

# Optimum Design

## Project 1

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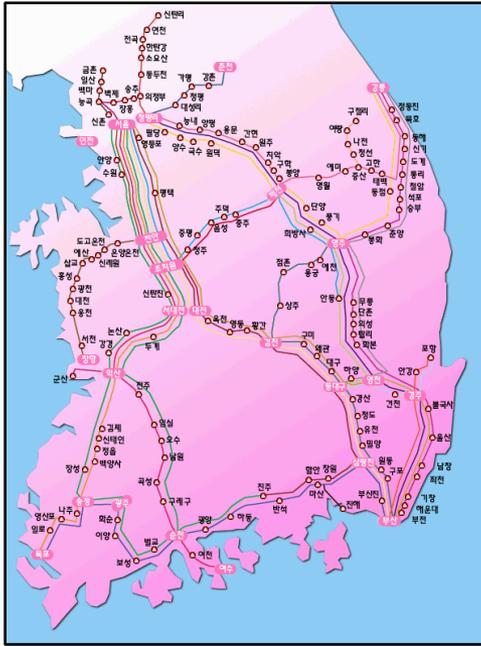
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# Formulation

- Step 1. Project/Problem Statement



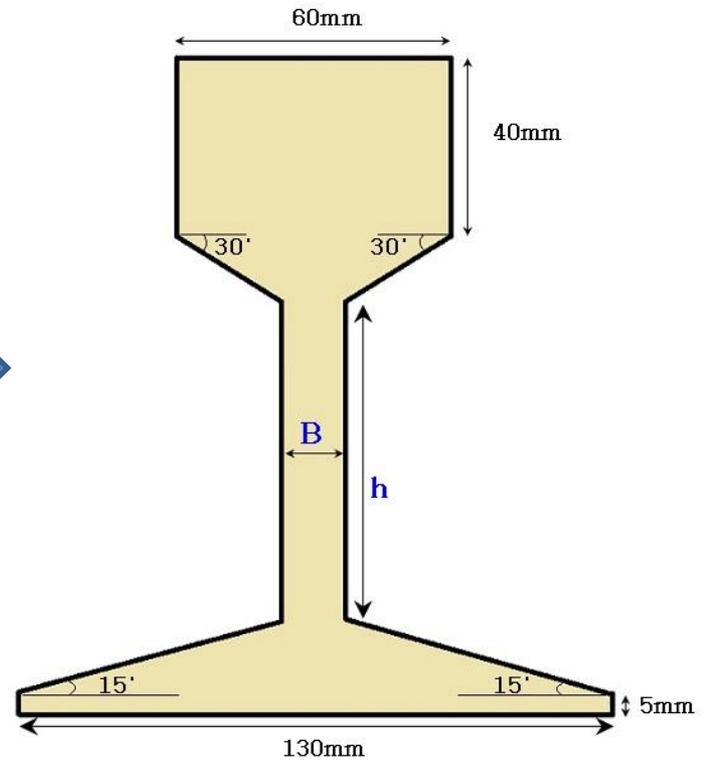
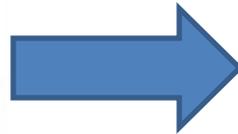
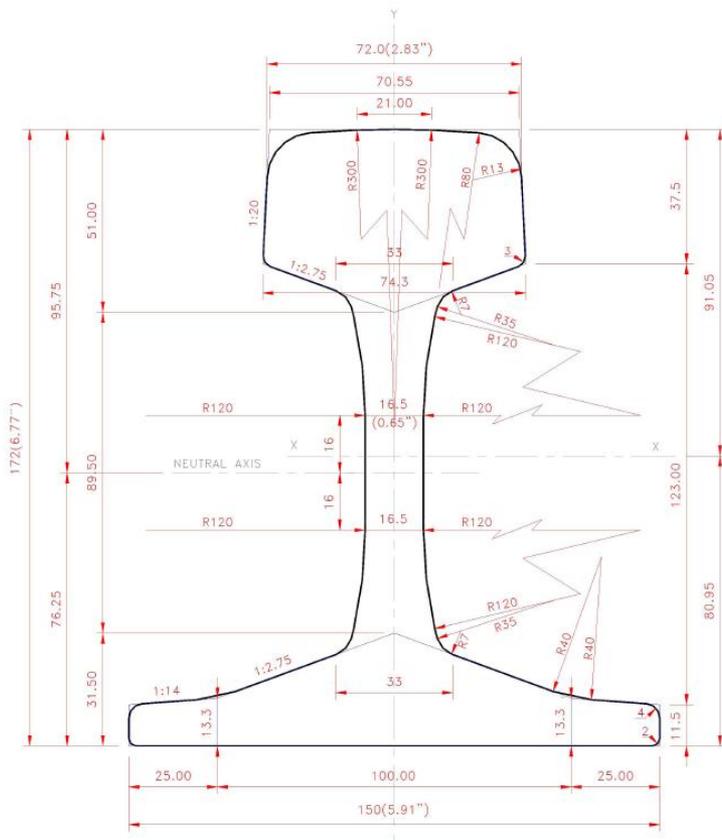
# Formulation

