



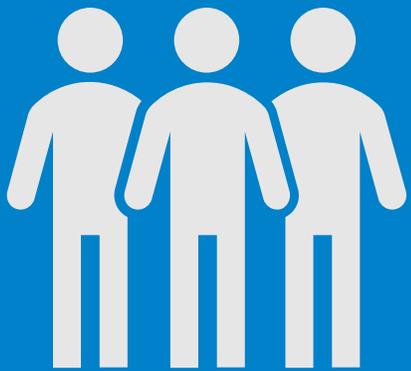
안녕, 평창 (HY- PyeongChang)

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2014012488 김혜연



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- 4 실제 무대에 적용



01 주제 및 팀명 선정 배경

주제 선정 배경



팀명 선정 배경



안녕, 평창 (HY-PyeongChang)

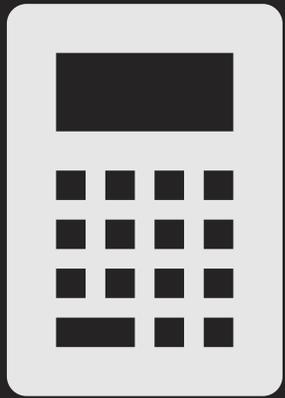
Project 목표

안전한 무대 및 lifter 만들기

- 2개의 critical point(A점, E점)

Point A. mesh의 정확성을 높여가며 analytic solution과의 수렴 여부 판단

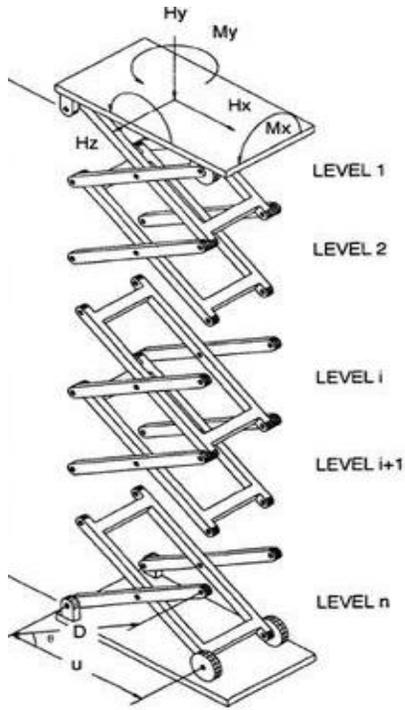
Point E. 가수들의 x축 위치 변화에 따른 응력 계산



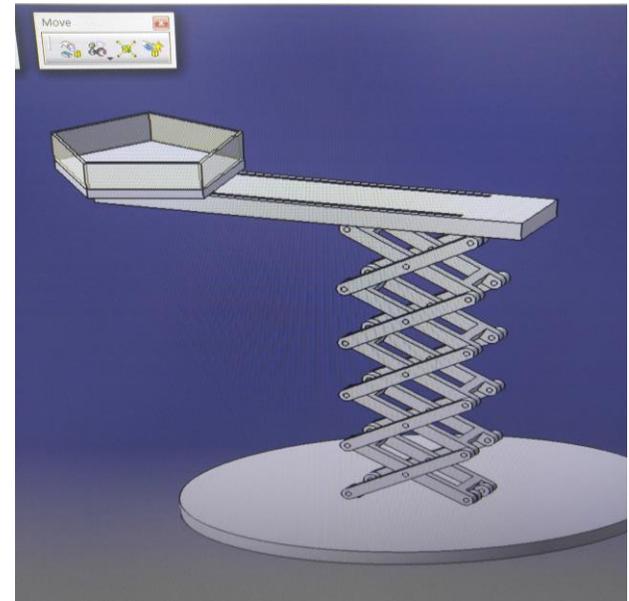
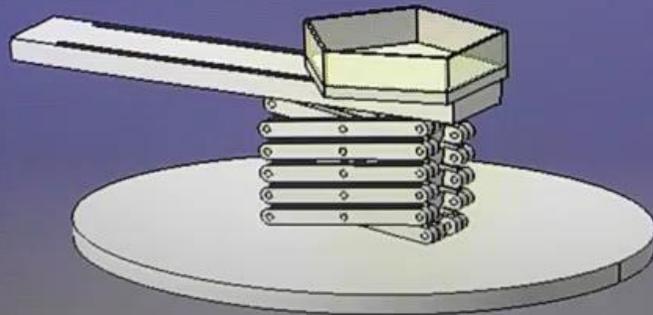
02 Design & MATLAB Calculate

Design & MATLA

1. 리프트 초기 모델링



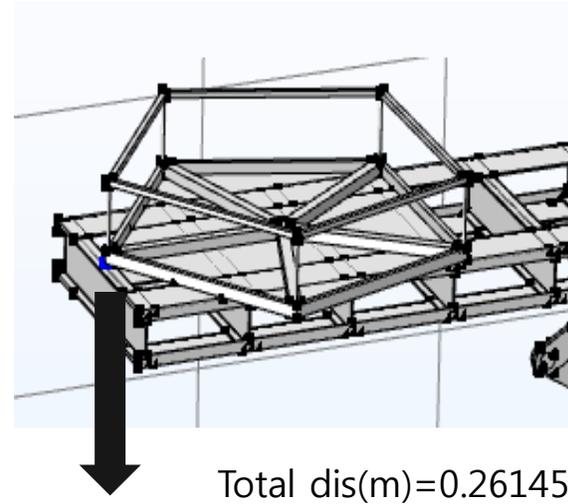
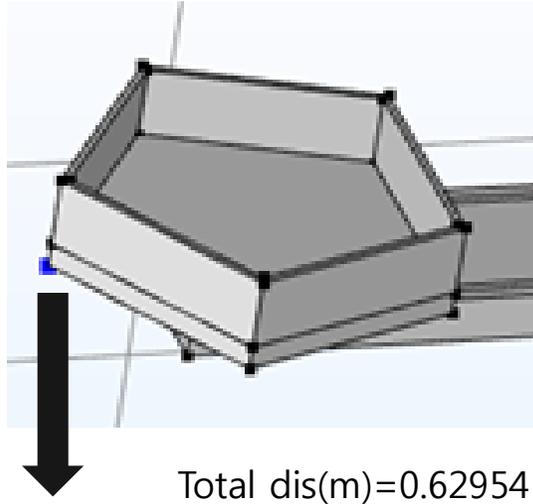
CONCEPT



PRODUCT

Design & MATLAB Calculate

2. 초기 모델의 문제점

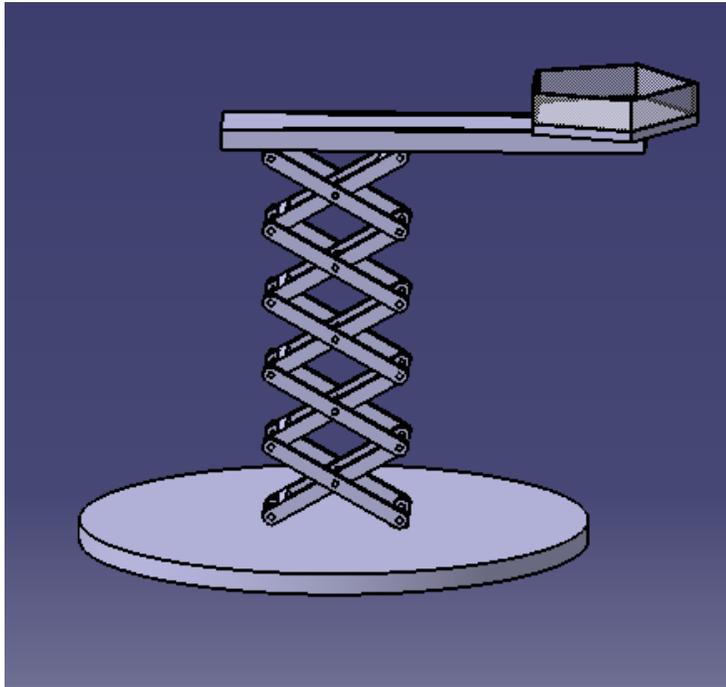


무대 끝의 displacement(0.6m) 값이 커 무대 안정도에 영향을 줄 수 있다.

Displacement의 값(0.26m)이 감소했음을 확인할 수 있다.

