



# 자전거 스포크 휠의 최적화



Team All A+

2004008471 조 재현

2005006423 김 기홍





# Contents

---



1. Abstract
2. Introduction of base model
3. Formulation based on 5-step
4. Predicted conclusion and To do.





# Abstract



❖ 자전거 휠 회전 관성 최소화 의 중요성 :

❖ 가속 및 감속에 유리

❖ 에너지 소모의 최소화

❖ 회전관성 감소 효과 >> 구조 경량화 효과





# Introduction of Base model



- ❖ 20 Hole
- ❖ Pre-tension  
spoke wheel
- ❖ Spoke diameter  
 $d=2[\text{mm}]$
- ❖ Spoke length  
 $\ell=287[\text{mm}]$





# Problem Statement

---



- ❖ Necessary Condition
  - Do not exceed allowable stress
  
- ❖ Optimization
  - Minimize the rotational stiffness





# Data and Information (1)



- ❖ Assumption
  - ❖ 2-D System analysis
  - ❖ Section shape of spoke is circle
  - ❖ Amount of pre-tension :  $20000/n$  [N]
  - ❖ Maximum Load applied on wheel : 120[kgf]

- ❖ Materials

|                         | Rim(6061-T6) | Spoke(348STS) |
|-------------------------|--------------|---------------|
| $\rho(kg / m^3)$        | 3000         | 8000          |
| $\sigma_{fatigue}(MPa)$ | 330          | 1000          |