- **Step 2. Data and Information Collection**

<b>KS60 Property</b>		
Yield Strength	Tension	900 MPa
	Shear	350 MPa
Elasticity		200 GPa
Thermal Expansion Coefficient		$1.2 \cdot 10^{-6}$ m/m'c
Area		7750 mm <sup>2</sup>

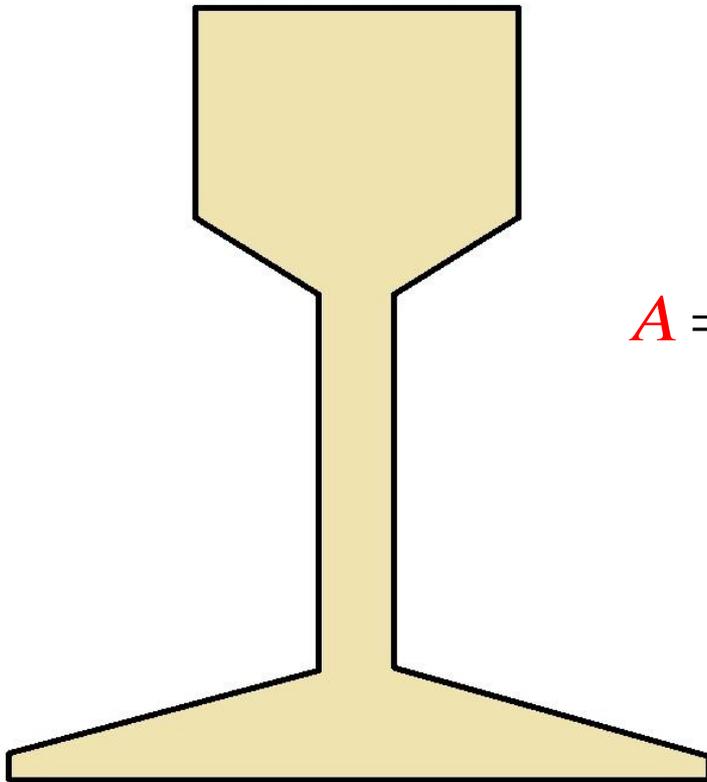
# Formulation

- Step 3. Identification/Definition of Design Variables



# Formulation

- Step 4. Identification of a Criterion to be optimized

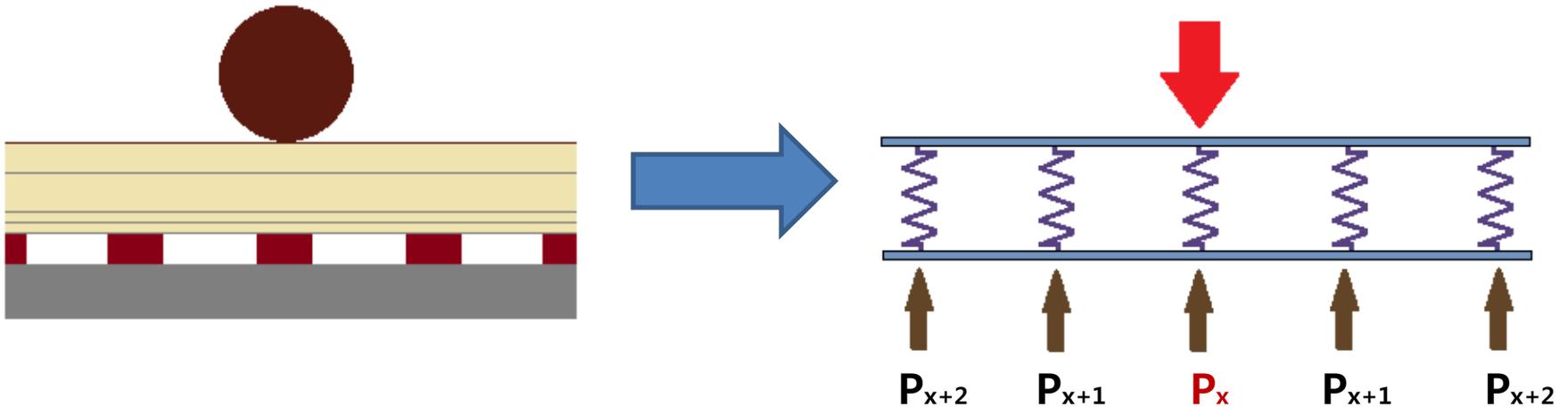


Area minimize

$$A = -0.317b^2 + 0.0174b + bh + 0.0097$$

# Formulation

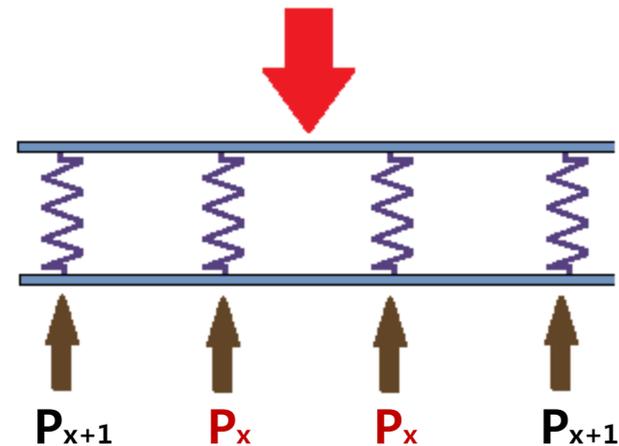
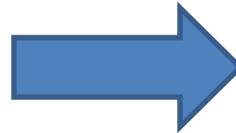