Design & MATLAB Calculate

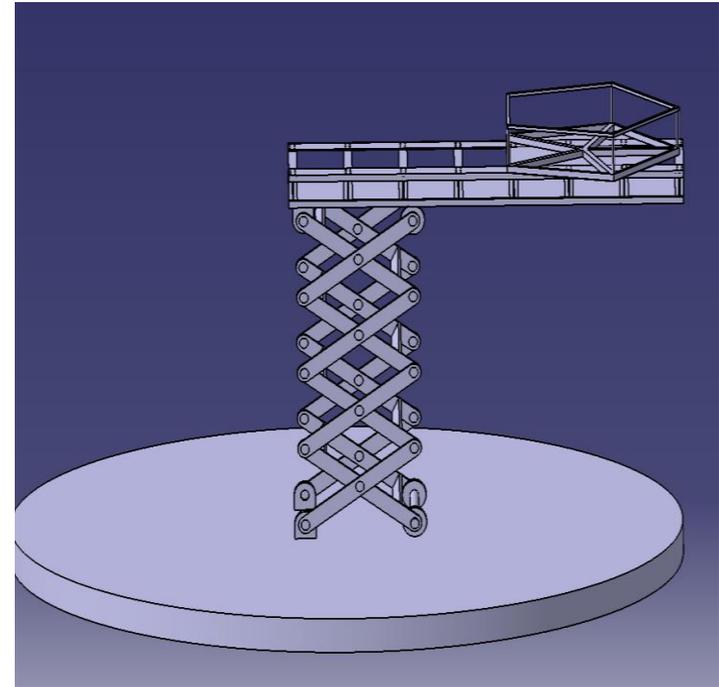
3. CATIA 모델의 변경



무대와 갑판의 경량화

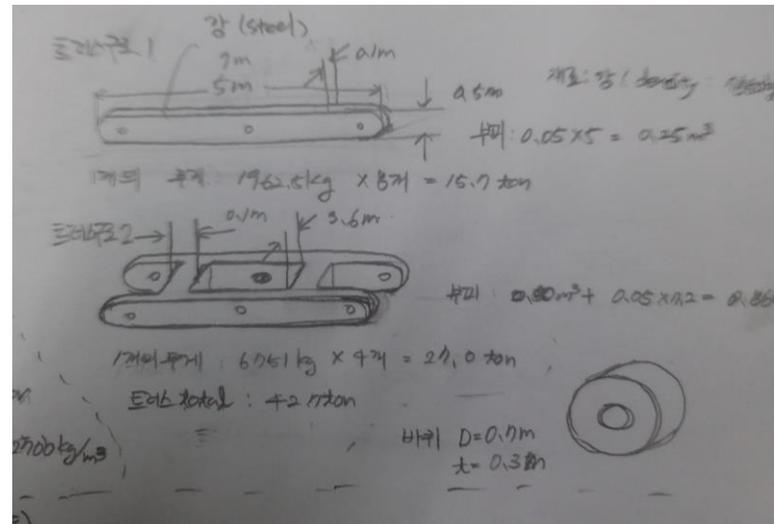
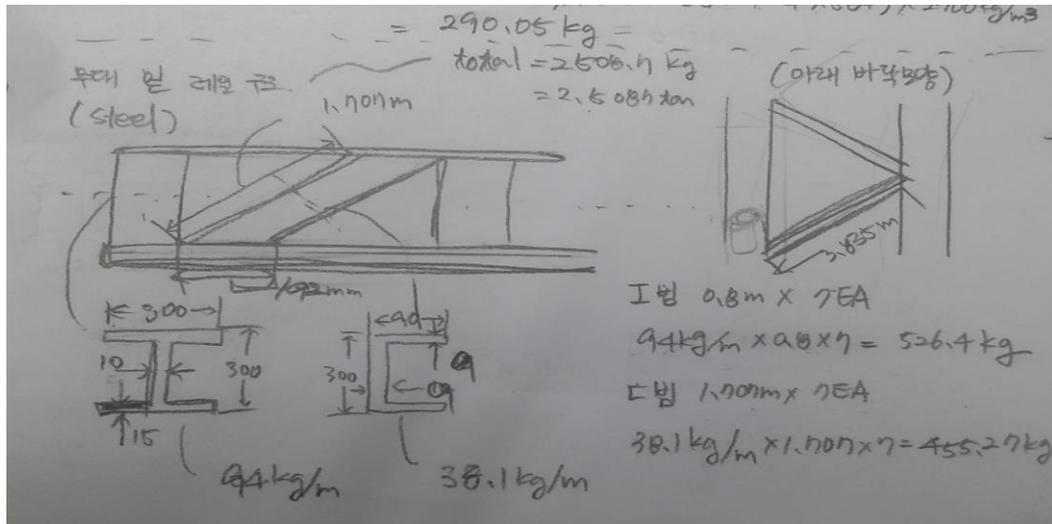


리프트 구조의 경량화

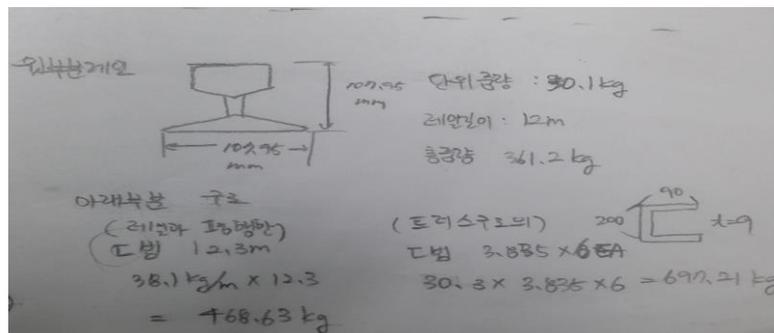
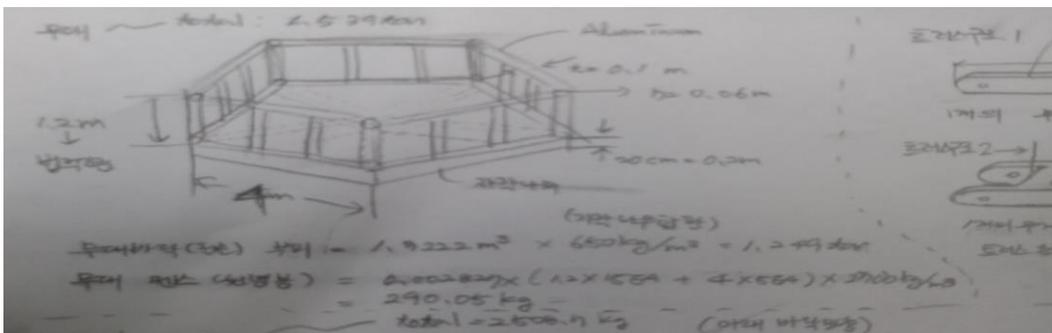


Design & MATLAB Caculate

4. 각 부분의 수치와 재질, 밀도의 선정

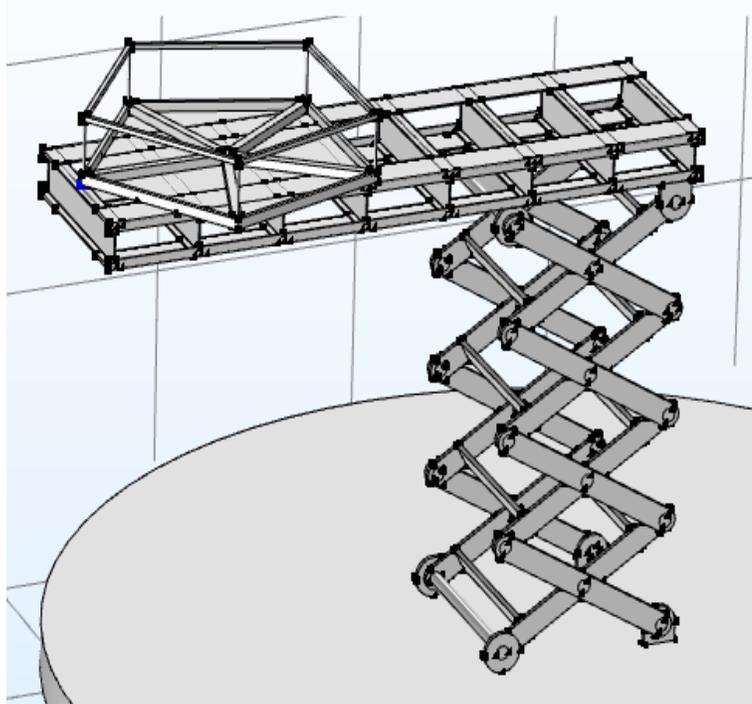


AISI 4340
밀도 : 7850 kg/m^3



Design & MATLAB Caculate

5. MATLAB을 통한 parameter 지정



```

2 - g= 9.81;
3 - sy1 = 580*10^6 ; % AISI1050CD %
4 - sy2 = 807*10^6 ; % AISI1050QNT %
5 - sy3 = 1590*10^6 ;% AISI4340QNT %
6 - dp = 0.31; % 핀의 지름%
7 - ds = 0.3; %샤프트의 지름%
8 - hb = 0.5; %바의 폭%
9 - ht = 0.1; %바의 두께%
10 - hl = 5;%바의 길이%
11 - l1 = 9.7;
12 - l2 = 4.3;%리프트의 폭%
13 - l = l1+l2;
14 - ls = 6.155;%무대 폭%
15 - fp = 600*g;%사람 및 무대장치의 무게%
16 - fs = 1539*g;%무대의 무게%
17 - f1 = 1738.17*g;%l1 만큼의 철골 구조 무게%
18 - f2 = 770.53*g;%l2 만큼의 철골 구조 무게%
19 - fl = f1+f2;%철골 전체의 무게%
20 - wt = hb*ht*hl*7850*g; %바 하나의 무게%
21 - x = 1%무대 중심에서 부터의 거리%
22 - lt = 4% 무대의 한 변%

```

Design & MATLAB Calculate

6. 각 부분에 작용하는 하중, 모멘트, 비틀림 계산

Handwritten engineering notes and diagrams for a beam analysis. The diagrams show a 3D view of a beam with forces F_p , F_s , F_z and a distributed load $16wx$. A simplified 2D beam diagram shows reaction forces F_a and F_b .

Equations for equilibrium:

$$F_a' + F_b' = F_s + F_z + 16wx \quad \text{--- ①}$$

$$M_a: F_s(l_1 - a \cos \alpha) + F_z(l_1 - \frac{l_1 + l_2}{2}) + F_b' \cdot l_2 - 16wx \cdot \frac{l_1}{2} = 0 \quad \text{--- ②}$$

$$M_b: F_s(l_1 + l_2) + F_z(\frac{l_1 + l_2}{2}) + 16wx \cdot \frac{l_2}{2} - F_a' \cdot l_2 = 0 \quad \text{--- ③}$$

Final force calculations:

$$F_a' = \frac{F_s(l_2 + a \cos \alpha) + \frac{F_z}{2}(l_1 + l_2) + 8wx \cdot l_2}{l_2}$$

$$F_b' = \frac{F_s}{l_2}(l_1 - a \cos \alpha) + \frac{F_z}{2l_2}(l_1 - l_2) - 8wx$$

Additional notes include a small diagram of a beam with forces F_1 and F_2 and a table of force components:

F_1	F_2
$F_1 = \frac{l_2 + 2a}{2l_2} F_p$	$F_2 = \frac{l_2 + 2a}{2l_2} F_p$
$F_1 = \frac{l_2 - 2a}{2l_2} F_p$	$F_2 = \frac{l_2 - 2a}{2l_2} F_p$

Handwritten engineering notes and diagrams for a truss structure. The diagrams show a truss with forces F_p , F_s , F_z and reaction forces N_z and b_z .