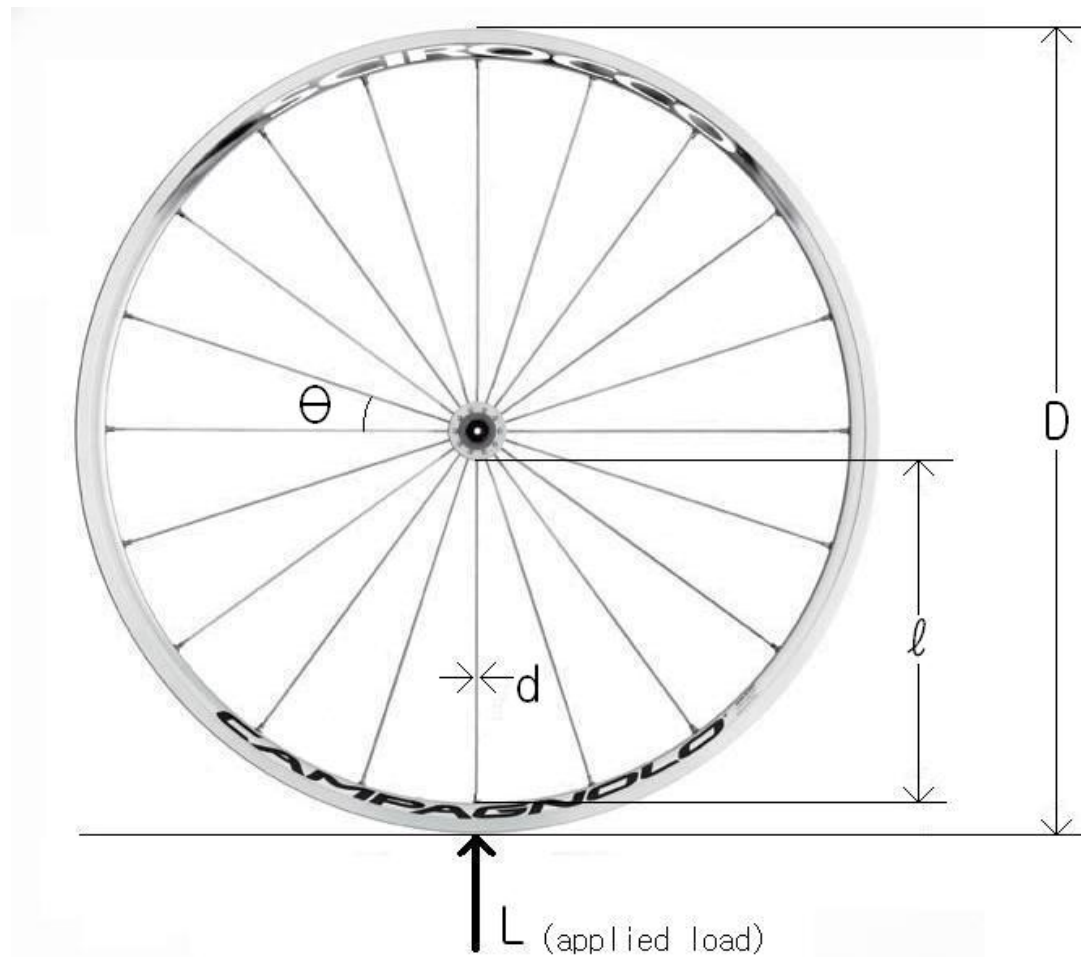




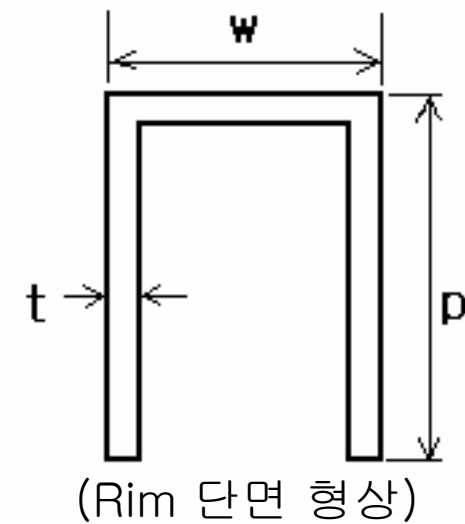
## Data and Information (2)



### ❖ Wheel Dimensions



$D=700[\text{mm}]$   
 $w=20[\text{mm}]$





## Data and Information (3)



❖ Spoke에 걸리는 최대 응력 계산

$$L_{\max}(n) = \frac{2gL}{D} \sin\left(\frac{\pi}{n}\right) + F_{pre}$$

$$A(d) = \frac{\pi}{4} d^2$$

$$F_{pre} = 20000 / n$$

$$\sigma_{spoke \max} = \frac{(2gL) \sin(\pi / n) + 20000 / n}{(\pi / 4) D d^2}$$







## Data and Information (4)



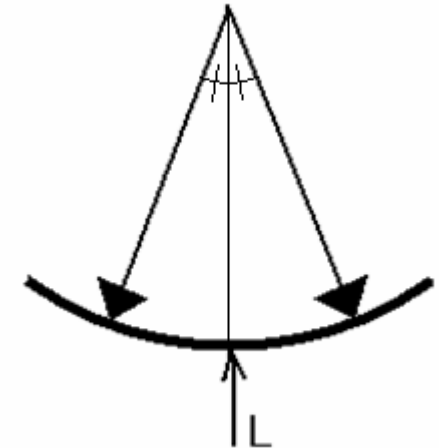
- ❖ Rim에 걸리는 최대 응력 계산  
(Beam으로 가정)

$$\bar{y} = \frac{p^2 + 0.5(w - 2t)t}{2p - 2t - w}$$

$$I = \frac{tp^3}{6} + 2pt\left(\bar{y} - \frac{p}{2}\right)^2 + t^2(w - 2t)\left\{\frac{t}{12} + \left(\bar{y} - \frac{t}{2}\right)^2\right\}$$

$$M_{\max} = \frac{DLg\pi}{2n}$$

$$\sigma_{\text{Rim max}} = \frac{M(p - \bar{y})}{I}$$





# Design Variables

---



- ❖ Spoke diameter :  $d$
- ❖ Number of spoke :  $n$ 
  - 스포크의 개수는 짝수만 허용하는 이산변수
- ❖ Rim depth :  $p$
- ❖ Rim thickness:  $t$





# Constraints



$$\diamond N_{safe} \sigma_{spoke \max} \leq \sigma_{348STS\_fatigue}$$

$$\diamond N_{safe} \sigma_{Rim \max} \leq \sigma_{alu\_fatigue}$$

$$\diamond 10 \leq n \leq 32 \text{ (n은 짝수)}$$

$$\diamond d \geq 1 [\text{mm}]$$

$$\diamond t \geq 0.8 [\text{mm}]$$





# Identification of a Criterion to be Optimized



Minimize the rotational stiffness :