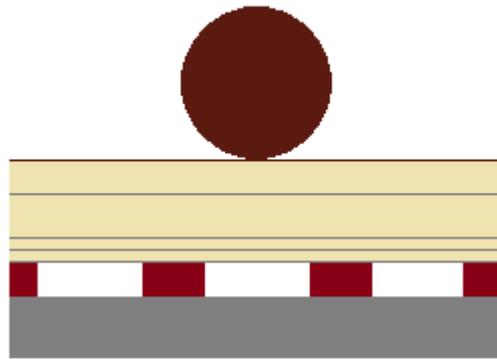
- Step 5. Identification of Constraints



$$P_1 = \int_0^a ky dx = \frac{W}{2} [1 - \phi(\beta a)] \quad \left( \text{where } \phi(x) = e^{-x} \cos x, \quad a = \text{침목간격}, \quad \beta = \sqrt{\frac{k}{4EI_x}} \right)$$

# Formulation

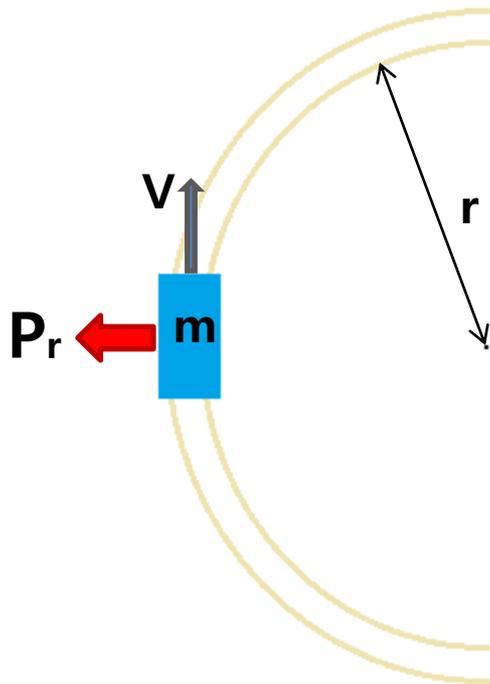
- Step 5. Identification of Constraints



$$P_2 = 2 \int_0^{\frac{a}{2}} ky dx = W \left[ 1 - \phi\left(\frac{a}{2} \beta\right) \right] \quad \left( \text{where } \phi(x) = e^{-x} \cos x, \quad a = \text{침목간격}, \quad \beta = \sqrt{\frac{k}{4EI_x}} \right)$$