Equations for equilibrium:

$$2(N_z + b_z) = F_1 + F_2 + F_p + F_s \quad \text{--- ①}$$

$$0.45 \cdot l_1 \cdot F_s + \frac{1}{2} \cdot F_1 + (1 + \frac{l_2}{2}) F_2 = 2(l_1 N_z + l_1 + l_2 b_z) \quad \text{--- ②}$$

Final force calculations:

$$N_z = 48225N$$

$$b_z = -25425N$$

Additional notes include a diagram of a beam with forces F_p , F_s , F_z and a table of force components:

F_a	F_b	F_c
$F_a = \frac{F_p}{2} + F_s$	$F_b = \frac{F_p}{2} + F_s$	$F_c = \frac{F_p}{2} + F_s$

Design & MATLAB Calculate

```
%-----사람무게 고려 x-----%
fa1= (fs/l2*(l1+l2-0.85*lt)+f1/2/l2*(l1+l2)+8*wt)/2;
fb1= -(fs/l2*(l1-0.85*lt)+f1/2/l2*(l1-l2)-8*wt)/2;
fc1= fb1;
fd1= fa1;
```

```
%-----사람위치에 따른 사람 및 무대 무게만 고려했을때의 걸리는 힘-----%
fa2 = fp/(1-l1/(l1+l2)-l1*(lt+2*x)/(l1+l2)/(lt-2*x)+(lt+2*x)/(lt-2*x));
fb2 = -fa2*l1/(l1+l2);
fd2 = fa2*(lt+2*x)/(lt-2*x);
fc2 = -fd2*l1/(l1+l2);
```

```
%-----%
%----모멘트----%
me = fs*(l1-0.85*lt)+f1*(l1-(l1+l2)/2)+fp*l1;
ma=lt/2*(fs+f1)+lt*8*wt+fp*(lt/2+x)-(fc+fc)*lt;
mb= ma;
```

```
mbar =(fa-wt)*cos(pi/6)*hl+wt*cos(pi/6)*hl/2;
```

```
%----토크----%
```

```
te=lt/2*(fs+f1)+fp*(lt/2+x)+lt*(fg-fh);
```

```
%-----전체-----%
```

```
fa = fa1+fa2;
fb = fb1+fb2;
fc = fc1+fc2;
fd = fd1+fd2;
```

```
fe1= (fs/l2*(l1+l2-0.85*lt)+f1/2/l2*(l1+l2))/2;
ff1 =-(fs/l2*(l1-0.85*lt)+f1/2/l2*(l1-l2))/2;
fg1 = ff1;
fh1 =fe1;
```

```
fe2 =fa2;
ff2 =fb2;
fg2 =fc2;
fh2 =fd2;
```

```
fe = fe1+fe2;
ff = ff1+ff2;
fg = fg1+fg2;
fh = fh1+fh2;
```

Load
Moment
Torque

Design & MATLAB Calculate

```
%%----우리가 관심 있는 부분 a,b,e의 응력----%%
```

```
%---a---%  
stress_ma = ma*32/pi/dp^3;  
stress_va = 4*fa/(3*pi*dp^2/4);  
von_a = sqrt(stress_ma^2+3*stress_va^2);
```

Stress

```
%---b---%  
  
stress_mb = mb*32/pi/dp^3;  
stress_vb = 4*fb/(3*pi*dp^2/4);  
von_b = sqrt(stress_mb^2+3*stress_vb^2);
```

```
%---e---%  
stress_fe =fe/0.000486;  
stress_me = me*10^6/309*150/10^3;  
stress_te = te*0.0252/((309+644)/10^6);  
  
von_e =sqrt((stress_fe+stress_me)^2+3*stress_te^2);
```

```
%----bar-----%  
stress_fbar = fbar/(hb*ht);  
stress_mbar = mbar*hb/2/(ht*hb^3/12);  
von_bar = sqrt((stress_fbar+stress_mbar)^2);
```

```
% safety factor%  
safety_a1=sy1/von_a;  
safety_a2=sy2/von_a;  
safety_a3=sy3/von_a;  
safety_b1=sy1/von_b;  
safety_b2=sy2/von_b;  
safety_b3=sy3/von_b;  
safety_e1=sy1/von_e;  
safety_e2=sy2/von_e;  
safety_e3=sy3/von_e;  
safety_bar1=sy1/von_bar;  
safety_bar2=sy2/von_bar;  
safety_bar3=sy3/von_bar;
```

Safety factor

Design & MATLAB Calculate

8. Parameter 와 Result 값

dp	0.3100
ds	0.3000
f1	1.7051e+04
f2	7.5589e+03
fa	1.2044e+05
fa1	1.1565e+05
fa2	4.7909e+03
fb	5.4903e+04
fb1	5.8222e+04
fb2	-3.3194e+03
fbar	6.0220e+04
fc	4.8264e+04
fc1	5.8222e+04
fc2	-9.9583e+03
fd	1.3002e+05
fd1	1.1565e+05
fd2	1.4373e+04
fe	4.3431e+04
fe1	3.8640e+04
fe2	4.7909e+03
ff	-2.2106e+04
ff1	-1.8786e+04
ff2	-3.3194e+03

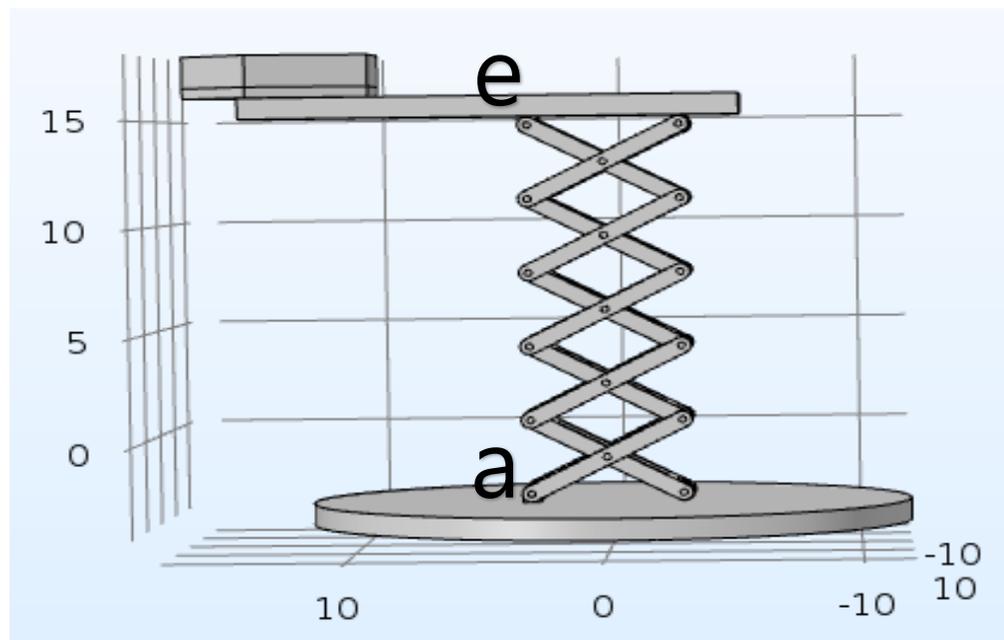
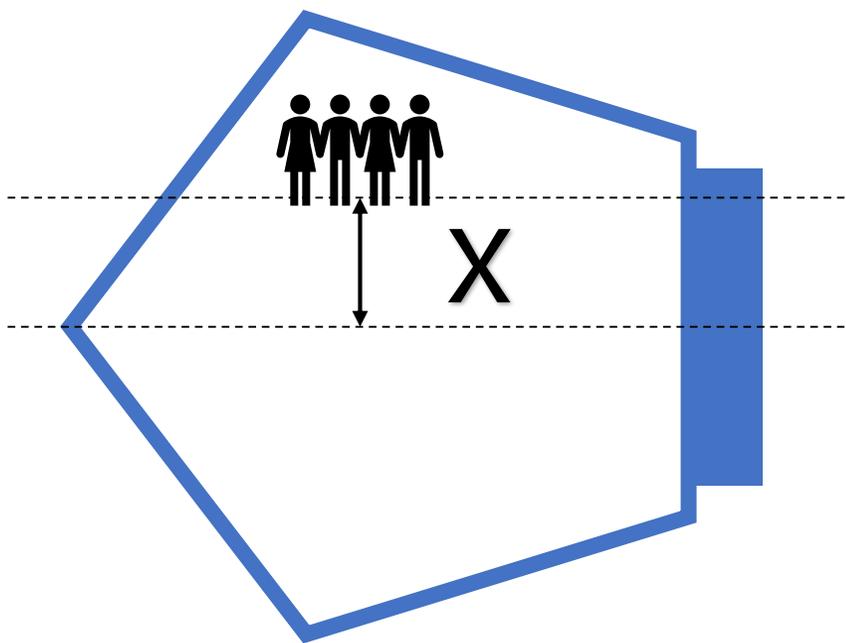
fg	-2.8745e+04
fg1	-1.8786e+04
fg2	-9.9583e+03
fh	5.3013e+04
fh1	3.8640e+04
fh2	1.4373e+04
fl	2.4610e+04
fp	5886
fs	1.5098e+04
g	9.8100
hb	0.5000
hl	5
ht	0.1000
l	14
l1	9.7000
l2	4.3000
ls	6.1550
lt	4
ma	3.2703e+05
mb	3.2703e+05
mbar	4.7984e+05
me	2.1866e+05

stress_ma	1.1182e+08
stress_mb	1.1182e+08
stress_mbar	1.1516e+08
stress_me	1.0614e+08
stress_te	-6.0807e+06
stress_va	2.1276e+06
stress_vb	9.6988e+05
sy1	580000000
sy2	807000000
sy3	1.5900e+09
te	-2.2996e+05
von_a	1.1188e+08
von_b	1.1183e+08
von_bar	1.1637e+08
von_e	1.9579e+08
wt	1.9252e+04
x	1

safety_a1	5.1843
safety_a2	7.2133
safety_a3	14.2121
safety_b1	5.1865
safety_b2	7.2164
safety_b3	14.2182
safety_bar1	4.9843
safety_bar2	6.9351
safety_bar3	13.6639
safety_e1	2.9623
safety_e2	4.1217
safety_e3	8.1208
stress_fbar	1.2044e+06
stress_fe	8.9365e+07

