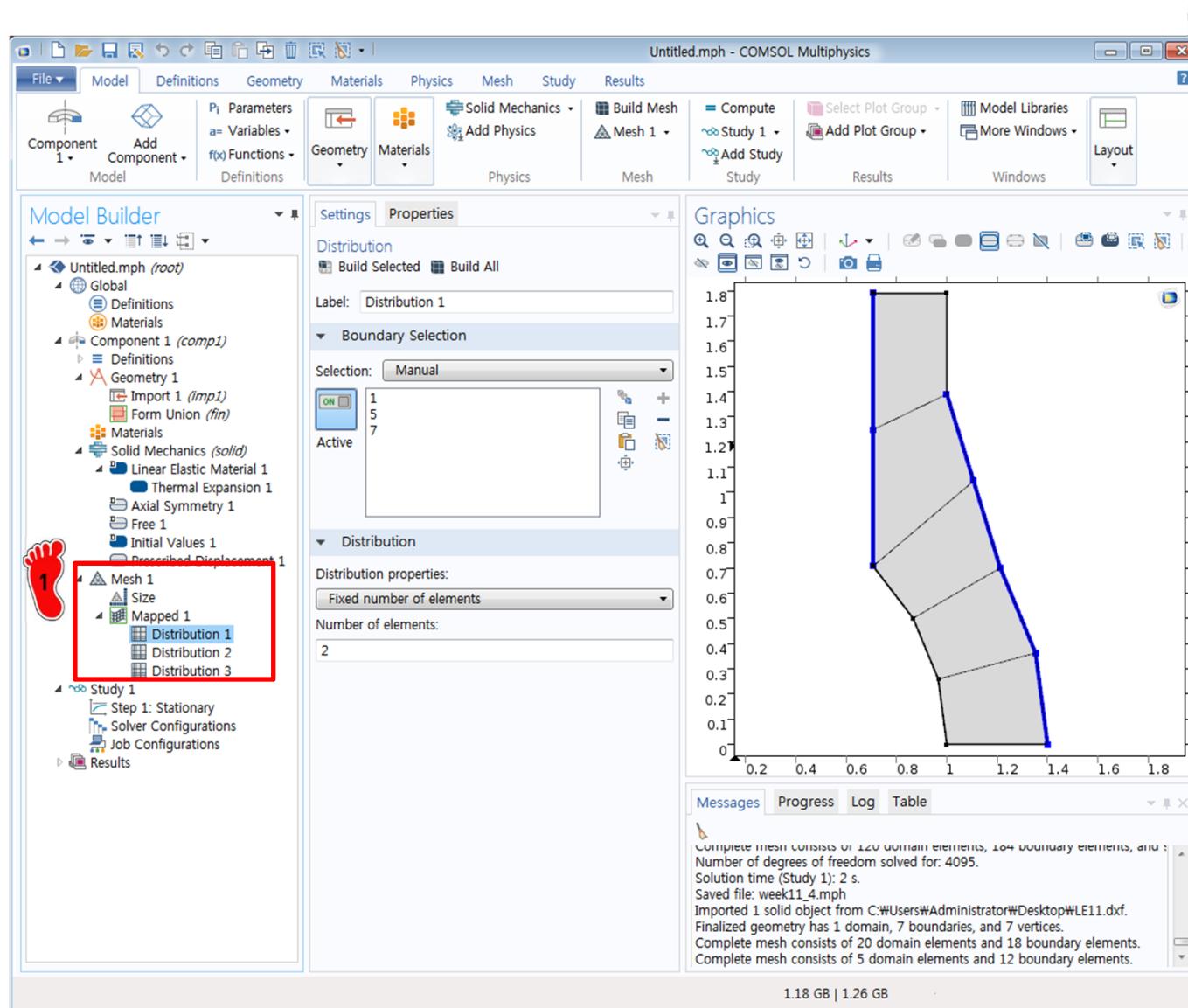
BOUNDARY CONDITION



1 Prescribed Displacement
경계조건 생성

2, 3번 경계 선택 후 z 방향
변위에 0 입력

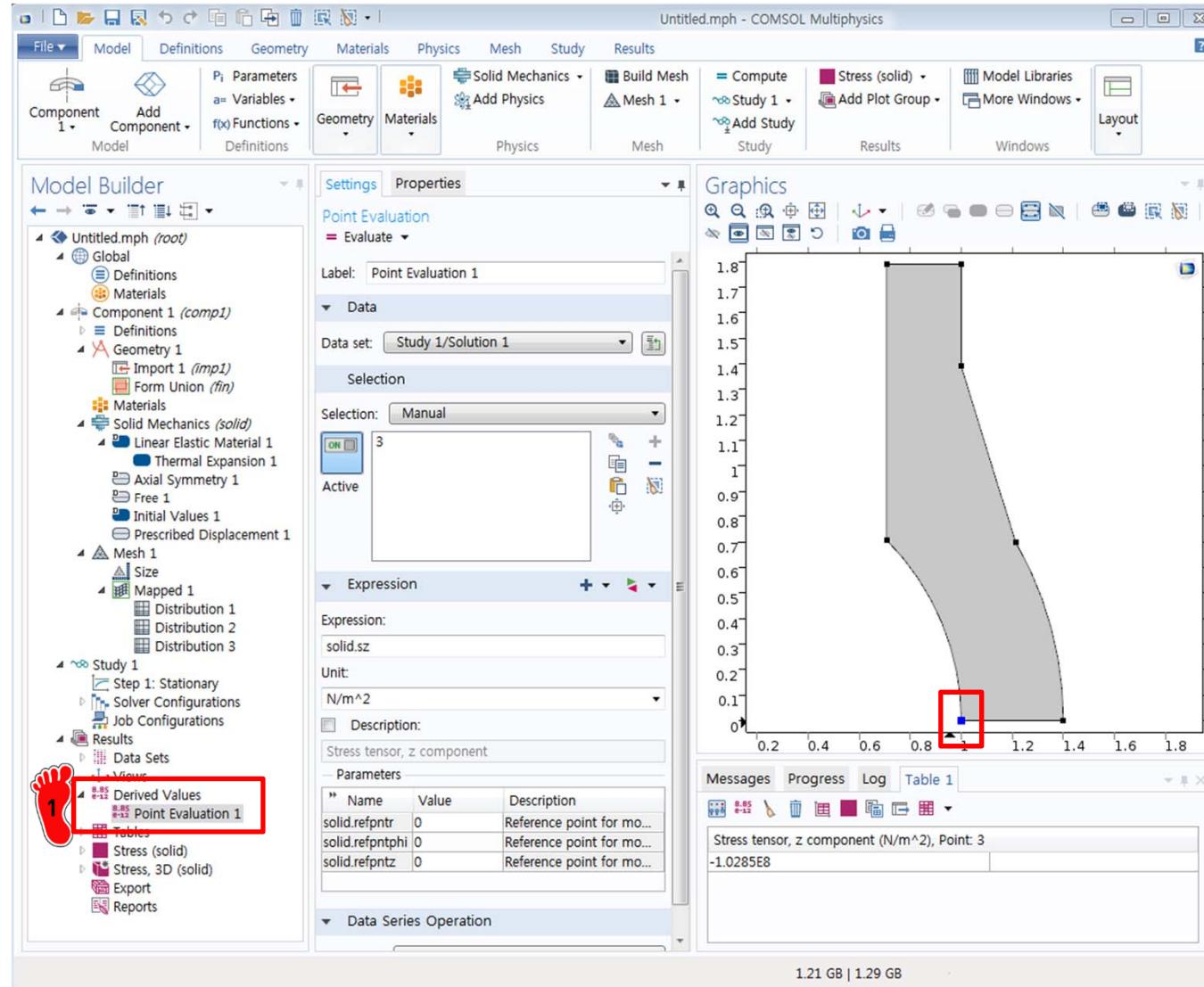
MESH



1 Mapped mesh 기능을 이용
하여 요소 생성

그 후, 해석실행

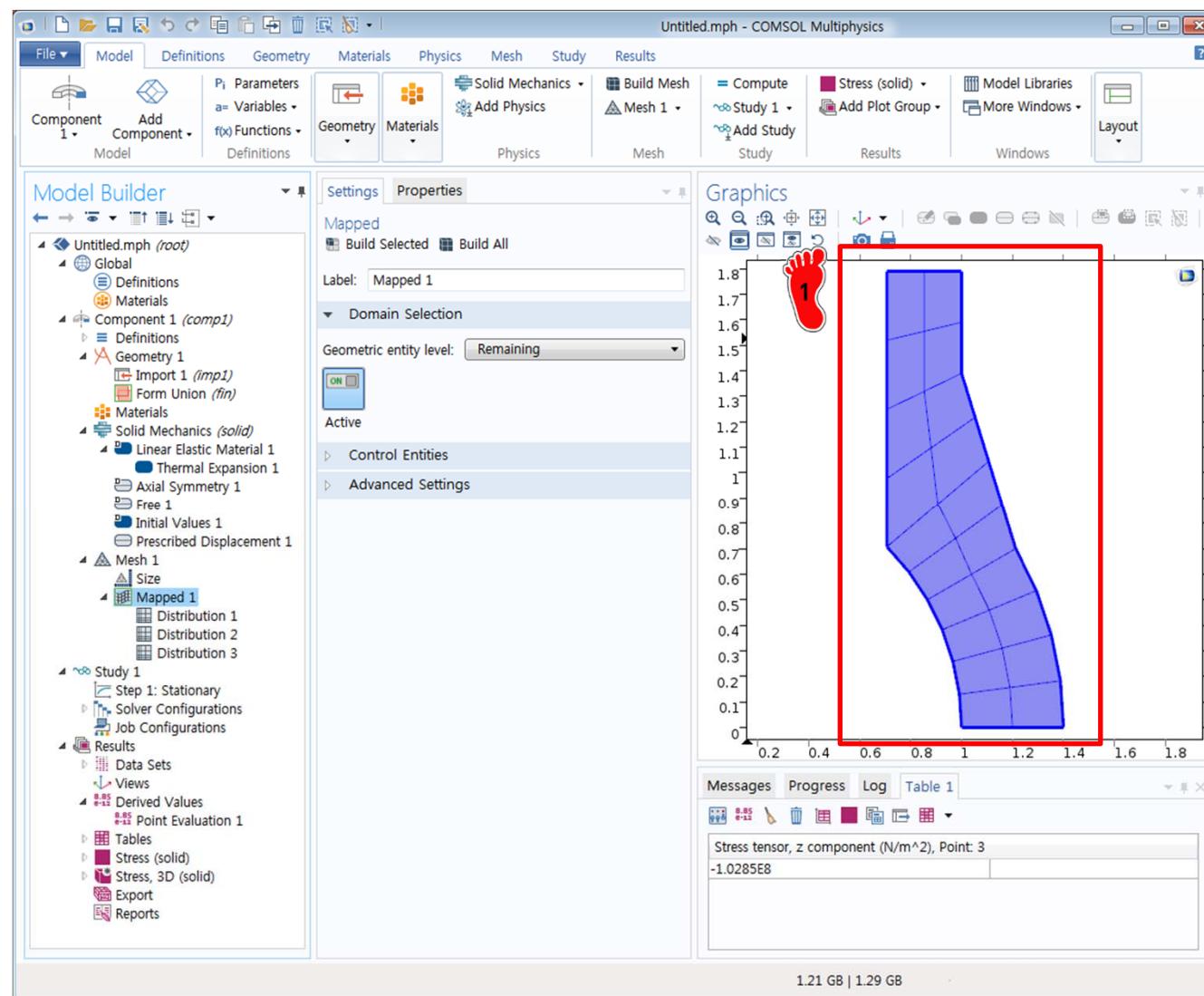
POST-PROCESSING



Derived Values 의 Point Evaluation 기능을 이용하여 3번 절점의 solid.sz 응력을 계산

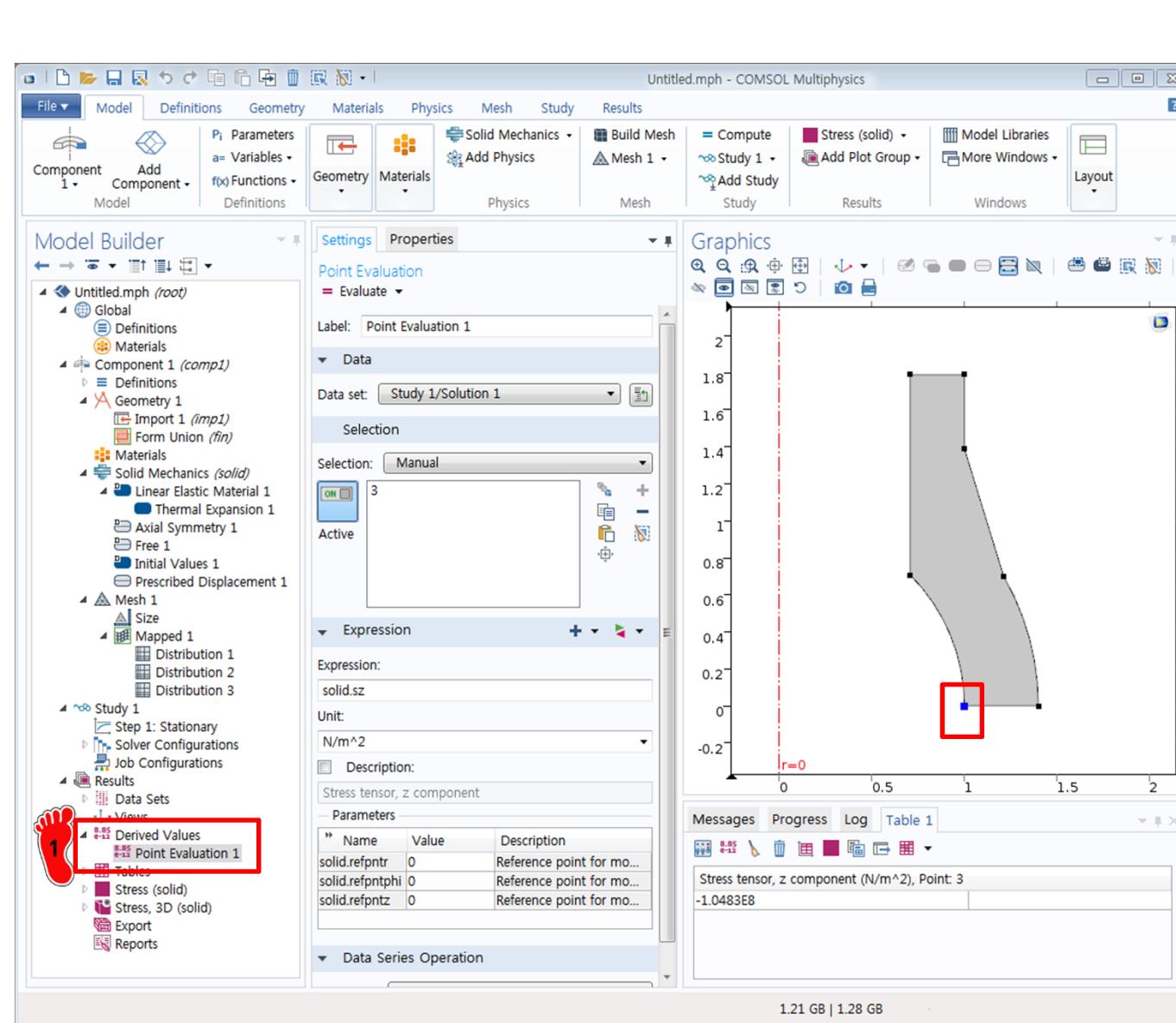
결과값: -102.85 MPa