# Formulation

- Step 5. Identification of Constraints

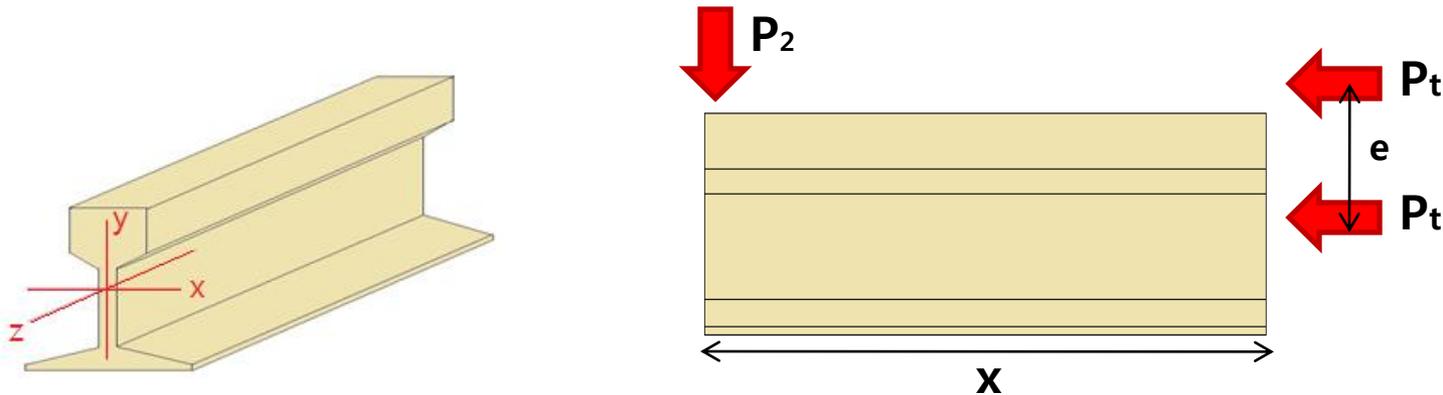


$$P_r = \frac{mV^2}{r}$$

# Formulation

- Step 5. Identification of Constraints

## 1) X-axis buckling (by thermal stress & $P_2$ )

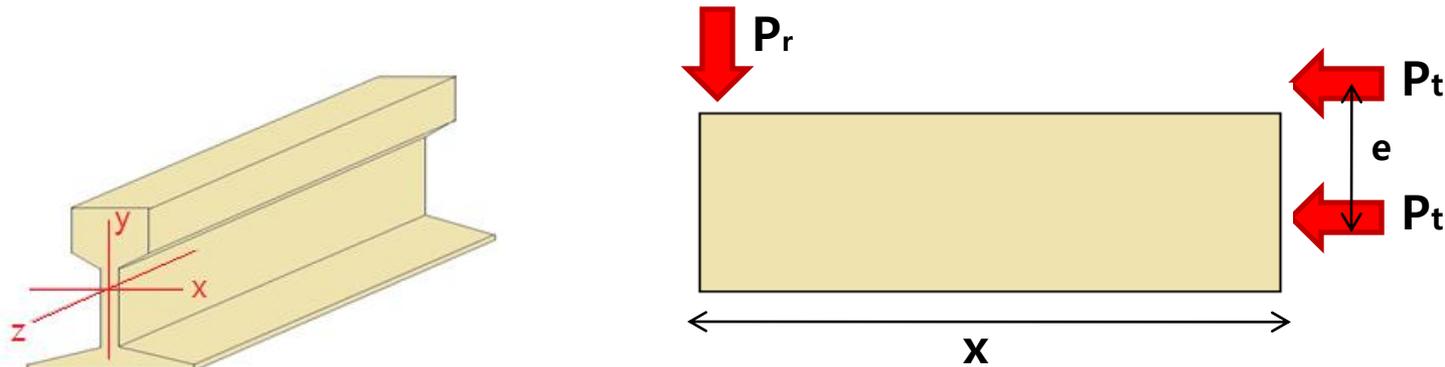


$$P_{cr} = P_t \left[ 1 + \frac{ce}{r^2} \sec \left( \sqrt{\frac{P_t}{EI_x}} \frac{L_e}{2} \right) \right] \quad \left( \text{where } e = \frac{P_2 x}{P_t}, \quad r^2 = \frac{I_x}{A} \right)$$

# Formulation

- Step 5. Identification of Constraints

## 2) Y-axis buckling (by thermal stress & $P_r$ )

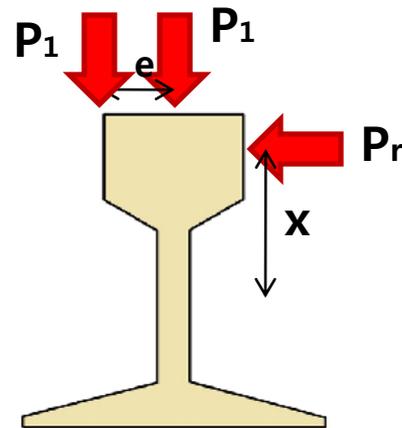
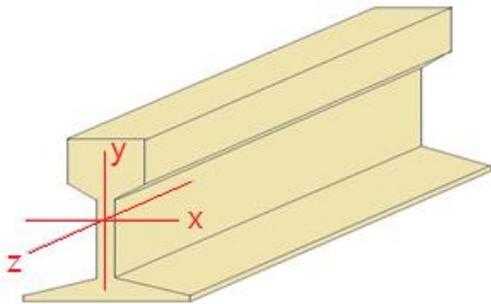