Design & MATLAB Calculate

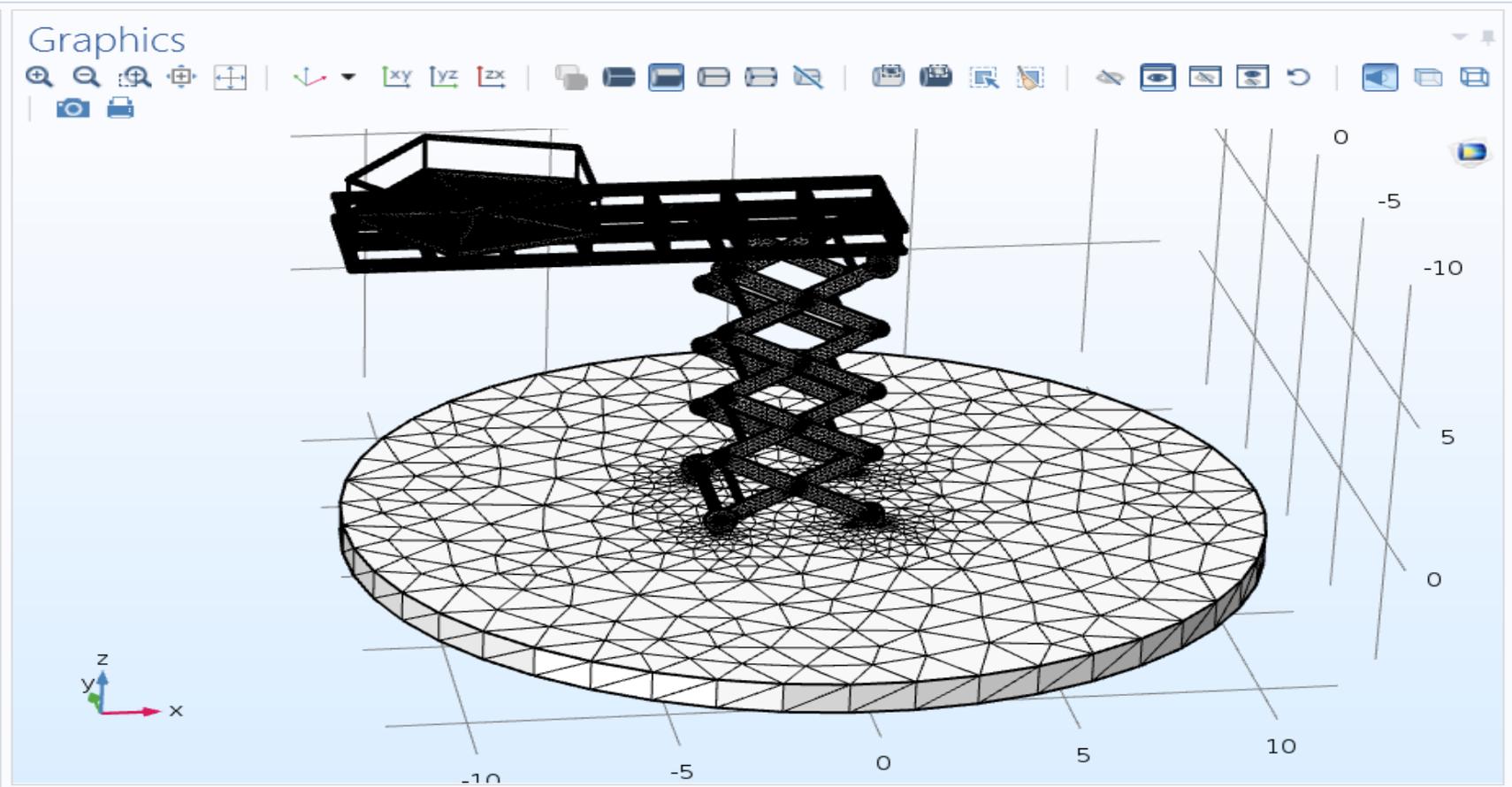
9.(사람의 위치) 변화에 따른 응력 분석





03 COMSOL Interpretation

초기 mesh 모델



Number of boundary elements :111705

Number of elements:180483

Free meshing time: 30.69s

Messages Progress **Log** Evaluation 3D

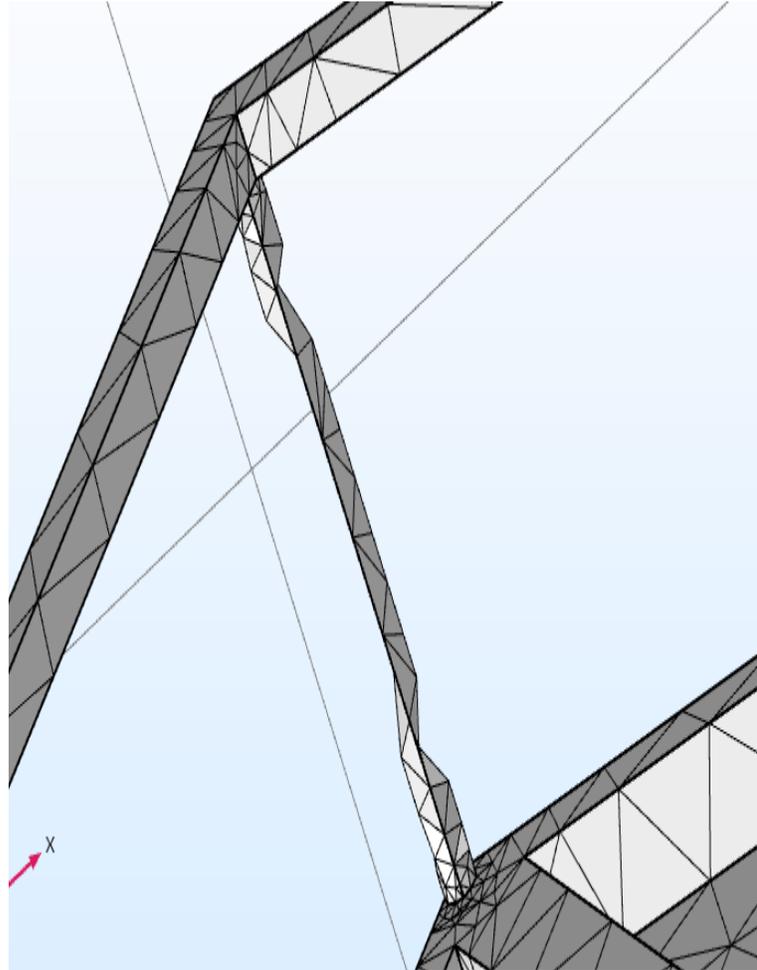
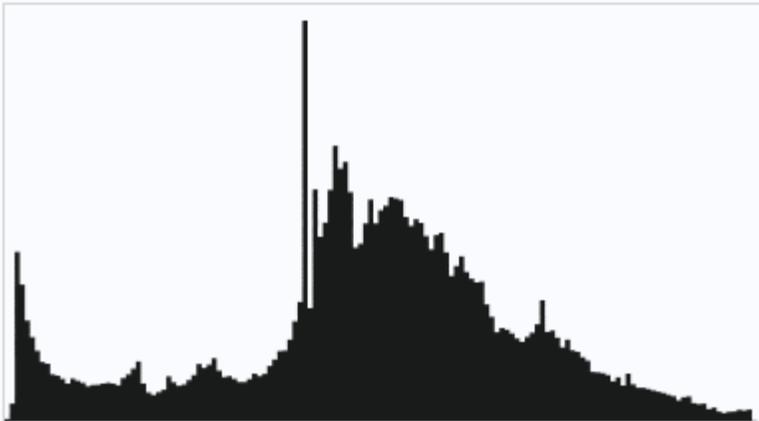
```
Number of boundary elements: 111705
Number of elements: 180483
Free meshing time: 30.69s
Minimum element quality: 0.0002843
```

문제점

— Domain element statistics —

Number of elements:	138521
Minimum element quality:	2.722E-4
Average element quality:	0.4808
Element volume ratio:	1.58E-6
Mesh volume:	461.6 m ³
Maximum growth rate:	5.893
Average growth rate:	2.082

