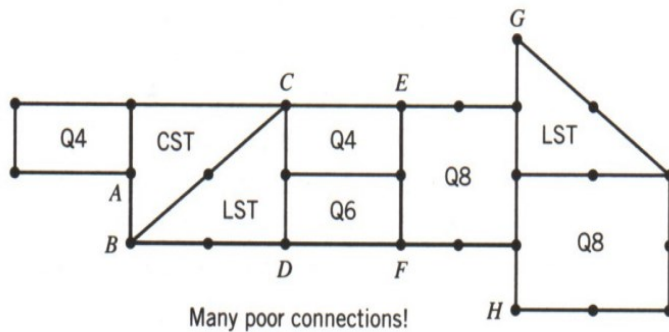


1. (30 pts) Answer the following:

- (1) What is the difference between rod(bar, truss) and beam elements?
- (2) If an element has a quadratic shape function in  $x$ , will the strain also be a quadratic function of  $x$ ? (If not, why?)
- (3) What is the advantage of using isoparametric elements?
- (4) List two causes of singular stiffness matrices.
- (5) Sketch the shape function for the edge node of a quadratic quadrilateral.
- (6) Explain rank-sufficient Gauss rules for 3-node triangular and 4-node quadrilateral elements.

2. (15 pts) Identify four problems with the mesh shown.



Name	Element	Number of nodes
CST	Constant Strain Triangle	3
LST	Linear Strain Triangle	6
Q4	Quadrilateral	4
Q6	bilinear quadratic Quadrilateral	4
Q8	quadratic Quadrilateral	8

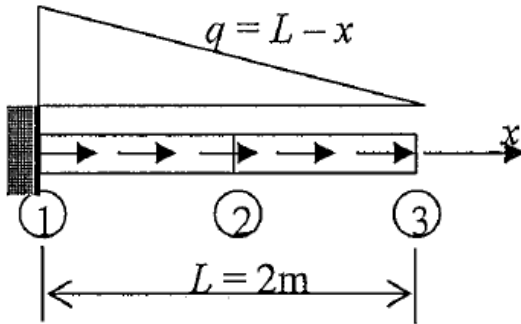
3. (15 pts) Suppose that the assembled stiffness equations for a one-dimensional finite element model before

imposing constraints are 
$$\begin{bmatrix} 2 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 2 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix} = \begin{bmatrix} 1 \\ 0 \\ 2 \end{bmatrix}.$$
 This system is to be solved subject to the multipoint

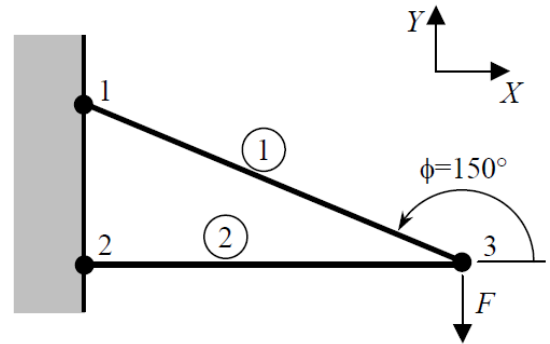
constraint  $u_1 = u_3$ .

- (1) Impose the constraint by the master-slave method taking  $u_1$  as master, and solve the resulting 2x2 system of equations.
- (2) Impose the constraint by the penalty function method, leaving the weight  $w$  as a free parameter. Write down the equation only and do not solve.
- (3) Impose the constraint by the Lagrange multiplier method. Show the 4x4 multiplier-augmented system of equations.

4. (10 pts) A bar of  $L = 2m$  carries linearly varying axial loads, as shown in the figure. Consider two equal-length bar elements, using axial displacements as nodal DOFs. Calculate work-equivalent nodal loads  $\{F\} = \{F_1, F_2, F_3\}$ .

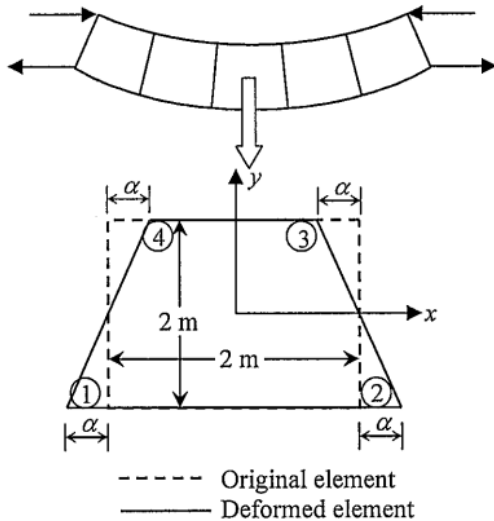


(Problem 4)



(Problem 5)

6. (20 pts) A beam under pure bending is modeled using Q6 elements. For a Q6 element shown, nodal displacements are given as  $v_1 = v_2 = v_3 = v_4 = 0$ ,  $u_1 = u_3 = -\alpha$ ,  $u_2 = u_4 = \alpha$ . Determine internal DOFs  $a_1, a_2, a_3$  and  $a_4$  such that the element satisfies the pure bending conditions: i.e.,  $\varepsilon_{xx}$  is a function of  $y$  only,  $\varepsilon_{yy} = \gamma_{xy} = 0$ .



$$\begin{cases} N_1 = \frac{1}{4}(1-x)(1-y), & N_2 = \frac{1}{4}(1+x)(1-y) \\ N_3 = \frac{1}{4}(1+x)(1+y), & N_4 = \frac{1}{4}(1-x)(1+y) \\ N_5 = 1-x^2, & N_6 = 1-y^2 \end{cases}$$

$$u = \sum_{i=1}^4 N_i u_i + N_5 a_1 + N_6 a_2$$

$$v = \sum_{i=1}^4 N_i v_i + N_5 a_3 + N_6 a_4$$