$$P_{cr} = P_t \left[ 1 + \frac{ce}{r^2} \sec \left( \sqrt{\frac{P_t}{EI_y}} \frac{L_e}{2} \right) \right] \quad \left( \text{where } e = \frac{P_r x}{P_t}, \quad r^2 = \frac{I_y}{A} \right)$$

# Formulation

- Step 5. Identification of Constraints

## 3) Z-axis buckling (by $P_1$ & $P_r$ )

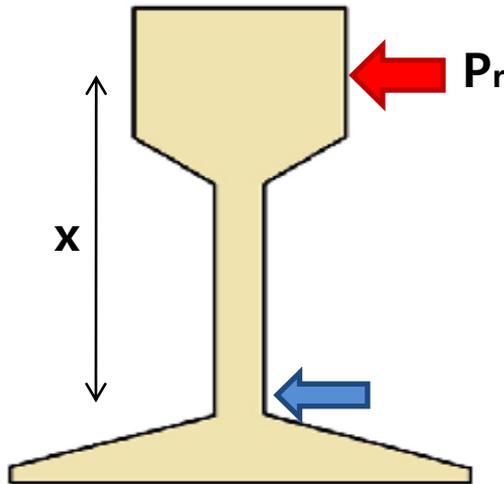


$$P_{cr} = P_t \left[ 1 + \frac{ce}{r^2} \sec \left( \sqrt{\frac{P_t}{EI_z}} \frac{L_e}{2} \right) \right] \quad \left( \text{where } e = \frac{P_r x}{P_t}, \quad r^2 = \frac{I_z}{A} \right)$$

# Formulation

- Step 5. Identification of Constraints

## 4) Maximum bending stress (by $P_r$ )



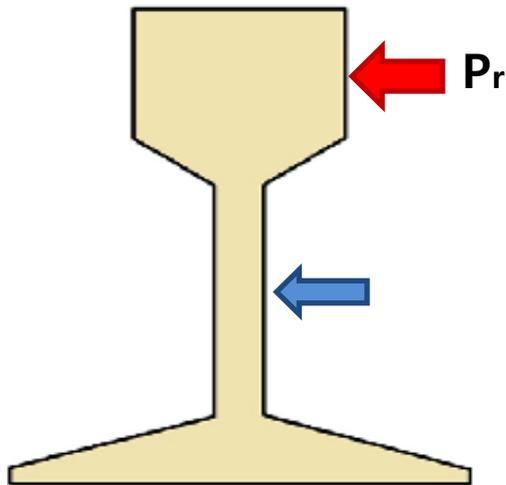
$$\sigma_{\max} = k \frac{Mc}{I_z}$$

$$\left( \text{where } c = \frac{b}{2}, \text{ } k = \text{intensity factor} \right)$$