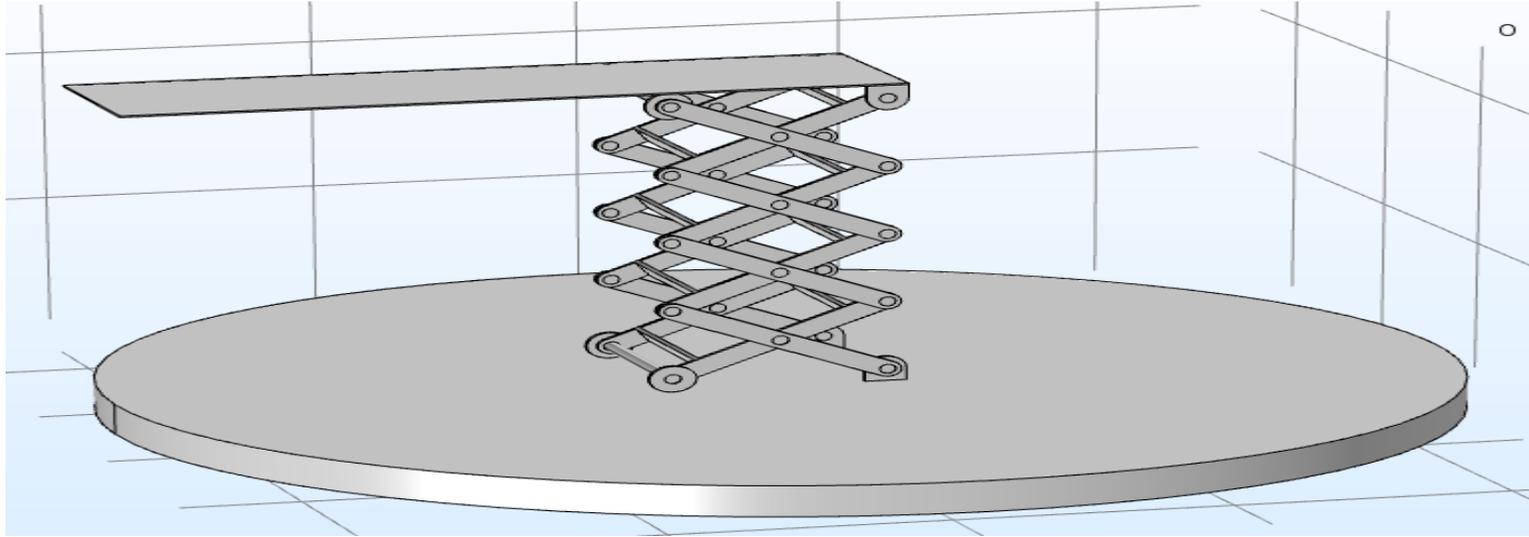
Element Quality Histogram



- CATIA를 이용하여 import

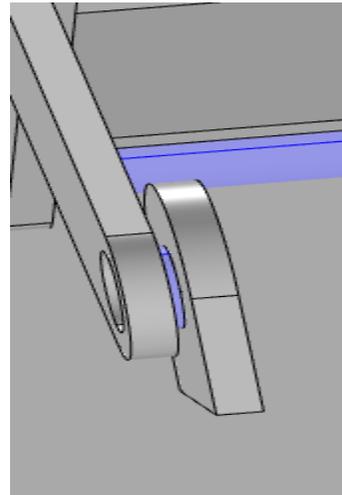
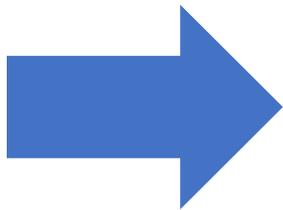
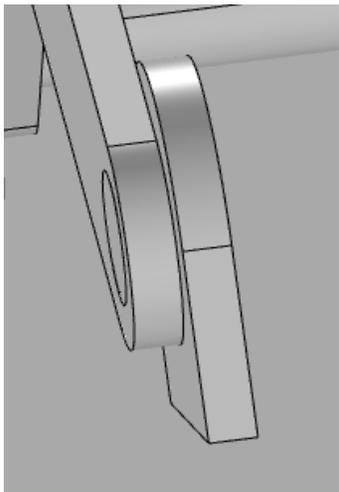
- 정확도(average element quality)가 떨어지는 문제점

Simplified modeling for A point



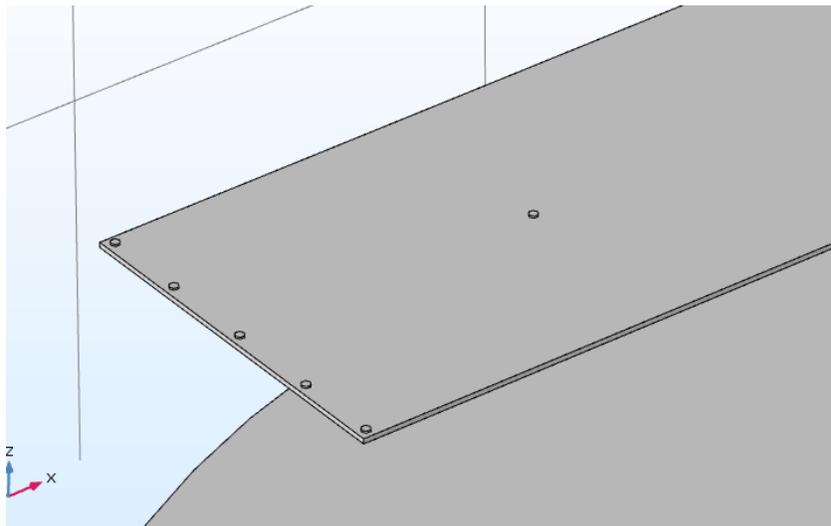
-무대를 없애고 point load 로 변경

-철골구조를 같은 무게의 판으로 변경

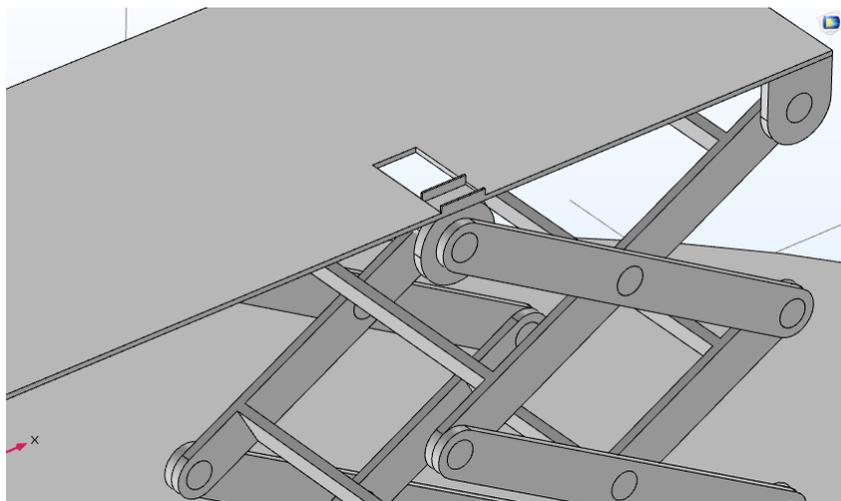


a점 shaft의 응력을 정확하게 보기
위해서 모델링을 변경하였다.

Simplified modeling for B point



- 마찬가지로 무대를 없애고 point load 로 변경

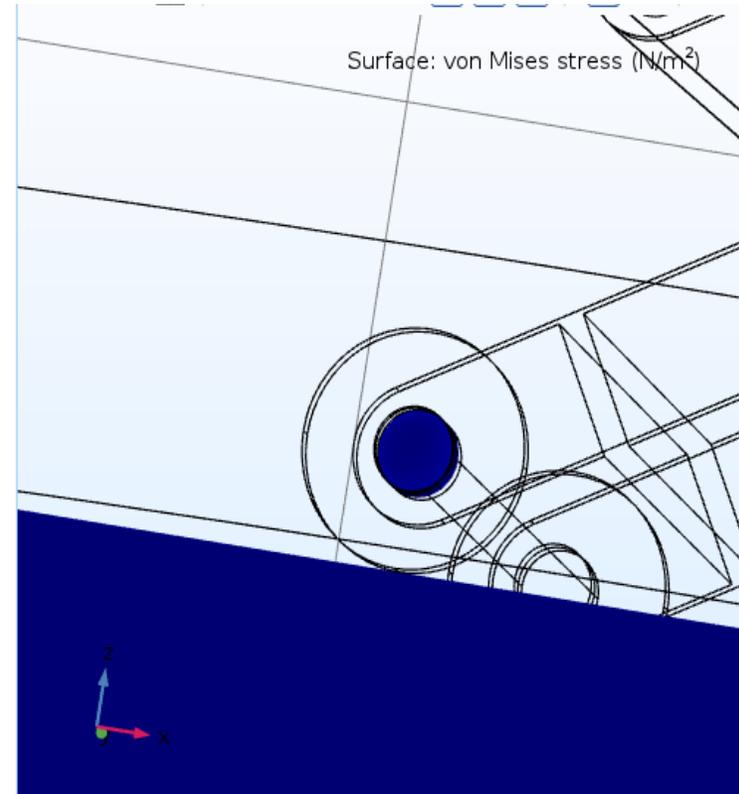
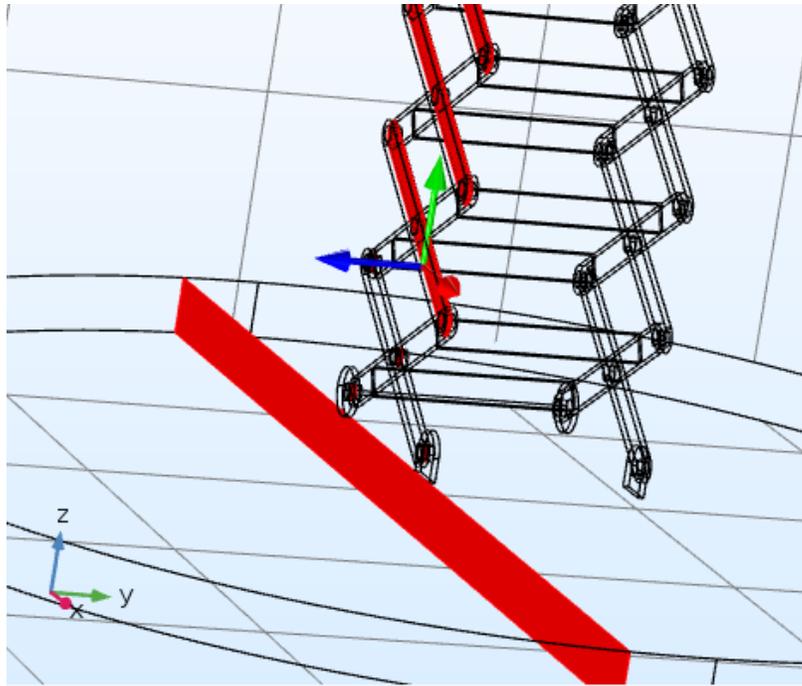


- □ beam 의 응력 해석을 위해 modeling

목표 1. mesh 변화를 통한 A점의 응력 해석

- Results
 - Data Sets
 - Study 1/Solution 1 (sol1)
 - Cut Plane 1
 - Derived Values
 - Tables
 - Stress (solid)
 - Surface 1

- A점 해석을 위해 cut plane을 사용



Mesh (1)