MESH



1 Fine mesh로 변경 후 해석
실행

POST-PROCESSING



Derived Values 의 Point Evaluation 기능을 이용하여 3번 절점의 solid.sz 응력을 계산

결과값: -104.83 MPa

SUMMARY

| Software | Order | Element | Coarse mesh | | Fine mesh | |
|-----------------|--------------|----------------|--------------------|--------------|------------------|--------------|
| | | | Result | Error | Result | Error |
| COMSOL | 2 | axisymmetry | -102.85 | 2.05% | -104.83 | 0.16% |
| COMSOL | 2 | hexahedron | -101.91 | 2.94% | -104.24 | 0.72% |

HOMEWORK: SOLID MECHANICS

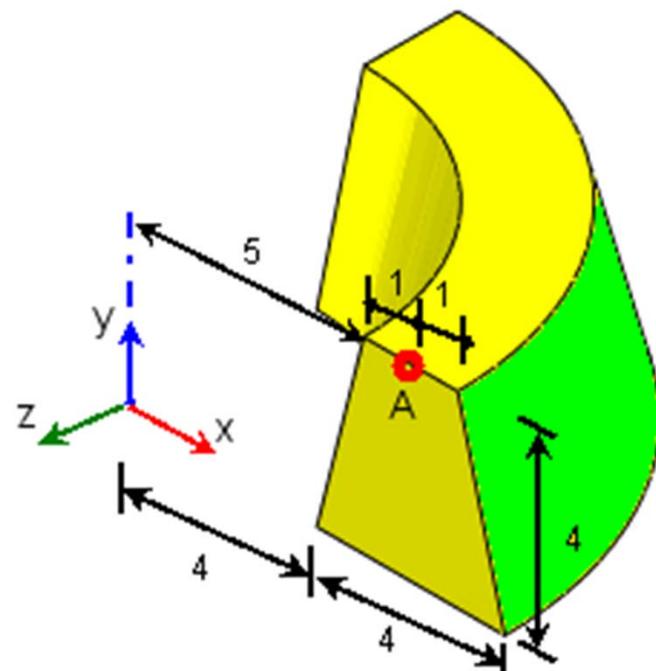
Tapered Thick Shell under Pressure and Gravity Loads

Determine the hoop stress at point A of a tapered thick shell subjected to the following loading conditions:

Study A. Uniform normal pressure of 100 MPa on outer face highlighted in green.

Study B. Uniform acceleration - 9.81 m/s² in global y-direction.

The bottom face on the xz plane is constrained in the vertical y-direction.



Modulus of elasticity (E) = 210 GPa
Poisson's ratio (ν) = 0.3
Density(ρ): 7000 kg/m³

Von Mises stress at point A

Study A : 211.6 MPa , Study B : 0.031 MPa

Study case 1 : 3D Solid Model

Study case 2 : 2D Axisymmetric Model