# Formulation

- Step 5. Identification of Constraints

## 5) Maximum Shear stress (by $P_r$ )

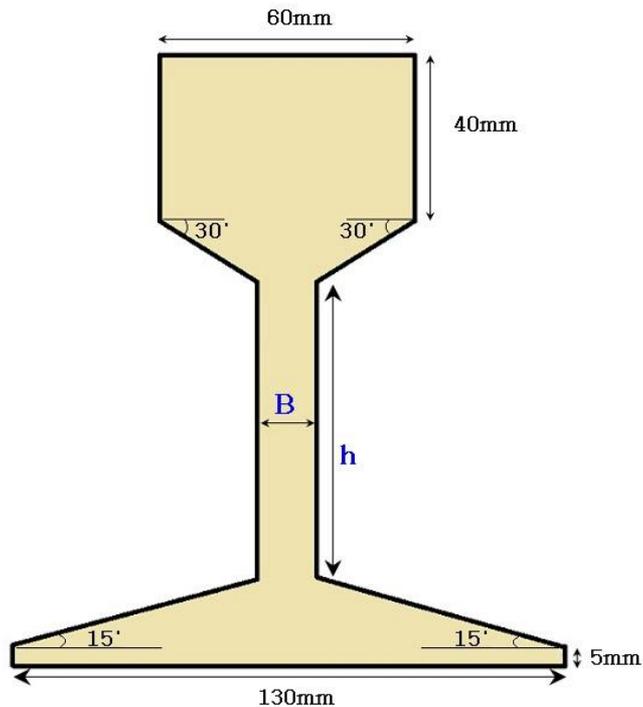


$$\tau_{\max} = \frac{V}{A}$$

# Formulation

- Step 5. Identification of Constraints

## 6) Geometric Shape



$$b \leq \frac{h}{5}$$

# Graphical optimization

- **Constraints**

1) X-axis buckling

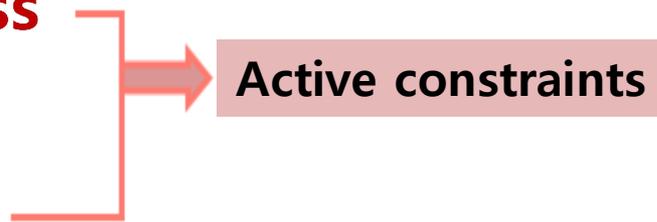
2) Y-axis buckling

3) Z-axis buckling

4) Max bending stress