Element Size

Calibrate for:
General physics

Predefined Fine

Custom

Element Size Parameters

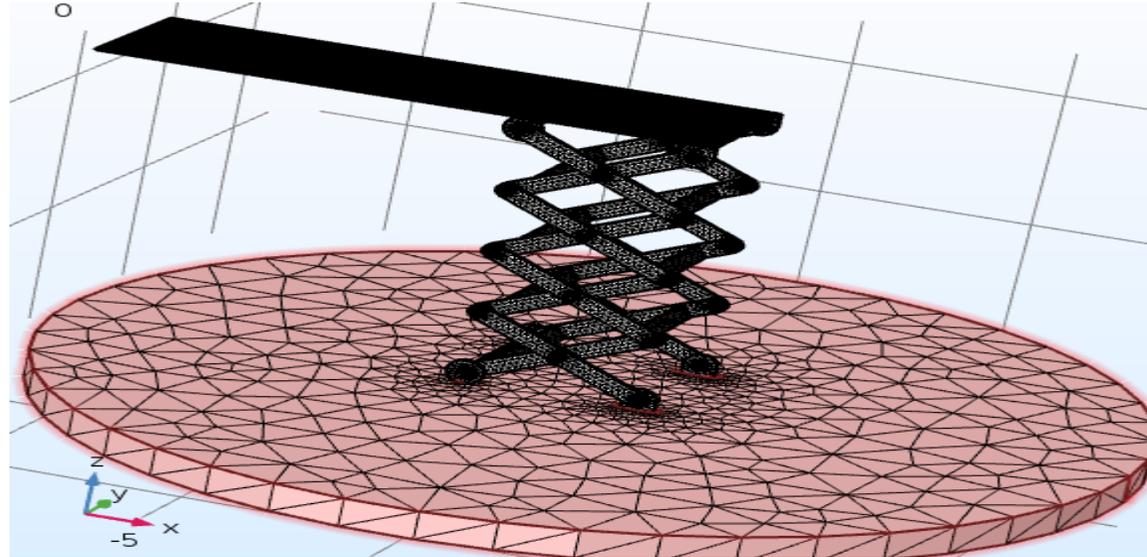
Maximum element size:
1.92 m

Minimum element size:
0.09 m

Maximum element growth rate:
1.45

Curvature factor:
0.5

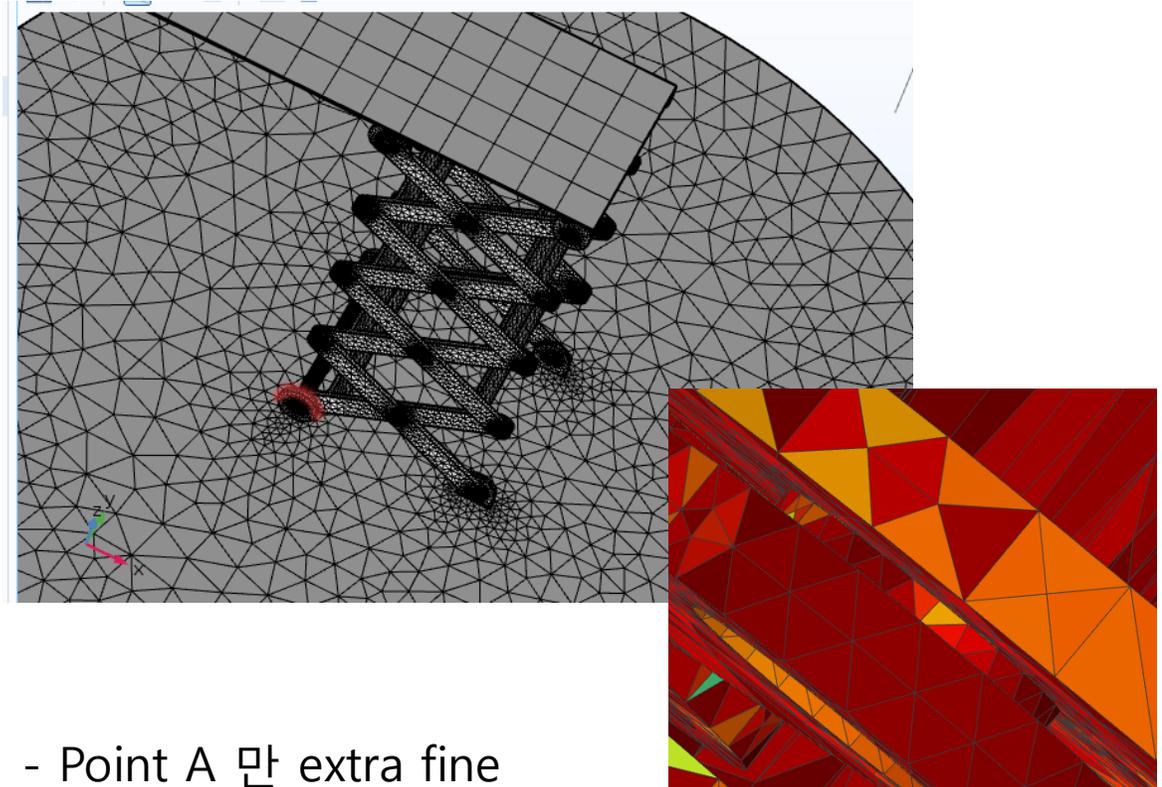
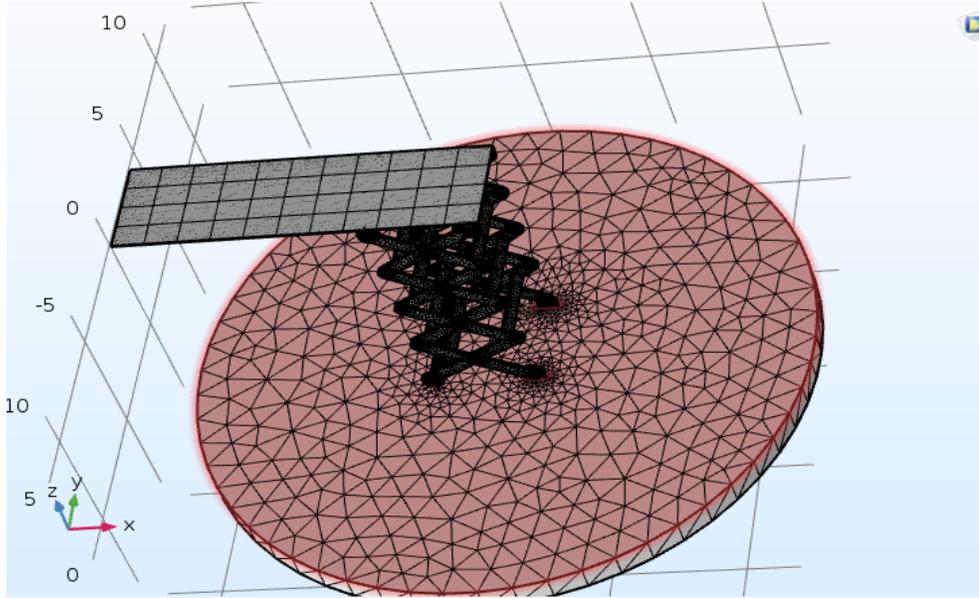
Resolution of narrow regions:
0.6



-Free tetrahedral mesh 로 사이즈 fine 으로 mesh 를 형성

-이 경우 해석이 필요치 않은 위쪽 부분에 과도한 mesh가 형성됨을 알 수 있다.

Mesh (2) ,(3)

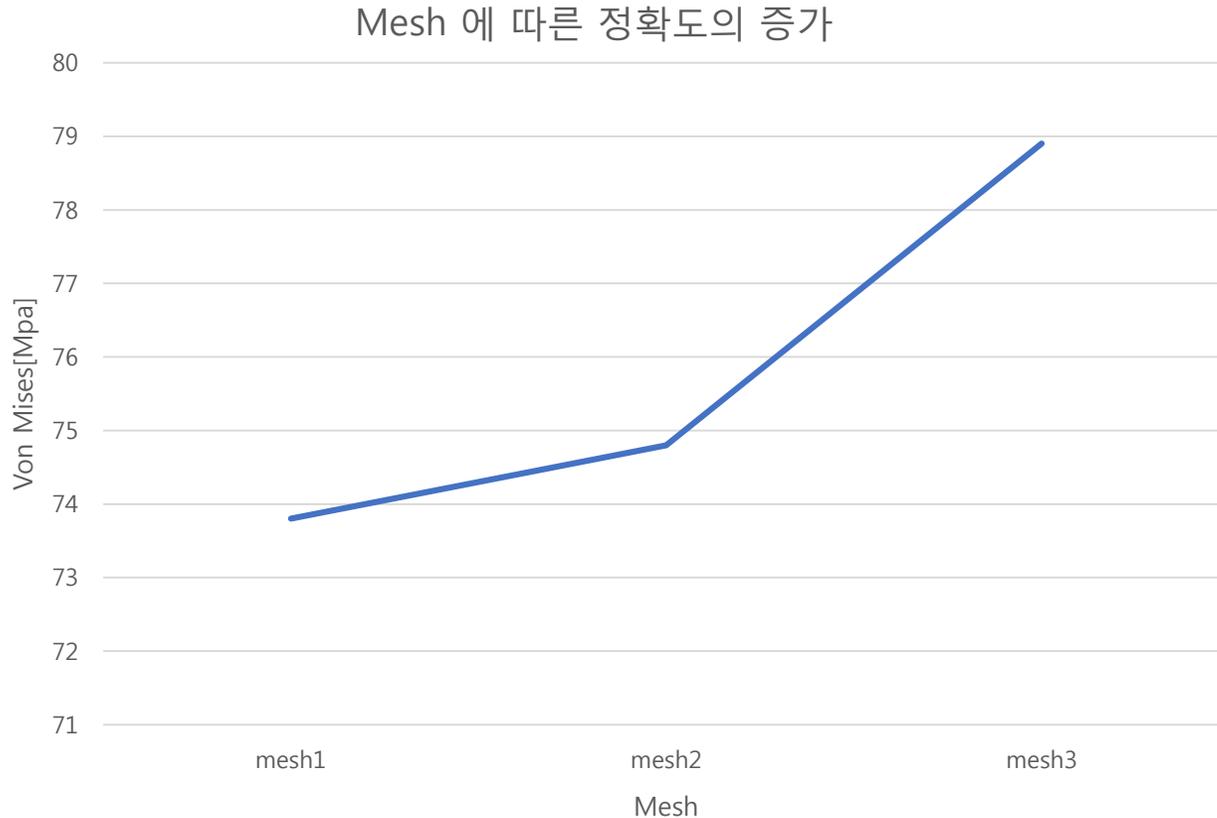


- 위 부분 mapped를 이용
- 나머지 부분 finer

- Point A 만 extra fine



결과값



Matlab 값 = 80.6Mpa (analytic)