$$m_{spoke}(n, l, d) = \rho_{348STS} \left(\frac{\pi}{4}\right) d^2 l$$

$$J_{spoke}(n, l, d) = m_{spoke} l^2 / 3$$

$$m_{Rim}(t, p) = \rho_{alu} \{2wt + (w - 2t)t\} (l + \bar{y})$$

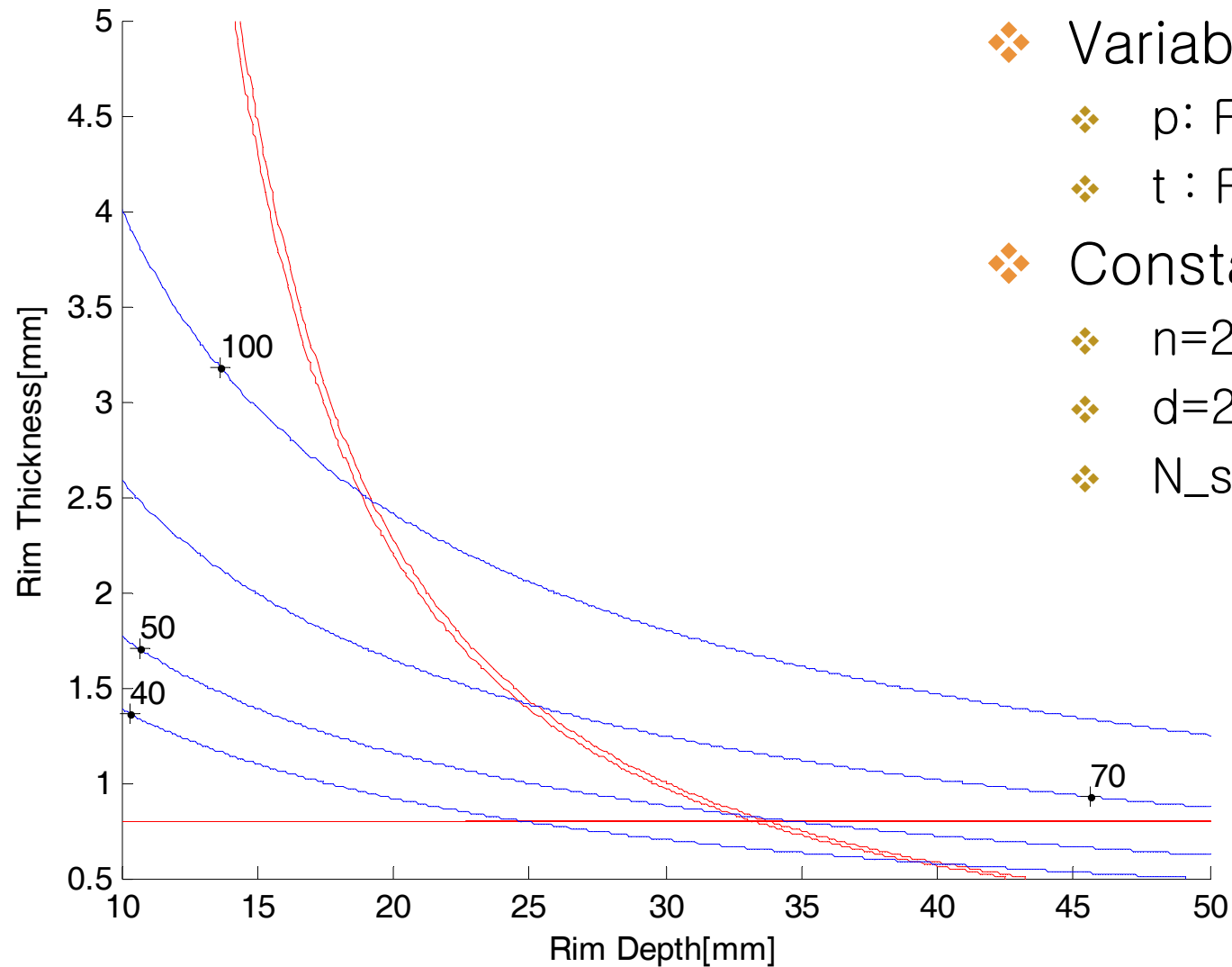
$$J_{Rim}(t, p) = m_{Rim} \left(\frac{D}{2} - p + \bar{y}\right)^2$$

$$J_{total}(n, d, t, p) = n \times J_{spoke} + J_{Rim}$$





# Graphical Solution



## Variables

- ❖  $p$ : Rim depth
- ❖  $t$ : Rim thickness

## Constants

- ❖  $n=20$
- ❖  $d=2\text{mm}$
- ❖  $N_{\text{safe}} = 2$





## 진행방향



- ❖ Spoke의 개수와 지름도 변수로 두어 총 4개의 변수로 최적화