5) Max shear stress

6) Geometric shape



# Graphical optimization

- Active constraints

**G1 : Max bending stress**

$$b \geq 0.0000926\sqrt{2258h + 180} - 0.00041$$

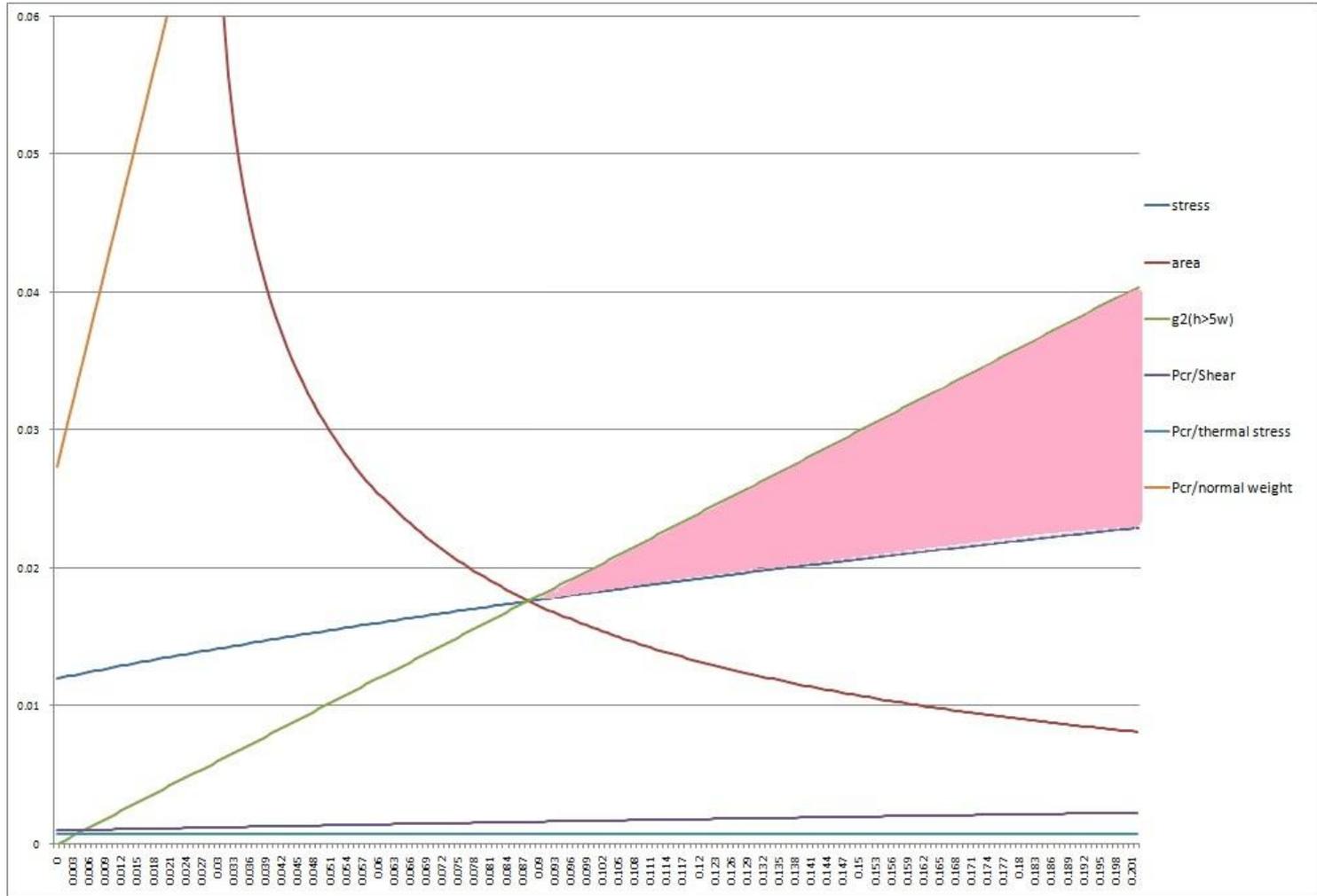
**G2 : Geometric shape**

$$b \leq \frac{h}{5}$$

**F : Area minimize**

$$b = 158h - 158\sqrt{h^2 + 0.035h - 127A + 0.0052} + 0.027$$

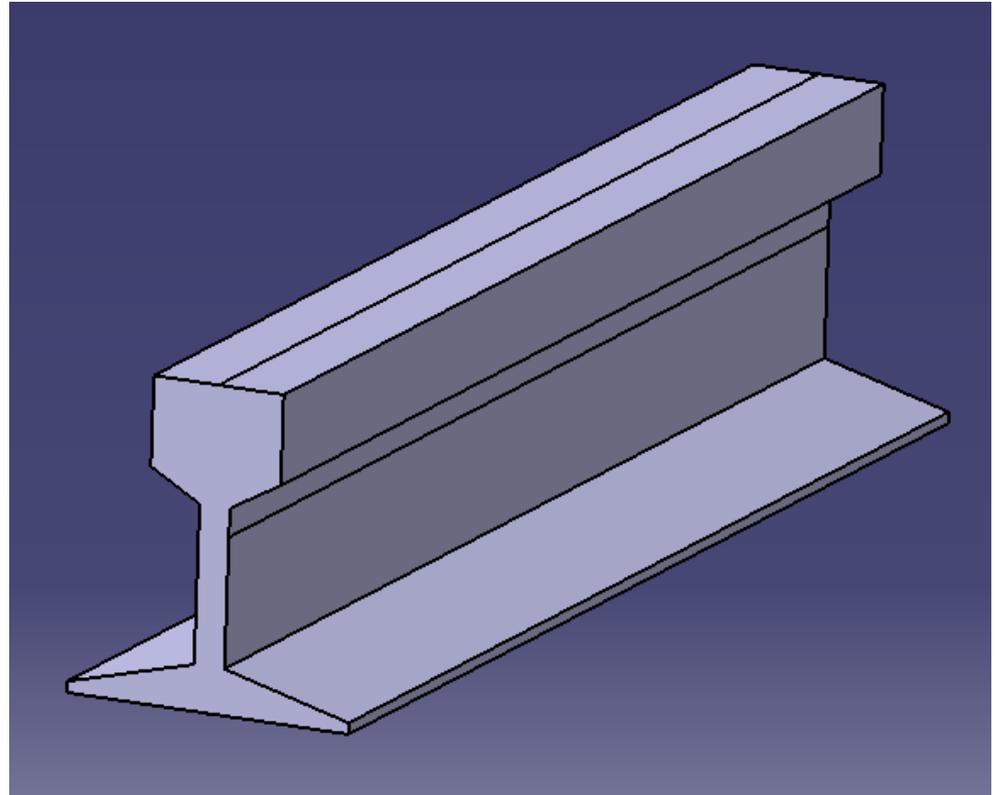
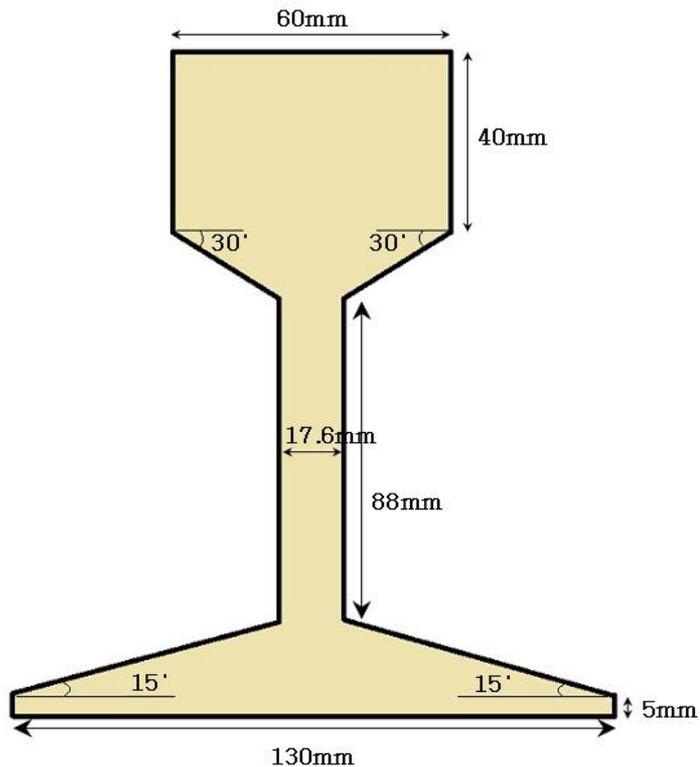
# Graphical optimization