Mesh1. Comsol 값 = 73.8Mpa

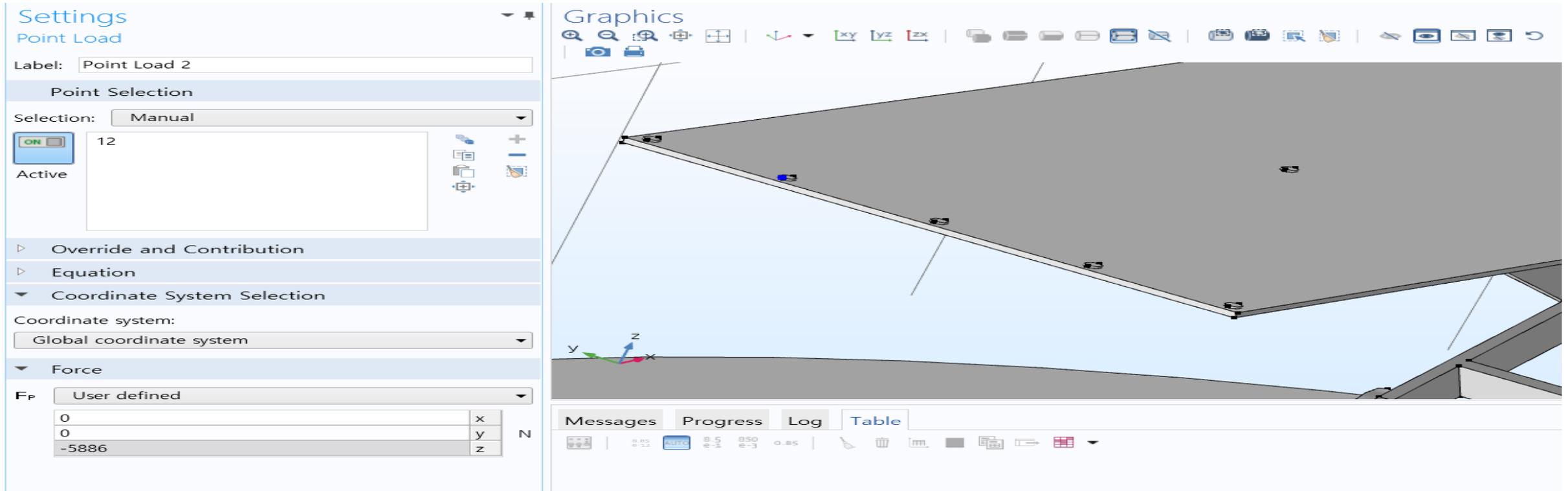
Mesh2. Comsol 값 = 74.8Mpa

Mech3. Comsol 값 = 78.9Mpa

- Mesh 를 정교하게 구성할 수 록 이론값에 가까워짐을 확인할 수 있다.

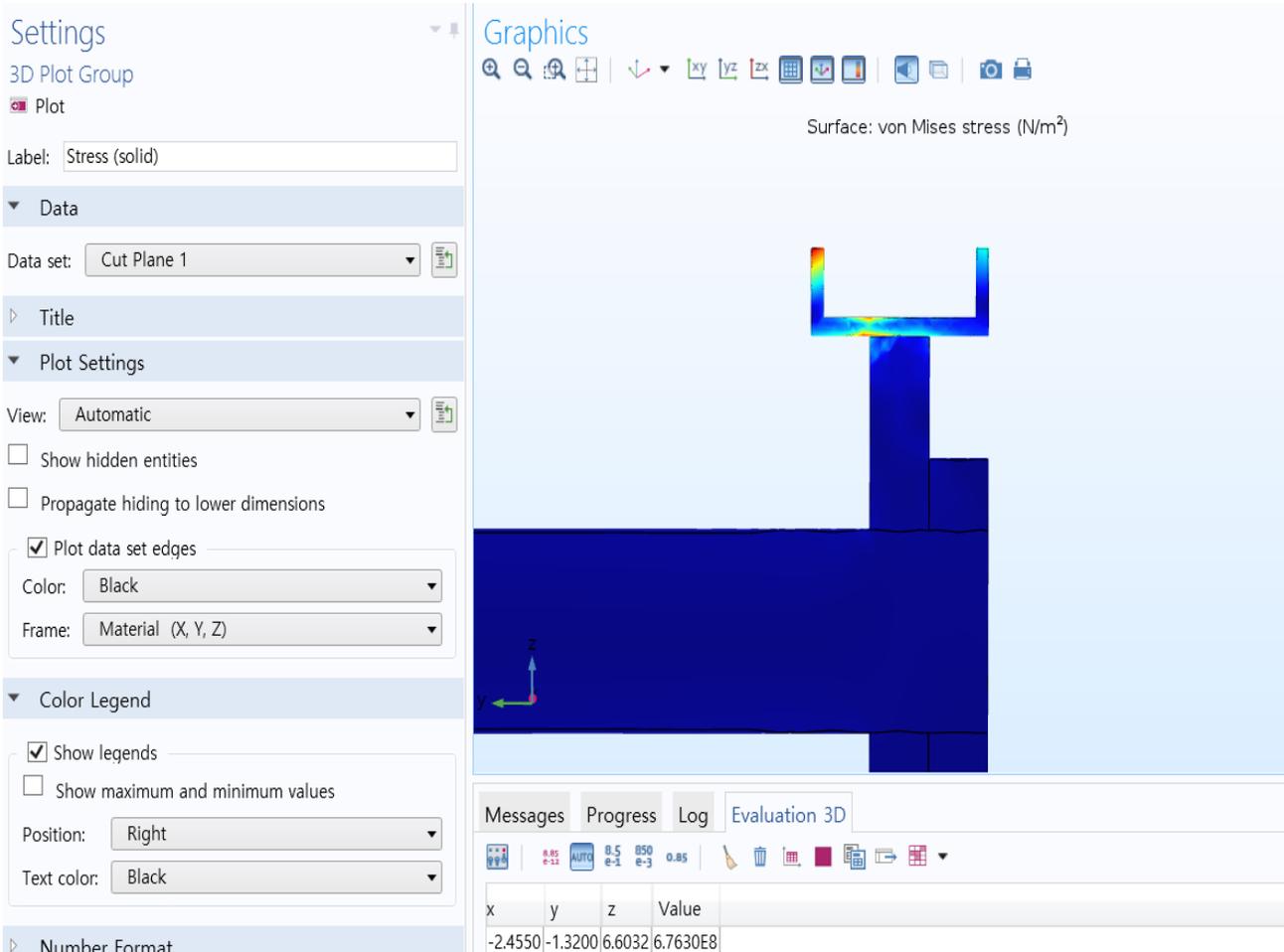


목표2. load의 X축 변화에 따른 응력 해석



무대의 무게 $F_s(=15098\text{N})$ 과 공연 가수들의 무게 $F_p(=5886\text{N})$ 을 해당하는 점에 point load 로 주었다.

X=0



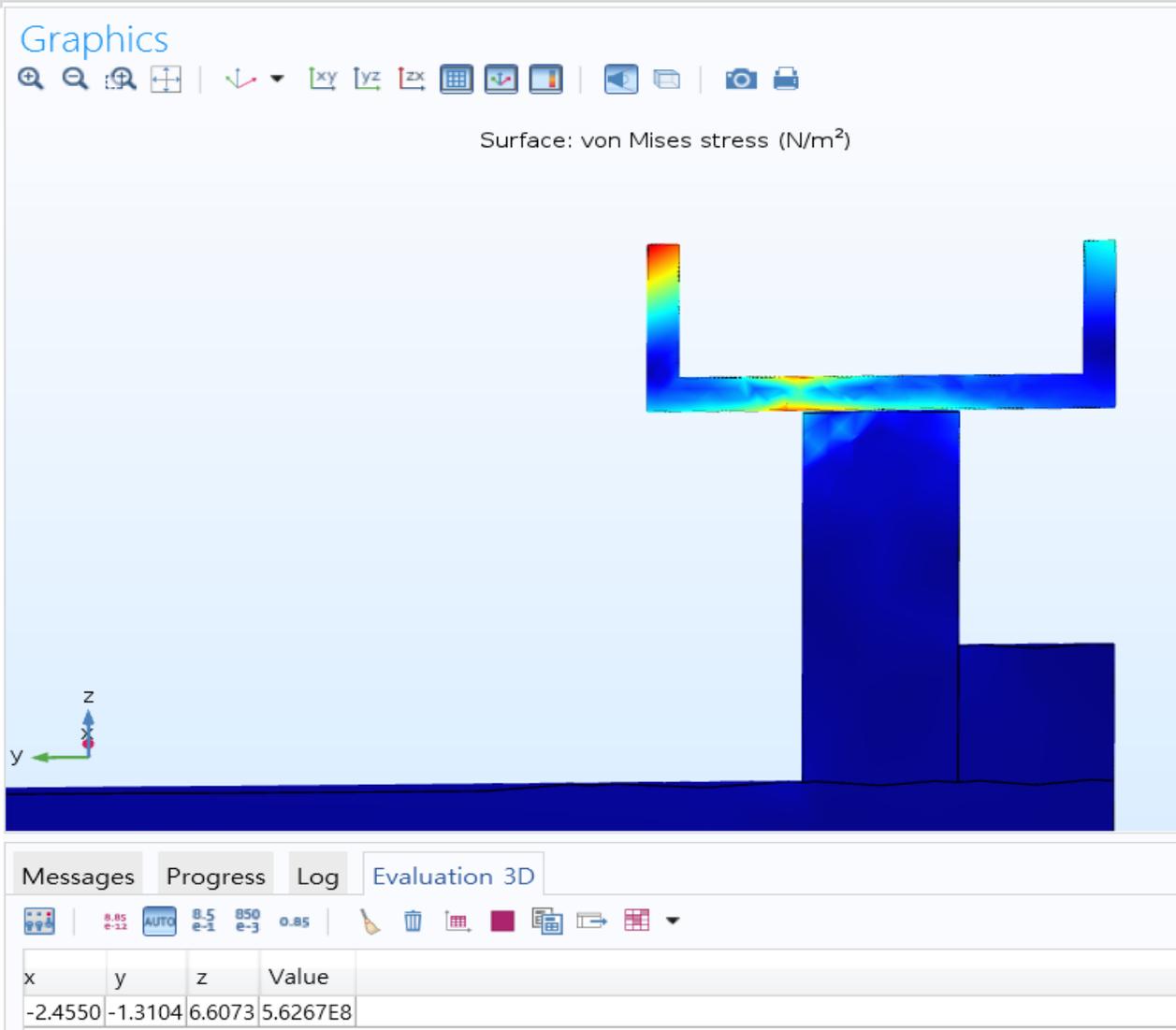
Matlab 값 : 65.7Mpa

Comsol 값 : 67.6Mpa

von_e

6.5785e+08

X=2



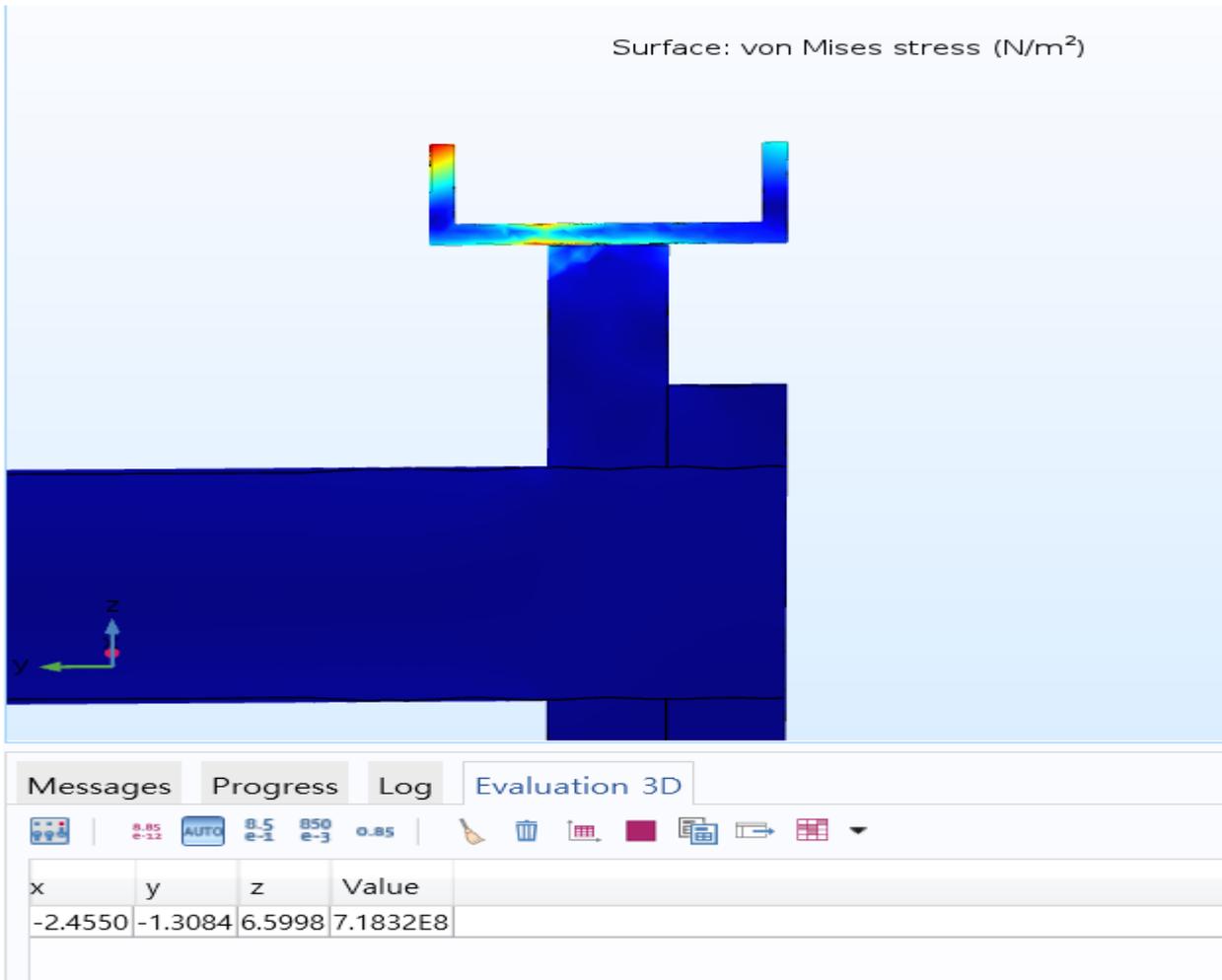
Matlab 값 : 59.8Mpa

Comsol 값 : 56.2Mpa

von_e

5.9840e+08

X=-2



Matlab 값 : 72.6Mpa

Comsol 값 : 71.8Mpa

 von_e

7.2060e+08



04 실제 무대에 적용

가수의 공연을 설계한 무대에 적용



$X = 0$



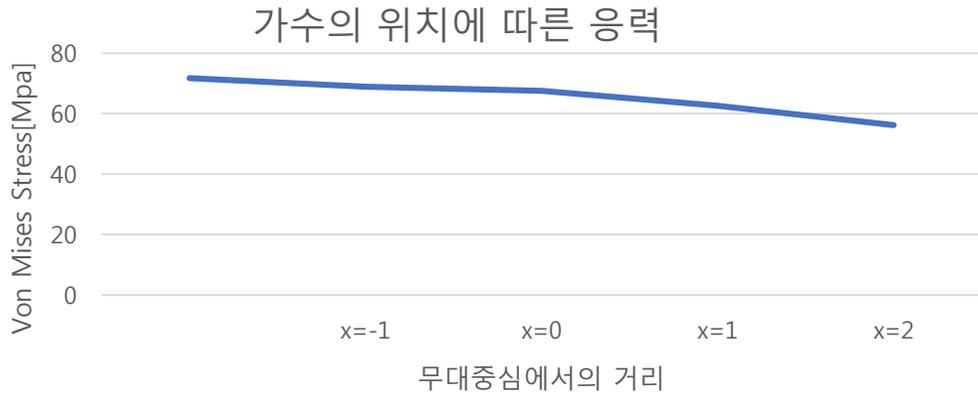
$X = -2$



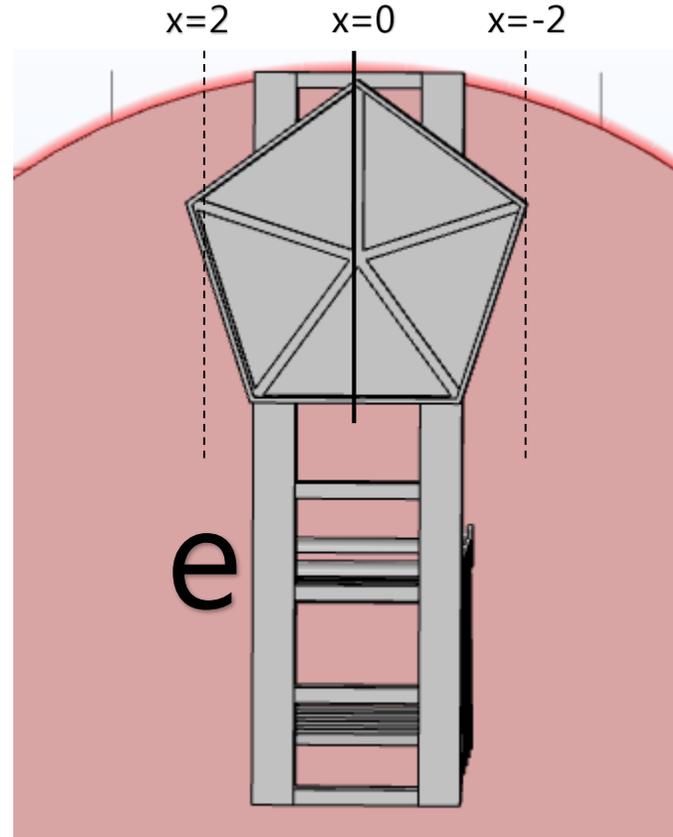
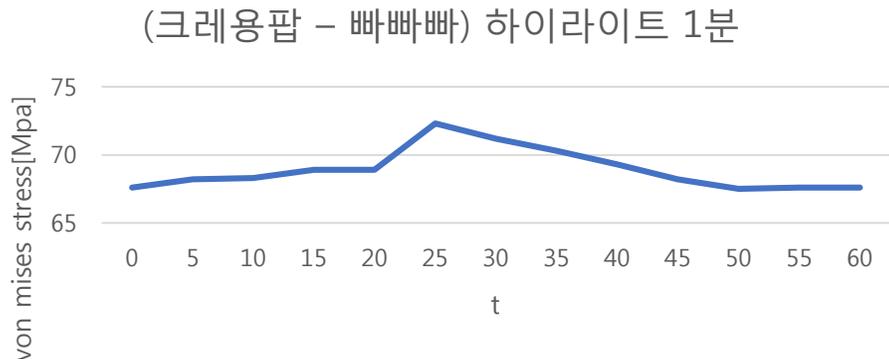
$X = 2$

평창동계올림픽 개.폐막식에서 가수가 공연을 할 때 critical point인 d 점의 von-mises 응력이 어떻게 작용하는지 살펴본다.

결과



E점에서 오른쪽으로 거리가 증가할 때 torsion이 증가하기 때문에 응력이 커짐을 알 수 있다.



하이라이트 1분의 데이터를 살펴보면 22~30 초 동안 무대 오른쪽에 가수들이 위치해있다고 볼 수 있다.

감사합니다