# Optimized solution

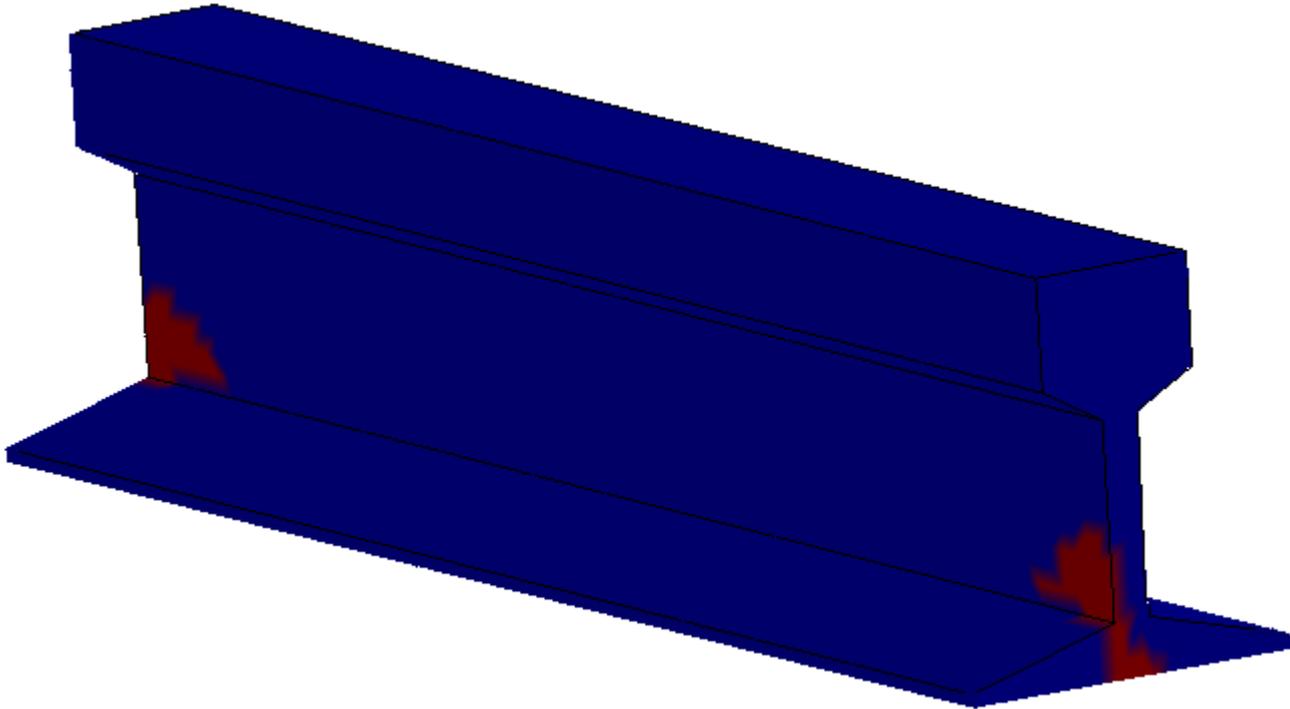
$b=17.6 \text{ mm}$   $h=88 \text{ mm}$

$A= 5635.7 \text{ mm}^2$  ( $<7750 \text{ mm}^2$ )



# Simulation

- Simulation by solidworks



# Comments

1. KS60의 면적보다 좋은 결과 값 도출
2. 가정한 사실이 많아 실제로 적용 시키기는 무리
3. Simulation 결과 아랫부분에서 집중응력 발생

# Reference

1. 선로공학 / 서사범 저 / 북갤러리
2. 재료역학 / 임장근 외 3명 저 / 병진
3. 재료역학 / Beer 저 / McGrawhill
4. 한국 철도 기술 연구원 <http://www.krri.re.kr/>

# Q n A

